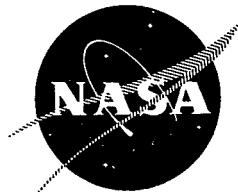


11-14
NASA CR - 72740

R - 8385



FLEXIBLE ROTORDYNAMIC ANALYSIS
FINAL REPORT

by

F. A. Shen and E. Mogil

114
N 70 - 42476
(ACCESSION NUMBER)
15
150
(PAGE)
150
(CODE)
A Division of North American Rockwell Corporation
(CATEGORY)

Facility Form 602

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

NASA Lewis Research Center

Contract NAS 3-13219

David P. Fleming, Project Manager



50T

NOTICE

This report was prepared as an account of Government-sponsored work. Neither the United States, nor the National Aeronautics and Space Administration (NASA), nor any person acting on behalf of NASA:

- A.) Makes any warranty or representation, expressed or implied, with respect to the accuracy, completeness, or usefulness of the information contained in this report, or that the use of any information, apparatus, method, or process disclosed in this report may not infringe privately-owned rights; or
- B.) Assumes any liabilities with respect to the use of, or for damages resulting from the use of, any information, apparatus, method or process disclosed in this report.

As used above, "person acting on behalf of NASA" includes any employee or contractor of NASA, or employee of such contractor, to the extent that such employee or contractor of NASA or employee of such contractor prepares, disseminates, or provides access to any information pursuant to his employment or contract with NASA, or his employment with such contractor.

Requests for copies of this report should be referred to

National Aeronautics and Space Administration
Scientific and Technical Information Facility
P.O. Box 33
College Park, Md. 20740

FINAL REPORT

FLEXIBLE ROTOR DYNAMIC ANALYSIS

by

F. A. Shen and E. Mogil

POWER SYSTEMS DIVISION - ROCKETDYNE
A Division of North American Rockwell Corporation
6633 Canoga Avenue
Canoga Park, California

prepared for

NATIONAL AERONAUTICS AND SPACE ADMINISTRATION

June 1, 1970

CONTRACT NAS 3-13219

Technical Management
NASA Lewis Research Center
Cleveland, Ohio
David P. Fleming

FORWARD

This report was prepared by Rocketdyne -
Power Systems Division, A Division of
North American Rockwell Corporation, under
National Aeronautics and Space Administration
Contract NAS 3-13219.

ABSTRACT

A digital computer program was developed to analyze a flexible rotor-bearing system. The program can be used to predict the dynamic performance under various operating conditions. An experimental test program was conducted utilizing a Mark-25 pump rotor, in a vacuum, to generate spin test data to be used for correlation with results of computer simulations of similar rotor unbalance conditions. The mathematical formulations, the computer program verification, the simulation model and test results are presented and discussed.

TABLE OF CONTENTS

	Page No.
SUMMARY	1
INTRODUCTION	3
MATHEMATICAL FORMULATION	5
EQUATIONS	6
NOMENCLATURE	22
VERIFICATION OF THE COMPUTER PROGRAM	27
CAPABILITY AND LIMITATION OF THE CURRENT COMPUTER PROGRAM	33
COMPUTER PROGRAM USER'S INSTRUCTIONS	35
DESCRIPTION OF THE COMPUTER PROGRAM	35
INPUT PROCEDURE	41
Input	42
Special Notes on Input	52
Namelist Input Procedure and Sample Input Data Sheet	54
Non-Linear Bearing Stiffness Characteristics Representation	60
Output Format	63
EXPERIMENTAL PROGRAM	66
TEST SET-UP	67
Hardware	67
Instrumentation	67
TEST PROGRAM	69
Balancing Tests	69
Investigative Tests	69
PROCEDURE FOR TEST SET-UP	71
ROTORDYNAMICS SPIN TEST PROCEDURE	74
RESULTS	76
SIMULATION OF EXPERIMENTAL TESTS	113
SIMULATION MODEL	113
RESULTS OF THE COMPUTER SIMULATION	113
CONCLUSIONS AND RECOMMENDATIONS	126
APPENDIX A - TYPICAL COMPUTER PROGRAM OUTPUT	130
APPENDIX B - COMPUTER PROGRAM LISTING	144

TABLE OF CONTENTS
(Continued)

	<u>Page No.</u>
Figure 1 - Relation Between the Secondary Frame of Reference and an Inertial Frame of Reference	18
Figure 2 - Schematic of the General Physical Model	19
Figure 3 - Rotating and Secondary Coordinate Systems Used In the Mathematical Model	21
Figure 4 - Data from a 5-Mass 95.5 rpm Flexible Rotor in a Steady-State Operation (Rotor Displacement)	31
Figure 5 - Data from a 5-Mass 95.5 rpm Flexible Rotor in a Steady-State Operation (Whirl-to-Spin Frequency Ratio)	32
Figure 6 - General Flow Diagram Relating Various Subroutines and the MAIN Program	36
Figure 7 - Sample Input Data	50
Figure 8 - Bearing Stiffness Curve	61
Figure 9 - Mark-25 Rotor High Speed Test Set-Up	68
Figure 10 - Mark 25 Rotordynamic Data Inducer Forward End Displacement vs Angles	70
Figure 11 - Mark-25 Rotordynamic Data Inducer and Coupling Ends	72
Figure 12 - Bently and Balance Planes, Mark-25 Pump Rotor	79
Figure 13 - Test No. 1136 - Bently's 2, 8 and 16 Deflections vs Speed	81
Figure 14 - Test No. 1137 - Bently's 2, 8 and 16 Deflections vs Speed	82
Figure 15 - Test No. 1138 - Bently's 2, 8 and 16 Deflection vs Speed	83
Figure 16 - Test No. 1139 - Bently's 2, 8 and 16 Deflection vs Speed	84
Figure 17 - Test No. 1142 - Bently's 2, 8 and 16 Deflection vs Speed	85
Figure 18 - Test No. 1143 - Bently's 2, 8 and 16 Deflection vs Speed	86

TABLE OF CONTENTS
(Continued)

	<u>Page No.</u>
Figure 19 - Test No. 1144 - Bently's 2, 8 and 16 Deflections vs Speed	87
Figure 20 - Test No. 1145 - Bently's 2, 8 and 16 Deflections vs Speed	88
Figure 21 - Test No. 1136 at 26,000 rpm, Deflection vs Station .	89
Figure 22 - Test No. 1136 at 28,000 rpm, Deflection vs Station .	90
Figure 23 - Test No. 1136 at 30,000 rpm, Deflection vs Station .	91
Figure 24 - Test No. 1137 at 26,000 rpm, Deflection vs Station .	92
Figure 25 - Test No. 1137 at 28,000 rpm, Deflection vs Station .	93
Figure 26 - Test No. 1137 at 30,000 rpm, Deflection vs Station .	94
Figure 27 - Test No. 1138 at 28,000 rpm, Deflection vs Station .	95
Figure 28 - Test No. 1138 at 30,000 rpm, Deflection vs Station .	96
Figure 29 - Test No. 1138 at 32,000 rpm, Deflection vs Station .	97
Figure 30 - Test No. 1139 at 28,000 rpm, Deflection vs Station .	98
Figure 31 - Test No. 1139 at 30,000 rpm, Deflection vs Station .	99
Figure 32 - Test No. 1139 at 32,000 rpm, Deflection vs Station .	100
Figure 33 - Test No. 1142 at 28,000 rpm, Deflection vs Station .	101
Figure 34 - Test No. 1142 at 30,000 rpm, Deflection vs Station .	103
Figure 35 - Test No. 1142 at 32,000 rpm, Deflection vs Station .	104
Figure 36 - Test No. 1143 at 26,000 rpm, Deflection vs Station .	104
Figure 37 - Test No. 1143 at 28,000 rpm, Deflection vs Station .	105
Figure 38 - Test No. 1143 at 30,000 rpm, Deflection vs Station .	106
Figure 39 - Test No. 1144 at 26,000 rpm, Deflection vs Station .	107
Figure 40 - Test No. 1144 at 28,000 rpm, Deflection vs Station .	108
Figure 41 - Test No. 1144 at 30,000 rpm, Deflection vs Station .	109
Figure 42 - Test No. 1145 at 26,000 rpm, Deflection vs Station .	110
Figure 43 - Test No. 1145 at 28,000 rpm, Deflection vs Station .	111
Figure 44 - Test No. 1145 at 30,000 rpm, Deflection vs Station .	112

TABLE OF CONTENTS
(Continued)

	<u>Page No.</u>
TABLE 1 - 5-STATION ROTOR-BEARING CHECKOUT COMPARATIVE RESULTS	28
TABLE 2 - COMPARISON OF MARK 25, 17 MASS MODEL NEW PROGRAM VS OLD PROGRAM	29
TABLE 3 - UNBALANCE AND UPPER SPEEDS FOR EXPERIMENTAL TESTS ...	73
TABLE 4 - RELATIVE MATHEMATICAL AND EXPERIMENTAL MODEL DESCRIPTORS	77
TABLE 5 - MARK-25 PUMP ROTOR MASS AND INERTIA PROPERTIES	116
TABLE 6 - MARK-25 PUMP SIMULATION DATA	117
TABLE 7 - TEST 1136, SIMULATION AT 26,000 rpm	118
TABLE 8 - TEST 1137, SIMULATION AT 26,000 rpm	119
TABLE 9 - TEST 1138, SIMULATION AT 32,000 rpm	120
TABLE 10 - TEST 1139, SIMULATION AT 32,000 rpm	121
TABLE 11 - TEST 1142, SIMULATION AT 32,000 rpm	122
TABLE 12 - TEST 1143, SIMULATION AT 32,000 rpm	123
TABLE 13 - TEST 1144, SIMULATION AT 32,000 rpm	124
TABLE 14 - TEST 1145, SIMULATION AT 26,000 rpm	125

SUMMARY

A 10-month study contract, NAS 3-13219, was initiated to update the digital computer program that evolved from a previous study, NAS 3-7996. The work was conducted in three parts. They were 1) update the flexible rotor digital computer program, 2) perform experimental spin tests to generate data, and 3) simulate the experimental tests using the computer program and determine the degree of correlation.

A flexible-rotor dynamic analysis computer program based on Newton's laws of dynamics has been written and computation results verified. The rotor spin and whirl motion were treated as independent but dynamically interacting parameters. Accordingly various whirl-to-spin frequency ratios can be developed as the dynamics relationship dictates. The spin speed of the rotor is controlled by torque applied and combined stiffness, dissipation and inertial loading functions in spin and transverse motion. Both in-phase and out-of-phase stiffness and damping functions are included at all rotor stations. Computation results are of both printout and CRT graph types. After debugging, the computer program was verified in three separate verification steps and the computation results were found to be in good agreement with those from manual and other machine computations. User's instructions are also provided.

The experimental test program, utilizing a Mark-25 pump rotor, was performed to obtain data for correlation with the results of the computer program. The test program consisted of a series of high-speed balancing tests and the eight investigative tests. The investigative tests were simulated with the computer program, and the correlation between theoretical and test results was not very close. Some degree of correlation existed for unbalance conditions at the inducer end of the rotor. Apparently the rotor modeling was not sufficiently descriptive.

The computer program as developed under this study contract can be used as a tool in predicting the dynamic performance of a flexible rotor-bearing system under various operating conditions.

Recommendations are made for further studies to improve the degree of correlation and to increase the computer program capability.

INTRODUCTION

As a continuing effort in advancing the state-of-the-art in rotordynamic analysis, a 10 month program was initiated to develop a computer program to permit more accurate analysis of rotating assemblies, and to apply this program to predictions of the dynamic behavior of the Rocketdyne Mark-25 pump rotor.

The program was accomplished in three tasks: (1) revising and updating the computer program, (2) performing spin tests, and (3) simulation of the experimental data utilizing the computer program.

The mathematical formulation used for the effort reported in this contract, although based on the same general principle as the previous study (Contract NAS3-7996), has been completely rewritten to include the following parameters required by the contract

1. Rotor flexibility due to both shear and bending
2. Damping functions
3. Non-isotropic bearing mounts
4. Nonlinear bearing spring rates
5. Multiple bearing supports

and the following additional items:

6. Linear, speed-sensitive, support bearing stiffness;
7. Enlarged maximum number of rotor-stations to 25, from 12,
8. A displacement vector rotating coordinate system (whirl), as opposed to the spin-speed rotating coordinate used in the 1967 contract; and
9. Up-dating the CRT graphs to include multiple-bearing output data plots.

In addition, spin tests utilizing the Mark-25 pump rotor were performed in the Rocketdyne rotor dynamics facility, for correlation with the

results of the flexible rotor dynamics computer program.

Simulation of the spin test and high speed balancing data was performed with the computer program, utilizing as program inputs the experimentally introduced unbalances.

A checked-out operating computer card deck for the flexible rotor dynamics computer program was delivered to the NASA Project Manager.

This final report presents the results of the research program conducted 4 September 1969 through 1 May 1970. During the course of development of the computer program, certain shortcomings in the program were noted. Conclusions and recommendations are given towards further improving the program computational-to-real time ratio and other rotordynamic parameters.

MATHEMATICAL FORMULATION (THEORY)

In writing the governing equations of a dynamic system, two general approaches may be taken, the energy approach and the momentum approach. The former is generally accomplished by means of Lagrange equations and the latter is carried out according to Newton's Laws of Dynamics. When the traction vs displacement relationship of a dynamic system can be clearly seen, it is faster and more direct to use the momentum approach as it is adopted in this analysis.

The basic assumptions made in the formulation of the general mathematical model are:

1. Representation of a continuous, elastic and distributed mass rotor by a set of discrete masses connected by weightless, elastic shaft sections. The polar and diametral mass moments of inertia and the mass of a rotor section are suitably lumped at the adjacent mass stations. The analysis of the transverse rotordynamic problem is based on a lumped parameter model. With an adequate number of discrete mass stations, the dynamic performance of a rotor can be accurately simulated.
2. Small lateral displacement of a rotor in comparison with the sectional length of a rotor*. This assumption is quite realistic for all rotor dynamics analysis. Under this assumption, the linear moduli of elasticity of the rotor may be used in establishing load and deflection relationship.

* Sectional length of a rotor refers to the length of a rotor section between adjacent discrete mass stations.

3. An axisymmetric rotor geometry and rotor mechanical properties are assumed. While a nonaxisymmetric rotor section will not introduce any solution difficulties, it would result in a very time-consuming computation process. The inclusion of the non-axisymmetric rotor design consideration in a computer analysis is generally not justified as most rotor geometries are or can be considered axisymmetric.
4. The foundation of the rotating machinery is anchored to the secondary frame of reference XYZ in Fig. 1. The secondary frame of reference is permitted to have only translatory motion at a constant velocity with respect to the inertial axes $X_oY_oZ_o$. The surface of the earth may be considered as a secondary inertial frame of reference with a constant velocity relative to the inertial frame of reference $X_oY_oZ_o$. Small angular motion of the secondary frame of reference from the earth rotation will have negligible effects on the dynamics of rotors except those of a gyroscope type of instrument.

Based on the above assumptions, a set of rotordynamics governing equations may be written. Figure 2 depicts a dynamic configuration of a rotor and its casing, and defines certain notations used in the equations. Figure 3 shows that the coordinates X_i and Y_i refers to the geometrical center of a rotor section.

EQUATIONS

The following 22 equations constitute the mathematical formulation for the physical model shown in Figure 2:

$$\begin{aligned}
 \ddot{\phi} \sum_i^n \left[I_{p_i} + m_i e_i^2 \right] + \sum_i^n \left\{ m_i e_i \left[(\ddot{y}_i + g_y) \cos(\phi + \alpha_i) \right. \right. \\
 \left. \left. - (\ddot{x}_i + g_x) \sin(\phi + \alpha_i) \right] \right\} + \sum_i^n \left[C_{z1i} \dot{\phi}^{C_{z1i}} + C_{z2i} \dot{\phi} \right] \\
 - \left[M_{z1} \dot{\phi}^{M_z} + M_{z2} \dot{\phi} + M_{z3} \right] = 0 \quad (1)
 \end{aligned}$$

Eq. (1) defines the torsional moment equilibrium relationship about z-axis. The first group of terms,

$$\ddot{\phi} \sum_i^n \left[I_{p_i} + m_i e_i^2 \right]$$

represents the inertia torque from the rotor polar mass moments of inertia about the rotor elastic centers. The second group,

$$\sum_i^n \left\{ m_i e_i \left[(\ddot{y}_i + g_y) \cos(\phi + \alpha_i) - (\ddot{x}_i + g_x) \sin(\phi + \alpha_i) \right] \right\}$$

denotes the torque derived from the rotor mass inertia forces. The third group

$$\sum_i^n \left[C_{z1i} \dot{\phi}^{C_{z1i}} + C_{z2i} \dot{\phi} \right]$$

is a summation of all rotor spin speed sensitive and constant dissipative torques, and

$$- \left[M_{z1} \dot{\phi}^{M_z} + M_{z2} \dot{\phi} + M_{z3} \right] = 0$$

The last group represents the spin speed sensitive and constant rotor drive torques.

Eqs. (2) and (3) summarize all forces specified at rotor station i along X and Y axis, respectively.

$$\begin{aligned}
 -F_{xi} = m_i \left\{ \ddot{x}_i + g_x - e_i \left[\ddot{\phi} \sin(\phi + \alpha_i) + (\dot{\phi})^2 \cos(\phi + \alpha_i) \right] \right\} \\
 + \left[K_i x_i + C_i \dot{x}_i + K_{pi} y_i + C_{pi} \dot{y}_i \right] + \left[K_{HDI} y_i (\dot{\phi} - K_{FL} \omega_i) \right. \\
 \left. + C_{HDI} \dot{y}_i (\dot{\phi} - C_{FL} \omega_i) \right] + A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_o) \right] \left[N_{Bik} \right. \right. \\
 \left. + B_{Bik} \left[\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right] \right] + K_{Bik} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{Bik}-1}{2}} \right. \\
 \left. + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi} + C_{Bxi} \dot{x}_i \right\} \quad (2)
 \end{aligned}$$

$$\begin{aligned}
 -F_{yi} = m_i \left\{ \ddot{y}_i + g_y + e_i \left[\ddot{\phi} \cos(\phi + \alpha_i) - (\dot{\phi})^2 \sin(\phi + \alpha_i) \right] \right\} \\
 + \left[K_i y_i + C_i \dot{y}_i - K_{pi} x_i - C_{pi} \dot{x}_i - K_{HDI} x_i (\dot{\phi} - K_{FL} \omega_i) \right. \\
 \left. - C_{HDI} \dot{x}_i (\dot{\phi} - C_{FL} \omega_i) \right] + A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_o) \right] \left[N_{Bik} \right. \right. \\
 \left. + B_{Bik} \left[\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right] \right] + K_{Bik} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{Bik}-1}{2}} \right. \\
 \left. + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] Y_{Bi} + C_{Byi} \dot{y}_i \right\} \quad (3)
 \end{aligned}$$

The first group in Eq. (2) and similarly with Eq. (3).

$$m_i \left\{ \ddot{x}_i + g_x - e_i \left[\ddot{\phi} \sin(\phi + \alpha_i) + (\dot{\phi})^2 \cos(\phi + \alpha_i) \right] \right\}$$

defines the inertia and gravity and/or G-loading forces along X-axis.
The second group,

$$[K_i x_i + C_i \dot{x}_i + K_{pi} y_i + C_{pi} \dot{y}_i]$$

denotes the rotor surface forces at a rotor station.

$$K_i x_i, C_i \dot{x}_i, K_{pi} y_i \text{ and } C_{pi} \dot{y}_i$$

are in-phase stiffness force, in-phase damping force, out-of-phase stiffness force and out-of-phase damping force respectively. The in-phase force is in line but opposite to the displacement or velocity vector and the out-of-phase force leads 90° from the displacement or velocity vector. The third group

$$[K_{HDi} y_i (\dot{\phi} - K_{Fi} \omega_i) + C_{HDi} \dot{y}_i (\dot{\phi} - C_{Fi} \omega_i)]$$

represents the rotor surface forces at a rotor station. They are the combined rotor-spin and whirl speed sensitive out-of-phase stiffness and damping forces. These forces may be generated in a non-synchronous whirl motion such as that encountered in a fluid-film bearing or similar components. The last group,

$$A_i \left\{ \left[(\dot{\phi} - \dot{\phi}_o) [N_{Bik} + B_{Bik} \left(\sqrt{X_{Bi}^2 + Y_{Bi}^2} - \rho_{Bik} \right)] + K_{Bik} \right] \right\}$$

$$\left[\left(\bar{X}_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{1}{2}} - 1 + D_{Bik} + \frac{E_{Bik}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi} + C_{Bxi} \dot{X}_i \right\}$$

denotes the nonlinear bearing forces at a bearing station. $\dot{\phi}_0$ is a rotor spin speed constant and $\dot{\phi}$ is the actual rotor spin speed. $(\dot{\phi} - \dot{\phi}_0) N_{BiK}$ is used to simulate the spin speed sensitive bearing stiffness at zero journal displacement. B_{BiK} is the spin speed and journal displacement sensitive stiffness coefficient. X_{Bi} and Y_{Bi} are the journal displacement coordinates and ρ_{BiK} is the lower bearing displacement limit for K th bearing stiffness section of bearing i. K_{BiK} is the journal displacement sensitive bearing stiffness.

$$K_{BiK} \left[\left(X_{Bi}^2 + Y_{Bi}^2 \right)^{\frac{H_{BiK}-1}{2}} + D_{BiK} + \frac{E_{BiK}}{\sqrt{X_{Bi}^2 + Y_{Bi}^2}} \right] X_{Bi}$$

is the nonlinear bearing stiffness force function, and $C_{BXi} \dot{X}_i$ the X-component damping force at the bearing.

Eqs. (4) and (5) describe all the specified moments defined about Y and X axis respectively.

$$\begin{aligned}
 - \left(l_{i-1} + l_i \right) M_{Yi} &= I_{Di} \left[\ddot{X}_{i+1} \ddot{X}_{i-1} - \beta_i (l_{i-1} + l_i) \left(\dot{\phi} \sin(\phi + \gamma_i) \right. \right. \\
 &\quad \left. \left. + (\dot{\phi})^2 \cos(\phi + \gamma_i) \right) \right] \\
 &\quad + I_{Qi} \dot{\phi} \left[(\dot{Y}_{i+1} - \dot{Y}_{i-1}) + \beta_i (l_{i-1} + l_i) \dot{\phi} \cos(\phi + \gamma_i) \right] \\
 &\quad + K_{\phi i} (X_{i+1} - X_{i-1}) + C_{\phi i} (\dot{X}_{i+1} - \dot{X}_{i-1}) \\
 &\quad + K_{\phi \rho i} (Y_{i+1} - Y_{i-1}) + C_{\phi \rho i} (\dot{Y}_{i+1} - \dot{Y}_{i-1}) \\
 &\quad + K_{\phi HDi} (\dot{\phi} - K_{\phi Fi} \omega_i) (Y_{i+1} - Y_{i-1}) + C_{\phi HDi} (\dot{\phi} - C_{\phi Fi} \omega_i) (\dot{Y}_{i+1} - \dot{Y}_{i-1})
 \end{aligned} \tag{4}$$

$$\begin{aligned}
(l_{i+1} + l_i) M_{X_i} = & I_{Di} \left[\left(\ddot{Y}_{i+1} - \ddot{Y}_i \right) + \beta_i (l_{i+1} + l_i) \left[\ddot{\phi} \cos(\phi + \gamma_i) \right. \right. \\
& \left. \left. - (\dot{\phi})^2 \sin(\phi + \gamma_i) \right] \right] \\
& - I_{\phi i} \dot{\phi} \left[(\dot{X}_{i+1} - \dot{X}_i) - \beta_i (l_{i+1} + l_i) \dot{\phi} \sin(\phi + \gamma_i) \right] \\
& + K_{\phi i} (Y_{i+1} - Y_i) + C_{\phi i} (\dot{Y}_{i+1} - \dot{Y}_i) \\
& - K_{\phi \rho i} (X_{i+1} - X_i) - C_{\phi \rho i} (\dot{X}_{i+1} - \dot{X}_i) \\
& - K_{\phi HDi} (\dot{\phi} - K_{\phi \rho i} \omega_i) (X_{i+1} - X_i) - C_{\phi HDi} (\dot{\phi} - C_{\phi \rho i} \omega_i) (\dot{X}_{i+1} - \dot{X}_i) \tag{5}
\end{aligned}$$

The first group in Eq. (4), and similarly with Eq. (5)

$$I_{Di} \left[\ddot{X}_{i+1} - \ddot{X}_i - \beta_i (l_{i+1} + l_i) \left[\ddot{\phi} \sin(\phi + \gamma_i) + (\dot{\phi})^2 \cos(\phi + \gamma_i) \right] \right]$$

denotes the inertial moment resulting from the transverse mass moment of inertia at a rotor station. In calculating the bending moments at a rotor station the slopes and their time derivatives are approximated by that of the chord joining the rotor centers at the adjacent stations. Using the chord approximations of the slopes the number of unknowns involved in the solution of the rotordynamics equations was substantially reduced. Two additional unknown variables at each station, i.e., the force and moment slope coefficients would otherwise have to be included in the rotordynamics equations. β_i and γ_i are the initial misalignment and angular orientation of the axis of the mass moments of inertia with respect to rotor elastic center line and initial rotor spin angular position respectively.

The second group,

$$I_{\phi i} \dot{\phi} \left[\left(\dot{y}_{i+1} - \dot{y}_{i-1} \right) + \beta_i (l_{i-1} + l_i) \dot{\phi} \cos(\phi + \gamma_i) \right]$$

represents the inertia moment from the polar mass moment of inertia at a rotor station.

The third group

$$K_{\phi i} (x_{i+1} - x_{i-1}) + C_{\phi i} (\dot{x}_{i+1} - \dot{x}_{i-1})$$

denotes the rotor surface forces and they are the in-phase stiffness and damping moment due to a rotor misalignment motions with respect to the rotor casing or bearing at a rotor station.

The fourth group,

$$K_{\phi Pi} (y_{i+1} - y_{i-1}) + C_{\phi Pi} (\dot{y}_{i+1} - \dot{y}_{i-1})$$

similarly represents the rotor surface moments consisting of the out-of-phase stiffness and damping moment from rotor misalignment motions.

The last group,

$$K_{\phi HDi} (\dot{\phi} - K_{\phi Fi} \omega_i) (y_{i+1} - y_{i-1}) + C_{\phi HDi} (\dot{\phi} - C_{\phi Fi} \omega_i) (\dot{y}_{i+1} - \dot{y}_{i-1})$$

denotes the rotor surface moments. They are the out-of-phase, rotor spin and whirl sensitive stiffness and damping moments, similar to the third group in Eq. (2) or (3).

Eqs. (6) and (7) define the force and moment and equilibrium relationships in XZ and YZ planes. This is where the force and moment influence coefficients e_{ij} and b_{ij} are used to solve the basic rotordynamics equations

for every rotor station except that for the first and last non-linear bearings. The basic rotordynamics equations for the first and last bearing stations are Eqs. (8) through (11). To solve the system of rotordynamics equations, first convert into the rotating coordinates and then substitute F_{xi} , F_{yi} , M_{yi} and M_{xi} functions in Eqs. (2) through (5) into Eqs. (6) through (11). The values of c_{ij} and b_{ij} used are computed in subroutine "INFLCO."

Eqs. (1) and (6) through (11) are solved simultaneously for the acceleration terms by means of ISIMDD subroutine. Using the current values of the acceleration and velocity terms, integration over a time interval will be made to obtain the solution for a rotor motion corresponding to a new real time. The process repeats for each time interval until a desired total time period for rotor motion is covered.

The influence coefficients are calculated by finding the rotor deflections (in.) at various stations when a unit transverse force (1 lb) or unit transverse moment (1 in-lb) is applied at a station. Rotor deflections considering rotor shear and bending elasticity are separately computed and then combined. The influence coefficients are based on the rotor-configuration supported on two knife-edge bearings located at the first and last support bearings. The rotor dynamic relationship at these two bearings are defined by Eqs. (8) through (11). Sample results in force and moment influence coefficients are shown in Appendix A. The meaning of row and column numbers in relation to the definitions of the influence coefficients is explained.

$$\sum_i^n \left[F_{xi} C_{ij} + M_{yi} b_{ij} \right] + X_{bi} + \left(X_{b2} - X_{bi} \right) \frac{Z_j - S}{L} - X_j = 0 \quad (6)$$

$$\sum_i^n \left[F_{yi} C_{ij} - M_{xi} b_{ij} \right] + Y_{bi} + \left(Y_{b2} - Y_{bi} \right) \frac{Z_j - S}{L} - Y_j = 0 \quad (7)$$

Eqs. (8), (9), (10) and (11) denote the moment and force equilibrium relationships for the first and last nonlinear bearing station in X-Z and Y-Z plane, respectively. These equations are solved simultaneously with Eqs. (6) and (7).

$$\sum_i^n \left[(Q - Z_i) F_{x_i} - M_{Y_i} \right] = 0 \quad (8)$$

$$\sum_i^n F_{x_i} = 0 \quad (9)$$

$$\sum_i^n \left[(Q - Z_i) F_{Y_i} + M_{X_i} \right] = 0 \quad (10)$$

$$\sum_i^n F_{Y_i} = 0 \quad (11)$$

Eqs. (12) and (14) denote the force equilibrium relationships at the bearing-and-mount station along X and Y axis, respectively. The bearing and mount damping forces are not included. Since zero bearing mass assumption is used, the stiffness force in the bearing should only equal that in the mount and similar relationship exists for the damping forces.

$$A_i \left\{ \left[[\dot{\phi} - \dot{\phi}_o] [N_{Bik} + B_{Bik} (\rho_{Bxi} - \rho_{Bik})] + K_{Bik} \right] \left(\rho_{Bxi}^{(H_{Bik}-1)} \right. \right. \\ \left. \left. + D_{Bik} + \frac{E_{Bik}}{\rho_{Bxi}} \right) X_{Bi} - K_{Mxi} [x_i - x_{Bi}] \right\} = 0 \quad (12)$$

Eqs. (13) and (15) are expressions for ρ_{Bxi} and ρ_{Byi} used in Eqs. (12) and (14) respectively.

$$\rho_{Bxi} = \sqrt{X_{Bi}^2 + \left[\frac{Y_i}{\frac{K_{Mxi}}{K_{Myi}} \left(\frac{X_i}{X_{Bi}} - 1 \right) + 1} \right]^2} \quad (13)$$

$$A_i \left\{ \left[[\dot{\phi} - \dot{\phi}_o] [N_{Bik} + B_{Bik} (\rho_{Byi} - \rho_{Bik}) + K_{Bik}] \right] \left(\rho_{Byi}^{(H_{Bik}-1)} \right. \right. \\ \left. \left. + D_{Bik} + \frac{E_{Bik}}{\rho_{Byi}} \right) Y_{Bi} - K_{Myi} [y_i - y_{Bi}] \right\} = 0 \quad (14)$$

$$\rho_{Byi} = \sqrt{Y_{Bi}^2 + \left[\frac{X_i}{\frac{K_{Myi}}{K_{Mxi}} \left(\frac{Y_i}{Y_{Bi}} - 1 \right) + 1} \right]^2} \quad (15)$$

Eqs. (12) through (15) establish the nonlinear bearing and mount stiffness forces along X and Y axes. The purpose of these equations is to solve for the nonlinear bearing journal displacements through an interative process, a faster method than the closed form solution.

Eqs. (16) through (21) define the various coordinates used in the basic rotordynamics equations. Under the zero bearing mass assumption, the bearing damping can be expressed in terms of the combined bearing-and-mount damping forces. The relationships are shown in Eqs (16) through (19).

$$\dot{X}_{Bi} = \frac{C_{BXi} \dot{X}_i}{C_{Bi}} \quad (16)$$

$$\dot{Y}_{Bi} = \frac{C_{BYi} \dot{Y}_i}{C_{Bi}} \quad (17)$$

$$\dot{X}_{Mi} = \frac{C_{BXi} \dot{X}_i}{C_{MXi}} \quad (18)$$

$$\dot{Y}_{Mi} = \frac{C_{BYi} \dot{Y}_i}{C_{MYi}} \quad (19)$$

Also based on zero-bearing-mass assumption the journal and mount displacement are collinear as shown in Eqs. (20) and (21).

$$X_{mi} = X_i - X_{bi} \quad (20)$$

$$Y_{mi} = Y_i - Y_{bi} \quad (21)$$

Eq. (22) defines the whirl speed of shaft center at station i.

$$\omega_i = \frac{\dot{Y}_i X_i - \dot{X}_i Y_i}{X_i^2 + Y_i^2} \quad (22)$$

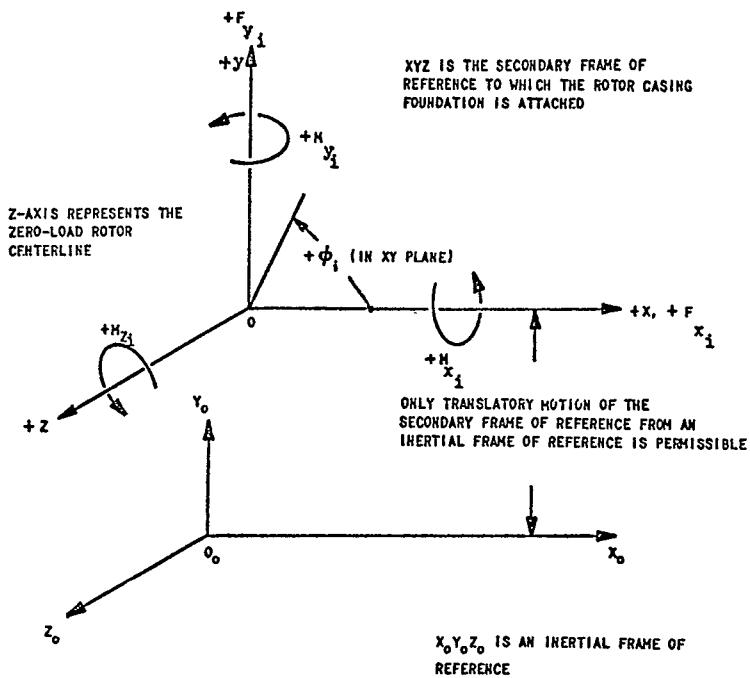


Figure 1 - Relation Between the Secondary Frame of Reference and an Inertial Frame of Reference

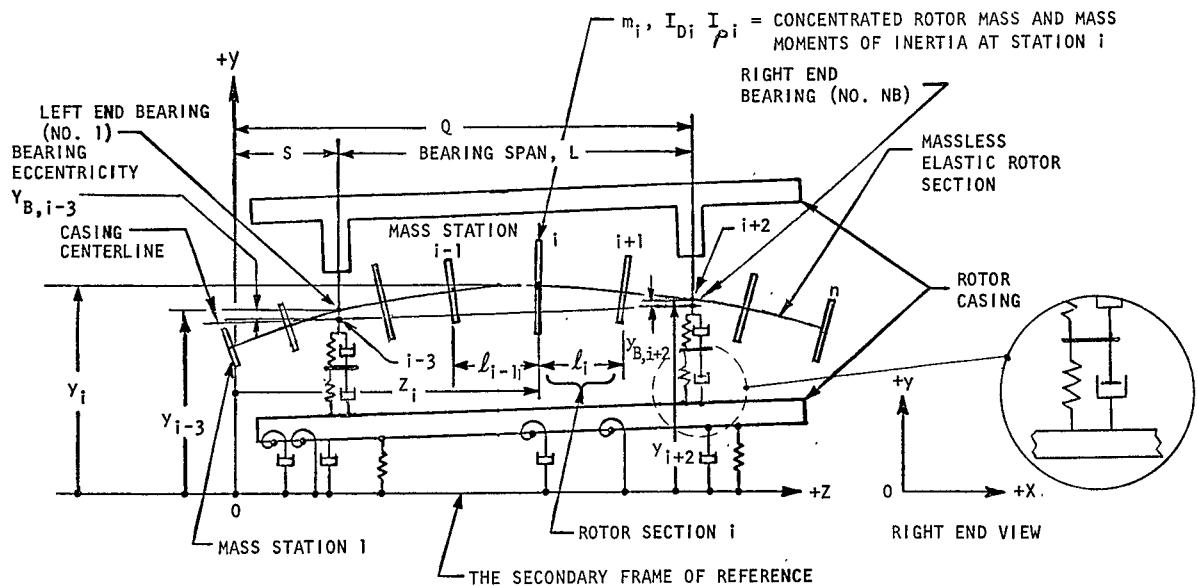


Figure 2. Schematic of the General Physical Model

The rotating coordinate system (Fig. 3) is used to reduce the integration time as it provides lower acceleration magnitude compared with that from conventional stationary coordinate system. The origin of the rotating coordinates coincides with the static rotor center at a station.

The static rotor center corresponds to that caused by g_x and g_y for the rotor with zero spin speed. The rotating coordinates used are synchronous with the rotor displacement vector at that rotor station. Let the static deflections at a rotor station be ΔX_i and ΔY_i ; and ρ_i and

Θ_i be the displacement vector and angular displacement of the rotating coordinates respectively. Then

$$\begin{aligned} X_i &= \rho_i \cos \theta_i + \Delta X_i \\ Y_i &= \rho_i \sin \theta_i + \Delta Y_i \end{aligned}$$

Differentiate once and twice with respect to time and using appropriate substitution, one obtains,

$$\begin{aligned} \dot{X}_i &= \dot{\rho}_i \cos \theta_i - \rho_i \dot{\theta}_i \sin \theta_i \\ \dot{Y}_i &= \dot{\rho}_i \sin \theta_i + \rho_i \dot{\theta}_i \cos \theta_i \\ \ddot{X}_i &= -\ddot{\theta}_i \dot{Y}_i - \dot{\rho}_i \dot{\theta}_i \sin \theta_i + \ddot{\rho}_i \cos \theta_i - \ddot{\theta}_i \rho_i \sin \theta_i \\ \ddot{Y}_i &= \dot{\theta}_i \dot{X}_i + \dot{\rho}_i \dot{\theta}_i \cos \theta_i + \ddot{\rho}_i \sin \theta_i + \ddot{\theta}_i \rho_i \cos \theta_i \end{aligned}$$

The variables X_i , Y_i , and their time derivatives in the rotordynamic equations are then replaced with the equivalent rotating coordinates prior to the solution of the rotordynamics equations.

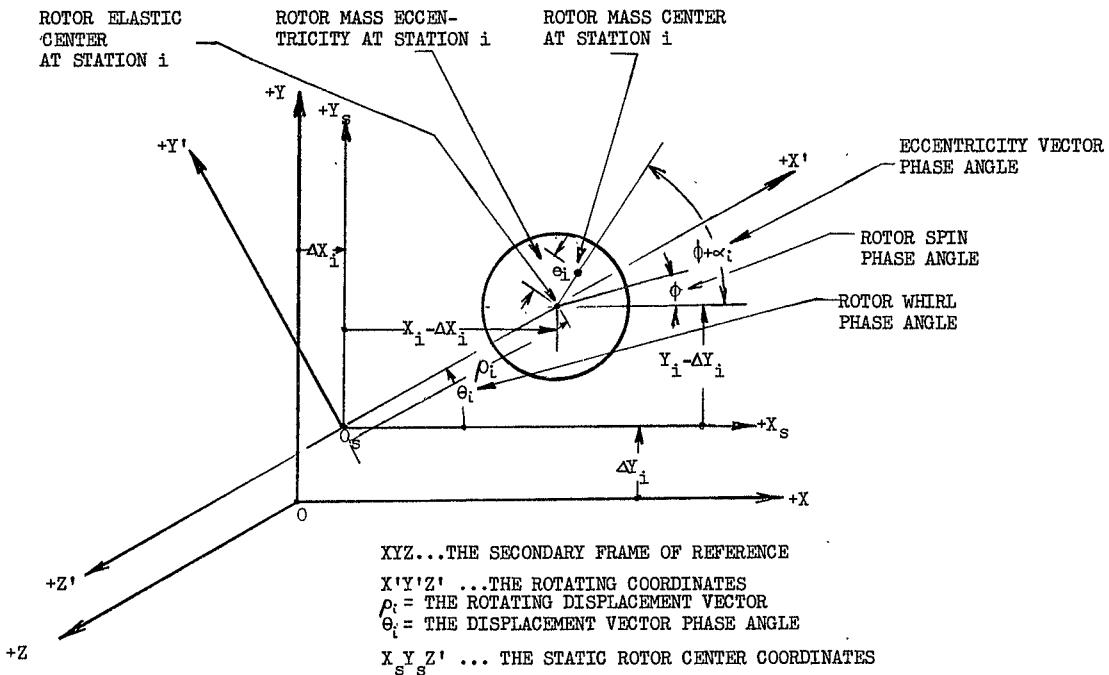


Figure 3 Rotating and Secondary Coordinate Systems Used in the Mathematical Model

NOMENCLATURE

A_i = bearing location notation, $A_i = 1$ implies a bearing at station i ; $A_i = 0$, no bearing at station i dimensionless

b_{ij}, c_{ij} = moment and force influence coefficient, in/(in-lb) and in/lb, respectively

B_{Bik} = rotative speed sensitive bearing stiffness displacement coefficient for k th stiffness section of bearing i ; for dimension, see Note 2

c_{Bi} = bearing damping coefficient for \dot{x}_{Bi} and \dot{y}_{Bi} , (lb-sec)/in

c_{Bxi}, c_{Byi} = equivalent bearing-and-mount damping coefficients for \dot{x}_i, \dot{y}_i , respectively, (lb-sec)/in

c_{Fi} = whirl frequency factor for damping force coefficient, dimensionless

c_{Hdi} = out-of-phase whirl-frequency sensitive damping force coefficient, $(lb\cdot sec^2)/(in\cdot radian)$

$c_i, c_{\phi i}$ = in-phase damping force and moment coefficient due to relative motion between rotor and casing $(lb\cdot sec)/in$ and $(in\cdot lb\cdot sec)/rad$, respectively

c_{Mxi}, c_{Myi} = non-isotropic mount damping coefficient for $\dot{x}_{Mi}, \dot{y}_{Mi}$, respectively, $(lb\cdot sec)/in$

$c_{pi}, c_{\phi pi}$ = out-of-phase damping force and moment coefficient due to relative motion between rotor and casing $(lb\cdot sec)/in$ and $(in\cdot lb\cdot sec)/rad$, respectively

c_{Zi} = torsional friction exponent, Eq. (1), dimensionless

C_{Zli} , C_{Z2i} = torsional friction coefficients, Eq. (1),
 dimension of $C_{Zli} \dot{\phi}^{CZi}$ or $C_{Z2i} \dot{\phi}$ in in-lb

$C_{\dot{\phi}i}$ = damping moment coefficient, (lb-in-sec)/rad.

$C_{\dot{\phi}^2i}$ = whirl-frequency factor for damping moment coefficient,
 dimensionless

$C_{\dot{\phi}HDi}$ = out-of-phase, whirl frequency sensitive, damping moment
 coefficient, (lb-in-sec²)/radian²

D_{BiK} , E_{BiK} = nonlinear bearing stiffness coefficients for kth stiffness
 section of bearing i. For dimension, see Note 1.

e_i = mass eccentricity at rotor station i, in

F_{xi} , F_{yi} = forces in X and Y direction, lb

g_x , g_y = gravity or g-loading in X and Y direction, in/sec²

H_{BiK} = nonlinear bearing stiffness exponent for kth stiffness
 section of bearing i. For dimension, see Note 1

I_{Di} , I_{Pi} = rotor diametral and polar mass moments of inertia at
 station i, lb-in-sec², respectively

K_{BiK} = rotative-speed sensitive bearing stiffness coefficient for
 Kth stiffness section of bearing i, lb/in

K_{Fi} = whirl frequency factor for stiffness force coefficient,
 dimensionless

K_i , $K_{\dot{\phi}i}$ = in-phase stiffness force and moment coefficient due to
 relative motion between rotor and casing at station i,
 lb/in and (in-lb)/rad, respectively.

K_{HDi} , $K_{\dot{Q}HDi}$ = out-of-phase hydrodynamic force and moment coefficient of balance piston, (lb-sec)/(in-rad) and (lb-in-sec)/rad², respectively

K_{MXi} , K_{MYi} = non-isotropic stiffness coefficients for X_{Mi} , Y_{Mi} , respectively, lb/in

K_{pi} , $K_{\dot{\phi}pi}$ = out-of-phase stiffness force and moment coefficient due to the relative motion between rotor and casing at station i , lb/in and in-lb/rad, respectively

$K_{\dot{\phi}Fi}$ = whirl frequency factor for stiffness moment coefficient, dimensionless

l_i = rotor section length between the adjacent mass stations i and $i + 1$, in

L = length between bearings No. 1 and NB, in

m_i = rotor mass at station i , (lb-sec²)/in

M_{xi} , M_{yi} , M_{zi} = moments about X, Y, and Z axes, respectively, in/lb

M_Z = exponent for speed sensitive rotor drive torque coefficient

M_{Z1} , M_{Z2} , M_{Z3} = coefficients for rotor drive torque, dimensions of $M_{Z1}\dot{\phi}^{M_Z}$, $M_{Z2}\dot{\phi}$, or M_{Z3} are in-lb

n = total number of rotor discrete mass stations, dimensionless

N_{Bik} = rotative-speed sensitive bearing stiffness coefficient for K th stiffness section of bearing i . For dimension, see Note 2.

NB = The last bearing number, or total number of non-linear stiffness bearings.

q = length between No. NB bearing and mass station 1, in

S	= length between No. 1 bearing and the mass station 1, in
t	= real time, sec
$X_{Bi}, Y_{Bi}, \dot{X}_{Bi}, \dot{Y}_{Bi}$	= bearing displacement and velocity coordinates, in and in/sec, respectively
$X_{Mi}, Y_{Mi}, \dot{X}_{Mi}, \dot{Y}_{Mi}$	= mount displacement and velocity coordinates, in and in/sec, respectively
X_i, Y_i	= displacement of the rotor geometrical center from its zero-load position in X and Y direction, respectively, at station i, in
XYZ	= the secondary frame of reference to which the rotor casing is attached (Fig. 3)
$X_o Y_o Z_o$	= an inertial frame of reference (Fig. 1)
Z_i	= Z coordinate of ith mass from first mass, in
α_i	= orientation of the ith mass eccentricity vector from that of the first rotor mass measured in the direction of rotation, rad
ω_i	= angular whirl velocity of the rotor geometric center at station i, rad/sec
$\dot{\phi}_o$	= a spin speed sensitive bearing stiffness parameter, rad/sec as defined in Eq. (2), (3), (12) and (14)
ϕ	= spin speed angular displacement of the torsionally rigid rotor, measured from the positive X-axis, rad

NOTE 1: Group dimension of K_{Bik} $\left[\frac{H_{Bik}^{-1}}{(x_{Bi}^2 + y_{Bi}^2)^2} + D_{Bik} + \frac{E_{Bik}}{x_{Bi}^2 + y_{Bi}^2} \right]$
is lb/in.

NOTE 2: Group dimension of $(\dot{\phi} - \phi_0)$ $N_{Bik} + B_{Bik} (\sqrt{x_{Bi}^2 + y_{Bi}^2} - \rho_{Bik})$
is the same as that of K_{Bik} .

ϕ_i = initial misalignment between the axis of the mass moments of inertia and the elastic axis of rotor section i , radians

γ_i = angular position of the X-Y plane projection of the axis of i th mass moments of inertia measured from that of the first, radians

ρ_{Bik} = lower bearing displacement limit for k th bearing stiffness section of bearing i , in

ρ_{Bxi}, ρ_{Byi} = as defined by Eqs. (13) and (15), respectively, in.

Subscripts

i = pertinent to the i th station or i th rotor section

$bl, b2$ = pertinent to first and last nonlinear bearing stations respectively

Time Derivative Notation

$\dot{x}, \ddot{x} \dots = \frac{dx}{dt}, \frac{d^2x}{dt^2} \dots$ respectively, in/sec, in/sec²

$\dot{\phi}, \ddot{\phi} = \frac{d\phi}{dt}, \frac{d^2\phi}{dt^2}$, respectively, rad/sec, rad/sec²

VERIFICATION OF THE COMPUTER PROGRAM

After debugging the computer program three stages of verification of the program were made. The detailed description of each stage of verification follows:

Verification of the Basic Mathematical Formulation

As a preliminary checkout of the "Fortran" coding of the mathematical formulation, a 5-station flexible rotor-bearing configuration, operating under a steady-state condition was chosen. Hand computed, initial rotor motion conditions simulating a steady-state operation were used. The hand calculation was based on the equilibrium between the centrifugal loading and bearing restoring forces. The computer result in rotating coordinates indicated a very small residual acceleration on the order of 10^{-9} of the magnitude of the centrifugal acceleration.

The residual net acceleration, not being zero, is due to input data not having a sufficient number of significant figures and computation round-off errors. The comparative results are shown in Table 1.

Verification of the Computation Results

The computation results of a 17-station rotor model operating under a steady-state condition were compared with that from an existing rotor-dynamic response program based on a matrix iteration technique and they were found in good agreement within a maximum of 1.5% deviation for the case studied. Two different approaches in mathematical formulations are used in these two types of computer programs. The accuracy of the flexible rotor dynamic computer program is considered to be adequate. Table 2 lists the comparative results.

Evaluation of the Accuracy and Convergence of the Computation Process

Using a 5-station rotor test model with a spin speed of 95.5 rpm, a computation with many integration steps was performed. A reasonably large

TABLE 1

5-STATION ROTOR-BEARING CONFIGURATION CHECKOUT
COMPARATIVE RESULTS

Station No.	1	2	3	4	5
<u>DISPLACEMENT VECTORS</u>					
Manually Calculated	.0017355"	.0009999"	.00077360"	.0009999"	.0017355"
Computer Program	.0017356"	.001"	.00077365"	.001"	.0017356"
<u>COMPUTER* CALCULATED ACCELERATION IN ROTATING COORDINATES</u>					
Radial	-3.716×10^{-9} in/sec	3.440×10^{-9}	$.1986 \times 10^{-9}$	3.440×10^{-9}	-3.716×10^{-9}
Tangential	8.116×10^{-16}	$+1.970 \times 10^{-14}$	1.315×10^{-14}	-2.503×10^{-14}	8.116×10^{-16}

* Predicted Acceleration = zero.

TABLE 2

COMPARISON OF MARK 25, 17 MASS MODEL
NEW PROGRAM VS OLD PROGRAM

Station	Deflection at Station $\times 10^{-5}$		% Difference
	New Transient Program	Old Steady-State Response Program	
1 (Inducer) End	.3554	.3548	.17
2	.3145	.3140	.16
3	.2028	.2023	.25
4	.1419	.1415	.28
5	.0845	.08414	.43
6	.04133	.04098	.85
7	.02310	.02276	1.50
8	.05076	.05099	.45
9	.1377	.1377	0
10	.1908	.1907	.05
11	.2364	.2363	.04
12	.3302	.3300	.06
13	.3517	.3514	.09
14	.3963	.3960	.08
15	.4159	.4156	.07
16	.5093	.5090	.06
17	.9782	.9784	.02

integration tolerance of 10% was used. The result in rotor displacement appears to stay close to the exact value. The instantaneous whirl to spin frequency ratio indicated a substantial oscillation at times, although the trend of convergence appeared definite. With a smaller integration tolerance as is usual (less than 10%), the magnitude of oscillations should be reduced accordingly. The integrated whirl displacement versus time function (not shown here) was reasonably smooth. Two graphs showing the variation of a deflection vector and whirl-to-spin frequency ratio are attached for reference (Figs. 4 and 5).

As evident from Fig. 4 and 5, the instantaneous whirl to spin frequency ratio appears to be coupled to the rotor displacement motion. The energy variation in the displacement motion is complemented by that in whirl motion through the mechanism similar to that of Coriolis acceleration. Because of the radius of whirl motion being rather small the angular velocity change is greatly magnified for a small change in rotary kinetic energy. Since the computer program for this rotor configuration was not run for a sufficient time period, it cannot be certain whether the relatively large oscillations between 0.12 and 0.14 second of real time were manifestation of numerical instability. It, however, does not appear likely to be the case.

The minimum DT (Delta Time) to avoid instability depends on the rotor mass-to-stiffness parameter ratio and no definite such relationships have yet been established.

Rotor Displacement at Station #3, Inches $\times 10^{-5}$

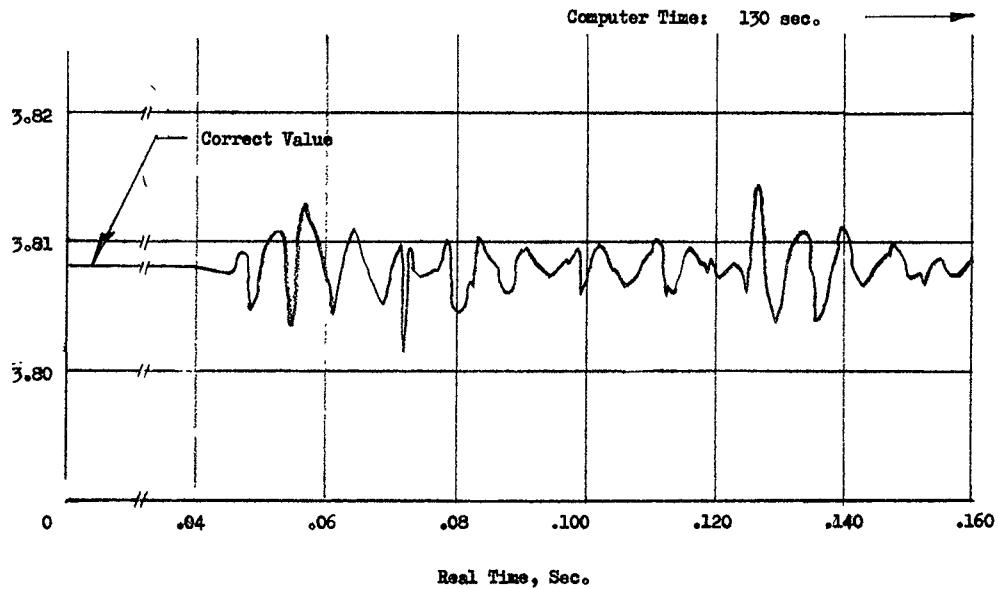


Figure 4 Data from a 5-Mass 95.5 rpm Flexible Rotor
in a Steady-State Operation

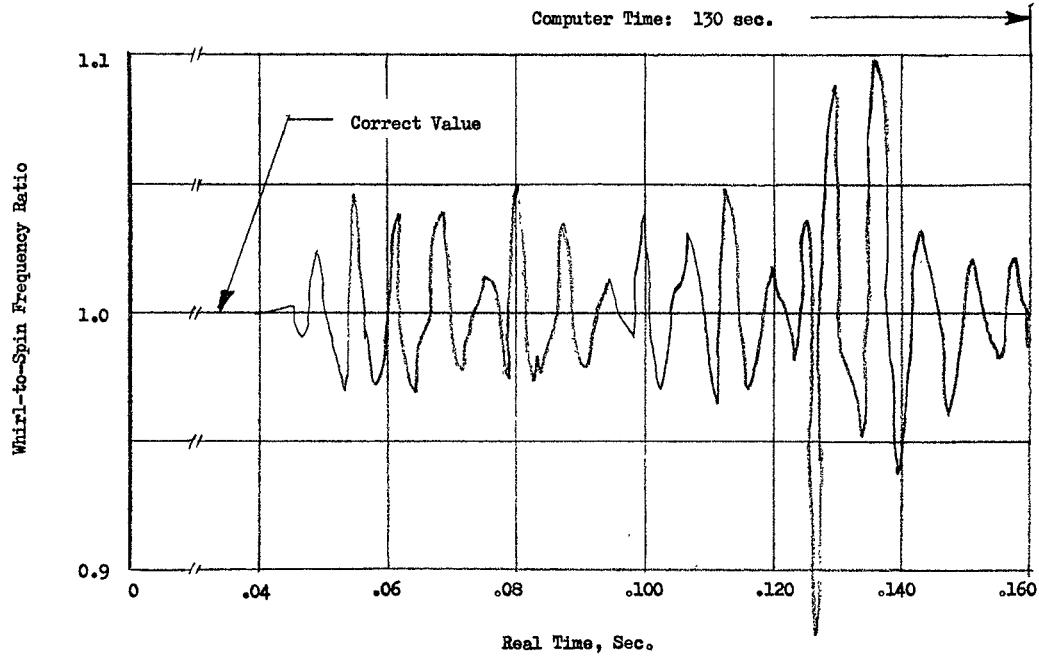


Figure 5 Data From a 5-Mass 95.5 rpm Flexible Rotor
in a Steady-State Operation

CAPABILITY AND LIMITATIONS OF THE CURRENT COMPUTER PROGRAM

Capability of the Computer Program

The Computer Program was written to include effects of:

1. Rotor elasticity in shear and bending
2. A multiplicity of non-linear stiffness bearings each in turn supported on a non-isotropic stiffness and damping characteristic mount
3. General in-phase and out-of-phase stiffness and damping functions
4. Mass and mass moments of inertia unbalance vectors
5. Drive torque and dissipation functions

The application of the program may be illustrated as follows:

1. Transient spin speed analysis through critical speed range using speed sensitive drive torque and/or dissipation functions
2. Various whirl-to-spin frequency mode simulations in transient, steady-state or quasi (cyclic) steady-state operations
3. Study of the effects of rotor transverse gravity or constant acceleration
4. Investigation of the effects of general fluid dynamic or some electromagnetic excitation function with in-phase or out-of-phase, whirl and/or spin velocity sensitive, stiffness and/or damping functions.
5. Non-linear bearing stiffness effects
6. Study the effects of shear-and/or bending-wise flexible couplings on rotordynamic performance
7. Analysis of a general rotor motion in a non-axisymmetric and non-steady state operation.

Limitations

Several major assumptions in the mathematical formulation of the simplified computer program were made. The assumptions and resulting limitations are itemized as follows:

Torsional Rigid Rotor Assumed. For most rotor design properties this assumption appears to be realistic. Deviations from the real solution would occur, only when substantial torsional oscillations coupled with transverse rotor motion are encountered. A multiple-rotor system with small dissipation coefficient and having pulsating drive torque of a frequency at, or near, a torsional critical speed of the rotor would be the case that a torsionally flexible rotor model would provide more accurate results.

Rotor Casing Parameter not Considered. For rotors with large casing-to-rotor mass ratio or reasonably rigid attachment between the casing and the foundation, the effects of the assumption are minimal.

Rotor Hysteretic Damping Including the Effects of Bolted or Press-fit Rotor Joints not Considered. This assumption does not materially affect the computation results unless an elastically bent rotor of substantial hysteretic property operates under a non-synchronous whirl condition, or under a substantial gravity or transverse acceleration loading.

COMPUTER PROGRAM USER'S INSTRUCTIONS

DESCRIPTION OF THE COMPUTER PROGRAM

The Simplified, Flexible Rotor, Transient-Speed Rotordynamics Analysis Computer program consists of a MAIN program, 12 subroutines and a major library subroutine (ISIMDD). The program is written for IBM 360, Model 65 and Fortran H compiler language. The CRT plotting language is to be compatible with that available at NASA Lewis Research Center, Cleveland, Ohio*. A brief description of the functions of the MAIN program and subroutines and a general flow diagram, Figure 6, is provided, as a guide for the use of the program.

*CRT Plotting Reference, Robert G. Kannenberg, Lewis Research Center, Cleveland, Ohio: "Cinematic-Fortran Subprograms for Automatic Computer Microfilm Plotting, NASA TM X-1866" November 1969.

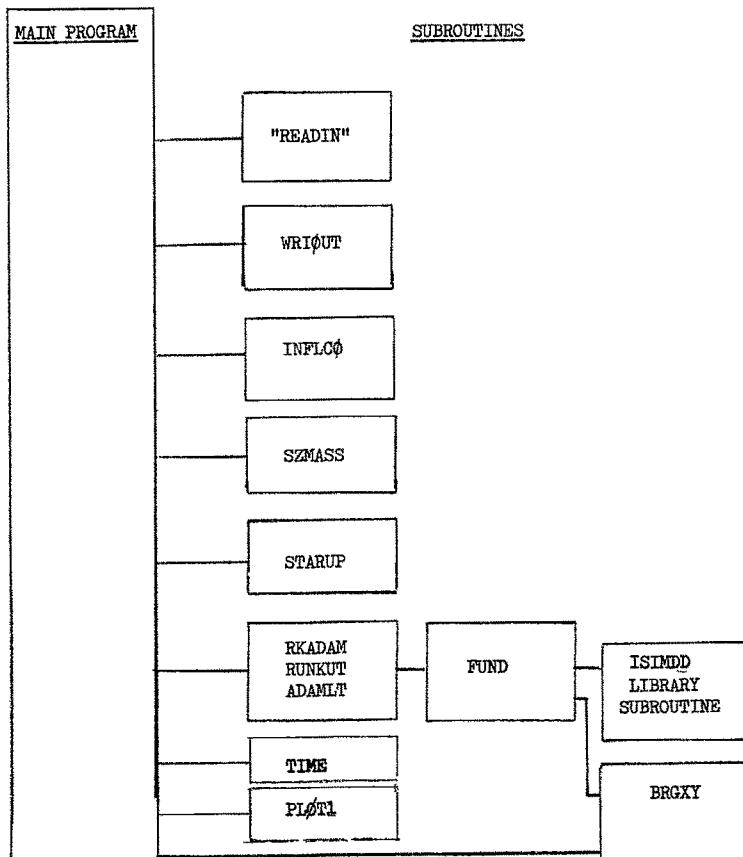


Figure 6 General Flow Diagram Relating Various Subroutines and the MAIN Program

<u>PROGRAM OR SUBROUTINE NAME</u>	<u>FUNCTION</u>
MAIN	<p>This is the basic controlling program which originates the basic "call" statement. It also provides output write-out and CRT generating procedures. The rotor motion in stationary coordinates is converted into rotating coordinates before transmission to the integration subroutines. Similarly, the results of integration are first converted back to stationary coordinates before they are used in print-outs and CRT graphs.</p>
READIN	<p>This function is basically to *read input other than providing built-in average data values. "READIN" is called by "MAIN."</p> <ol style="list-style-type: none"> 1. 2-card job descriptive title 2. NAMELIST/DATA 1/, the data which must be input 3. NAMELIST/DATA 2/, optional, when the input data in the data 2 group are not provided, the built-in average data values will be used. 4. Read punched cards from previous analysis for continued study. The read-in of the punch cards becomes effective when C₀NTIN = 1 is specified in data as opposed to C₀NTIN = 0 condition.

* Detail READIN information will be discussed in subsequent section.

<u>PROGRAM OR SUBROUTINE NAME</u>	<u>FUNCTIONS</u>
WR10UT	<p>This is to print out all the input data in groups according to their uses. Descriptive definitions and dimensional units are provided for each data set. "WR10UT" is called by "MAIN."</p>
STARUP	<p>This subroutine is to generate accurate initial rotor motion conditions for starting computations in the basic program. Using the "STARUP" subroutine, computations in the basic program can proceed in an efficient manner. The present "STARUP" is in its simplified version; it includes only linear bearing and rotor excitation stiffness functions. Torque, damping and out-of-phase force parameters are not included. It is in a tentative status subject to updating to include all leftover parameters. "STARUP" is a steady-state subroutine and is called by "MAIN."</p>
FUND	<p>This subroutine provides Time derivatives required by the integration subroutines "RKADAM," "RUNKUT," and ADAMLT." To compute the derivatives, "FUND" obtains input data through "READIN," influence coefficient data from "INFLCϕ," and bearing data from "BRGXY."</p>

<u>PROGRAM OR SUBROUTINE NAME</u>	<u>FUNCTIONS</u>
RKADAM, RUNKUT, & ADAMLT	<p>These integration subroutines are based on an adjustable step, predictor-corrector, Adam-Moulton integration procedure. The starting points are established through forward difference Runge-Kutta approach although option in backward difference procedure is also provided. "TOLI" is an input data name which is used to regulate the computation tolerance in integration accuracy as may be desired. Higher accuracy may be obtained with smaller "TOLI" value such as .001 or .0001. Also, higher accuracy means longer computation time. Thus, an optimum balance between the degree of accuracy and computation time should be made to suit the circumstances.</p> <p>"RKADAM," "RUNKUT" and "ADAMLT" are called by "MAIN."</p>
INFLC \emptyset	<p>This generates the required influence coefficients considering both bending and shear elasticity of the rotor. It is called by "MAIN" and the influence coefficients are also used by "STARUP" and "FUND."</p>
ISIMDD	<p>This is a simultaneous linear equation solution library subroutine which computes individual derivatives required by the integration subroutines. It is called by "FUND."</p>

<u>PROGRAM OR SUBROUTINE NAME</u>	<u>FUNCTIONS</u>
SZMASS	This is a subroutine which computes the rotor mass, transverse and polar moments of inertia, and rotor length coordinates. It is called by "MAIN."
BRGXY	This is a subroutine which computes journal displacements from their respective zero-load position The computations appropriately consider the nonlinear stiffness characteristics of the bearings, and non-isotropic mount force characteristics. "BRGXY" is called by "MAIN" AND "FUND."
TIME	This is to furnish real time for the iteration process from the tentative "real time" output of the integration subroutines.
PILOT1	It is to provide multiple bearing data for CRT plotting. It is used to generate journal deflections and bearing forces for a maximum number of 12 bearings. It is called by "MAIN."

INPUT PROCEDURE

In using the computer program a continuous rotor configuration is to be simulated by a discrete-mass rotor with appropriate massless elastic members in between adjacent rotor masses. In general, the minimum number of discrete masses used should be such that the rotordynamic mode shape can be possibly sustained. For instance, if a shaft operates in its third critical speed range the mode shape for a low stiffness bearing would be ; to sustain this mode a minimum of 5 masses is required. To obtain good accuracy in general, several times the minimum number of mass requirements are used. In rotor motion predominately influenced by mass eccentricity, damping, and stiffness function, the mode shape in a critical speed range may be substantially modified from that of a pure critical speed mode shape. Judgement must hence be exercised in selecting the number of discrete masses in representing a rotor configuration.

The rotor to be studied is first divided into consecutively numbered stations. The total number of stations may vary from 5 to 25, inclusively. Rotor sections between adjacent rotor stations are labeled with the same numbers as that of the left adjacent stations. The rotor property input data are appropriately subscripted according to the rotor station or section numbers. For non-linear stiffness bearing data two-dimensional subscripts are used. The first subscripts define their rotor station location and the second define the non-linear bearing stiffness sections.

The input data is printed prior to the writing of computational results

A detailed description of the input, output and usage of the program appears

in the following sections. To supplement the input procedure description, a sample input code sheet is shown in Figure 7.

Input

A namelist input procedure is adopted as the major input format due to its flexibility in selecting input parameters and the liberal use of built-in average values when desired. For preliminary analysis, by making use of built-in data, the input data volume can be drastically reduced.

The complete input listing is as follows:

1. Read Title. Two consecutively located 80-character spaced cards must be used for job description title. The title description may use the first 72 spaces of each card and 73 through 80th space may be used for card identification number only.
2. NAMELIST/DATA 1/. The data group consists of key input data which must all be read in Namelist format. This sequence of read-in data is immaterial. Because the Data 1 parameters pertain to the rotor geometry and other essential descriptive information, average-value data cannot be built into the program. The physical input of these parameters is necessary. The names and their definitions contained in the NAMELIST/DATA 1/ are on page 44.
3. Name List/Data 2/. The majority of the input data is included in the "Data 2" namelist. All parameters contained in "Data 2" are optional input data; i.e., they need not be inputed if the built-in average values are adequate. However, a card with the name Data 2 is necessary even if the data field is empty.

The input names for Data 2, their definitions, and the built-in values are on page 45.

4. Reading Punched Cards. If continuation of a previous analysis is desired specify "CONTIN=1" and attach the cards providing starting data from a previous computation.

CAUTION: If K is greater than 1 for one or more non-linear stiffness bearings, the input of double-subscripted $BR\phi(I,K)$ for appropriate K values must be input. Where I is the rotor station for the non-linear bearing, K varies from 1 through K for appropriate non-linear bearings. The input of $BR\phi_B(I,1)$ would be necessary when a different than built-in value of 0.005 inch is desired.

NAMELIST/ DATA 1/

Name Used In The Computer Program	Equivalent Name in The Mathe- matical Model	Stored Value	Description
NS	N		Number of rotor stations (allowable range: $5 \leq NS \leq 25$).
TMAX			Total real time to be run, sec.
TST δ P			The computer time allowed for each set of data, minutes.
DD(NS-1)*			Outside diameters of rotor sections between adjacent rotor stations, in.
QL(NS-1)	l_i		Rotor section lengths between adjacent rotor stations, in.
NB			Number of non-linear stiffness bearings. (Allowable range: $2 \leq NB \leq 12$)
IB(NB)			Rotor station numbers of non-linear stiffness bearings
K(NB)			Total number of stiffness sections for each of the non-linear stiffness bearings. (Allowable range: $1 \leq K \leq 6$)
ICOND			ICOND=1 Means starting a new rotor-bearing configuration. ICOND=0 Means read-in alternate initial conditions for the same rotor-bearing configuration.
C δ NTIN			C δ NTIN=0 Means starting a new rotor dynamics analysis with initial conditions provided by the startup subroutine C δ NTIN=1 Means continuation of a previous analysis by using the previous results on punched cards as the initial conditions.
FD δ T			Initial rotor spin frequency, rpm.
WHIVEL			Initial rotor whirl frequency, rpm.

* (NS), (I) or (I, J) after a name specifies the name being an array of one or two dimension. The values of the indexes specify the current size of the array.

NAMELIST/ DATA 2/

<u>Name Used In The Computer Program</u>	<u>Equivalent Name in The Mathe- matical Model</u>	<u>Stored Value</u>	<u>Description</u>
T		0	Initial real time, sec.
DT		.00001	Estimated initial integration step (real) time, sec. The computer program may modify the value of DT to suit tolerance requirements.
FD		0	Rotor spin angular displacement coordinate, degrees, (This value is added to data).
IASIGN		1	Rotor station at which whirl/spin frequency ratio will be plotted on CRT.
NP0RPM		1	The number of spin speeds in rpm at or near which 3-dimensional absolute rotor mode shape CRT graphs are required. The spin speed rpm values are listed under INPRPM array. (Allowable range: $0 \leq NP0RPM \leq 50$).
NP0INT		25	The number of points (one per integration step) for each CRT graph. (Allowable range: $1 \leq NP0INT \leq 50$). It applies to all CRT graphs except the 3-dimensional rotor mode shape plot.
CRT		0	CRT=0 Means CRT is not required, CRT=1 Means CRT is required.
ACCEL		0	ACCEL=0 Means a 3-dimensional steady-state rotor mode shape CRT would be provided if concurrently CRT=1. ACCEL=1 Means the steady-state rotor mode shape will not be provided.
INPRPM(NP0RPM)		0	The rotor spin speed rpm values at or near which CRT graphs for 3-dimensional absolute rotor mode shapes are required.
T0LI		.01	Integration tolerance, fraction.
T0LB		.001	Tolerance in computing bearing displacements, fraction.
GX	g_x	0	Gravity or G-loading in X direction, in/sec ² .

NAMELIST/DATA 2/

Name Used In The Computer Program	Equivalent Name in The Mathe- matical Mode	Stored Value	Description
GY	g_y	0	Gravity, or G-loading in Y direction, in/sec ² .
TMZ	M_z	0	Exponent for speed sensitive rotor drive torque
TMZ1	M_{z1}	0	Coefficient for rotor drive torque
TMZ2	M_{z2}	0	Coefficient for rotor drive torque, (in-lb-sec)/rad
TMZ3	M_{z3}	0	Coefficient for rotor drive torque, in/lb.
D(NS-1)		0	Inside diameters of rotor sections between adjacent rotor stations, in.
DN(NS-1)		.283	Material densities of rotor sections between adjacent stations, lb/in ³ .
EE(NS-1)		3×10^7	Young's moduli of rotor sections between adjacent stations, lb/in ² .
GG(NS-1)		1.15×10^7	Shear moduli of rotor sections between adjacent stations, lb/in ² .
EI(NS-1)		0	Direct input of the products of Young's moduli and area moments of inertia of rotor sections between adjacent stations, lb-in ² .
*GAK(NS-1)		0	Direct input of the products of shear moduli, cross-sectional areas and reciprocals of shear stress concentration factors between adjacent stations, lb.
AM(NS)		1×10^{-16}	Additional rotor masses at rotor stations, (lb-sec ²)/in.
AID(NS)		0	Additional rotor transverse mass moments of inertia at rotor stations, (lb-in-sec ²).
AIRφ(NS)		1×10^{-16}	Additional rotor polar mass moments of inertia at rotor stations, (lb-in-sec ²).

* GAK stands for (GG)(Rotor Cross-section Shear Area), where K denotes

$$\text{Maximum to average shear stress ratio expressible as: } \frac{4}{3} \frac{[(DD)^3 - D^3]}{[(DD)^2 + D^2](DD - D)}$$

NAMELIST/DATA 2/

<u>Name Used In the Computer Program</u>	<u>Equivalent Name in The Mathe- matical Model</u>	<u>Stored Value</u>	<u>Description</u>
ECC(NS)	e_i	1×10^{-16}	Rotor mass eccentricities at rotor stations, in.
ALFA(NS)	α_i	0	Phase angles for rotor mass eccentricity vectors at rotor stations measured from that of the first rotor station, degrees.
BETA(NS)	β_i	0	Initial misalignments between the axes of the mass moments of inertia and the elastic axes at rotor stations, degrees.
GAMMA(NS)	γ_i	0	Angular positions of the X-Y plane projections of the axes of mass moments of inertia at rotor stations measured from that at the first rotor station, degrees.
CZ(NS)	C_{Zi}	0	Torsional friction exponents at rotor stations, dimensionless.
CZ1(NS)	C_{Z1i}	0	Torsional friction coefficients at rotor stations, dimension of $C_{Z1i} (FD\theta T)C_{Zi}$ in-lb.
CZ2(NS)	C_{Z2i}	0	Torsional friction coefficients at rotor stations, (in-lb-sec)/rad.
XKF(NS)	$K_{\phi i}$	0	Whirl-frequency factors for stiffness force coefficients at rotor stations, dimensionless.
XCF(NS)	$C_{\phi i}$	0	Whirl-frequency factors for damping force coefficients at rotor stations, dimensionless.
XKFF(NS)	$K_{\phi Fi}$	0	Whirl-frequency factors for stiffness moment coefficients at rotor stations.
XCF(NS)	$C_{\phi Fi}$	0	Whirl-frequency factors for damping moment coefficients at rotor stations, dimensionless.
QK(NS)	K_i	0	In-phase stiffness force coefficients at rotor stations, lb/in.
QC(NS)	C_i	0	In-phase damping force coefficients at rotor stations, (lb-sec)/in.

NAMELIST/DATA 2/

Name Used In The Computer Program	Equivalent Name in The Mathe- matical Model	Stored Value	Description
QKP(NS)	K_{Pi}	0	Out-of-phase stiffness force coefficients at rotor stations, lb/in.
QCP(NS)	C_{Pi}	0	Out-of-phase damping force coefficients at rotor stations, (lb-sec)/in.
QKHD(NS)	K_{HDi}	0	Out-of-phase whirl and spin velocity sensitive stiffness force coefficients at rotor stations, (lb-sec)/(in-rad).
QCHD(NS)	C_{HDi}	0	Out-of-phase whirl and spin velocity sensitive damping force coefficients at rotor stations, (lb-sec ²)/(in-rad).
QKF(NS)	K_{Fi}	0	In-phase stiffness moment coefficients at rotor stations, in-lb.
QCF(NS)	C_{Fi}	0	In-phase damping moment coefficients at rotor stations, in-lb-sec.
QKPF(NS)	$K_{\phi Pi}$	0	Out-of-phase stiffness moment coefficients at rotor stations, in-lb.
QCPF(NS)	$C_{\phi Pi}$	0	Out-of-phase damping moment coefficients at rotor stations, in-lb-sec.
QKHDF(NS)	$K_{\phi HDi}$	0	Out-of-phase whirl and spin velocity sensitive stiffness moment coefficients at rotor stations, (lb-in-sec)/rad.
QCHDF(NS)	$C_{\phi HDi}$	0	Out-of-phase whirl and spin velocity sensitive damping moment coefficients at rotor stations, (lb-in-sec ²)/rad.
BKMX(NB)	K_{mx1}	10^{10}	Non-isotropic mount stiffness coefficients in X and Y-direction, respectively, for non-linear stiffness bearings, lb/in.
BKMY(NB)	K_{my1}		
BCM _X (NB)	C_{mx1}	1×10^{-16}	Non-isotropic mount damping coefficients in X-direction for non-linear stiffness bearings, (lb-sec)/in.
BCM _Y (NB)	C_{my1}	1×10^{-16}	Non-isotropic mount damping coefficients in Y-direction for non-linear stiffness bearings, (lb-sec)/in.
BCB(NB)	C_{Bi}	1×10^{-16}	Bearing damping coefficients for non-linear stiffness bearings, (lb-sec)/in.

Name Used In The Computer Program	Equivalent Name in The Mathe- matical Model	Stored Value	Description
*FD \emptyset FIX	\emptyset_o	0	A bearing stiffness speed parameter, rpm
*BNB(NBK)	B_{NLK}	0	Rotor spin-speed sensitive bearing stiffness coefficients for K stiffness sections of non-linear stiffness bearing.
*BBB(NB, K)	B_{BiK}	0	Rotor spin-speed sensitive bearing stiffness coefficients for K stiffness sections of bearings for non-linear stiffness bearing.
*BR \emptyset B(NB, K)		.005 (for the 1st stiffness sec- tions of all bearings)	Upper bearing-displacement limits for K stiffness sections of bearing, in.
*BKB(NB, K)	K_{BiK}	1×10^6	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.
*BHB(NB, K)	H_{BiK}	1	Non-linear bearing stiffness exponents for K stiffness sections of bearing.
*BDB(NB, K)	D_{BiK}	0	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.
*BEB(NB, K)	E_{BiK}	0	Non-linear bearing stiffness coefficients for K stiffness sections of bearing.

*See equations on p. 60

FORTRAN FIXED 10 DIGIT DECIMAL DATA

DECK NO.

PROGRAMMER

DATE

PAGE ____ OF ____ JOB NO. ____

NUMBER		DESCRIPTION	
1	A	C, H, E, C, K, 0, U, T, , R	
13	0	T, O, R, C, O, N, F, I, G, U	
25	R	A, T, I, O, N, C, O, N, S, I	
37	S	T, S, 0, F, 5, S, T, A	
49	T	I, O, N, S, A, N, D, 5	
61	M	A, S, S, E, S, +, R, O, T, 0	
IDENTIFICATION		73	80
1	R	M, A, S, S, U, N, B, A, L	
13	A	N, C, E, S, I, N, C, L, U, D	
25	E	D, ., B, E, A, R, I, N, G, S	
37	A	R, E, A, T, , S, T, A, T	
49	I	O, N, S, 1, ., 5, ., A	
61	P	R, I, L, /, 2, 7, /, 1, 9, 7, 0	
IDENTIFICATION		73	80
1	&	D, A, T, A, 1, N, S, =, 5	
13	,	T, M, A, X, =, 1, , T, S, T, 0	
25	R	=, 1, , D, D, =, 4, *, 1, , Q	
37	L	=, 4, *, 1, , N, B, =, 2, , I	
49	B	=, 1, , 5, , I, C, O, N, D, =	
61	I	, C, O, N, T, I, N, =, 0, , F	
IDENTIFICATION		73	D, O, T, =, 1, 0, 0, , 80
1	W	H, I, V, E, L, =, 1, 0, 0, ,	
13	K	(, 1,), =, 1, , K(, 5,), =	
25	I	, &, E, N, D,	
37			
49			
61			
IDENTIFICATION		73	80

NUMBER		DESCRIPTION	
1	&, D, A, T, A, 2, C, R, T, =		
13	I	, A, C, C, E, L, =, 1, , I, N	
25	P	R, P, M, =, 2, , &, E, N, D	
37			
49			
61			
IDENTIFICATION		73	80
1	/	*	
13			
25			
37			
49			
61			
IDENTIFICATION		73	80
1			
13			
25			
37			
49			
61			
IDENTIFICATION		73	80
1			
13			
25			
37			
49			
61			
IDENTIFICATION		73	80
1			
13			
25			
37			
49			
61			
IDENTIFICATION		73	80

4. Read Punch-Card Input (Optional). The program has provision for continuing a previous rotordynamics analysis by using the results stored in punched cards from the previous analysis.

Double precision rotating coordinate rotor displacement and velocity values are stored in the punched cards. Each card contains 3 numerical values and a right adjusted card sequence identification number, beginning with 1. For this optional input, these cards must be placed consecutively and immediately after the regular input data of the first round, as shown below:

1. Title 2 cards
2. DATA 1/ which includes "C \emptyset NTIN=1"
and "IC \emptyset ND=1"
3. DATA 2/, optional
4. Punched Data Cards from previous computations when C \emptyset NTIN=1, and not to be used if C \emptyset NTIN=0
- Data for alternate initial conditions on same rotor. Do not include IC \emptyset ND data. (Includes data groups 1, 2, 3, and 4 as appropriate.)
- Data for a new rotor configuration must include IC \emptyset ND=1. (Includes data groups 1, 2, 3, and 4 as appropriate.)
- /* End Card

When read-in punched cards are desired "C \emptyset NTIN=1" must be included in /DATA 1/ located prior to the punched cards. When punched card read-in is effected, the regular, initial condition computation by "STARUP" will be by-passed.

Special Notes on Input

1. Alternate Provisions. In the input procedure there are two alternate input provisions. In certain rotor designs the flexibility of rotor may not be derived directly from the diameter and elastic modulus considerations, such as complex bolted rotor sections. In such circumstances, an optional combined EI or GAK may be more conveniently used in place of EE, DD, D or GG and shear stress concentration factor.

EI is the product of Young's modulus of elasticity and sectional moment of inertia, GAK is the product of shear elastic modulus, rotor cross-sectional area and reciprocal of maximum to average shear stress ratios, where

EE = Young's modulus of elasticity, psi

GG = = Shear modulus of elasticity, psi

DD = Outside rotor diameter, in

D = Inside rotor diameter, in

In using the option input, EI, EE set or GAK, GG set, only values of one of the two sets can be assigned. No values are to be assigned to the other set.

The built-in average values of EE and GG for all rotor sections are:

$$EE (I) = 3 \times 10^7$$

$$GG (I) = 1.5 \times 10^7$$

It should be noted that the option, EI or GAK, may be used for any rotor sections desired, and need not be used for all rotor sections.

2. Analysis with Various Alternate Operating Parameters. The program has the provision to analyze the effect of various sets of operating parameters on the performance of a given rotor-bearing

configuration. This can be accomplished by inputting; first, a complete data set including the rotor design and the initial set of values of operating parameters including "ICOND=1" in Data 1/Name List; the second set of data for different values of certain parameters, but not including "ICOND=1", may be input without repeating the same rotor design information. This procedure may be continued indefinitely until the analysis of a new rotor design is initiated. With the new rotor design data, "ICOND=1" must be included in the first data input set.

The rotor-bearing configuration, referred to above, is defined as:

- a. Rotor design, geometry and related physical properties which include the number of rotor stations (NS), transverse rotor gravity or accelerations (GX, GY), rotor diameters, length, density, elasticity, mass, and mass moments of inertia.
- b. Bearing locations and number of bearings used.

The parameters other than that of rotor-bearing configurations may be repeated with different numerical values for a given rotor-bearing configuration. They are listed as:

- a. Initial rotor motion conditions.
- b. Bearing and mount damping and stiffness characteristics, number of non-linear stiffness sections.
- c. Unbalance parameters for rotor masses and mass moments of inertia.
- d. Rotor drive torque and torsional damping parameters.
- e. All rotor in-phase and out-of-phase stiffness and damping parameters.

Namelist Input Procedure and Sample Input Data Sheet

According to the current North American Rockwell Corporation practice compatible with IBM 360 Fortran IV, compiler H language, the namelist input format is described in the following five pages.

A sample of a completed input data sheet is given in Appendix A.

NAMELIST INPUT-OUTPUT

Namelist offers a quick and relatively easy way of coding input-output. The programmer writes a NAMELIST statement, including one or more lists of variables and/or arrays he wishes to read in or print out, and assigns a name to each list. Thereafter, instead of rewriting an entire list, he references the list name in READ and WRITE statements.

On input, the number of items to be read and the place each one is to be stored is governed to a large extent by the input data cards. Each item is written in a format much like an arithmetic statement, with each variable explicitly named and set equal to the desired value. A few items or many can be read by the same READ statement, and the order of writing the data is unimportant.

On output, the members of the list will be printed in a standard format. The appearance of the printout is not suitable for formal reports, but is satisfactory as a substitute for a partial dump in error branches, for printout of input data, for intermediate results during checkout, or in any other case in which answers are needed and the format is not important.

TRANSMITTAL STATEMENTS

General Form:

READ (a, b, END = c, ERR = d)

WRITE (a, b)

Where: a is the data set reference number.

b is the Namelist name.

See Section 204.2, Sequential Input-Output, for a complete description of these statements.

Note that Namelist cannot be used with a PUNCH statement. However, the statement

WRITE (14, b)

can be used to produce punched output.

NAMELIST STATEMENT

General Form:

NAMELIST /x/ a, b, ..., c /y/ d, e, ..., f
/z/ g, h, ..., i

Where: x, y, and z are Namelist names.
a, b, c, ... are variable or array names.

The following rules apply to construction of a NAMELIST statement.

1. A Namelist name consists of from 1 through 6 alphanumeric characters, the first of which is alphabetic. The name is enclosed in slashes.
2. A Namelist name may be defined only once by its appearance in a NAMELIST statement, and it must be so defined before its use. After it is defined in the NAMELIST statement, the Namelist name may appear only in input or output statements thereafter in the program.
3. A NAMELIST name cannot be transmitted as an argument from a calling program to a subprogram. In a subprogram, dummy arguments cannot appear in a Namelist list.
4. The list of variable and array names belonging to a Namelist name ends with a new Namelist name (enclosed in slashes) or with the end of the NAMELIST statement.
5. A variable or array name may belong to one or more Namelist names. For example:

DIMENSION A(10), I(5), L(6)
NAMELIST /NAM1/ A, D, I, K, L
/NAM2/ A, C, K, M

means that the arrays A, I, and L, and the variables D and K are included in the list for NAM1, and the array A and the variables C, K, and M are associated with the Namelist name, NAM2.

6. The specification statement(s) defining the type, size, and relative storage locations of arrays and variables must precede any NAMELIST statement in which those arrays and variables are mentioned.

INPUT DATA

The input data must be in a special form in order to be read with a Namelist read statement. Following are the published IBM specifications for the data:

1. The first character of each input record (card image) must be blank.
2. The second character in the first record (of a group of records) must be an &.
3. The Namelist name must start in column 3 of the first record of the group.
4. The Namelist name must be followed by a blank; it must not contain embedded blanks.
5. Input data items follow the Namelist name and are separated by commas. There must not be embedded blanks in the variable names, array names, or constants.
6. Input data items can be continued on succeeding records. A comma after the last item of data on a record is optional. Each succeeding record must begin with a complete variable name, array name, or constant (not with an equals sign).
7. The end of a group of data items is signaled by &END, anywhere in a record except in the first character position.
8. Constants in the data items may take any of the following forms:
 - a. Integer
 - b. Real
 - c. Complex
 - d. Logical
 - e. Literal (input only)

Constants are written exactly as in a source program, except that logical constants may be written in the form T or F as well as .TRUE. or .FALSE..

9. The form of the data items may be:

Variable name = single constant

The variable name must be one of the names in the Namelist list. It may be a single variable name or it may be subscripted.

Array name = set of constants

The array name must be one of the names in the Namelist list.

The set of constants may be separated by commas, or may be in the form "k*constant" where k is an unsigned integer used to represent k constants.

If the array name is not subscripted, the first of the set of constants will be stored in the first element of the array, with subsequent constants stored in consecutive elements.

The number of constants must be less than or equal to the number of elements in the array.

10. If an item that appears in a Namelist list also appears in an EQUIVALENCE statement, only the name from the list may appear on an input record.

The following items are comments on the current IBM Namelist implementation (Release 15/16). As such, they are subject to change in future releases.

1. The first character of any input record is ignored.
2. Embedded and trailing blanks in a constant or exponent are treated as zeros. For example:

<u>Input Data Item</u>	<u>Is Interpreted As</u>
------------------------	--------------------------

I = 40,	I = 40,
J = 403,	J = 403,
E = 1.0E60,	E = 1.0E60,
K = ,	K = 0,

3. Hexadecimal constants, of the form Znnnnnnn, are accepted. (On output, they will be assigned the type of the name in the list.)
4. Literal constants, of the form 'aaaa', are accepted. (On output, they will be assigned the type of the name in the list.)

When using Namelist to read a literal constant into an array, all items in the array following the literal constant will be set to EBCDIC blanks. Thus, if there are three words in the array ARR, the input data item

ARR(1) = 'ABCD'

will store the characters ABCD in the first word and will fill the next two words with blanks. The input data item

ARR(1) = 'ABCDEFGHIJKL'

will fill all three words in the array with the characters shown.

5. The following form is permitted:

Subscripted array name = set of constants

When the array name is subscripted, the first item of the set of constants will be placed in the element indicated, with subsequent constants in consecutive elements.

6. The form

Array name = single constant

is interpreted to mean that the single constant is to be put in the first element of the array.

7. A repetition factor can be inserted anywhere in the list. Assume that the array TERSE has a dimension of 10. The input data items to fill this array could be:

TERSE = 4.0, 8*0.0, 6.0

The first and last elements of the array will be set to 4.0 and 6.0, respectively; the intervening elements will be set to 0.0.

8. If the size of an array is exceeded, the error messages appear irrelevant. Depending on the context, the following might be printed:

NAME NOT IN NAMELIST DICTIONARY
or
NAME LARGER THAN 8 CHARACTERS

9. If a real number is set equal to an integer, the decimal point is assumed to be at the end of the number.

<u>Input Data Item</u>	<u>Is Interpreted As</u>
REAL = 1234,	REAL = 1234.0,

10. If an integer is set equal to a real number, the decimal point is ignored.

<u>Input Data Item</u>	<u>Is Interpreted As</u>
INTGR = 2.345,	INTGR = 2345,

11. If an item is repeated in the group of Namelist input records, the last one read is used.

12. Since the last item on a card must be a data name or a constant, an identification field may not be punched in columns 73-80.

13. If EEE facilities (Region 226) are available, a CALL ERRSET statement will permit continuation of the program after Namelist errors are encountered. For example, the statement

CALL ERRSET (221, 20, 20, 0, 1, 224)

will permit up to 19 occurrences each of errors 221 through 224; the program will be terminated on the 20th error for any one of these error numbers. A message showing the card in error will be printed.

Note that the standard fixup ignores all remaining items in the group of input records. Not only the erroneous item, but all the following items in the group are ignored.

INPUT PROCEDURE

When a READ statement references a Namelist name, the input of data is begun. The first data card (or record) is read and examined to verify that its name is consistent with the Namelist name in the READ statement.

If the specified Namelist name is not found, additional records are examined consecutively until there is a successful match, or until all the data for the program are exhausted. (This provides a method for skipping undesired blocks of data in some branches of the program.)

Reading continues until an &END is encountered. Any information following the &END is ignored.

OUTPUT

When the arrangement of the printed page is not significant, Namelist relieves the programmer of the effort of setting up formats.

When a WRITE statement references a Namelist name:

1. All variables and arrays, and their values, belonging to the Namelist name will be written, each according to its type. Arrays are written in columnwise order.
2. The output data will be written such that:
 - a. The field for the data will be large enough to contain all significant digits.
 - b. The output can be read by an input statement referencing the Namelist name.

The PUNCH statement cannot be used with Namelist. However, a WRITE (14, nname) statement, where nname is a Namelist name, can be employed to produce punched output. A suitable DD statement is also required; one is provided in AFSLINK.

EXAMPLE

Assume that the program contains the following statements:

```
COMPLEX  COMPLEX (5)
REAL*8   DBLE (5)
REAL*4   REAL (10)
INTEGER   INTGR (10)
LOGICAL  LOGCL (20)

NAMELIST /INPUT/ REAL, INTGR,
           COMPLEX, DBLE, LOGCL
.
.
.
READ (5, INPUT)
.
```

Also assume the four data cards in the right column. (For the example, the input is written in the same order as the Namelist list. This is for convenience comparing the input to the output; the data need not be written in that order.)

The first record is read from the device associated with data set reference number 5 (normally the input stream). The record is searched for an & in column 2, immediately followed by the Namelist name, INPUT, and the required blank after the name.

Since the search is successful, the data items are converted and placed in core. If desired, only one or two of the variable names could have appeared in the input records.

The ten real constants will be placed in REAL(1) through REAL(10). The constant 2 will be placed in INTGR(1), the constant 6 will be stored in INTGR(2) through INTGR(9), and the constant 10 will be stored in INTGR(10).

The arrays COMPLEX and DBLE will be similarly filled as indicated. The first two items of the array LOGCL will be set to .TRUE.; the remainder of that array will be set to .FALSE..

INTGR(3), previously set to 6, will be reset to 9 by the last data item.

The &END signals termination of the input for this READ statement.

The example on the following page shows output produced by the statement:

```
WRITE (6, INPUT)
```

For the illustration, the output data are derived from the input shown below.

1	INPUT	
13	REAL = 1.1414,	
25	2.704, 3.945,	
37	85.31, 99.96,	
49	95.3, 333.E12	
61	, 1886E9,	
1	169017.E43,	
13	6627506,	
25	INTGR = 2,	
37	8*6, 10,	
49	COMPLEX = (1.0,	
61	2.0)	
1	4*(2.2986,	
13	4.77),	
25	DBLE = 1.04,	
37	1.5578552,	
49	1.41427854,	
61	.13534D06,	
1	.1484167D9,	
13	LOGCL = T, .TRU	
25	E., 18*F,	
37	INTGR(3) = 9,	
49		
61	\$END	

EXAMPLE OF NAMELIST OUTPUT

Standard Namelist printing was modified by a job control statement to reduce the width of the output to fit on this page. The JCL statement used was:

//G.FT06F001 DD SYSOUT=A,DCB=(RECFM=VBA,LRECL=90,BLKSIZE=990)

If this technique is employed to reduce the width of the output, all records written on the data set defined by data set reference number 6 must fit within the reduced record length.

The maximum CTC block size for an ASP system is 1012.

Non-Linear Bearing Force Characteristic Representation

Provision is made in the program to simulate non-linear bearing stiffness characteristics; however, linear damping is used in the program. The non-linear bearing stiffness characteristics including simultaneously linear speed sensitive stiffness parameters may be represented with a mathematical formulation as follows: (To eliminate translation of parametric notations from that used in the mathematical formulation to that used in the computer program, the latter notations are used here).

$$\left[(FD\phi T - FD\phi FIX) \left(BNB + BBB (\rho - BR\phi B) \right) + BKB \right] (\rho^{BBB} + (BDB)\rho + BEB) = F$$

where ρ is the resultant journal displacement from its zero-load position and F is the corresponding resultant bearing force.

BNB used in the computer program was denoted in the mathematical formulation portion of this report by N_{Bik} , and similarly

BBB by B_{Bik}

$BR\phi B$ by ρ_{Bik}

BKB by K_{Bik}

BHB by H_{Bik}

BDB by D_{Bik}

BEB by E_{Bik}

$FD\phi T$ by ϕ

$FD\phi FIX$ by ϕ_0

" $FD\phi FIX$ " is a non-subscripted name in the current program. It could be extended to a subscripted array in the future extension of the computer program. To simplify the expression of the above equation, the subscripts

i , k are omitted. The constants FDOFIX, BNB, BBB, etc., may be considered as that for a particular non-linear stiffness section of a bearing station. To briefly describe the method of determining the constants an arbitrary non-linear bearing stiffness characteristic is used as shown in Fig. 8.

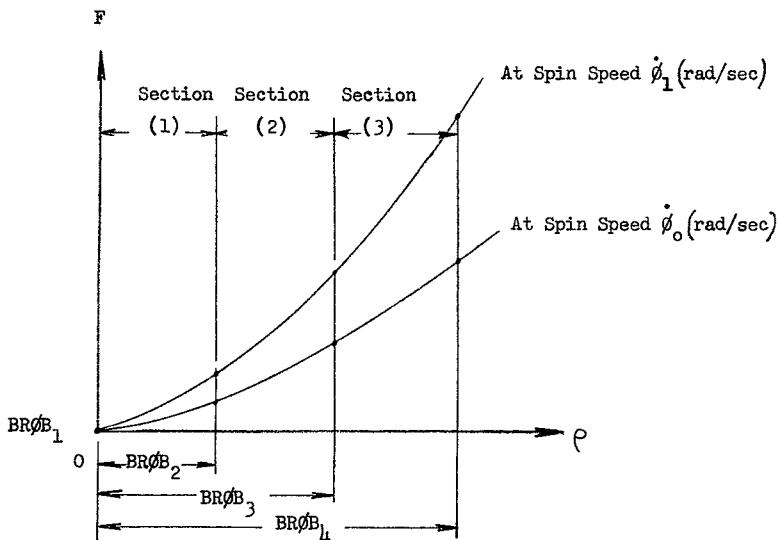


Fig. 8. Bearing Stiffness Curves

Assume the non-linear bearing stiffness section (2) is to be simulated. First use the non-linear stiffness characteristic at spin speed $\dot{\phi}_0$. Determine BHB according to the curvature, then solve for the three unknowns from the three equations,

$$BKB \left[\rho_i^{BHB} + (BDB) \rho_i + BEB \right] = F_i \quad i = 1, 2, 3$$

corresponding to three selected points on the characteristic curve.

The reason for using BKB as a common factor is that BKB may be directly used as a linear stiffness factor for a linear bearing when BHB = 1, and BDB = BEB = 0. Having determined the values of BKB, BHB, BDB and BEB, the values of BNB and BBB can be readily computed from selected two ρ values on the upper curve at spin speed $\dot{\phi}_1$ in Fig. 8. The equations used to obtain the solutions are:

$$\left\{ \left(\dot{\phi}_1 - \dot{\phi}_0 \right) \left[BNB + BBB(\rho_1 - BR\rho_2) \right] + BKB \right\} \left[\rho_1^{BHB} + (BDB) \rho_1 + BEB \right] = F_1$$

$$\left\{ \left(\dot{\phi}_1 - \dot{\phi}_0 \right) \left[BNB + BBB(\rho_2 - BR\rho_1) \right] + BKB \right\} \left[\rho_2^{BHB} + (BDB) \rho_2 + BEB \right] = F_2$$

In a similar manner the non-linear bearing stiffness representation of all other sections may be made.

Output Format

To provide a permanent record, the input data are printed out prior to the computation results.

The results of computations are printed with CRT graphs as optional additional output.

1. Print-Out. The computation results are printed out in the following sequence:
 - a. Total rotor weight, lb
 - b. Total rotor mass, the sum of additional rotor mass (AM) and the rotor mass computed from the rotor geometry $(\text{lb} - \text{sec}^2)/\text{in}$
 - c. Total rotor polar mass moment of inertia, the sum of additional rotor polar mass moment of inertia (AIR ϕ) and the polar mass moment of inertia computed from the rotor geometry.
 - d. Force and moment influence coefficients
 - e. Initial rotor motion conditions
 - f. Elements in rotating coordinates consisting of:
 - 1) NS number of rotor displacement vector length, in
 - 2) NS number of rotor whirl displacements, rad
 - 3) NS number of rotor displacement vector velocities, in/sec
 - 4) NS number of rotor whirl velocities, rad/sec
 - 5) Rotor spin displacement rad
 - 6) Rotor spin velocity rad/sec
 - g. The appropriate derivatives consisting of:
 - 1) NS number of rotor displacement vector velocities in/sec
 - 2) NS number of rotor whirl velocities rad/sec
 - 3) NS number of rotor displacement vector radial accelerations in/sec²

4)	NS number of rotor whirl accelerations	rad/sec ²
5)	Rotor spin velocity	rad/sec
6)	Rotor spin acceleration	rad/sec ²
h.	X displacement array in stationary coordinates, (in.)	
i.	Y displacement array in stationary coordinates, (in.)	
j.	Rotor deflection vector array, in.	
k.	Phase angle array for rotor deflection vectors from x-axis in the direction of rotation, degrees	
l.	X velocity array in stationary coordinates, in.	
m.	Y velocity array in stationary coordinates, in.	
n.	Whirl frequency array, rpm	
o.	Whirl-to-spin frequency ratio array	
p.	Total number of revolutions	
q.	Spin speed, rpm	
r.	X bearing displacement coordinate, in.	
s.	Y bearing displacement coordinate, in.	
t.	X bearing velocity coordinate, in/sec.	
u.	Y bearing velocity coordinate, in/sec.	
v.	Journal displacement vector from bearing-center array, in.	
w.	Journal displacement vector phase angle in the direction of rotation from X-axis, degrees	
x.	Bearing reaction array, lb.	
y.	Bearing reaction vector phase angle in the direction of rotation from x-axis, degrees	
z.	Bearing force to journal deflection ratio, or equivalent linear bearing stiffness, lb/in/bearing.	

A sample of printout of the computation results including that of input data is attached in Appendix A.

2. CRT Plotting Output. To facilitate the direct comprehension of the computation results and appreciation of the trend of rotordynamic

behavior, a series of 7 types of CRT are provided as an optional output controlled by an input word, "CRT." That is, CRT=1 for CRT output and CRT=0 for no CRT output.

The 7 types of CRT graphs are described as follows:

- a. Rotor 3-dimensional mode shape at or near preselected input speeds, "INPRPM," for a rotor operating with spin-speed acceleration or deceleration. For a steady-state rotor spin speed, one CRT of this type is provided during the run. The spin speed, at which the mode shape is depicted, is labeled on top of the graph. Three-dimensional information of the graph is provided by labeling the phase angle at corresponding displacement vector stations.
- b. Rotor spin speed versus time graph.
- c. Rotor whirl-to-spin speed ratio versus spin speed graph for a pre selected rotor station as designated by the input, "IASIGN."
- d. Bearing forces versus rotor spin speed graphs for all non-linear stiffness bearings.
- e. Bearing displacements versus spin speed graphs for all non-linear stiffness bearings.
- f. Maximum rotor displacement versus spin speed. The rotor station number for the maximum rotor deflection station for a spin speed is appropriately labeled for identification
- g. Rotor displacement at station "IASIGN" versus spin speed.

The number of points included in a frame of the CRT graph may vary from 1 through 50 as specified by the value of the input word, "NPPOINT."

EXPERIMENTAL PROGRAM

The purpose of Task II, Mark 25 Rotor Dynamics Tests, was to perform spin tests utilizing the Mark 25 pump rotor to obtain data for correlation with the results of the rotordynamics computer program.

The test program included an initial series of seven tests to minimize rotor unbalance, plus eight investigative tests on the Mark 25 pump rotor with an array of unbalance levels and locations. Instrumentation was provided to measure response amplitudes and phase angles.

MARK 25 TURBOPUMP

The Mark 25 pump is an axial-flow liquid hydrogen pump consisting of an inducer and four axial stages. The pump rotor is mounted by duplex paired bearings. The bearings used are 55-millimeter bore size. To maintain radial stiffness required for rotordynamic considerations (critical speed, rotor whirl, and minimum tip clearance without rubbing), the bearings are arranged in axially preloaded pairs, each pair free to move independently of the other.

The rotor is built-up rotor of eight disks clamped together with 18, (5/16) through bolts. The overhung inducer is mounted by 6, (3/8) bolts to the forward disk stub shaft. The rotor is driven through a ball spline coupling at the aft stub shaft. The overall rotor length is approximately 2 $\frac{1}{4}$ inches with a nominal diameter of 6.1 inches.

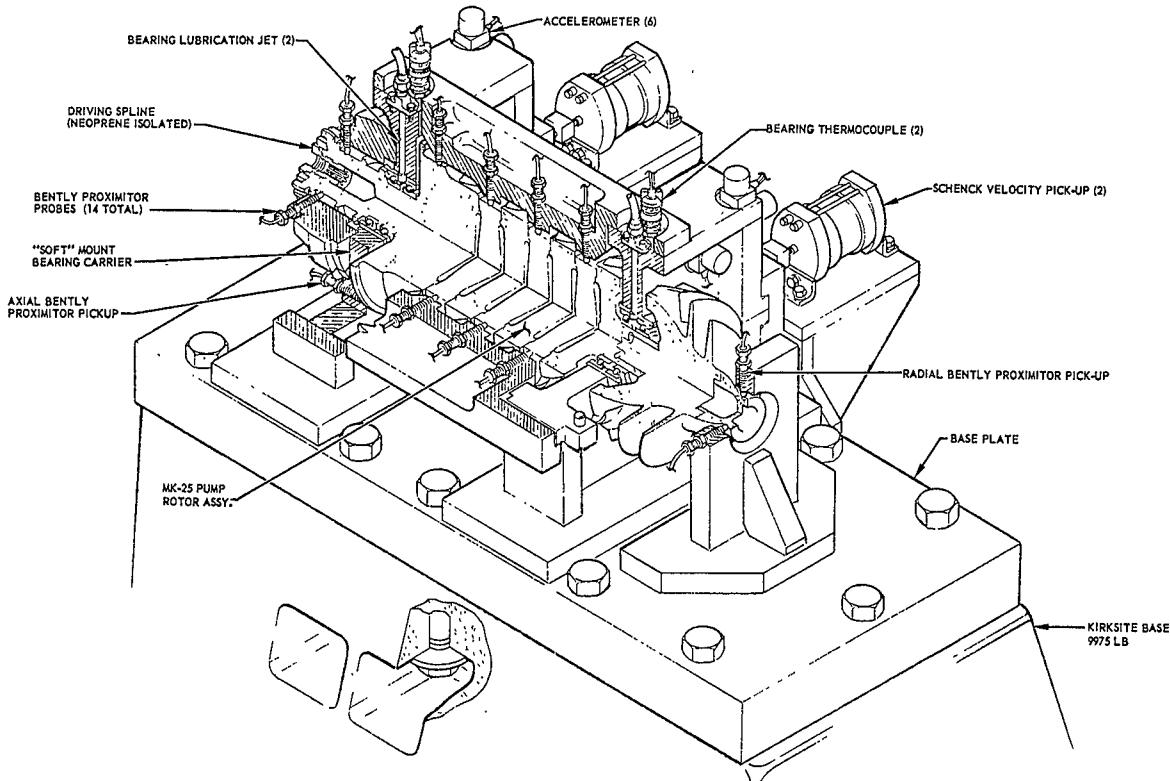


Figure 9. Mark-25 Rotor High Speed Test Set-Up

TEST SET-UP

Hardware

The test set-up consisted of a Mark 25 pump rotor mounted in a support pedestal which was bolted to a large seismic mass. The complete assembly was mounted inside a large vacuum chamber. The pump bearings were cooled by a Freon 21 system. Rotating speeds up to 60,000 rpm were provided by a prime mover (300 hp dynamometer) and a 10:1 gear ratio speed increaser. The rotor was driven by means of an aluminum quill shaft and a splined adapter with a neoprene isolator to minimize transmission of gear noises.

Instrumentation

The basic instrumentation consisted of the following:

- 1 channel, rotor speed
- 14 channels, rotor displacements
- 2 channels, pedestal vibration
- 2 channels, bearing temperatures

The primary instrumentation parameters were the rotor displacement measurements which were made with Bently proximity transducers. A typical installation of these transducers is shown in Fig. 9.

Two mounting bars for the displacement transducers were located along the rotor axis in two 90 degree planes and bolted to the bearing pedestals. The fourteen non-contacting displacement indicators were used to monitor the radial deflection of the rotor in six radial planes and the axial position of the rotor. A calibration spot face was used to determine actual displacement values during test runs.

TEST PROGRAM

The test program was accomplished in two parts. The first part was a set of spin tests to establish an initial balancing condition and the second part was to perform spin tests to investigate unbalance conditions. These unbalance conditions would produce response data for correlation with the mathematical computer model results.

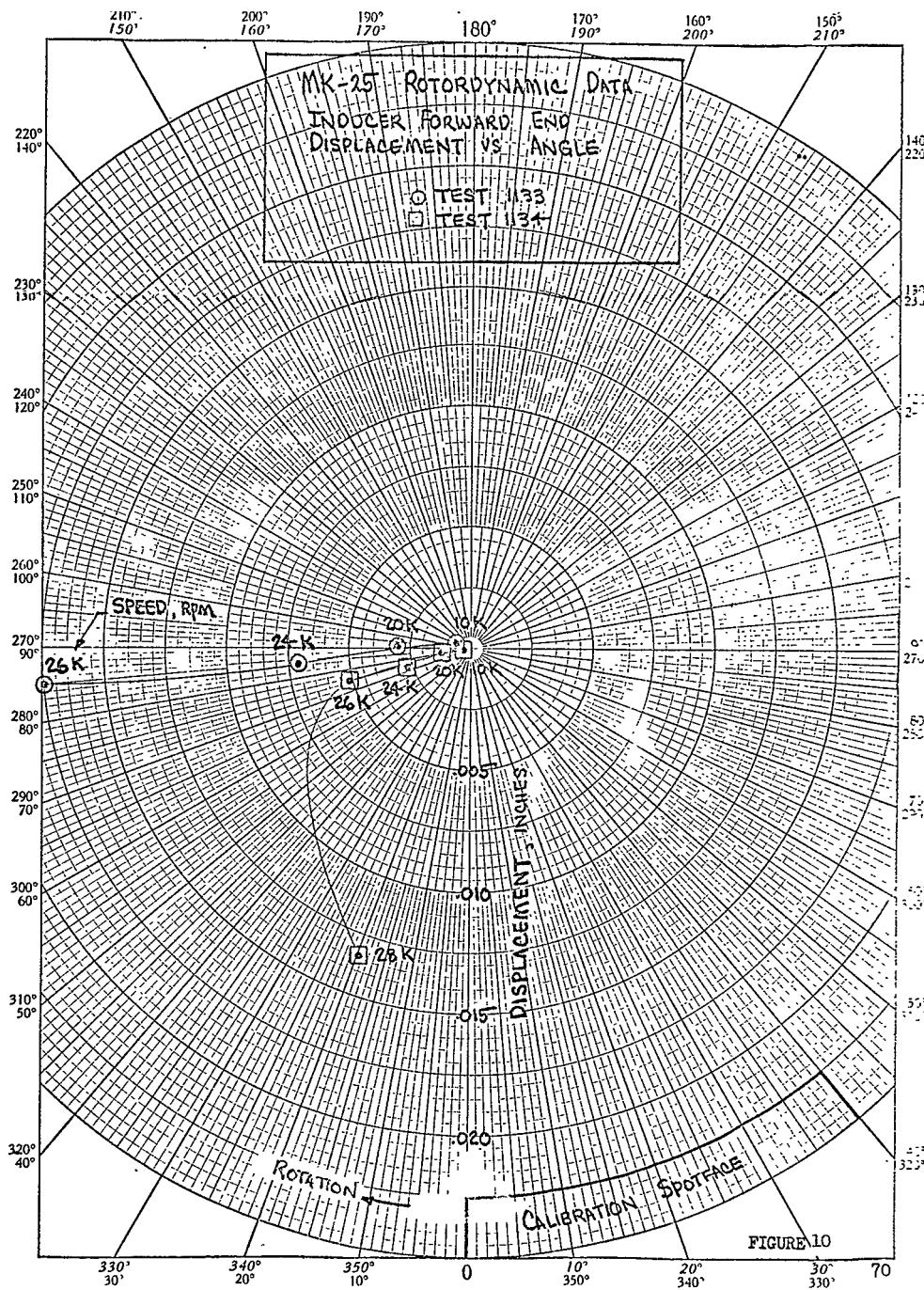
A total of 16 tests were performed as part of this contract study. The first three tests were performed without the inducer. The next three were performed with the inducer. The remaining 10 tests were investigative. One of the investigative tests, 1140, was not used because of loss of data due to a tape recorder malfunction.

Balancing Tests

Three tests (1130 through 1132) were performed without the inducer to obtain an initial balance on the basic rotor. The inducer was then installed and test 1133 performed. During test 1133, the maximum displacement at the forward (inducer) end was .0174 inches at 26,000 rpm. A Post-test examination revealed an inadequate inducer pilot depth. The pilot depth was increased and tests 1134 and 1135 were performed. The maximum displacement for test 1134 was .0113 at 28,000 rpm, but was only .0051 at 26,000 rpm which indicated a significant balance improvement over test 1133. Figure 10 presents the inducer forward displacement for tests 1133 and 1134. Test 1135 was an attempt to achieve an improved balance but was terminated at 24,000 rpm due to excessive deflections.

Investigative Tests

The eight investigative tests, 1136 through 1139 inclusive, and 1142



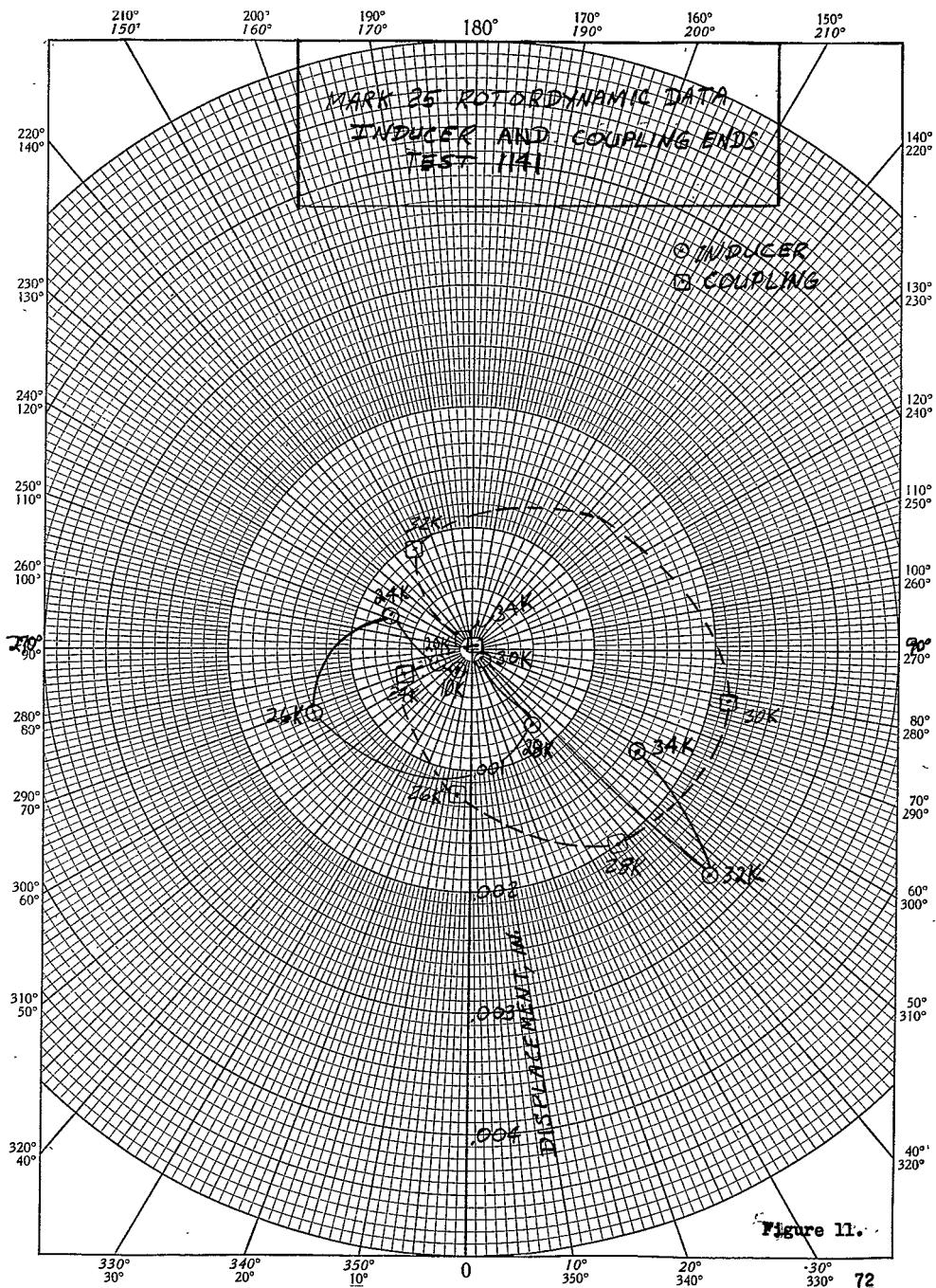
through 1145 were performed with various unbalance conditions. Test 1141 was selected as the reference run since it was the best balance achieved. Response deflections of the rotor for the other tests are given as vector changes from the deflections observed in 1141. The forward inducer end and the coupling end displacements for test 1141 are shown in Figure 11. In order to include the effect of critical speed only the tests that operated up to 30,000 rpm or greater were used to generate investigative data. The critical speed was calculated to be approximately 28,000 rpm.

Table 3 lists the test run, the unbalance condition and the upper speed limit achieved. The unbalance weights were selected to allow operation to at least 30,000 rpm without creating excessive bearing loads or allowing a deterioration problem to develop.

PROCEDURE FOR TEST SET-UP

Prior to installation in the rotordynamics test facility, the pump rotor assembly was dynamically balanced at low speed (1000 rpm) on the Gisholt balancing machine. The balancing sequence was as follows:

1. All individual rotors and rotor spacers were dynamic balanced on an arbor.
2. The inducer was balanced on an arbor.
3. The rotors and rotor spacers were assembled into a composite rotor assembly by means of thru bolts, and this assembly mounted in balance bearings was dynamic balanced on the Gisholt machine.
4. The balance bearings were removed, and the rotor was assembled to a pump configuration with duplex bearings, special bearing cartridges, and all rotating parts except the inducer, spinner, and mounting studs and fasteners. Assembled in this manner the rotor was again dynamic balanced. The inducer was then installed on the rotor and this assembly balanced. The spinner was then added and balance corrections made. All components of the rotor were matchmarked.



Figure

TABLE 3

UNBALANCE AND UPPER SPEEDS
FOR EXPERIMENTAL TESTS

Test No.	Unbalance *		Location		Speed rpm
	Amount Gram-In	Angle Degrees	Plane No. Shown in Fig. 12	Inches From Inducer	
1136	2.12 3.40	138 139	2 3	1.8 5.2	30,000
1137	2.74 4.25	195 196	2 3	1.8 5.2	30,000
1138	0.26 0.365	-24 -24	2 3	1.8 5.2	34,000
1139	0.64 1.10	-32 -30	2 3	1.8 5.2	34,000
1142	1.47	121	10	12.9	32,000
1143	.118	165	9	24.0	30,000
1144	.51	125	7	19.5	30,000
1145	.52	115	4	7.8	30,000

* Reference to Test 1141 and machined spot face on the coupler.

5. The inducer, spinner, and mounting studs were removed from the rotor assembly and the rotor minus the above parts was installed in the cradle mount in the rotordynamics test facility.
6. After the rotor had been installed, all Bently proximitron probes were gapped, rotor runouts were taken and recorded, and Freon lube jets, bearing thermocouples, and accelerometers were installed.
7. After the rotor was initially balanced, the inducer and spinner were installed so that the complete rotor assembly could be balanced.

ROTORDYNAMICS SPIN TEST PROCEDURE

Prior to the first test each day, the speed increasing gear box was heated to operating temperature by recirculating lube oil through heaters by electrically driven pumps in order to ensure vertical alignment of the center lines on gear box pinion shaft and test rotor. Also, all electronic equipment, oscilloscopes, tape recorders, etc. were turned on to warm up and be checked out. After the above was established, the sequence of events was as follows:

1. The lube oil pumps and heaters were turned off, and the lube oil run tank was charged.
2. The test chamber was closed and the vacuum pumps were started.
3. The Freon pressurized lube system for pump rotor bearings was charged.
4. All on-line monitoring instruments were set up and calibrated and the appropriate test parameter calibrations were recorded on tape.
5. All other systems and controls in the control room and test cell were made ready for test.

6. When chamber pressure reached approximately 40 mm Hg. pressure, the Freon-21 bearing lube system was turned on while rotating the pump rotor slowly to chill the bearings, rotor and cradle to a stabilized operating temperature.
7. When the pump rotor had been chilled, the Freon flow was stopped, bearing purge was turned on, and all Bently dectector potentiometers were set for proper voltage output.
8. The test area was cleared and all stations in the control room were manned. The bearing purges were turned off; lube oil and Freon lube systems were turned on and flows were established.
9. The tape recorders and Brush recorders were turned on.
10. The test was started and rotor speed was manually programmed to follow a speed ramp run schedule with a series of speed plateaus. In addition to recording test parameters, one channel on each tape recorder was used to monitor voice, and rotor speed was announced in approximate intervals of five seconds as well as when speed was being changed. This channel also monitored voice communication between the control room and data acquisition room.
11. During the test, Bently output signals were monitored on four two-channel oscilloscopes.
12. At the completion of a test, vacuum was broken in the test chamber, and the tape played back.
13. Phase angle in degrees and rotor motion in mils for each speed plateau was determined from Polaroid photographs and oscillograph records.
14. The rotor quill shaft was disengaged so that the rotor could be turned, by hand, to check bearings.
15. The appropriate balance corrections or unbalance conditions were made on the rotor assembly.

16. The above procedure was repeated until all required tests were performed.

RESULTS

The test results of the experimental program are given in radial deflection of the rotor at six axial stations along the rotor. The experimental results are presented in graphical CRT plots. There are two types of data presentations for each test:

1. deflections vs speed for a given station,
example: Figure 13
2. deflection vs station for a given speed,
example: Figure 21

For the first type of plot, the stations selected were stations 2 (Bently 16), the inducer end; station 9 (Bently 8), the center of the rotor; and station 17 (Bently 2), the coupler end of the pump. The second type of plot was selected at or near speeds of maximum deflection at one or more stations. Three speeds are shown for each test. The total maximum deflection and the X and Y plane components are shown relative to the spot face at station 17, which was the zero degree (0°) reference point on the rotor. Table 4 gives the relationship between the experimental and mathematical station descriptors.

In order to adjust the data to a form easily used for comparative purposes, the data was corrected to remove the effects of shaft runout and the unknown residual unbalance. The latter was accomplished by vectorially subtracting the data from the minimum response run (the reference run, 1141) from all other test runs. The unbalance condition listed in Table 3 is the "delta" unbalance between the test run and the minimum response run (1141). All angles referred to in the Table 3 and the graphical

TABLE 4

RELATIVE MATHEMATICAL AND EXPERIMENTAL
MODEL DESCRIPTORS

Bently Meas. Location	Math. Model Station	Rotor Axial Position (Ref. Inducer end) Inches
16	2	0.8
10	8	8.75
8	9	11.5
6	11	14.5
4	13	17.25
2	17	22.9

plots are referenced to a spot face at plane 8 (Bently #2), Fig. 12

On the experimental data plots the following nomenclature is used:

DEFL 2 = deflection of the rotor at Bently #2 (Math Model Sta. 17) vs speed

DEFL 8 = deflection of the rotor at Bently #8 (Math Model Sta. 9) vs speed

DEFL 16 = deflection of the rotor at Bently #16 (Math Model Sta. 2) vs speed

X PLANE = deflection of the rotor vs station at indicated speed in the X-plane with reference to the spot face at Station 17.

Y PLANE = deflection of the rotor vs station at indicated speed in the Y-plane with reference to the spot face at Station 17

TOTAL = total deflection of the rotor (maximum) vs station

All deflections given are in mils (10^{-3} inches).

Figures 13 to 20, inclusive, are the deflection vs speed plots for test runs 1136 through 1139 and 1142 through 1145, inclusive. The speed range plotted is from 10,000 to 30,000 rpm except for Test 1142 which had an upper speed limit of 32,000 rpm, and Tests 1138 and 1139 which achieved upper speed limits of 34,000 rpm.

Figures 21 to 44, inclusive, are the deflection of the rotor vs rotor stations for three speed cases for each of the investigative tests. The abscissa is the distance from the inducer end in inches. In most cases, the applied unbalance tended to cause the local section of the pump rotor to deflect significantly; i.e., when the unbalance was at the

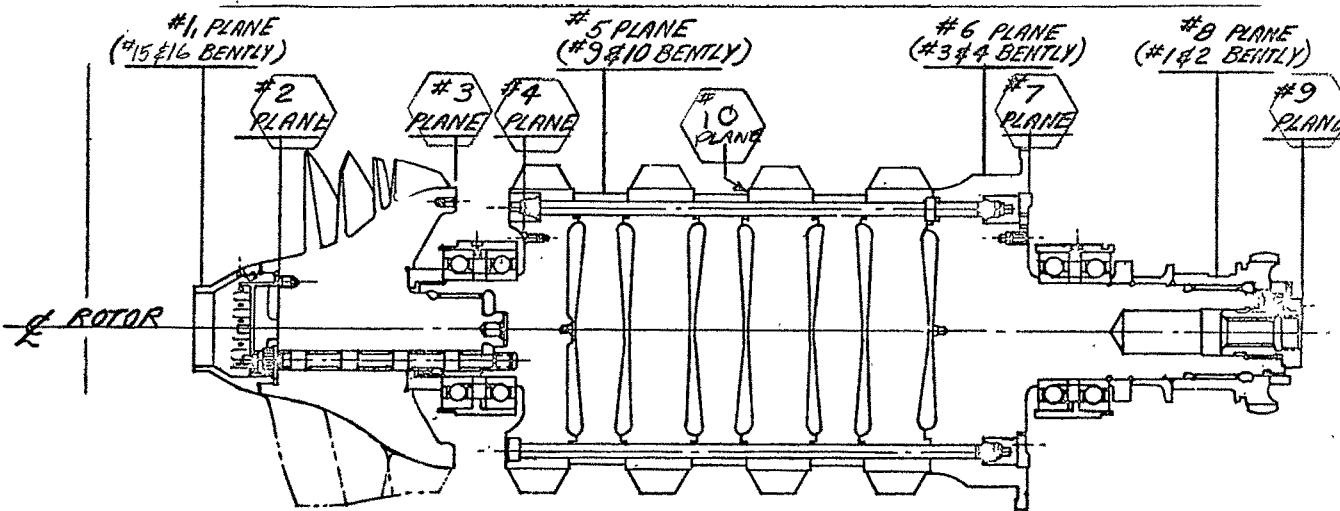


Figure 12. Bently and Balance Planes, Mark-25 Pump Rotor

inducer, the inducer end had the largest deflection; when the unbalance was in the center section (between the bearings) the center of the rotor deflected more. However, when the unbalance (a very small amount) was applied at the coupler end, the inducer displayed large deflections. This was primarily due to the inherent mode shape of the rotor in the speed range of interest.

TEST NO. 1136

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

0005 0000

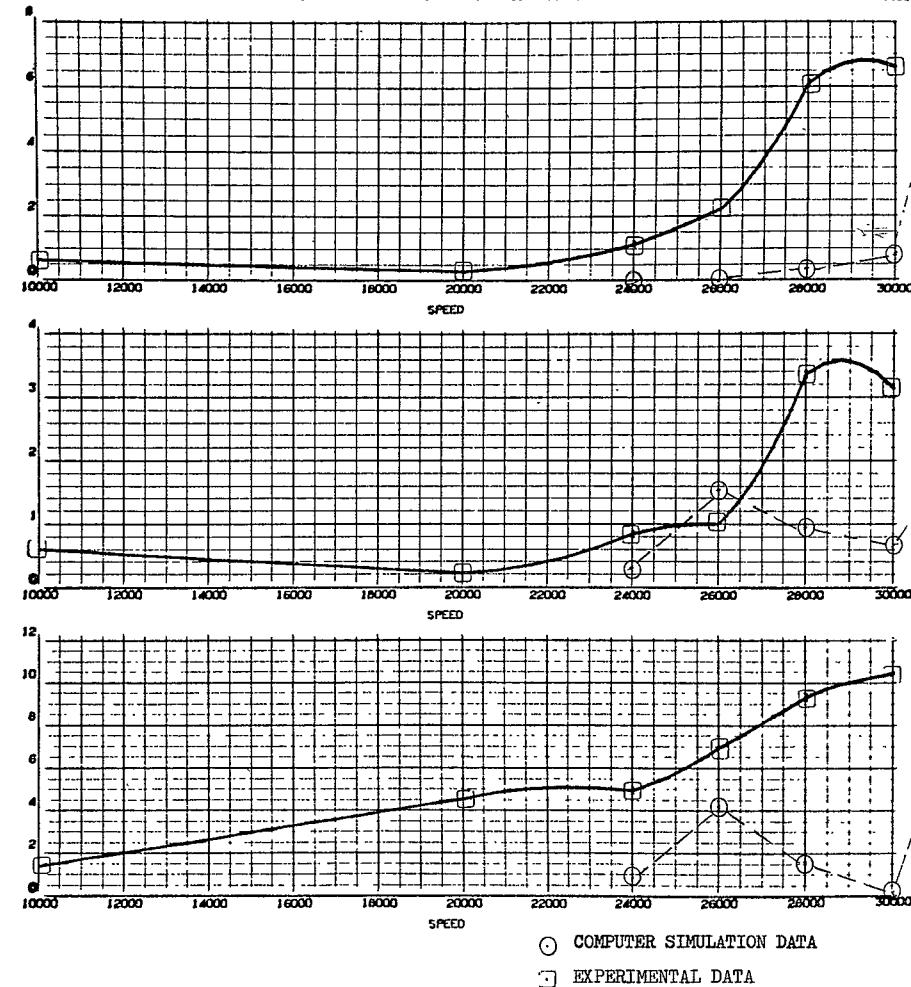


Figure 13

TEST NO. 1137.

BENTLY'S 2, 8 AND 16 DEFLECTIONS (INLS) VS SPEED (RPM)

843953
0007 0000

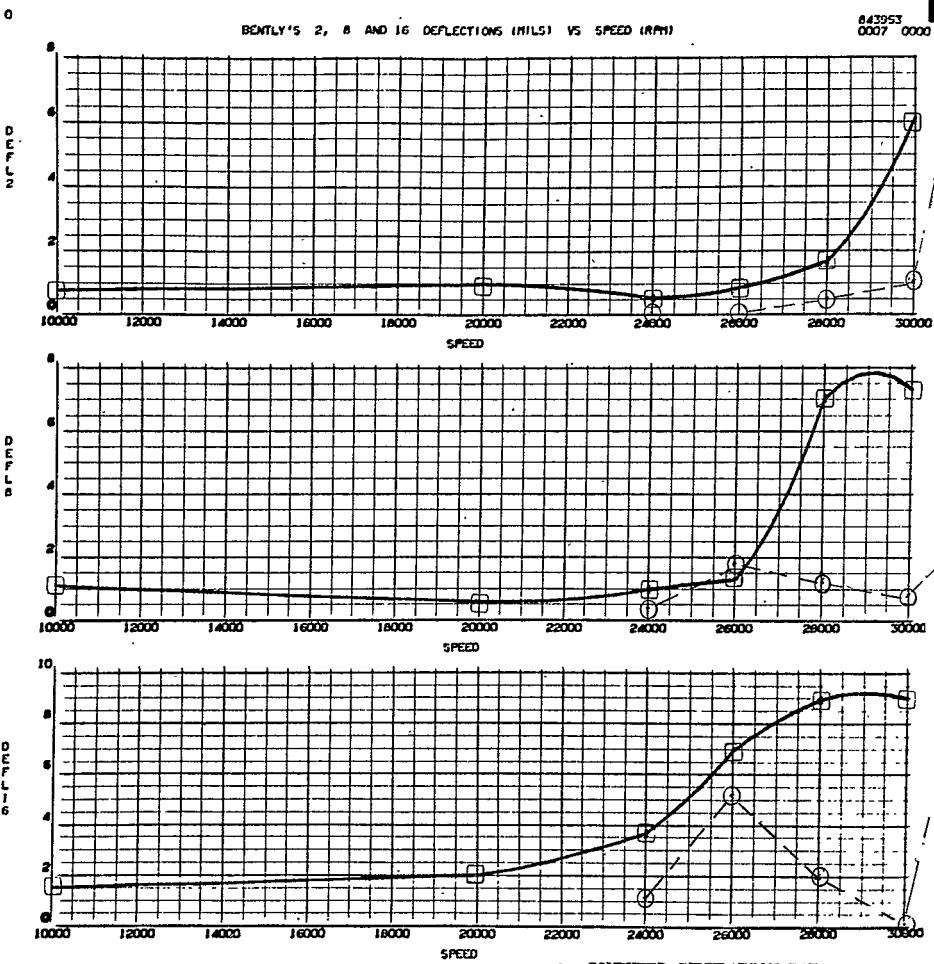


Figure 14

TEST 1138

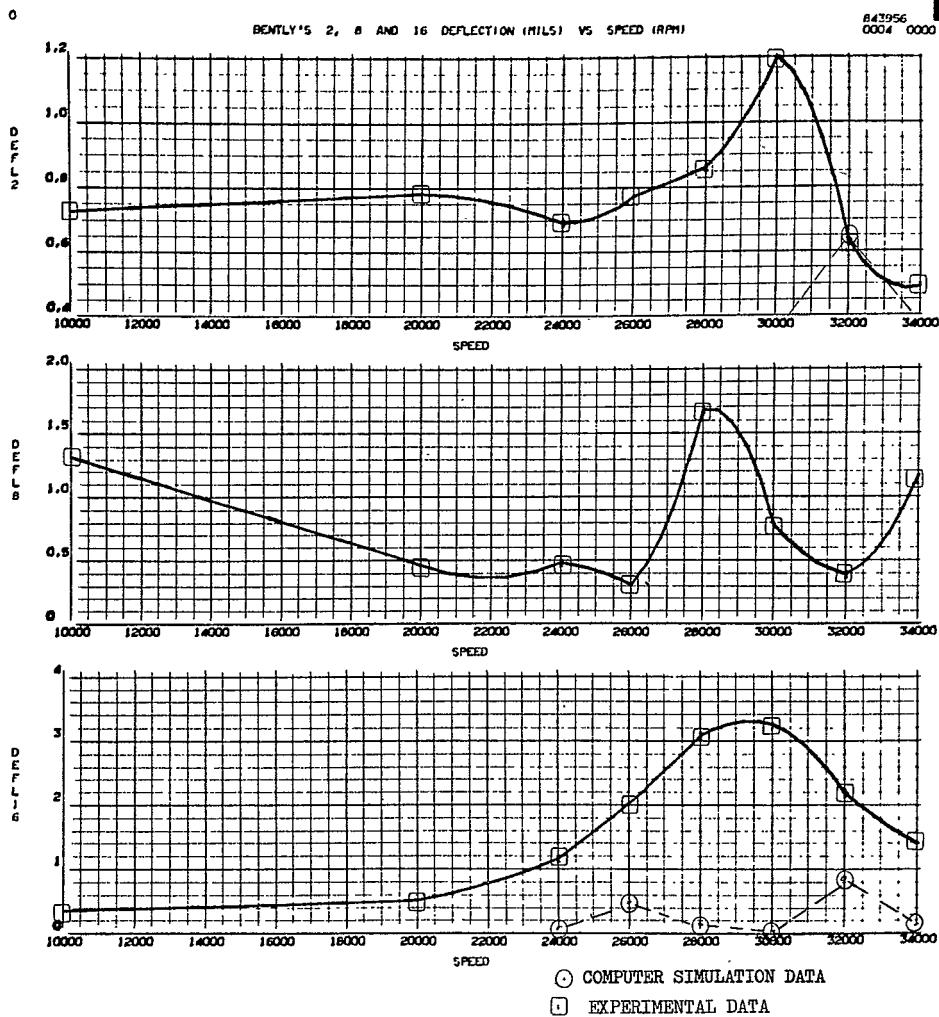


Figure 15

TEST 1139

BENTLY'S 2, 8 AND 16 DEFLECTION (MILS) VS SPEED (RPM)

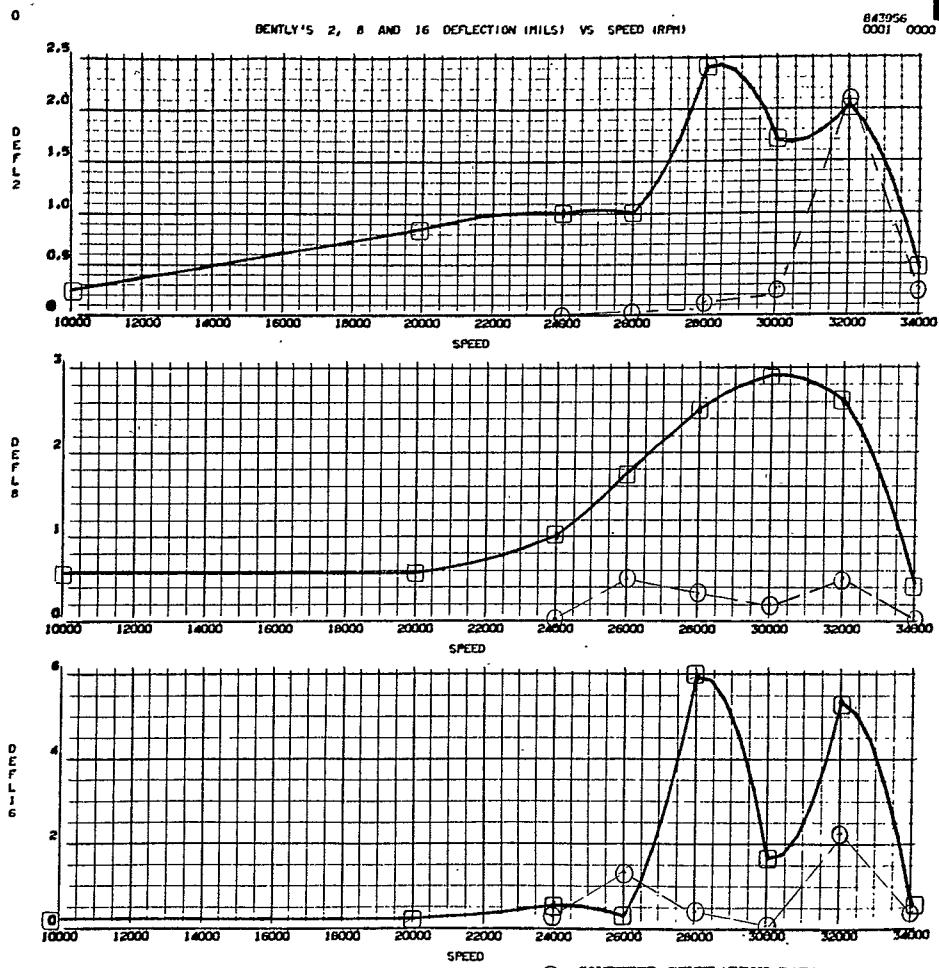
843956
0001 0000

Figure 16

TEST 1142

BENTLY'S 2, 8 AND 16 DEFLECTION (MILS) VS SPEED (RPM)

B43982
0001 0000

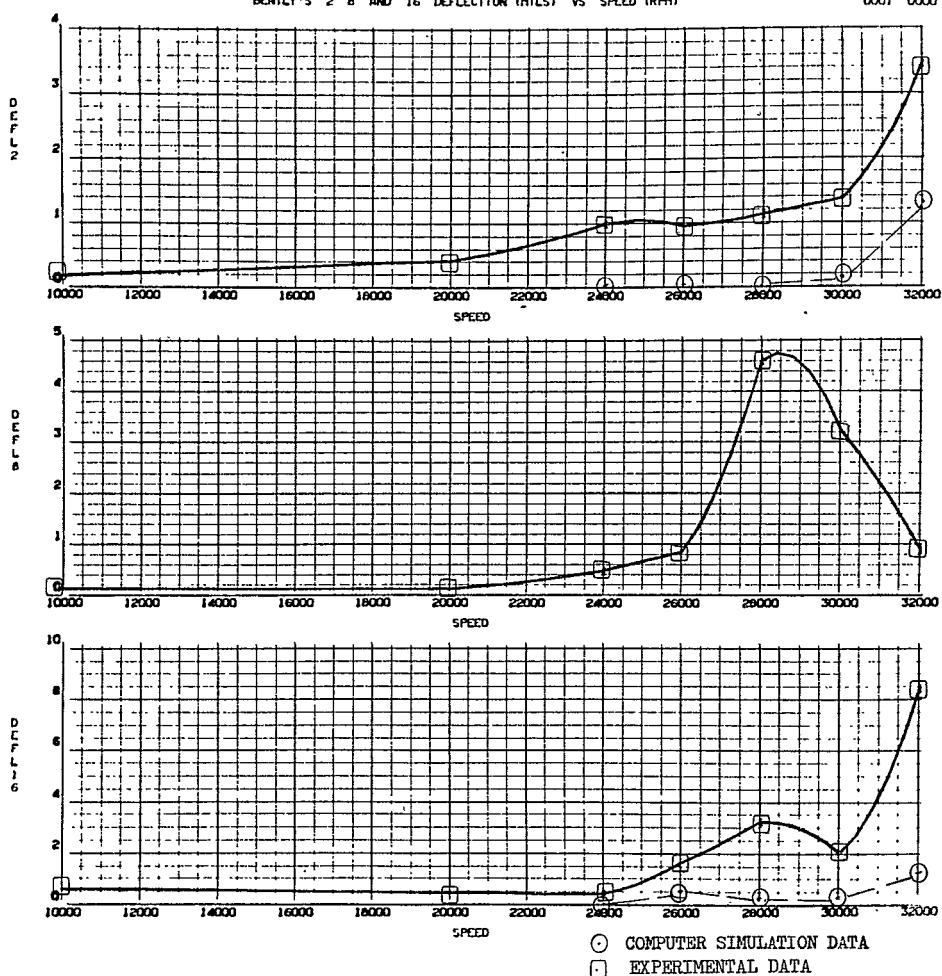
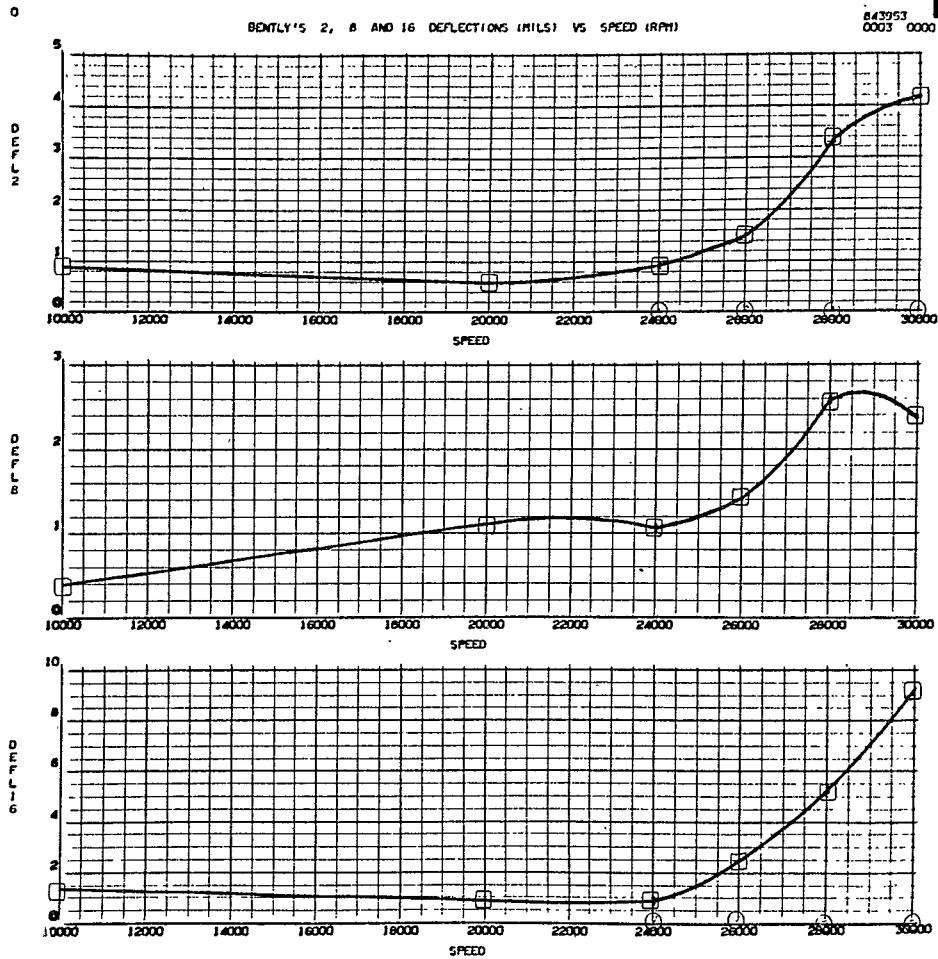


Figure 17

TEST NO. 1143

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

843953
0003 0000



○ COMPUTER SIMULATION DATA

□ EXPERIMENTAL DATA

Figure 18

TEST NO. 1144

BENTLY'S 2, 8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

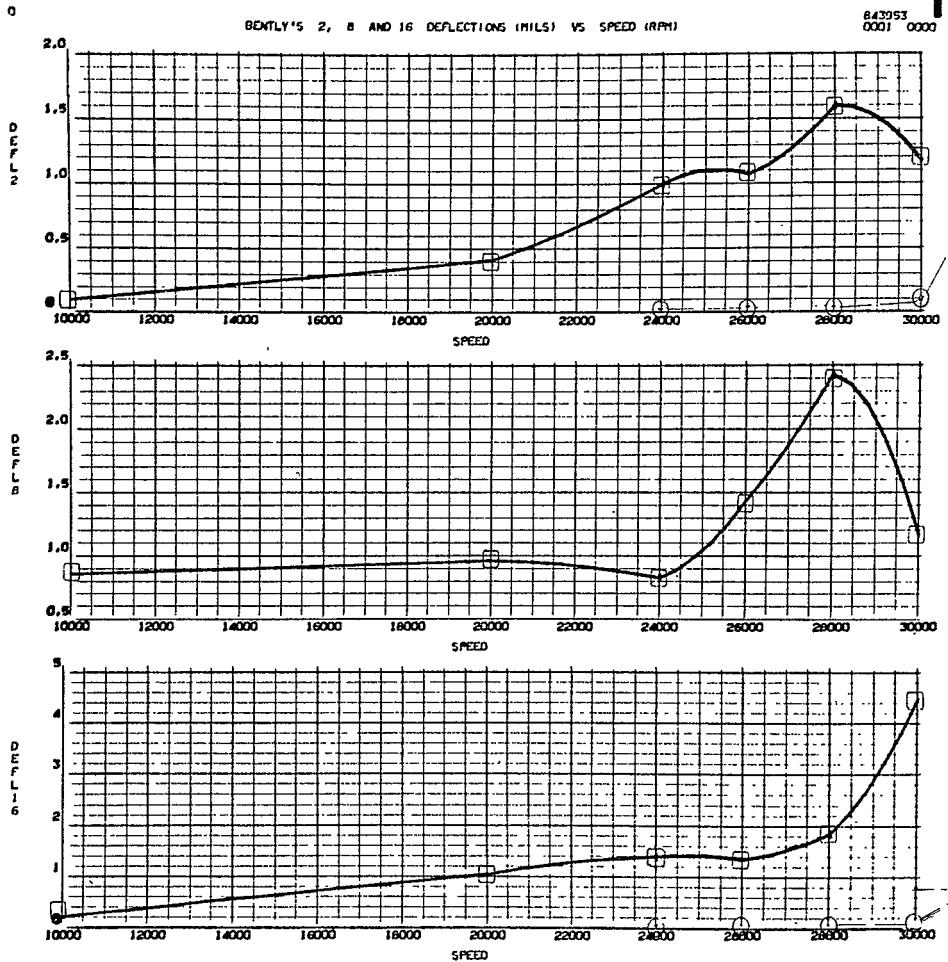
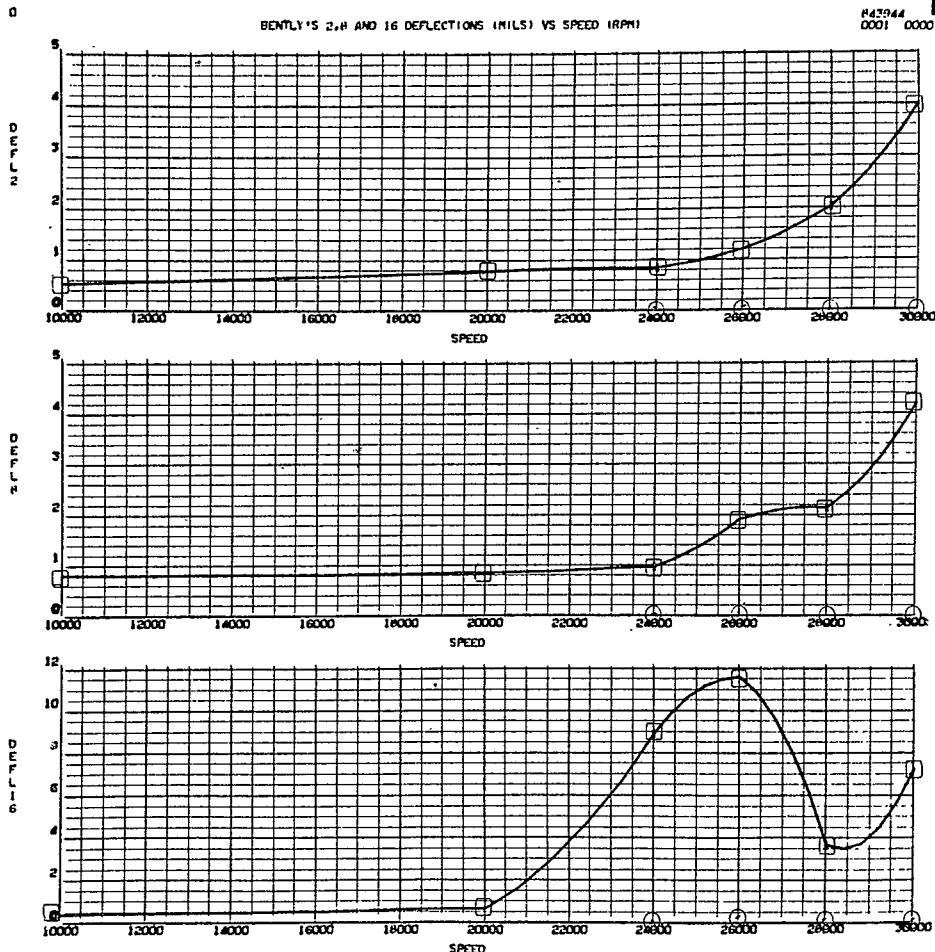
843953
0001 0000

Figure 19

TEST 1145

BENTLY'S 2.8 AND 16 DEFLECTIONS (MILS) VS SPEED (RPM)

P43944
0001 0000

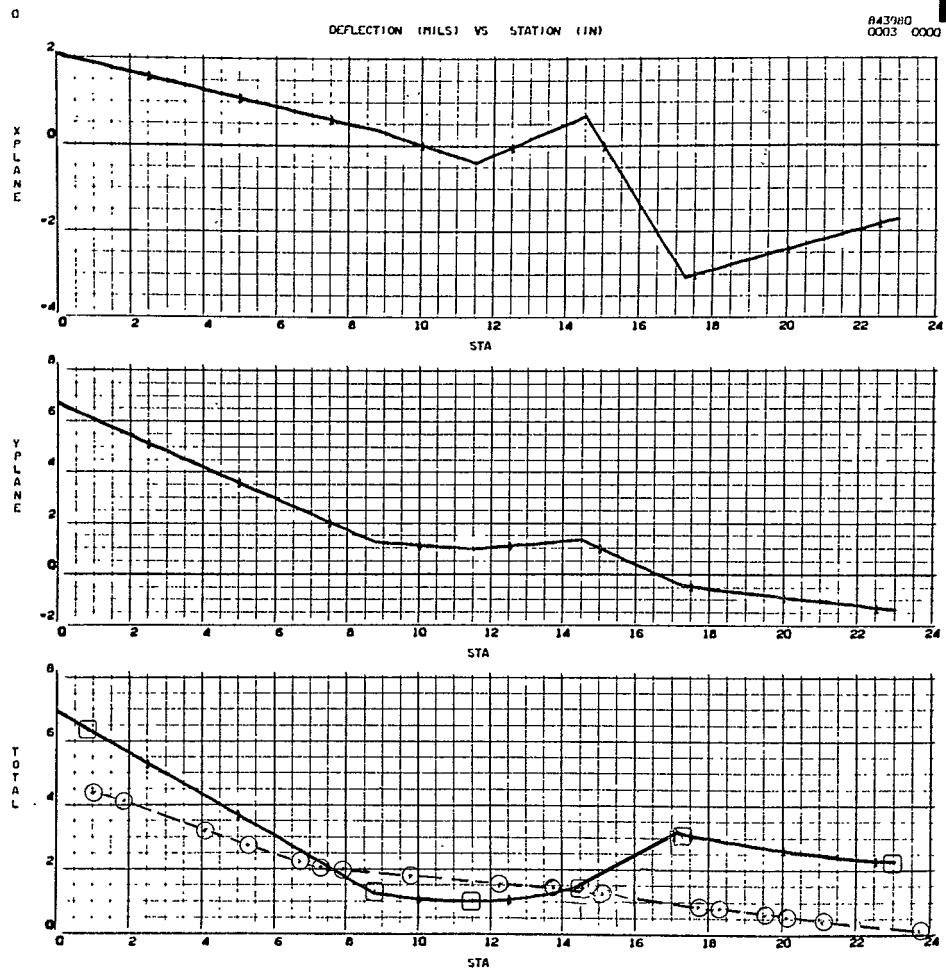


○ COMPUTER SIMULATION DATA

□ EXPERIMENTAL DATA

Figure 20

TEST 1136 26000 RPM



C COMPUTER SIMULATION DATA

EXPERIMENTAL DATA

Figure 21

TEST 1136 28000 RPM

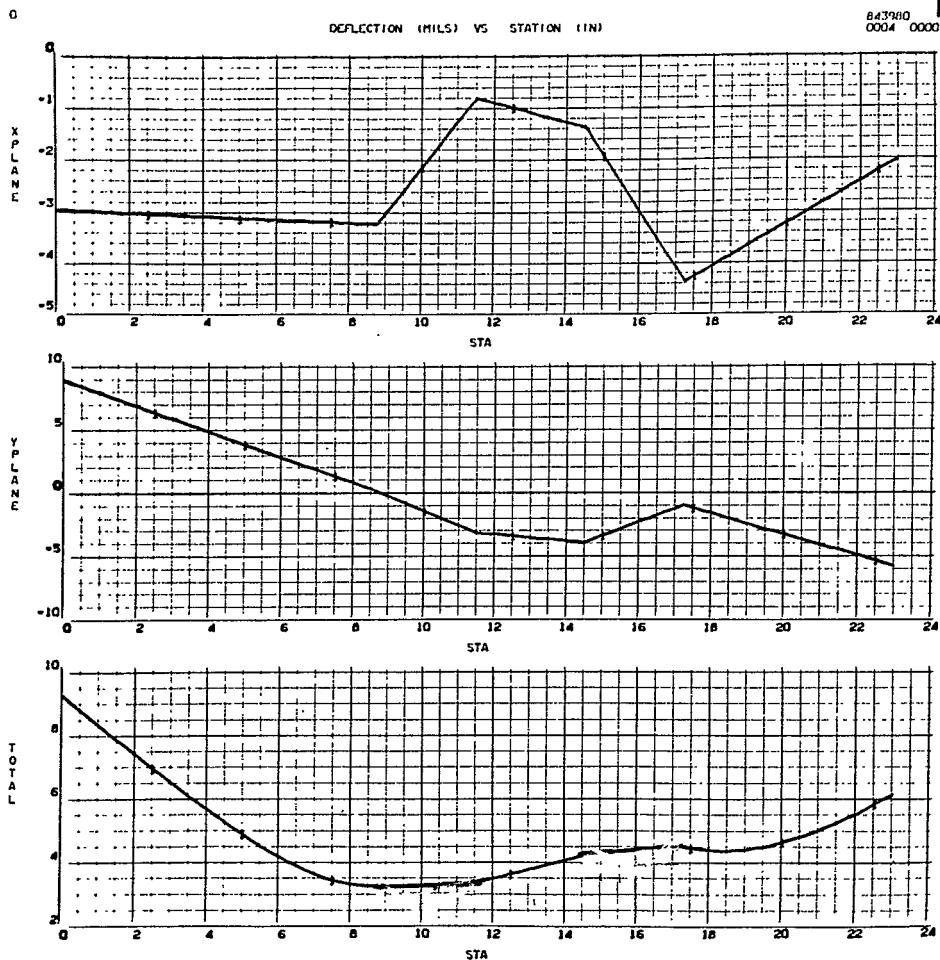


Figure 22

TEST 1136 30000 RPM

843900
0005 0000

DEFLECTION (MILS) VS STATION (IN)

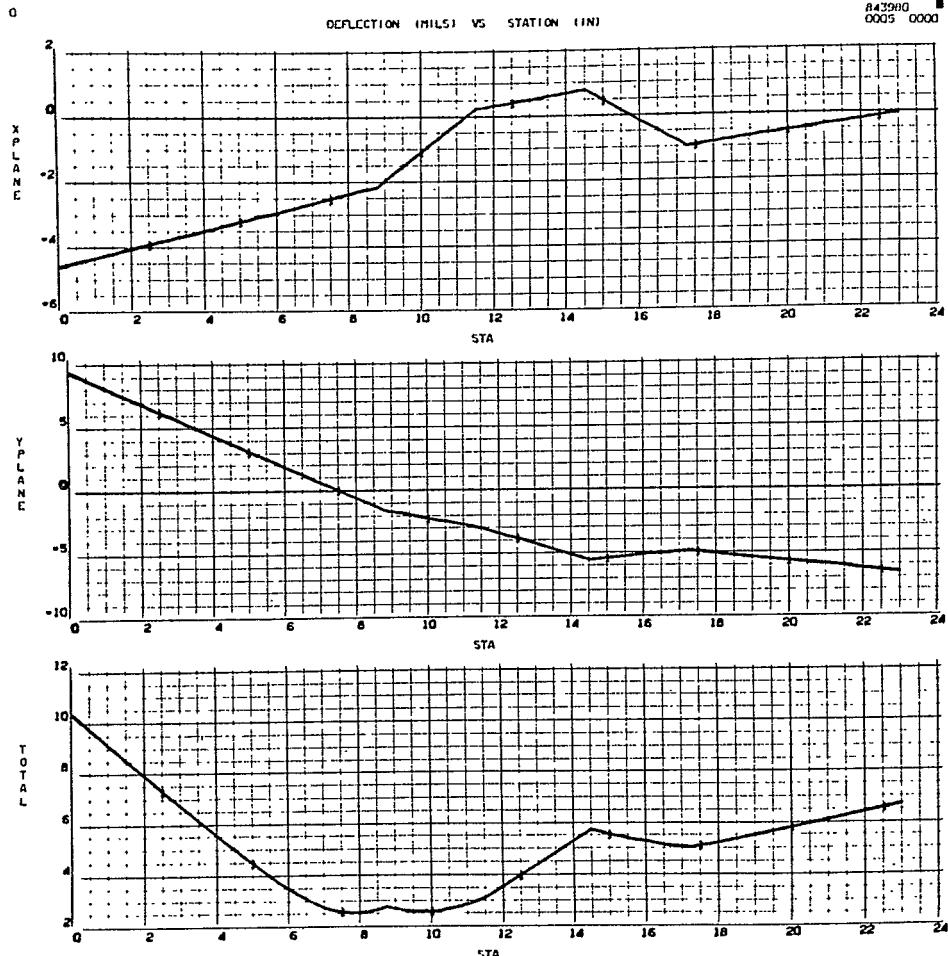
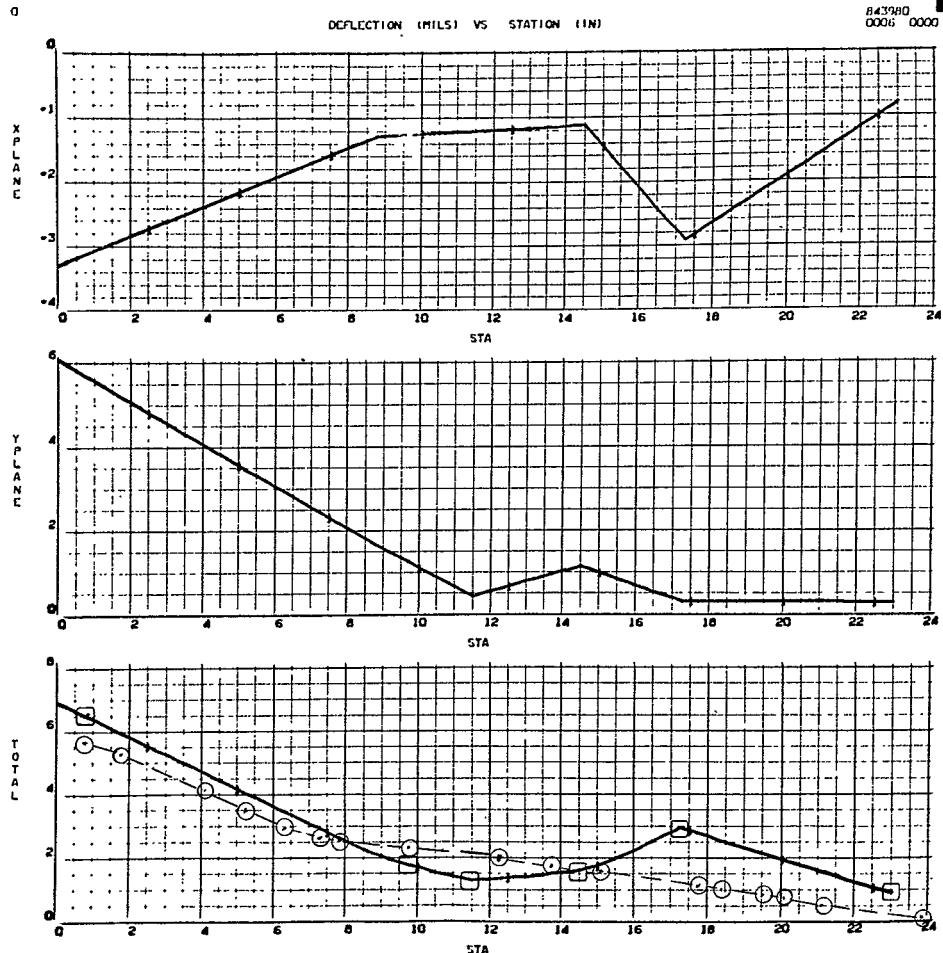


Figure 23

TEST 1137 26000 RPM

843980
0000 0000

DEFLECTION (MILS) VS STATION (IN)



○ COMPUTER SIMULATION DATA

□ EXPERIMENTAL DATA

Figure 24

TEST 1137 28000 RPM

DEFLECTION (INCHES) VS STATION (IN)

843980
0007 0000

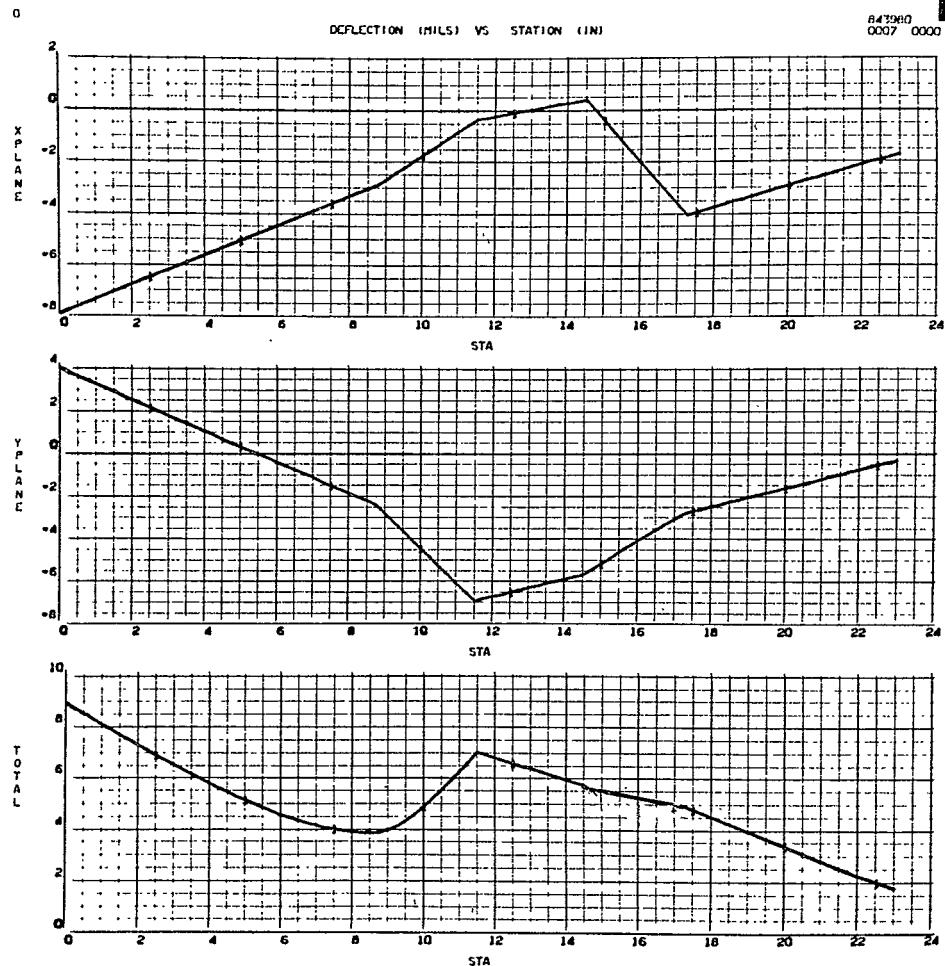


Figure 25

TEST 1137 30000 RPM

843980
0000 0000

DEFLECTION (MILS) VS STATION (IN)

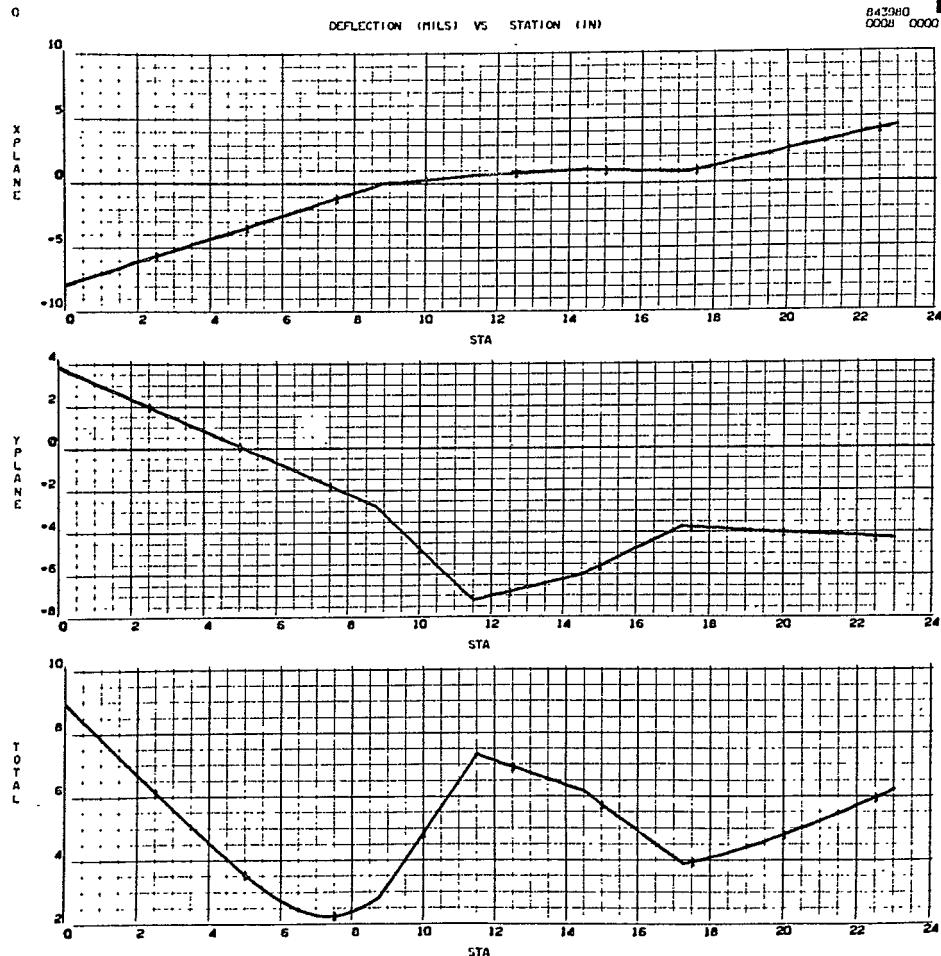


Figure 26

TEST 1138 28000 RPM

843980
0009 0000

DEFLECTION (MILS) VS STATION (IN)

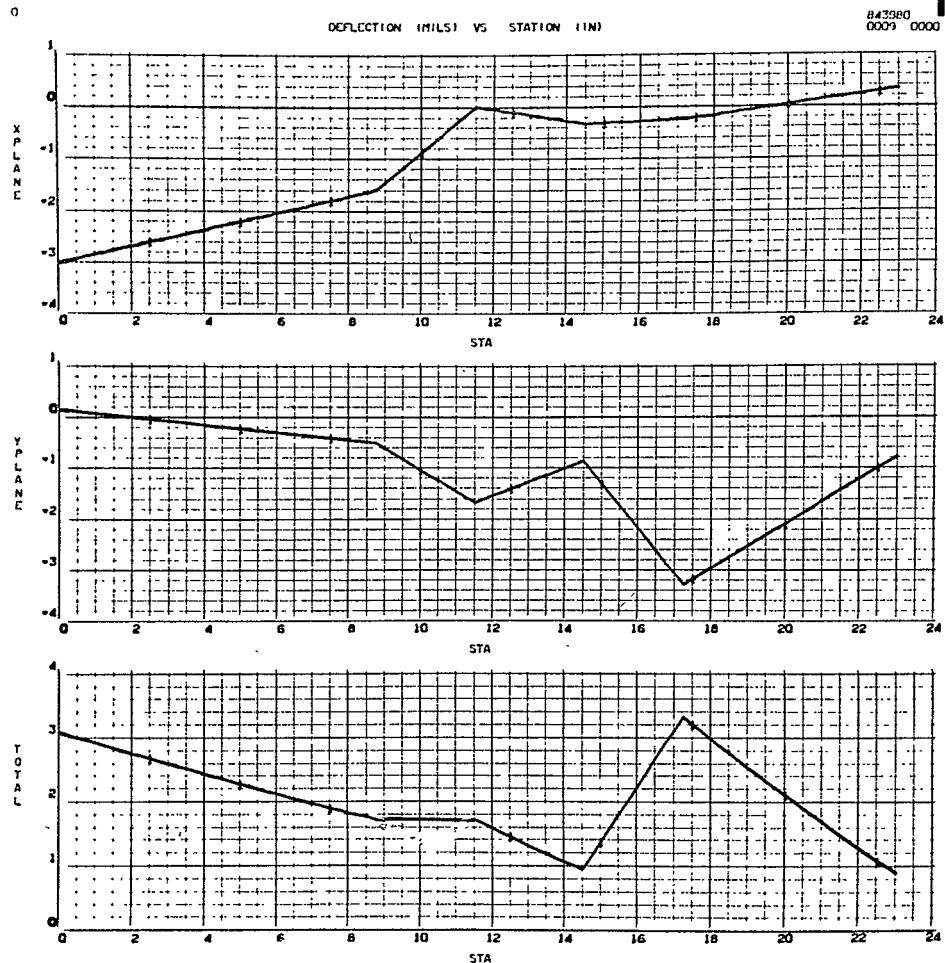


Figure 27

TEST 1138 30000 RPM

B47980
0010 0000

DEFLECTION (MILS) VS STATION (IN)

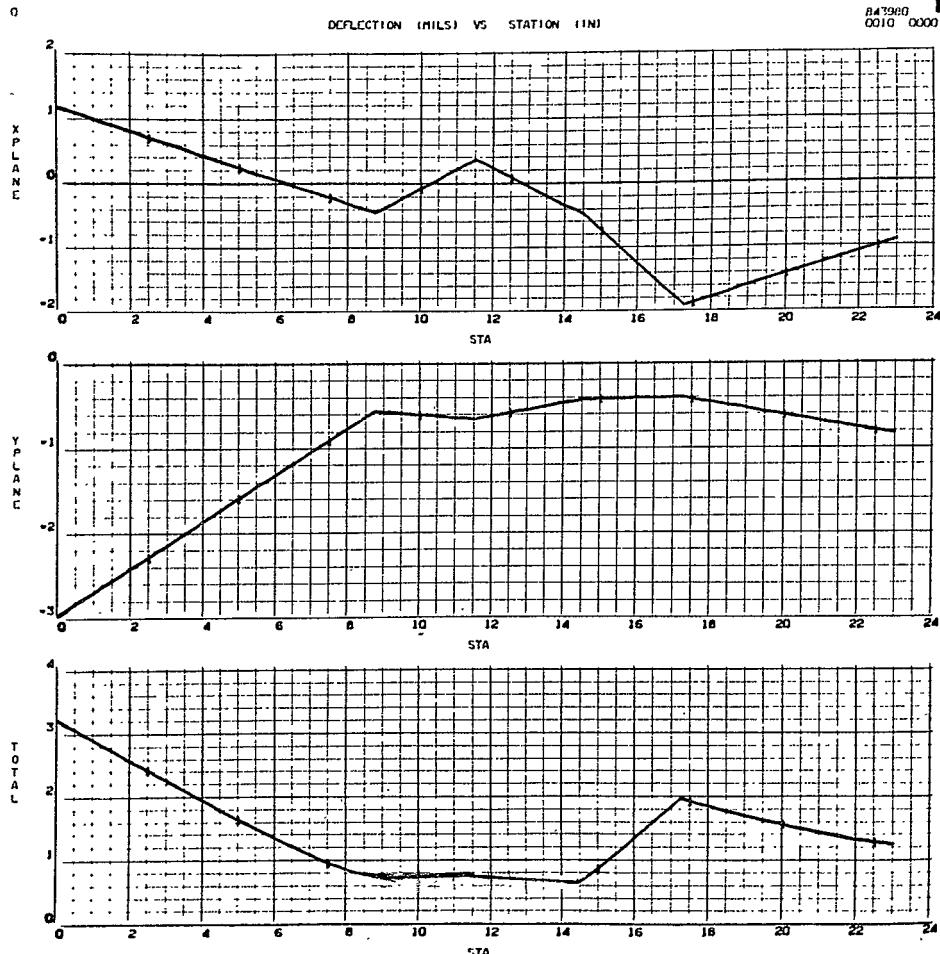
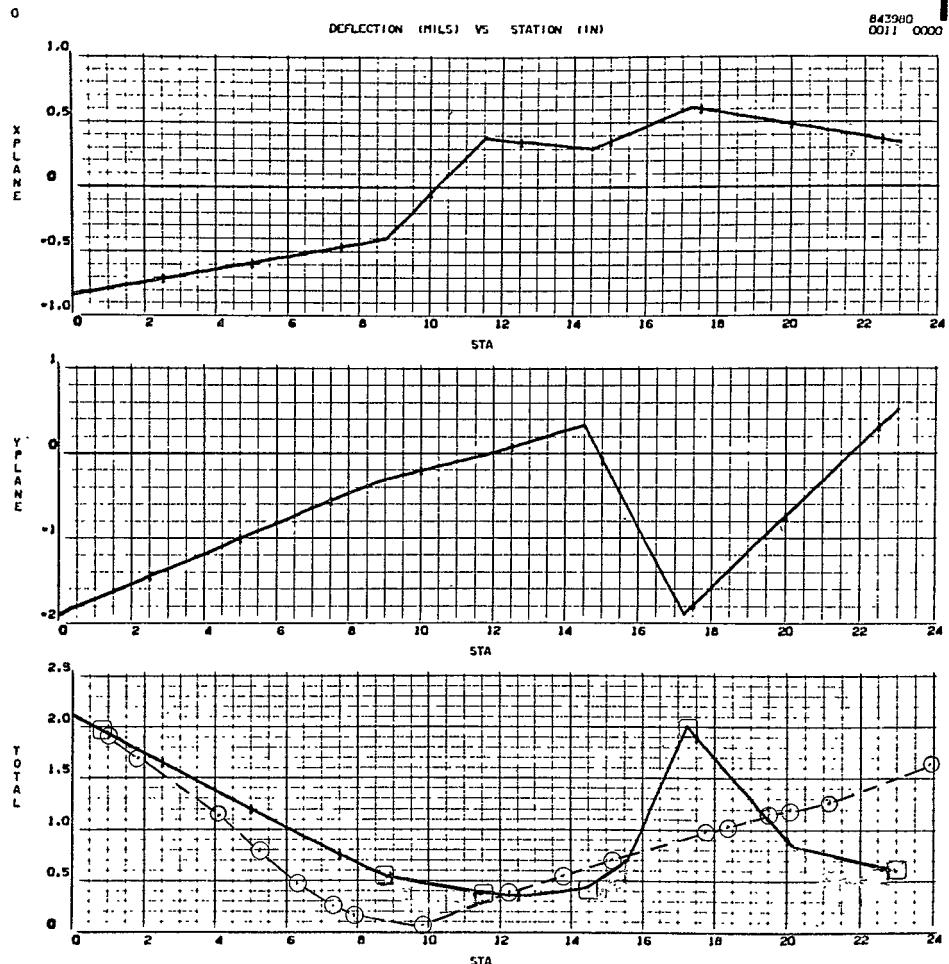


Figure 28

TEST 1138 32000 RPM

DEFLECTION (MILS) VS STATION (IN)

843980
0011 0000



○ COMPUTER SIMULATION DATA (x 2)

□ EXPERIMENTAL DATA

Figure 29

TEST 1139 28000 RPM

B43980
0012 0000

0
DEFLECTION (MILS) VS STATION (IN)

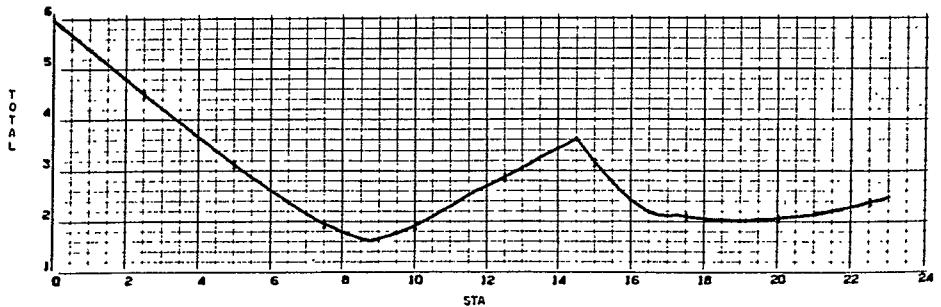
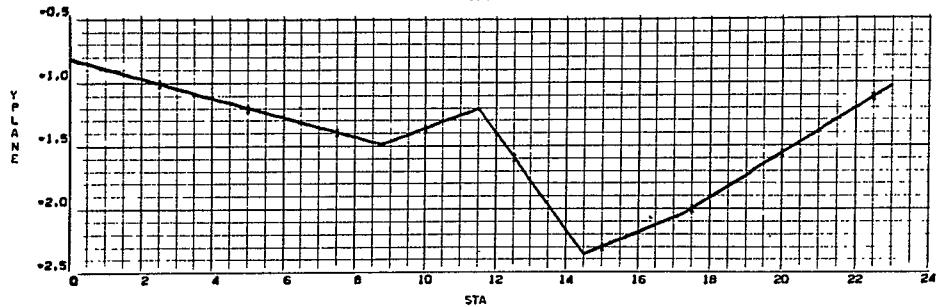
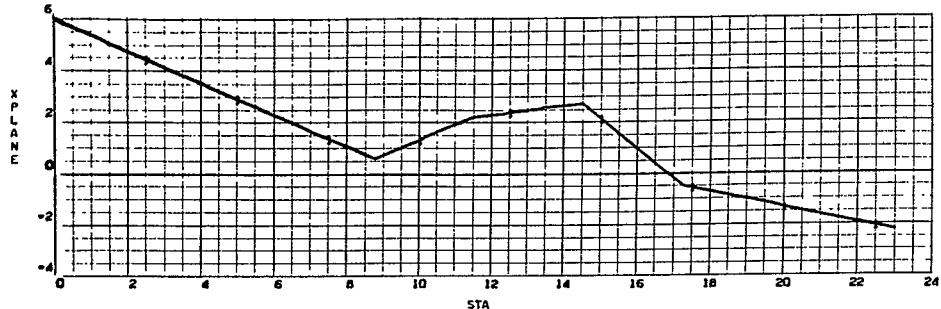


Figure 30

TEST 1139 30000 RPM

843980
0013 0000

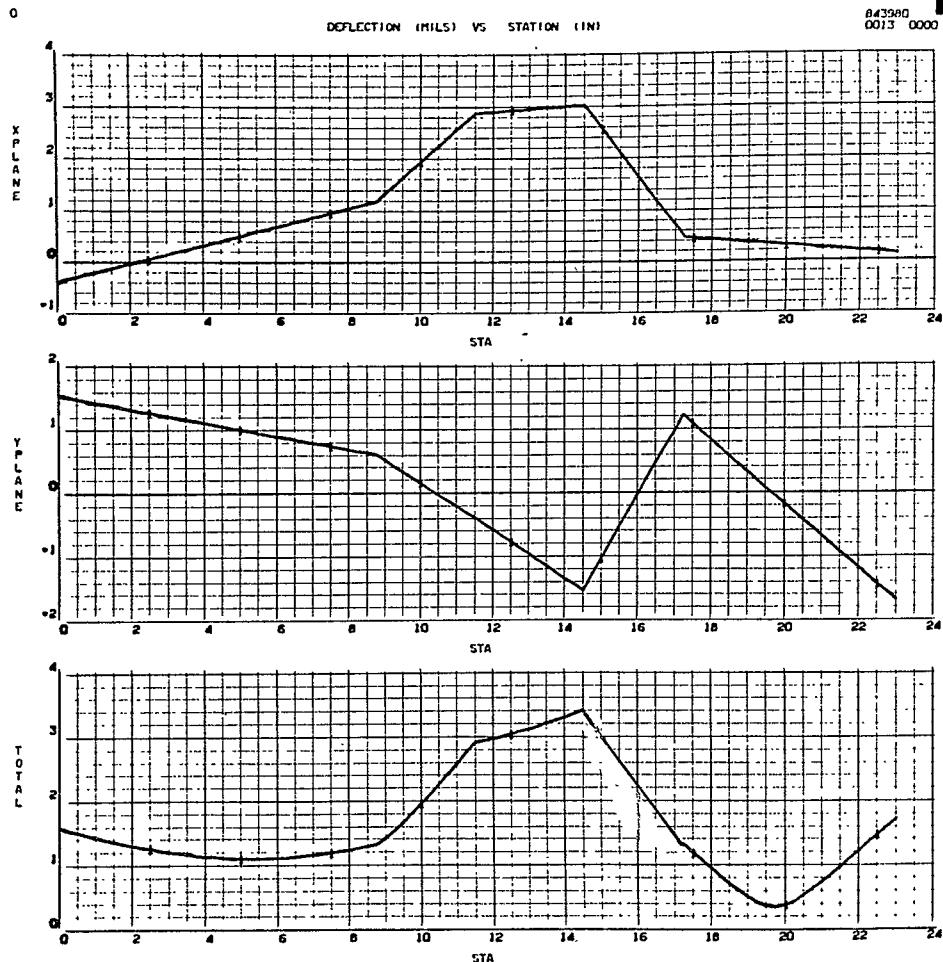


Figure 31

TEST 1139 32000 RPM

B43980
0014 0000

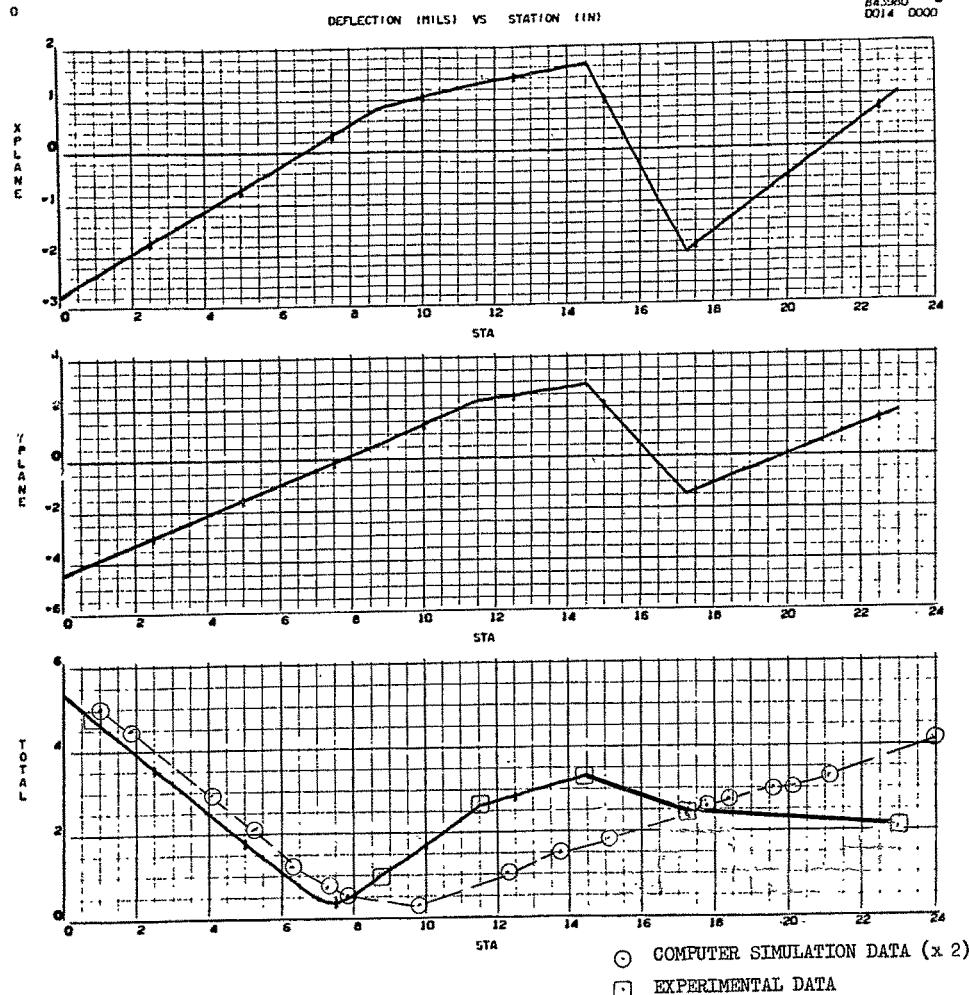


Figure 32

TEST 1142 28000 RPM

643980
0015 0000

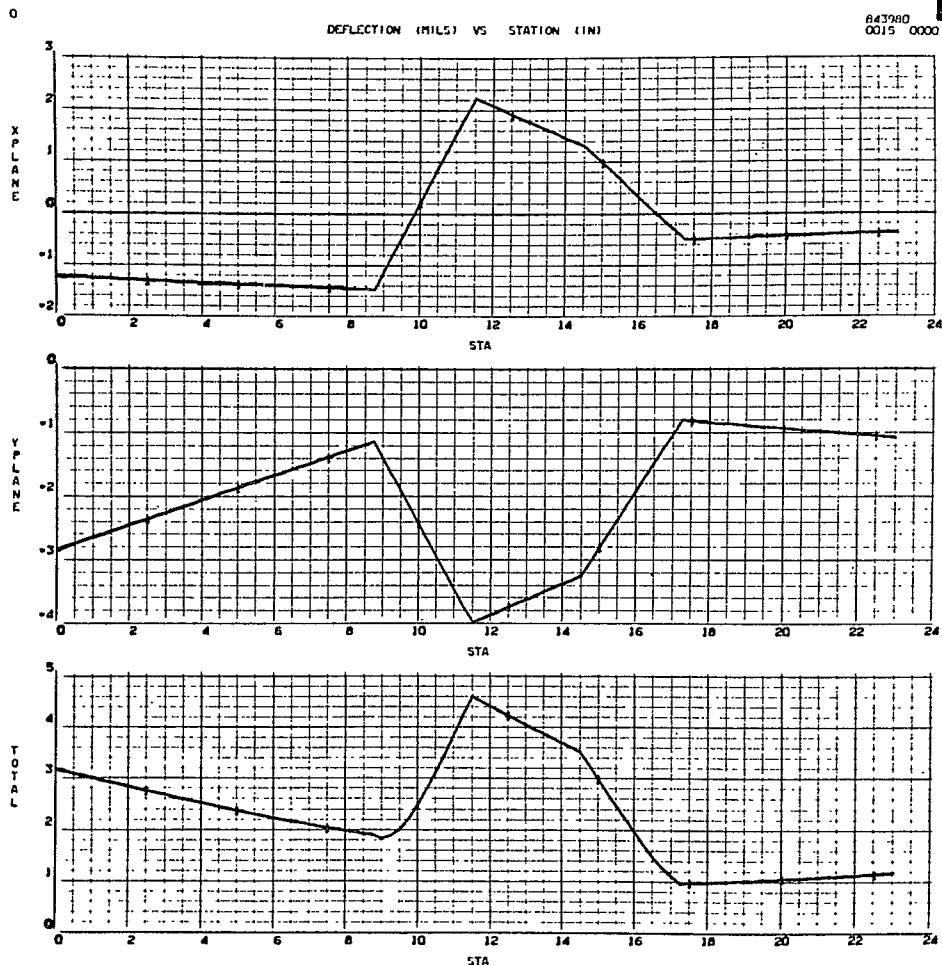


Figure 33

TEST 1142 30000 RPM

843380
0016 0000

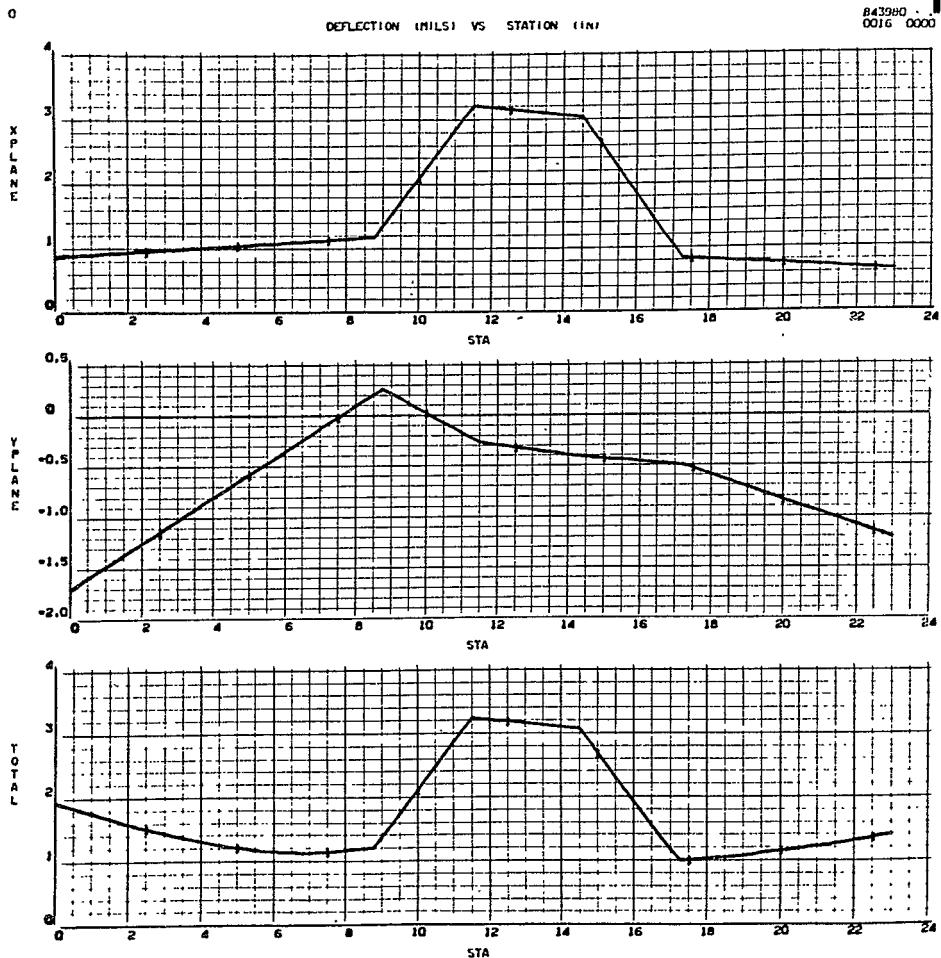


Figure 34

TEST 1142 32000 RPM

847910
0017 0000

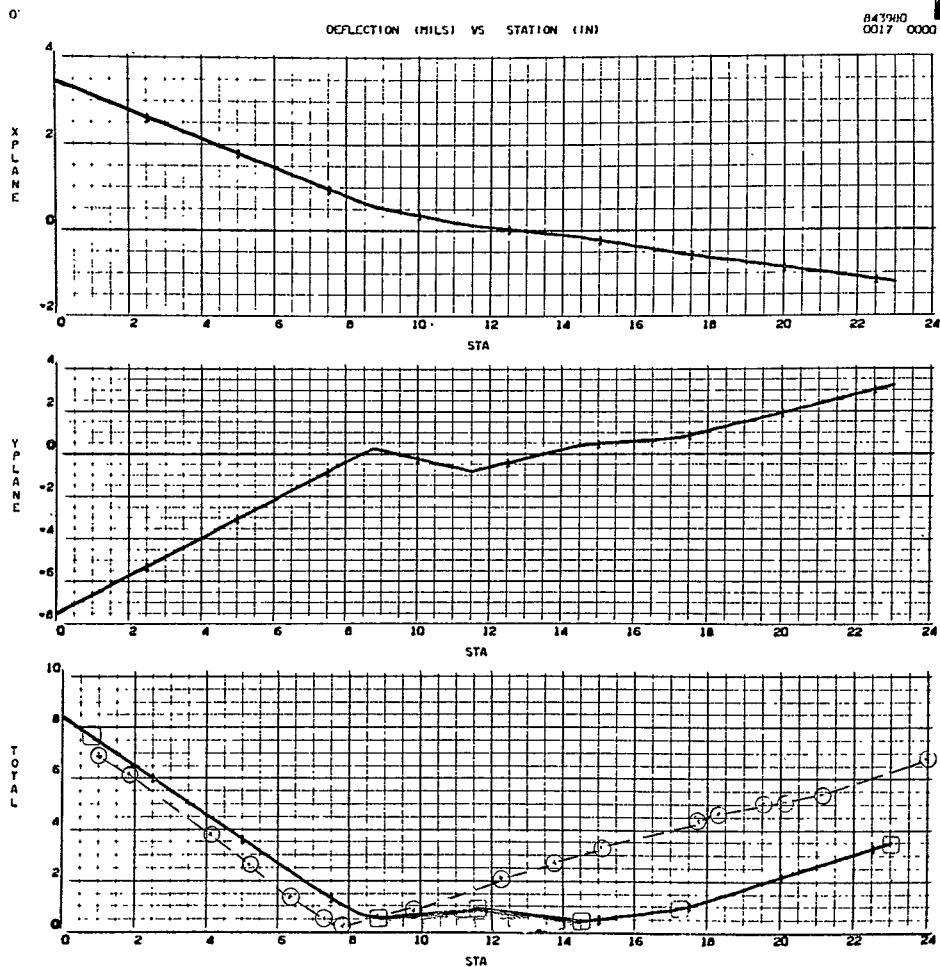


Figure 35

○ COMPUTER SIMULATION DATA (x 5)

□ EXPERIMENTAL DATA

TEST 1143 26000 RPM

842980
0042 0000

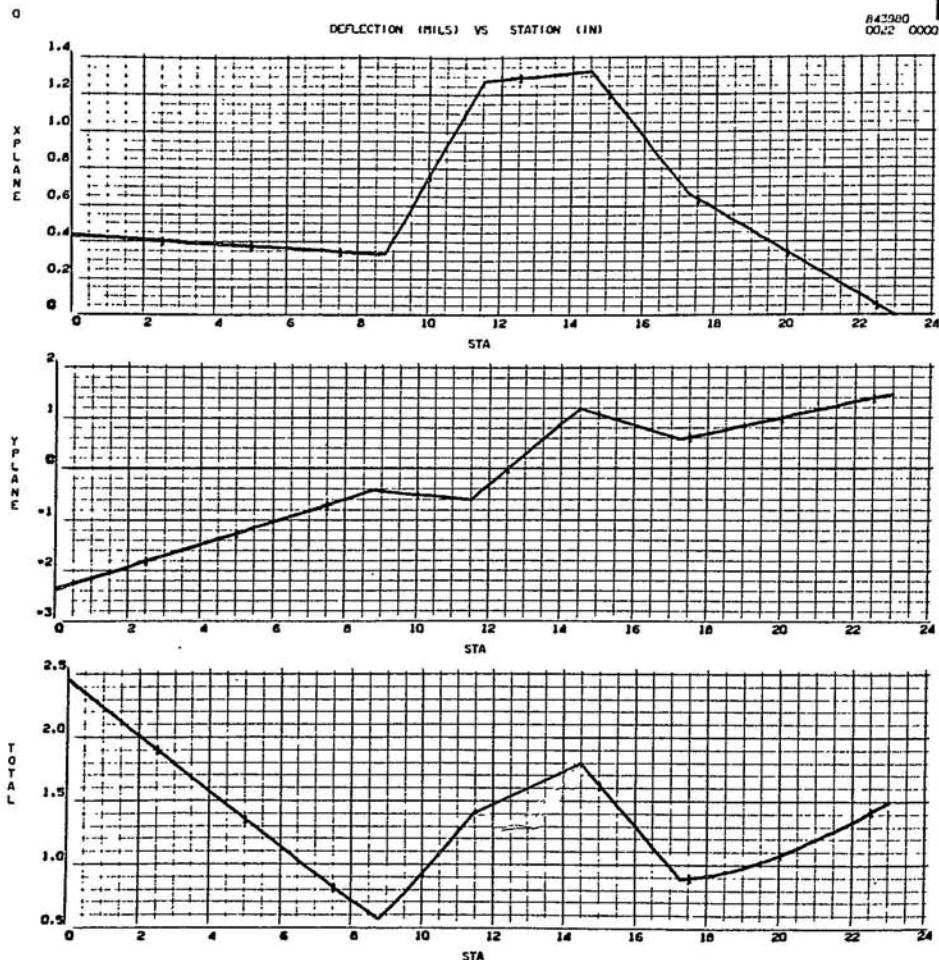


Figure 36

TEST 1143 28000 RPM

DEFLECTION (MILS) VS STATION (IN)

843980
0023 0000

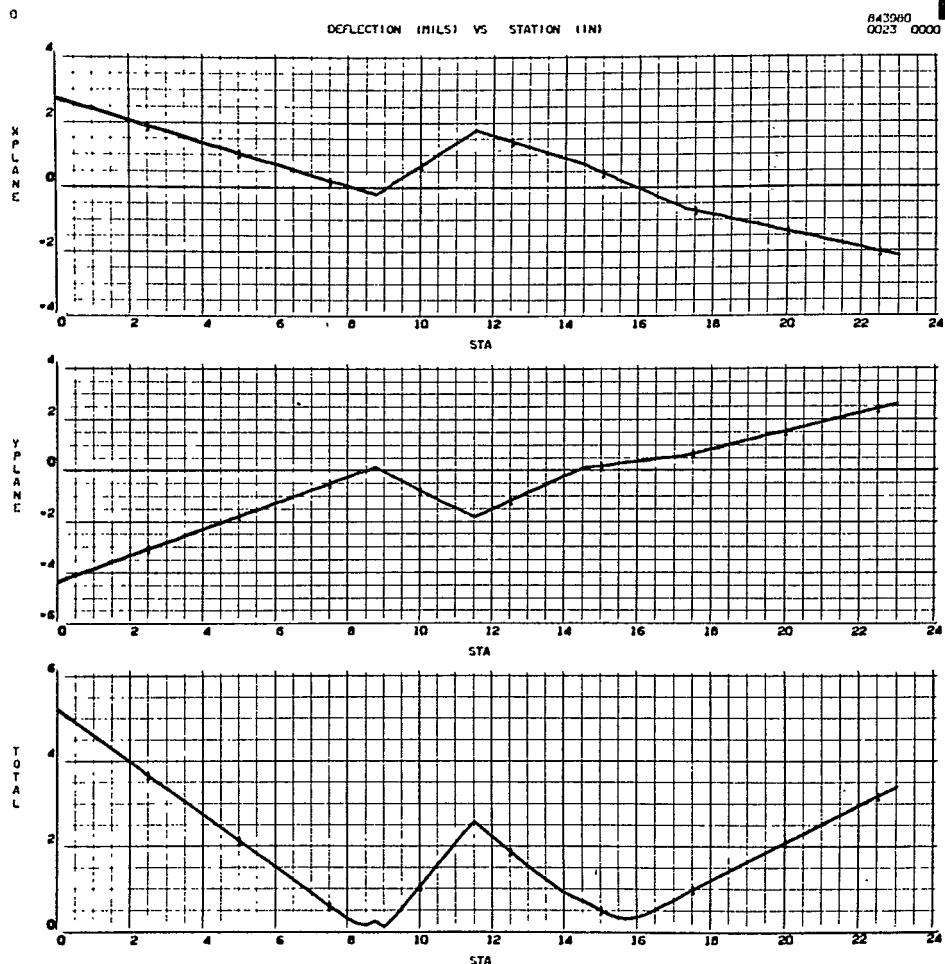


Figure 37

TEST 1143 30000 RPM

843980
0024 0000

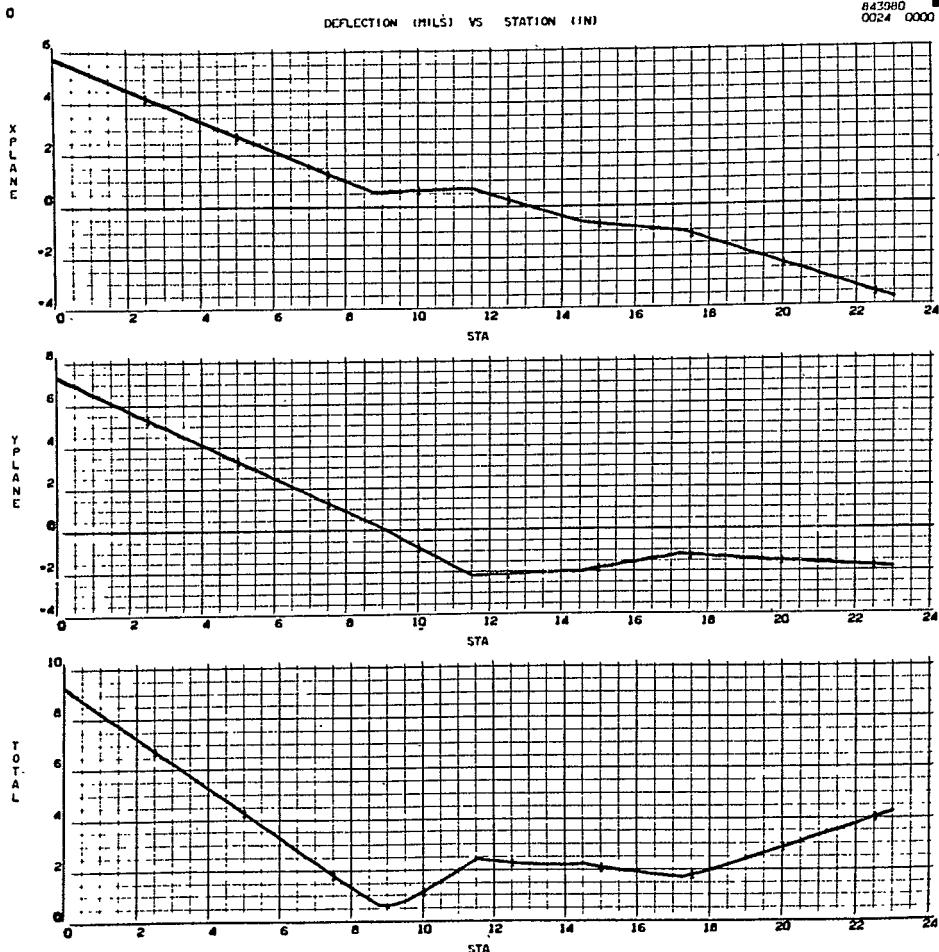


Figure 38

TEST 1144 26000 RPM

DEFLECTION (MILS) VS STATION (IN)

843960
0018 0000

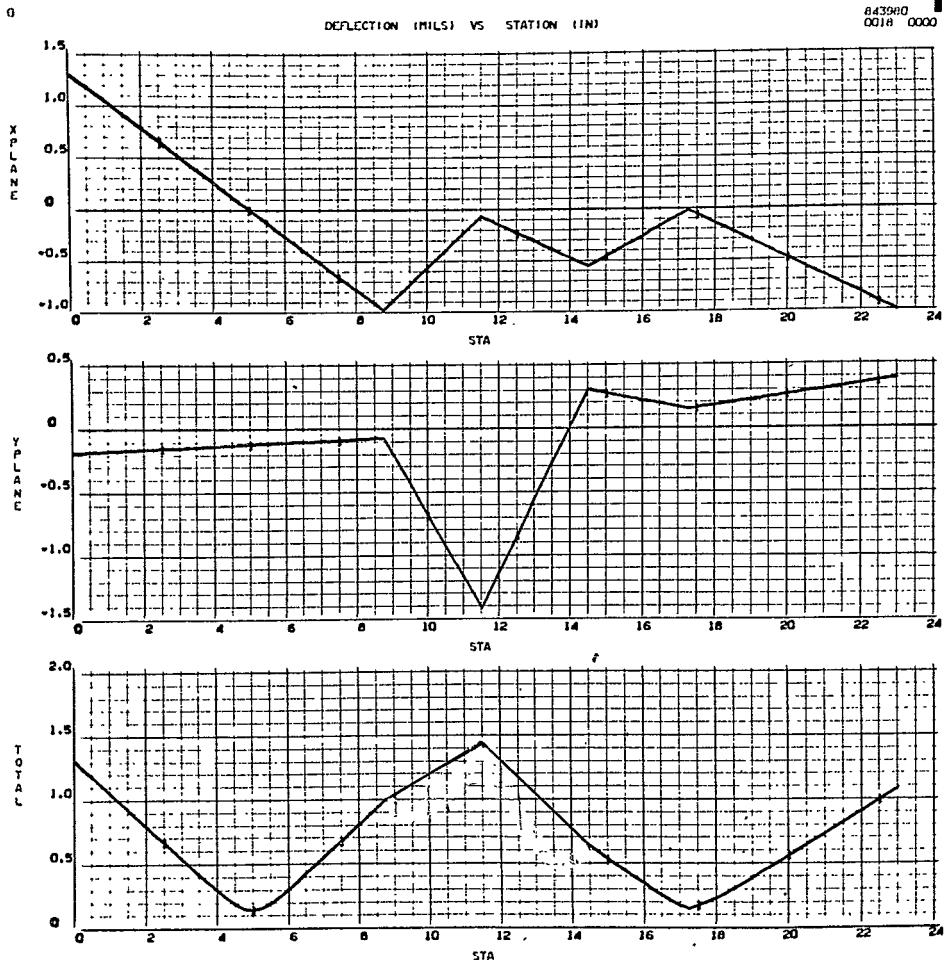


Figure 39

TEST 1144 28000 RPM

843960
0019 0000

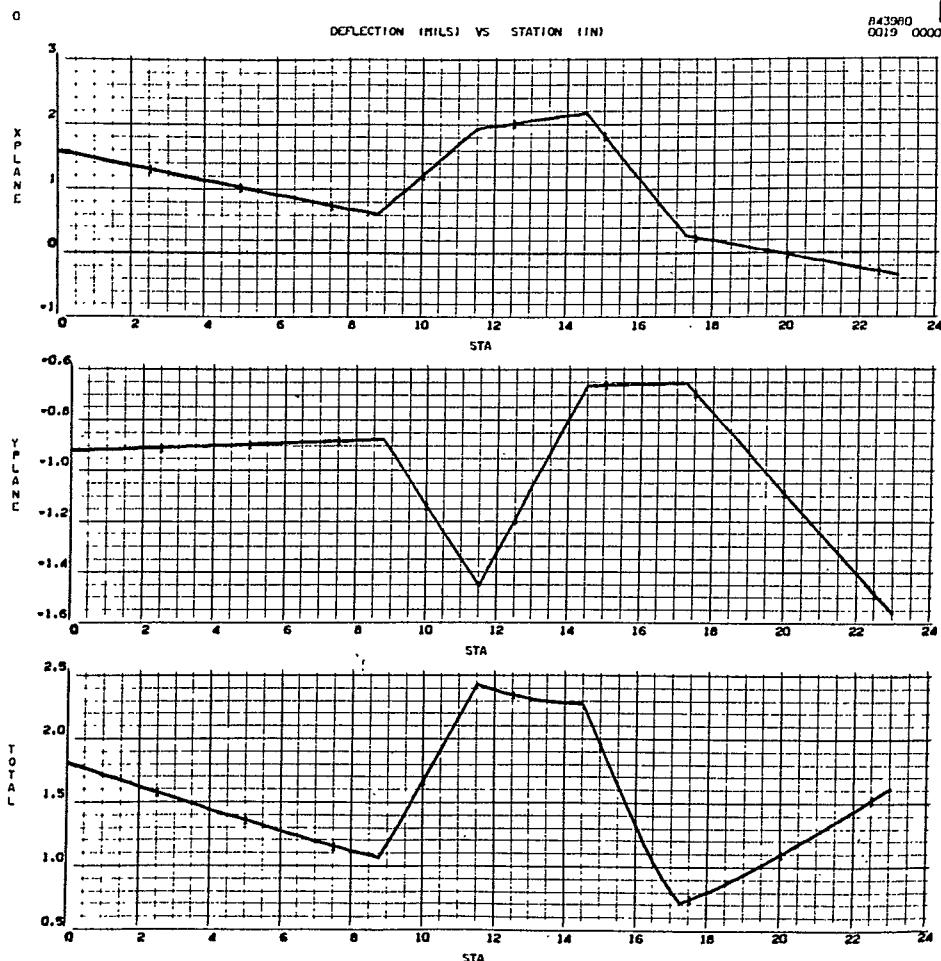


Figure 40

TEST 1144 30000 RPM

043900
000000

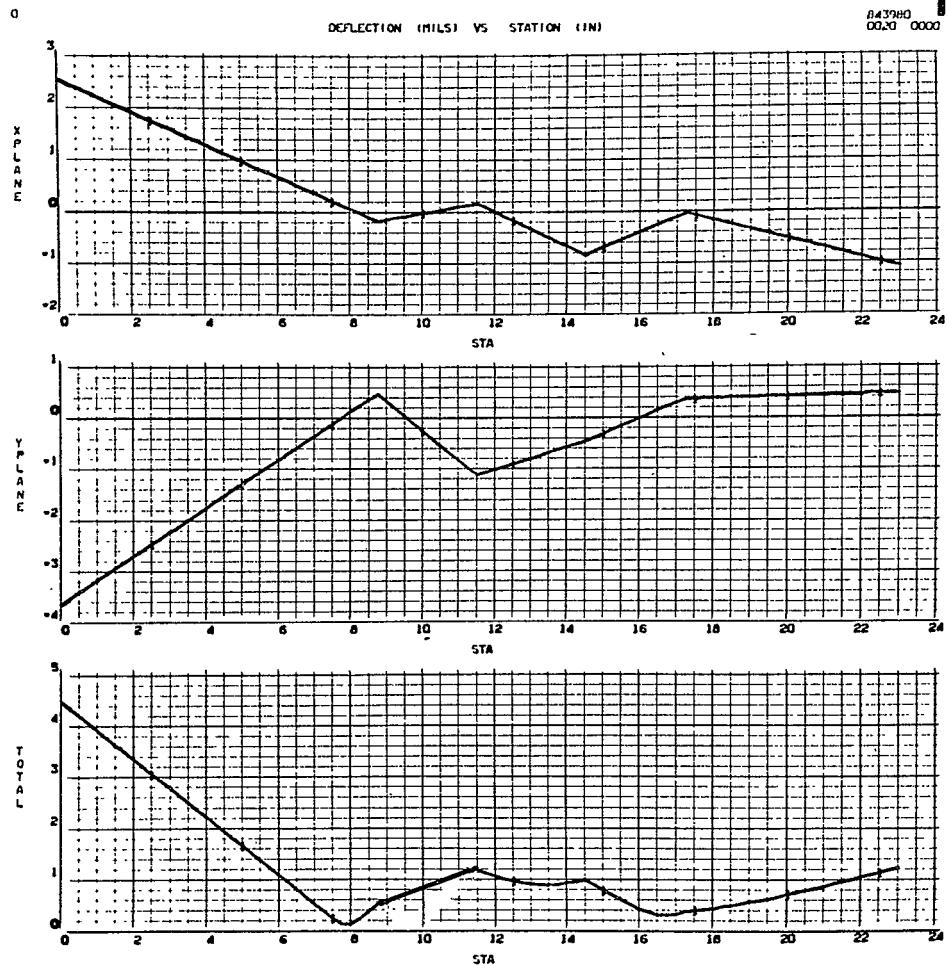


Figure 41

TEST 1145 26000 RPM

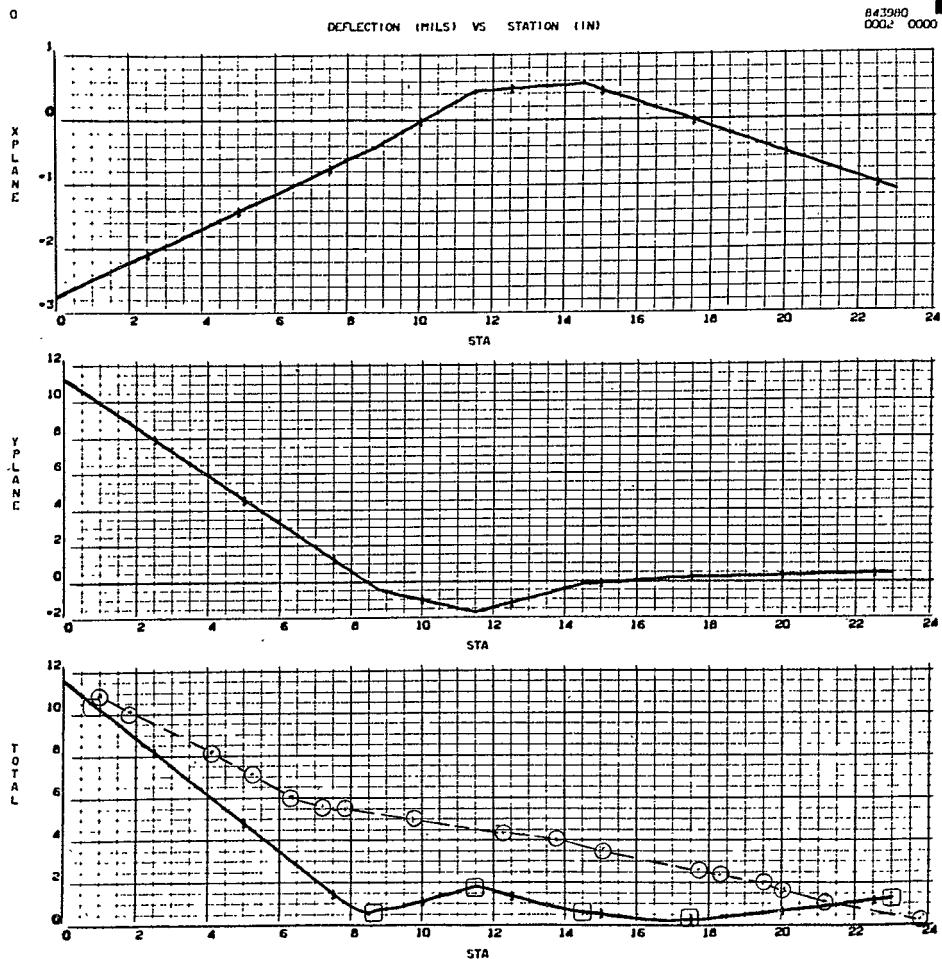


Figure 42.

TEST 1145 28000 RPM

043900
0001 0000

DEFLECTION (MILS) VS STATION (IN)

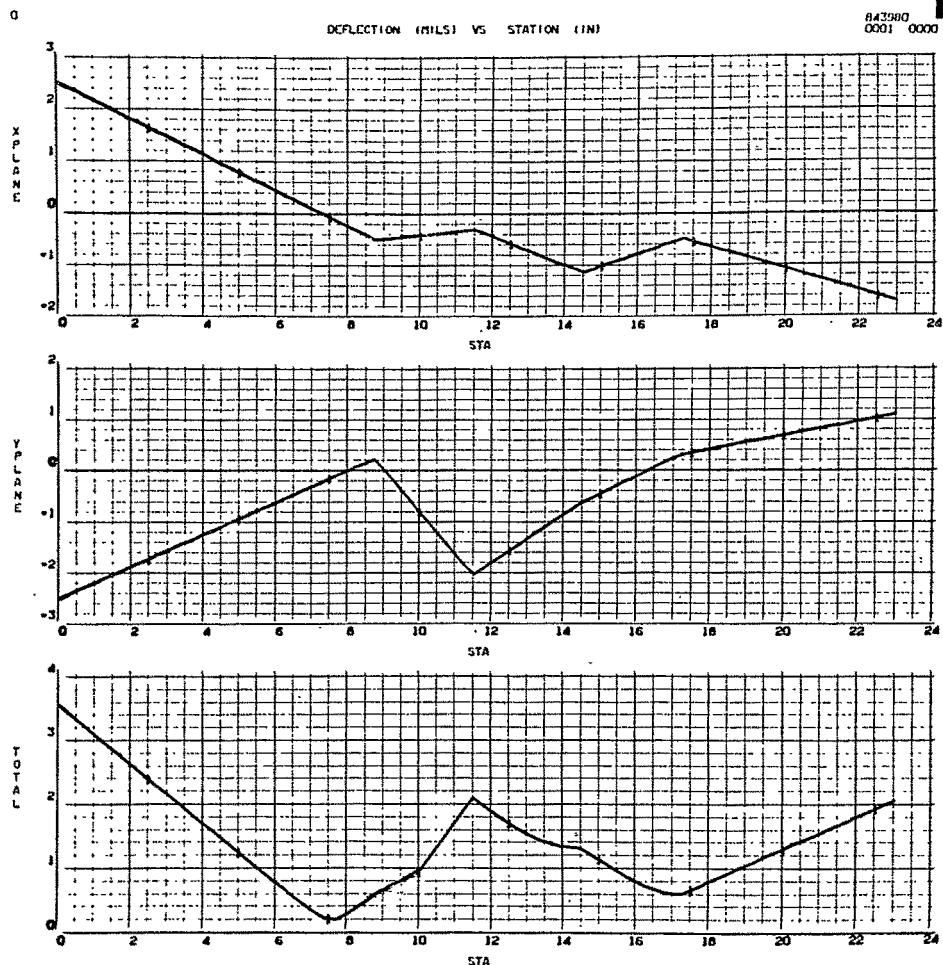


Figure 43

TEST 1145 30000 RPM

B43900
0021 0000

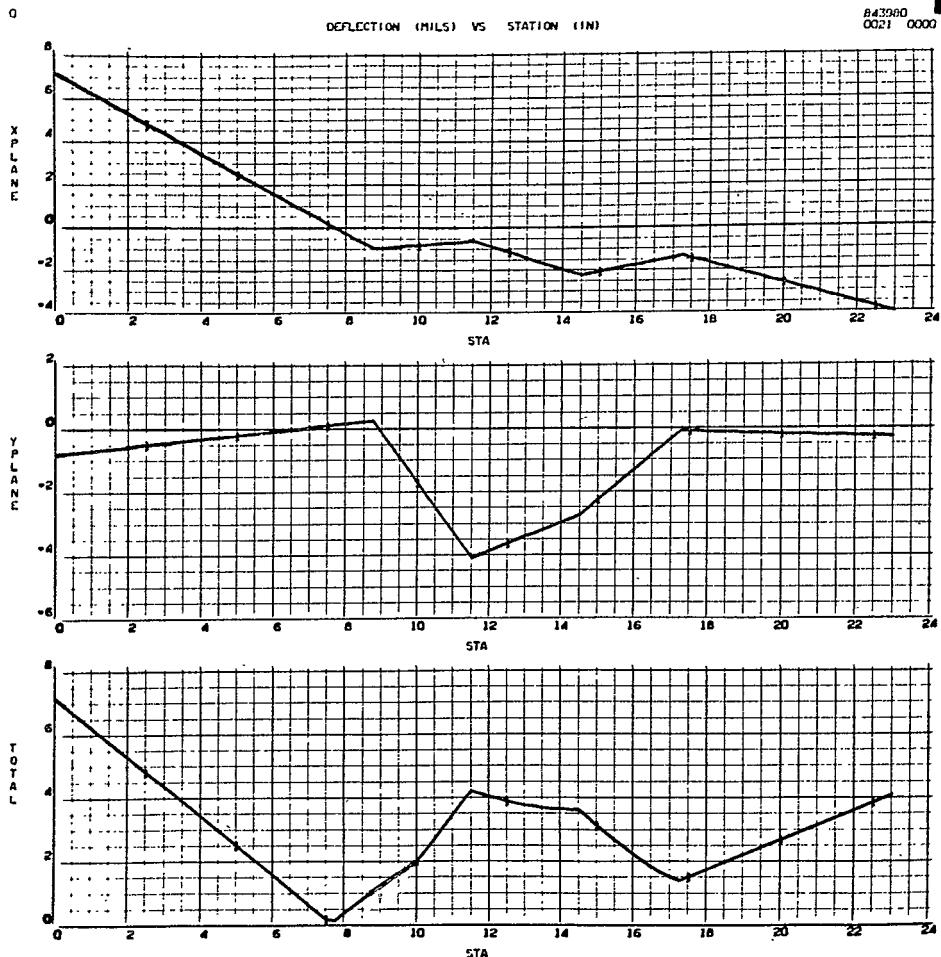


Figure 44

SIMULATION OF EXPERIMENTAL TESTS

SIMULATION MODEL

A discrete 17 mass model was constructed to describe the physical configuration of the Mark-25 pump rotor. Table 5 lists the mass and inertia properties for each of the 17 stations.

The four bearings, in two duplex pairs, were located at Stations 5, 6, 15 and 16. A speed sensitive linear stiffness characteristic was used for each of the four angular contract ball bearings.

Eight different cases of rotor mass unbalances corresponding to the experimental test runs were used in the computer program input data. These unbalance cases are listed in Table 6. Six steady state speed slices were simulated for each test run. The speeds were from 24,000 to 34,000 rpm in 2000 rpm increments.

RESULTS OF THE COMPUTER SIMULATION

The results of the computer simulation are listed in Tables 7 through 14. Although excellent correlation was achieved between the results of the new computer program versus a synchronous response program (see Table 2), large variances occurred in the simulation runs as compared to the experimental data.

For graphical comparison the simulation data is plotted on the experimental plots of deflection versus speed, and is identified by a small circle.* Also, comparison of the calculated vs. experimental mode shapes are shown in Figures 21, 24, 29, 32, 35 and 42. In some cases it was necessary to multiply the simulation data by a scale factor. However, the mode shapes compared favorably. The close agreement between the computer results and that from an existing synchronous rotordynamic response program suggests

*See Figures 13 through 20.

that the computer program is valid and able to generate accurate results. The discrepancies between the test and computer data should only result from inadequacies in test model simulation. Comparisons between the computer and test data (Figures 13 through 20) reveal,

1. Computer simulated deflections are generally smaller than the test data.
2. Within the speed range tested, reasonably negative curvatures (inverted U shape) of the deflection versus speed curves at the local maximum deflection regions are observed.
3. The local maximum deflection points from the computed data are in general at a higher speed than that from the test data.

Item 1 above suggests that the actual mass eccentricities of the test rotor may be greater than those used in the simulation. Since there are only a maximum of two unbalance locations for each unbalance configuration, it is possible that there are compensating local unbalances not detectable during the balance procedure. That is, the actual unbalance distribution of a rotor unbalance configuration is different from that used in the computer program. Although the postulate of actual eccentricities being larger than those used may tend to explain a good portion of the discrepancies between test and computer points, no such errors have been found in the test procedure.

Item 2 indicates that certain amount of damping exists. It appears that besides the small amount of inherent damping in the elastic bearing and mount components the high damping component Kirksite base for the test apparatus may contribute to the system as a significant damping parameter. These was no damping included in the computer analysis. The damping parameter has not been included in the STARUP subroutine.

Item 3 suggests that the support bearings stiffness and/or rotor rigidity are higher in the computer model than in the test mode, thus resulting in higher computed critical speeds. The test rotor is fabricated from individual disk components and is of reasonably complex structure. It is possible that the computer model may represent a more stiff rotor than the real one.

As far as the support bearing's stiffness simulation is concerned, the discrepancies between the actual value and that used in the computer program should not be very large. In the past experiences, the use of similar bearing stiffness appeared to give reasonable critical speed results comparable to those observed from related test data.

Tables 7 through 11 are copies of the actual printout from the computer simulation runs and list the following:

- a. X displacement array
- b. Y displacement array
- c. Rotor deflection vector array
- d. Rotor deflection vector phase angle array
- e. X velocity array
- f. Y velocity array
- g. Whirl frequency array
- h. Whirl to spin frequency ratio array

TABLE 5

MARK-25 PUMP ROTOR MASS AND INERTIA PROPERTIES

Station	Location Inches	Mass lb-sec ² /in	I_D , lb-in-sec ²	I_P , lb-in-sec ²	$I_D - I_P$
1	1.0	.0023293	.0018504	.0022752	- .0004218
2	1.8	.008456	.027589	.047474	- .019885
3	4.02	.093548	.142081	.244449	- .10241
4	5.22	.017604	.052015	.089431	- .037416
5	6.32	.0056835	.0028311	.0046549	- .0018238
6	7.32	.0021558	.0010739	.0017656	- .0006917
7	7.81	.017136	.04451	.079042	- .049534
8	9.83	.054180	.1450	.2515	- .15662
9	12.24	.0331708	.0934	.1585	- .08504
10	13.73	.040357	.11800	.1914	- .09338
11	15.03	.0361565	.1019	.1834	- .10151
12	17.72	.030516	.0854	.1593	- .10791
13	18.32	.030516	.0854	.1593	- .10791
14	19.53	.021697	.0608	.1175	- .07673
15	20.02	.002661	.0037044	.0023834	+ .0013210
16	21.13	.010644	.014818	.009534	+ .005289
17	23.98	.011584	.012806	.013423	- .000617

TABLE 6

MARK-25 PUMP SIMULATION DATA

<u>Run</u>	<u>Station</u>	<u>Mass₂</u> lb-sec ² /in	<u>Eccentricity, e</u>	<u>Phase, Degree</u>
1136	2	.008456	.00143"	138.0°
	5	.005684	.00341"	139.0°
1137	2	.008456	.00185"	195.0°
	5	.005684	.00425"	196.0°
1138	2	.008456	.000175"	-24.4°
	5	.005684	.000366"	-23.5°
1139	2	.008456	.000429"	-32.3°
	5	.005684	.0011"	-29.7°
1141	Base	Run	0.0"	0.0°
1142	10	.04036	.0002065"	121.0°
1143	17	.01158	.000058"	165.0°
1144	13	.03052	.0000964"	125.0°
1145	7	.01714	.000174"	115.0°

TABLE 7

TEST 1136 SIMULATION AT 26,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-3.37000D-03	-3.1209D-03	-2.4481D-03	-2.0794D-03	-1.7235D-03	-1.5516D-03	
-1.5324D-03	-1.3899D-03	-1.1926D-03	-1.0622D-03	-9.4029D-04	-6.7142D-04	
-6.10C5D-04	-4.8231D-04	-4.2161D-04	-2.9067D-04	2.9491D-05		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
2.8554D-03	2.6442D-03	2.0736D-03	1.7609D-03	1.4591D-03	1.3134D-03	
1.2971D-03	1.1763D-03	1.0920D-03	8.9876D-04	7.9555D-04	5.6788D-04	
5.1591D-04	4.0776D-04	3.5637D-04	2.4552D-04	-2.5563D-05		
ROTOR DEFLECTION VECTOR ARRAY, (IN.)						
4.4170D-03	4.0904D-03	3.2083D-03	2.7248D-03	2.2582D-03	2.0329D-03	
2.0077D-03	1.8209D-03	1.5623D-03	1.3914D-03	1.2317D-03	8.7937D-04	
7.9895D-04	6.3158D-04	5.5204D-04	3.8049D-04	3.9028D-05		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
8TT	1.3972D 02	1.3973D 02	1.3973D 02	1.3974D 02	1.3975D 02	1.3975D 02
	1.3975D 02	1.3976D 02	1.3976D 02	1.3976D 02	1.3977D 02	1.3978D 02
	1.3978D 02	1.3979D 02	1.3979D 02	1.3981D 02	3.1908D 02	
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-7.7745D 00	-7.1994D 00	-5.6458D 00	-4.7944D 00	-3.9726D 00	-3.5758D 00	
-3.5317D 00	-3.2029D 00	-2.7478D 00	-2.4471D 00	-2.1661D 00	-1.5462D 00	
-1.4047D 00	-1.1102D 00	-9.7030D-01	-6.6846D-01	6.9622D-02		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-9.1755D 00	-8.4973D 00	-6.6655D 00	-5.6616D 00	-4.6927D 00	-4.2249D 00	
-4.1722D 00	-3.7842D 00	-3.2470D 00	-2.8920D 00	-2.5601D 00	-1.8281D 00	
-1.6610D 00	-1.3132D 00	-1.1479D 00	-7.9143D-01	8.0276D-02		
WHIRL FREQUENCY ARRAY, RPM						
2.6000D 04	2.6000D-04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D-04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D-04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	

TABLE 8TEST 1137 SIMULATION AT 26.000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-5.3679D-03	-4.9705D-03	-3.8972D-03	-3.3091D-03	-2.7415D-03	-2.4676D-03	
-2.4369D-03	-2.2098D-03	-1.8957D-03	-1.6881D-03	-1.4942D-03	-1.0663D-03	
-9.6869D-04	-7.6546D-04	-6.6892D-04	-4.6066D-04	4.8728D-05		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-1.6599D-03	-1.5373D-03	-1.2059D-03	-1.0243D-03	-8.4901D-04	-7.6437D-04	
-7.5490D-04	-6.8470D-04	-5.8751D-04	-5.2327D-04	-4.6324D-04	-3.3079D-04	
-3.0056D-04	-2.3764D-04	-2.0773D-04	-1.4323D-04	1.4487D-05		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
5.6187D-03	5.2028D-03	4.0795D-03	3.4640D-03	2.8700D-03	2.5833D-03	
2.5511D-03	2.3135D-03	1.9846D-03	1.7674D-03	1.5643D-03	1.1165D-03	
1.0142D-03	8.0150D-04	7.0043D-04	4.8241D-04	5.0835D-05		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
6TT	1.9718D 02	1.9719D 02	1.9719D 02	1.9720D 02	1.9721D 02	1.9721D 02
	1.9721D 02	1.9722D 02	1.9722D 02	1.9722D 02	1.9723D 02	1.9723D 02
	1.9724D 02	1.9725D 02	1.9725D 02	1.9727D 02	1.6557D 01	
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)						
4.5156D 00	4.1856D 00	3.2834D 00	2.7889D 00	2.3120D 00	2.0817D 00	
2.0554D 00	1.8643D 00	1.5996D 00	1.4247D 00	1.2613D 00	9.0065D-01	
8.1833D-01	6.4702D-01	5.6558D-01	3.9004D-01	-3.9393D-02		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-1.4615D 01	-1.3533D 01	-1.0611D 01	-9.0097D 00	-7.4643D 00	-6.7184D 00	
-6.6349D 00	-6.0168D 00	-5.1614D 00	-4.5963D 00	-4.0682D 00	-2.9033D 00	
-2.6375D 00	-2.0841D 00	-1.8213D 00	-1.2542D 00	1.3268D-01		
WHIRL FREQUENCY ARRAY, RPM						
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01		

TABLE 9

TEST 1128 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-8.9574D-04	-7.9473D-04	-5.2262D-04	-3.7265D-04	-2.2741D-04	-1.2549D-04	
-3.7334D-05	4.1324D-05	1.8349D-04	2.6620D-04	3.3372D-04	4.6182D-04	
4.8891D-04	5.3972D-04	5.5193D-04	5.9336D-04	7.6412D-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
3.6902D-04	3.2740D-04	2.1530D-04	1.5352D-04	9.3674D-05	5.1662D-05	
3.5924D-05	-1.7135D-05	-7.5755D-05	-1.0986D-04	-1.3770D-04	-1.9052D-04	
-2.0169D-04	-2.2264D-04	-2.2767D-04	-2.4475D-04	-3.1516D-04		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
9.6876D-04	8.5953D-04	5.6523D-04	4.0303D-04	2.4594D-04	1.3571D-04	
9.4434D-05	4.4745D-05	1.9851D-04	2.8798D-04	3.6102D-04	4.9958D-04	
5.2887D-04	5.8384D-04	5.9755D-04	6.4185D-04	8.2656D-04		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
1.05761D 02	1.05761D 02	1.05761D 02	1.05761D 02	1.05761D 02	1.05762D 02	
1.05764D 02	3.3748D C2	2.3757D 02	3.03757D 02	3.03758D 02	3.03758D 02	
3.03758D 02	3.03758D C2	3.03758D 02	3.03758D 02	3.03759D 02	3.03759D 02	
X-VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)						
-1.2366D 00	-1.0971D 00	-7.2142D-01	-5.1453D-01	-3.1323D-01	-1.7316D-01	
-1.2043D-01	5.7400D-02	2.5383D-01	3.6812D-01	4.6139D-01	6.3824D-01	
6.7558D-01	7.4629D-01	7.6059D-01	8.2080D-01	1.0569D 00		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN/SEC)						
-3.0017D 00	-2.6632D 00	-1.7513D 00	-1.2487D 00	-7.6234D-01	-4.2052D-01	
-2.9264D-01	1.3852D-C1	6.1490D-01	8.9205D-01	1.1183D 00	1.5477D 00	
1.6385D 00	1.8085D CC	1.8566D 00	1.9881D 00	2.5602D 00		
WHIRL FREQUENCY ARRAY, RPM						
3.2000D 04	3.2000D C4	3.2000D C4	3.2000D 04	3.2001D 04	3.2000D 04	
3.2000D 04	3.2000D C4	3.2000D C4	3.2000D 04	3.2000D 04	3.2000D 04	
3.2000D 04	3.2000D C4	3.2000D C4	3.2000D 04	3.2000D 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.00000D 00	1.00000D 00	1.00000D 00	1.00000D 00	1.00000D 00	9.9999D-01	
9.9999D-01	1.00000D 00					
1.00000D 00	9.9999D-C1	1.00000D 00	9.9999D-01	9.9999D-01		

TABLE 10

TEST 1139 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

-2.1879D-03	-1.9412D-03	-1.2765D-03	-9.1032D-04	-5.5573D-04	-3.0770D-04
-2.1501D-04	9.7638D-05	4.4312D-04	6.4415D-04	8.0830D-04	1.1198D-03
1.1856D-03	1.3093D-03	1.3390D-03	1.4398D-03	1.8550D-03	

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

1.2669D-03	1.1241D-03	7.3520D-04	5.2711D-04	3.2173D-04	1.7785D-04
1.2403D-04	-5.7470D-05	-2.5802D-04	-3.7470D-04	-4.6998D-04	-6.5074D-04
-6.8896D-04	-7.6068D-04	-7.7794D-04	-8.3641D-04	-1.0774D-03	

ROTOR DEFLECTION VECTOR ARRAY (IN.)

2.5282D-03	2.2431D-03	1.4751D-03	1.0519D-03	6.4214D-04	3.5540D-04
2.4822D-04	1.1330D-04	5.1277D-04	7.4520D-04	9.3500D-04	1.2951D-03
1.3713D-03	1.5142D-03	1.5486D-03	1.6651D-03	2.1451D-03	

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES

121	1.4993D 02	1.4993D 02	1.4993D 02	1.4993D 02	1.4997D 02
	1.5002D 02	3.2952D 02	3.2979D 02	3.2981D 02	3.2982D 02
	3.2984D 02	3.2984D 02	3.2984D 02	3.2985D 02	3.2985D 02

X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-4.2455D 00	-3.7667D 00	-2.4770D 00	-1.7666D 00	-1.0768D 00	-5.9614D-01
-4.1573D-01	1.9254D-01	8.6457D-01	1.2556D 00	1.5748D 00	2.1803D 00
2.3081D 00	2.5498D 00	2.6017D 00	2.8042D 00	3.6118D 00	

Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-7.3317D 00	-6.5050D 00	-4.2778D 00	-3.0504D 00	-1.8631D 00	-1.0310D/00
-7.2045D-01	3.2721D-C1	1.4850D 00	2.1586D 00	2.7087D 00	3.7527D 00
3.9736D 00	4.3869D 00	4.49C2D 00	4.8240D 00	6.2150D 00	

WHIRL FREQUENCY ARRAY, RPM

3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04
3.2000D 04					
3.2000D 04					

WHIRL TO SPIN FREQUENCY RATIO ARRAY

1.0000D 00	9.9999D-01				
1.0000D 00					
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01	

TABLE 11

TEST 1142. SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

-7.5056D-04	-6.6020D-04	-4.16C1D-04	-2.8211D-04	-1.5348D-04	-6.0035D-05
-2.3716D-05	9.5442D-05	2.2516D-04	2.9994D-04	3.6124D-04	4.7614D-04
5.0023D-04	5.4497D-04	5.5465D-04	5.9021D-04	7.4478D-04	

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)

1.1668D-03	1.0263D-03	6.4671D-04	4.3856D-04	2.3859D-04	9.3327D-05
3.6865D-05	-1.4837D-04	-3.5022D-04	-4.6628D-04	-5.6157D-04	-7.4018D-04
-7.7764D-04	-8.4719D-04	-8.6223D-04	-9.1751D-04	-1.1578D-03	

ROTOR DEFLECTION VECTOR ARRAY (IN.)

1.3873D-03	1.2203D-03	7.6856D-04	5.2146D-04	2.8370D-04	1.1097D-04
4.3837D-05	1.7642D-04	4.1619D-04	5.5442D-04	6.6772D-04	8.8010D-04
9.2464D-04	1.0073D-03	1.0252D-03	1.0910D-03	1.3767D-03	

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES

122	1.2275D 02	1.2275D C2	1.2275D 02	1.2275D C2	1.2275D 02	1.2275D 02
	1.0275D 02	3.0275D C2	3.0275D 02	3.0275D 02	3.0275D 02	3.0275D 02
	3.0275D 02	3.0275D C2	3.0275D 02	3.0275D 02	3.0275D 02	3.0275D 02

X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-3.9099D 00	-3.4392D C0	-2.1671D 00	-1.4697D 00	-7.9891D-01	-3.1278D-01
-1.2359D-01	4.9718D-01	1.1729D 00	1.5625D 00	1.8818D 00	2.4802D 00
2.6056D 00	2.8391D C0	2.8871D 00	3.0752D C0	3.8806D 00	

Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)

-2.0515D 00	-2.2124D C0	-1.3941D 00	-9.4523D-01	-5.1536D-01	-2.0110D-01
-7.9393D-02	3.1986D-C1	7.5456D-01	1.0052D 00	1.2106D C0	1.5959D 00
1.6767D 00	1.8259D C0	1.8623D 00	1.9768D 00	2.4945D 00	

WHIRL FREQUENCY ARRAY, RPM

3.2000D 04	3.2000D C4	3.2000D 04	3.2000D 04	3.2001D 04	3.1999D 04
3.1999D 04	3.2000D 04				
3.2000D 04	3.2000D C4	3.2001D C4	3.2000D 04	3.2000D 04	

WHIRL TO SPIN FREQUENCY RATIO ARRAY

1.0000D 00	1.0000D C0	1.0000D 00	1.0000D 00	1.0000D 00	9.9998D-01
9.9998D-01	1.0000D 00	1.0000D C0	1.0000D 00	1.0000D 00	1.0000D 00
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01	

TABLE 12

TEST 1143 SIMULATION AT 32,000 rpm

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-3.0738D-04	-2.7112D-04	-1.7318D-04	-1.1944D-04	-6.7755D-05	-3.0700D-05	
-1.6576D-05	2.9509D-05	7.9637D-05	1.0850D-04	1.3181D-04	1.7536D-04	
1.8448D-04	2.0137D-04	2.0486D-04	2.1706D-04	2.6845D-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
7.2367D-05	6.3830D-05	4.0771D-05	2.8121D-05	1.5952D-05	7.2277D-06	
3.9025D-06	-6.9473D-06	-1.8749D-05	-2.5545D-05	-3.1031D-05	-4.1284D-05	
-4.3433D-05	-4.7408D-05	-4.8231D-05	-5.1102D-05	-6.3201D-05		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
3.1578D-04	2.7853D-04✓	1.7791D-04	1.2271D-04	6.9608D-05	3.1539D-05	
1.7029D-05	3.0316D-05✓	8.1814D-05✓	1.1147D-04	1.3541D-04✓	1.8015D-04	
1.8953D-04✓	2.0687D-04	2.1C46D-04	2.2299D-04	2.7578D-04✓		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
1.6675D 02	1.6675D 02✓	1.6675D 02	1.6675D 02	1.6675D 02	1.6675D 02	
1.6675D 02	3.4675D C2✓	3.4675D 02✓	3.4675D 02	3.4675D 02✓	3.4675D 02	
3.4675D 02✓	3.4675D C2	3.4675D 02	3.4675D 02	3.4675D 02✓		
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-2.4249D-01	-2.1388D-01	-1.3660D-01	-9.4266D-02	-5.3219D-02	-2.4235D-02	
-1.3096D-02	2.3274D-02	6.2819D-02	8.5592D-02	1.0397D-01	1.3827D-01	
1.4545D-01	1.5895D-01	1.6C80U-01	1.7147D-01	2.1208D-01		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-1.0300D 00	-9.0853D-01	-5.8C32D-01	-4.0025D-01	-2.2711D-01	-1.0287D-01	
-5.5542D-02	9.8887D-02	2.6687D-01	3.6360D-01	4.4169D-01	5.8764D-01	
6.1824D-01	6.7476D-01	6.8671D-01	7.2730D-01	8.9949D-01		
WHIRL FREQUENCY ARRAY, RPM						
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04	
3.2000D 04	3.2000D C4	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	
3.2000D 04	3.2000D C4	3.2001D 04	3.2000D 04	3.2000D 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	
9.9999D-01	1.0000D 00					
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01	9.9999D-01	

TABLE 13

TEST 1144 SIMULATION AT 32,000 RPM

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-2.0000D-04	-4.6778D-04	-2.9749D-04	-2.0408D-04	-1.1427D-04	-4.9597D-05	
-2.4787D-05	5.6315D-05	1.4456D-04	1.9539D-04	2.3644D-04	3.1327D-04	
3.2943D-04	3.5970D-04	3.6639D-04	3.9057D-04	4.9461D-04		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
7.0100D-04	6.2639D-04	3.9836D-04	2.7328D-04	1.5301D-04	6.6414D-05	
3.0192D-05	-7.5400D-05	-1.6357D-04	-2.6164D-04	-3.1660D-04	-4.1949D-04	
-4.4113D-04	-4.8167D-04	-4.0762D-04	-5.2300D-04	-6.6231D-04		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
8.8715D-04	7.8178D-04	4.9718D-04	3.4108D-04	1.9097D-04	8.2889D-05	
4.0142D-05	9.4116D-05	2.6159D-04	3.2654D-04	3.9514D-04	5.2356D-04	
2.5057D-04	6.0116D-04	6.1233D-04	6.5274D-04	8.2661D-04		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
124	1.2675D 02	1.2675D 02				
	1.0675D 02	3.0675D 02				
	3.0675D 02					
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-2.3815D 00	-2.0090D 00	-1.3349D 00	-9.0158D-01	-5.1233D-01	-2.2258D-01	
-1.5112D-01	2.5269D-01	6.4864D-01	8.7673D-01	1.0609D 00	1.4056D 00	
1.6478D 00	1.6142D 00	1.6426D 00	1.7530D 00	2.2199D 00		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC)						
-1.7768D 00	-1.5676D 00	-9.9695D-01	-6.8381D-01	-3.8351D-01	-1.6616D-01	
-8.3017D-02	1.8873D-01	4.8444D-01	6.5477D-01	7.9235D-01	1.0500D 00	
1.01042D 00	1.2052D 00	1.2299D 00	1.3082D 00	1.6567D 00		
WHIRL FREQUENCY ARRAY, RPM						
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04	
3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	3.2000D 04	
3.2000D 04	3.2000D 04	3.2001D 04	3.2000D 04	3.2000D 04		
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01	
9.9998D-01	1.0000D 00					
1.0000D 00	9.9999D-01	1.0000D 00	9.9999D-01	9.9999D-01		

TABLE 14**TABLE 1145 // SIMULATION AT 26,000 RPM**

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
-1.2147D-04	-1.1321D-04	-5.1C15D-05	-7.8748D-05	-6.6757D-05	-6.1523D-05	
-6.1292D-05	-5.6349D-05	-4.9176D-05	-4.4338D-05	-3.9749D-05	-2.9490D-05	
-2.7140D-05	-2.2215D-05	-1.9795D-05	-1.4632D-05	-2.3146D-06		
Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES, (IN.)						
2.4444D-04	2.2783D-04	1.8216D-04	1.5847D-04	1.3434D-04	1.2381D-04	
1.2335D-04	1.1340D-04	6.8963D-05	8.9226D-05	7.9992D-05	5.9347D-05	
5.4617D-05	4.4706D-05	3.9835D-05	2.9447D-05	4.6579D-06		
ROTOR DEFLECTION VECTOR ARRAY (IN.)						
2.7296D-04	2.5440D-04	2.0453D-04	1.7696D-04	1.5002D-04	1.3825D-04	
1.3773D-04	1.2663D-04	1.1C15D-04	9.9634D-05	8.9324D-05	6.6270D-05	
6.0988D-05	4.9921D-05	4.4482D-05	3.2882D-05	5.2013D-06		
PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES						
125	1.1642D 02	1.1642D 02				
	1.1642D 02	1.1642D 02				
	1.1642D 02	1.1642D 02				
X VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC.)						
-6.6555D-01	-6.2030D-01	-4.9869D-01	-4.3148D-01	-3.6577D-01	-3.3709D-01	
-3.3583D-01	-3.0875D-01	-2.6945D-01	-2.4294D-01	-2.1779D-01	-1.6158D-01	
-1.4670D-01	-1.2172D-C1	-1.0845D-01	-8.0175D-02	-1.2684D-02		
Y VELOCITY ARRAY IN STATIONARY COORDINATES, (IN./SEC.)						
-3.3072D-01	-3.0824D-C1	-2.4781D-01	-2.1441D-01	-1.8177D-01	-1.6754D-01	
-1.6688D-01	-1.5342D-01	-1.3389D-01	-1.2072D-01	-1.0823D-01	-8.0295D-02	
-7.3896D-02	-6.0484D-02	-5.3910D-02	-3.9839D-02	-6.2989D-03		
WHIRL FREQUENCY ARRAY, RPM						
2.6000D 04	2.6000D C4	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	2.6000D 04	
2.6000D 04	2.6000D C4	2.6000D C4	2.6000D 04	2.6000D 04	2.6000D 04	
WHIRL TO SPIN FREQUENCY RATIO ARRAY						
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	
1.0000D 00	1.0000D 00	1.0000D 00	1.0000D 00	9.9999D-01		

CONCLUSIONS AND RECOMMENDATIONS

CONCLUSIONS

The Rotordynamics Computer program, as developed under this contract, can serve as a useful tool in predicting the dynamic performance of a rotor-bearing system under various operating conditions. The drive and dissipation torque effects on an accelerating or decelerating rotor can be analyzed with this program.

In addition, the causes and effects of non-synchronous whirl rotor motion may also be studied by applying appropriate out-of-phase stiffness and positive or negative damping excitations. The computer program can be used as an analytical study tool for a general transient spin-speed and/or non-axisymmetric rotor motion.

The computer results of the simulation did not show good agreement, in general, with the test data. Some of the causes for the poor correlation may have been due to unknown characteristics not included in the simulation model. Some of these characteristics are:

1. unknown damping coefficient associated with bearing,
2. mount damping,
3. test mount pedestal stiffness and mass considerations,
4. bearing non-linear description and
5. internal hysteretic damping of the rotor.

In addition, the original intent to iterate the model was not funded; therefore, the time and funds to accomplish improved correlation between the computer results and the experimental data was not available. These items are discussed in the recommendations.

RECOMMENDATIONS

The period of the current contract provided adequate time to update the computer program and to obtain verification. An immediate follow-on effort is needed to study and correlate the integration process with the basic mathematical formulation such that a thorough understanding may be achieved. This would lead to possible improvement in computation-to-real time ratio. Other items needed to be included are the transverse dissipation, the out-of-phase stiffness and damping functions, and the non-linear stiffness bearing in the "STARUP" subroutine.

In addition, an iteration or parametric study is required to improve the Mark-25 model so that a better degree of correlation may be achieved between the simulation results and experimental data.

Recommendations of continued future efforts to improve capability of the computer program are divided into 3 groups, with group I representing the most important items, then group II, and then group III with the least important items.

Group I:

- a. In depth study of the integrations between the basic mathematical formulation and integration process in an attempt to improve computation-to-real time ratio.
- b. Up-date the "STARUP" subroutine to include other rotordynamics parameters such as:
 1. Rotor drive torque and dissipation functions.
 2. Rotor in-phase damping and in-phase and out-of-phase stiffness and damping parameters.

3. Non-linear bearing stiffness characteristics.

Group II:

- a. Inclusion of the Bearing Mass Parameter. The bearing mass will be simulated by a concentrated mass located between a bearing and its mount. For a bearing with soft mount design and particularly in high speed operation, the bearing mass effects should be considered. In addition to the bearing mass parameter, the bearing transverse mass moment of inertia also affects the rotordynamic performance. For normal small operating bearing misalignment, the effects of the transverse mass moment of inertia parameter are relatively small and its inclusion in the rotordynamic study would be optional.
- b. Formulation of Hydrodynamic Force Excitation Functions. Particular emphases are placed on the axial pump impeller to stator interactions. A linear hydrodynamic force characteristics should be sufficient at this stage of the program development. These force characteristics will include in-phase and out-of-phase excitations which may also be functions of spin and/or whirl frequencies. The hydrodynamic excitations of the close-clearance flow-control surfaces of the balance piston such as that used in the Mark-25 axial pumps were found to be small compared to other rotordynamic loads and hence they will not be included.
- c. Feasibility Study of Advanced Rotating-Coordinate Systems. A feasibility study is suggested to investigate the practicality of using advanced rotating coordinates to facilitate the integration process by smoothing out the second order rotordynamic function fluctuations. The second order function

fluctuations may result from cyclic force variations of the rotor mass unbalance in a non-synchronous whirl rotor motion or from non-isotropic bearing mount force characteristics.

Group III:

Inclusion of the below listed parameters as discussed:

- a. Rotor Casing Mass and Transverse Mass Moment of Inertia, and the Casing Support Stiffness and Damping Parameter. In these suggested parameters, a rigid casing is assumed. The casing is considered to be supported on an inertial frame of reference (foundation) through elastic and damping members. The stiffness and damping parameters include both force and moment functions of displacements and misalignments respectively.
- b. Rotor Material Hysteresis. This parameter includes also the effects of press-fit and/or bolted joints in an elastic rotor. This hysteresis parameter will include both in-phase and out-of-phase damping characteristics.
- c. Transverse Sinusoidal Ambient Vibrations. A fixed-amplitude and constant-frequency sinusoidal vibration is considered to be imposed on an otherwise fixed foundation. It appears that for a rotor-bearing system operating under a high-frequency ambient vibration condition, the computer time may become excessive unless only the steady-state rotordynamic performance is studied. An approximate treatment in including the ambient vibration parameter is to apply an equivalent cyclic G-loading directly to the rotor. This treatment would substantially reduce the complexity of the formulation required for the inclusion of the ambient vibration parameter.

APPENDIX A

A 5-STATION 5-MASS TEST ROTOR MODEL, BEARINGS LOCATED AT STATIONS 1 AND 5
LOS ANGELES, CALIF. APRIL 4/1970

NS = 5 NUMBER OF ROTOR STATIONS (ALLOWABLE RANGE: 5=<NS=<25)
IASIGN = 1 A ROTOR STATION NUMBER AT WHICH THE WHIRL/SPIN FREQUENCY RATIO WILL BE PLOTTED ON CPT
NOORPM = 1 THE NUMBER OF SPIN SPEEDS IN RPM AT OR NEAR WHICH 3-DIMENSIONAL ABSOLUTE ROTOR MODE SHAPE CPT GRAPHS ARE REQUIRED. THE SPIN SPEED RPM VALUES ARE LISTED UNDER INPRPM ARRAY.
(ALLOWABLE RANGE: 0=<NOORPM=<50)
NPOINT = 25 THE NUMBER OF POINTS (ONE PPF EACH INTEGRATION STEP) FOR EACH CPT GRAPH.
(ALLOWABLE RANGE: 1=<NPOINT=<50)
CRT = 0 CRT=0 MEANS CRT IS NOT REQUIRED
ACCEL = 0 CRT=1 MEANS CRT IS REQUIRED
ACCF=0 MEANS A 3-DIMENSIONAL ROTOR MODE SHAPE CRT CORRESPONDING TO THAT AT BEGINNING OF THE RUN WOULD BE PROVIDED IF CONCURRENTLY CRT=1.
ACCF=1 MEANS ONLY THE TRANSIENT-SPEED ROTOR MODE SHAPES AT OR NEAR INPRPM VALUES WILL BE PROVIDED IF CONCURRENTLY CRT = 1.
INPRPM ARRAY THE ROTOR SPIN SPEED RPM VALUES AT OR NEAR WHICH CPT GRAPHS FOR 3-DIMENSIONAL ABSOLUTE ROTOR MODE SHAPES ARE REQUIRED
0.0

T = 0.0 INITIAL REAL TIME, SEC.
DT = 1.00000D-05 ESTIMATED INITIAL INTEGRATION STEP (REAL) TIME, SEC.
TMAX = 2.0000D-02 TOTAL REAL TIME TO BE RUN, SEC.
TOLI = 1.0000D-02 INTEGRATION TOLERANCE, FRACTION
TOLR = 1.0000D-03 TOLERANCE IN COMPUTING BEARING DISPLACEMENTS, FRACTION
TSTOP = 5.0000D-01 THE COMPUTER TIME ALLOWED FOR EACH SET OF DATA, MINUTES.

GX = 0.0 GRAVITY OR G-LOADING IN X DIRECTION, IN/SEC**2
GY = 0.0 GRAVITY OR G-LOADING IN Y DIRECTION, IN/SEC**2
TM7 = 0.0 EXPONENT FOR SPEED SENSITIVE ROTOR DRIVE TORQUE
TMZ1 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE
TMZ2 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE, (IN-LB-SEC)/RAD.
TMZ3 = 0.0 COEFFICIENT FOR ROTOR DRIVE TORQUE, IN-LB

DD ARRAY OUTSIDE DIAMETERS OF ROTOR SECTIONS BETWEEN ADJACENT ROTOR STATIONS, IN.
 1.00000 01 1.00000 01 1.00000 01 1.00000 01

D ARRAY INSIDE DIAMETERS OF ROTOR SECTIONS BETWEEN ADJACENT ROTOR STATIONS, IN.
 0.0 0.0 0.0 0.0

QL ARRAY ROTOR SECTION LENGTHS BETWEEN ADJACENT ROTOR STATIONS, IN.
 1.00000 00 1.00000 00 1.00000 00 1.00000 00

DN ARRAY MATERIAL DENSITIES OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**3
 3.00000-01 3.00000-01 3.00000-01 3.00000-01

EE ARRAY YOUNGS MODULI OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**2
 0.0 0.0 0.0 0.0

GG ARRAY SHEAR MODULI OF ROTOR SECTIONS BETWEEN ADJACENT STATIONS, LB/IN**2
 1.15000 07 1.15000 07 1.15000 07 1.15000 07

ET ARRAY DIRECT INPUT OF THE PRODUCTS OF YOUNGS MODULI AND AREA MOMENTS OF INERTIA OF ROTOR
 SECTIONS BETWEEN ADJACENT STATIONS, LB-IN**2
 1.66670 04 1.66670 04 1.66670 04 1.66670 04

GAK ARRAY DIRECT INPUT OF THE PRODUCTS OF SHEAR MODULI, CROSS-SECTIONAL AREAS
 AND RECIPROCALS OF SHEAR STRESS CONCENTRATION FACTORS BETWEEN ADJACENT STATIONS, LB.
 0.0 0.0 0.0 0.0

AM ARRAY ADDITIONAL ROTOR MASSES AT ROTOR STATIONS, (LB-SFC**2)/IN.
 0.0 0.0 0.0 0.0

AID ARRAY ADDITIONAL ROTOR TRANSVERSE MASS MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)
 0.0 0.0 0.0 0.0

AIRD ARRAY ADDITIONAL ROTOR POLAR MASS MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)
 1.00000-16 1.00000-16 1.00000-16 1.00000-16

ECR ARRAY ROTOR MASS ECCENTRICITIES AT ROTOR STATIONS, IN.
 1.00000-03 1.00000-03 1.00000-03 1.00000-03

ALFA ARRAY PHASE ANGLES FOR ROTOR MASS ECCENTRICITY VECTORS AT ROTOR STATIONS MEASURED FROM THE
 INITIAL ROTOR SPIN ANGULAR POSITION, DEGREES
 4.50000 01 4.50000 01 4.50000 01 4.50000 01

BETA ARRAY	INITIAL MISALIGNMENTS BETWEEN THE AXES OF THE MASS MOMENTS OF INERTIA AND THE ELASTIC AXES AT ROTOR STATIONS, DEGREES			
0.0	0.0	0.0	0.0	
GAMMA ARRAY	ANGULAR POSITIONS OF THE X-Y PLANE PROJECTIONS OF THE AXES OF MASS MOMENTS OF INERTIA AT ROTOR STATIONS MEASURED FROM THAT AT THE FIRST ROTOR STATION, DEGREES			
0.0	0.0	0.0	0.0	
C7 ARRAY	TORSIONAL FRICTION EXPONENTS AT ROTOR STATIONS, DIMENSIONLESS			
0.0	0.0	0.0	0.0	
CZ1 ARRAY	TORSIONAL FRICTION COEFFICIENTS AT ROTOR STATIONS, DIMENSION OF CZ1(I)*F00T**CZ(I) IS IN-LB.			
0.0	0.0	0.0	0.0	
CZ2 ARRAY	TORSIONAL FRICTION COEFFICIENTS AT ROTOR STATIONS, (IN-LB-SEC)/RAD.			
0.0	0.0	0.0	0.0	
XKF ARRAY	WHIRL-FREQUENCY FACTORS FOR STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS			
0.0	0.0	0.0	0.0	
XCF ARRAY	WHIRL-FREQUENCY FACTORS FOR DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS			
0.0	0.0	0.0	0.0	
XKFF ARRAY	WHIRL-FREQUENCY FACTORS FOR STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS			
0.0	0.0	0.0	0.0	
XCFF ARRAY	WHIRL-FREQUENCY FACTORS FOR DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, DIMENSIONLESS			
0.0	0.0	0.0	0.0	
QK ARRAY	IN-PHASE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, LB/IN.			
0.0	0.0	0.0	0.0	
QC ARRAY	IN-PHASE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/IN.			
0.0	0.0	0.0	0.0	

QKD ARRAY	OUT-OF-PHASE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, LB/IN.			
0.0	0.0	0.0	0.0	
QCP ARRAY	OUT-OF-PHASE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/IN.			
0.0	0.0	0.0	0.0	
QKHDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE STIFFNESS FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC)/(IN-RAD)			
0.0	0.0	0.0	0.0	
QCHDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE DAMPING FORCE COEFFICIENTS AT ROTOR STATIONS, (LB-SEC**2)/(IN-RAD)			
0.0	0.0	0.0	0.0	
QKF ARRAY	IN-PHASE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB.			
0.0	0.0	0.0	0.0	
QCF ARRAY	IN-PHASE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.			
0.0	0.0	0.0	0.0	
QKPF ARRAY	OUT-OF-PHASE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB.			
0.0	0.0	0.0	0.0	
QCPF ARRAY	OUT-OF-PHASE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, IN-LB-SEC.			
0.0	0.0	0.0	0.0	
QKHDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE STIFFNESS MOMENT COEFFICIENTS AT ROTOR STATIONS, (LB-IN-SEC)/RAD.			
0.0	0.0	0.0	0.0	
QCHDF ARRAY	OUT-OF-PHASE WHIRL AND SPIN VELOCITY SENSITIVE DAMPING MOMENT COEFFICIENTS AT ROTOR STATIONS, (LB-IN-SEC**2)/RAD.			
0.0	0.0	0.0	0.0	

134

NR	=	2	NUMBER OF NON-LINEAR STIFFNESS BEARINGS. (ALLOWABLE RANGE: 2=<NB=<12)
TB ARRAY		5	ROTOR STATION NUMBERS FOR NON-LINEAR STIFFNESS BEARINGS
1			
K ARRAY			TOTAL NUMBER OF STIFFNESS SECTIONS FOR EACH OF THE NON-LINEAR STIFFNESS BEARINGS. (ALLOWABLE RANGE: 1=<K=<4)
1		1	
BKMX ARRAY			NON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN X-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, LB/IN.
8.0000D 04			8.0000D 04
BKMY ARRAY			NON-ISOTROPIC MOUNT STIFFNESS COEFFICIENTS IN Y-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, LB/IN.
8.0000D 04			8.0000D 04
RCMX ARRAY			NON-ISOTROPIC MOUNT DAMPING COEFFICIENTS IN X-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
1.0000D-16			1.0000D-16
RCMY ARRAY			NON-ISOTROPIC MOUNT DAMPING COEFFICIENTS IN Y-DIRECTION FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
1.0000D-16			1.0000D-16
BCB ARRAY			BEARING DAMPING COEFFICIENTS FOR NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.
1.0000D-16			1.0000D-16

NONLINEAR BEARING SPECIFICATIONS

BNB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
0.0 SECTIONS OF NON-LINEAR STIFFNESS BEARING NUMBER 1

BNB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
0.0 SECTIONS OF NON-LINEAR STIFFNESS BEARING NUMBER 2

BBB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
0.0 SECTIONS OF BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER 1

BBB ARRAY ROTOR SPIN-SPEED SENSITIVE BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
0.0 SECTIONS OF BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER 2

135 BDOB ARRAY UPPER BEARING-DISPLACEMENT LIMITS FOR K STIFFNESS SECTIONS OF BEARING
5.00000-03 NUMBER 1, IN.

 BDOB ARRAY UPPER BEARING-DISPLACEMENT LIMITS FOR K STIFFNESS SECTIONS OF BEARING
5.00000-03 NUMBER 2, IN.

 BKB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
8.00000 04 SECTIONS OF BEARING
NUMBER 1

 BKB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS
8.00000 04 SECTIONS OF BEARING
NUMBER 2

 BHB ARRAY NON-LINEAR BEARING STIFFNESS EXPONENTS FOR K STIFFNESS SECTIONS OF BEARING
1.00000 00 NUMBER 1

RHR ARRAY NON-LINEAR BEARING STIFFNESS EXPONENTS FOR K STIFFNESS SECTIONS OF BEARING
NUMBER 2
1.00000 1.0

RDB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
NUMBER 1
0.0

ROB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
NUMBER 2
0.0

RRR ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
NUMBER 1
0.0

REB ARRAY NON-LINEAR BEARING STIFFNESS COEFFICIENTS FOR K STIFFNESS SECTIONS OF BEARING
NUMBER 2
0.0

INITIAL ROTOR MOTION SPECIFICATIONS

ICOND = 1 ICOND=1 MEANS STARTING A NEW ROTOR-BEARING CONFIGURATION
ICOND=0 MEANS READ-IN ALTERNATE INITIAL CONDITIONS FOR THE SAME
ROTOR-BEARING CONFIGURATION

CONTIN = 0 CONTIN=C MEANS STARTING A NEW ROTORDYNAMICS ANALYSIS WITH INITIAL CONDITIONS
PROVIDED BY THE STARUP SUBROUTINE
CONTIN=1 MEANS CONTINUITION OF A PREVIOUS ANALYSIS BY USING THE PREVIOUS
RESULTS ON PUNCHED CARDS AS THE INITIAL CONDTIONS

FD = 0.0 ROTOR SPIN ANGULAR DISPLACEMENT COORDINATE, DEGREES.
FDOT = 1.0000D 02 ROTOR SPIN FREQUENCY, RPM.
WHIVEL = 1.0000D 02 ROTOR WHIRL FREQUENCY,RPM.
FDNIFX = 0.0 A BEARING STIFFNESS SPEED SENSITIVE PARAMETER, RPM

.....THE END OF INPUT DATA.....

137

TOTAL ROTOR WEIGHT,LB = 9.424RD 01
TOTAL ROTOR MASS,(LB*SEC**2)/IN = 2.4411D-01
TOTAL ROTOR POLAR MASS MOMENT OF INERTIA, LB*IN*SEC**2 = 3.0514D 00

FORCE INFLUENCE COEFFICIENTS

ROW 0.0	1	0.0	0.0	0.0	0.0
ROW 0.0	2	4.500110D-05	5.500073D-05	3.500036D-05	0.0
ROW 0.0	3	5.500073D-05	8.000146D-05	5.500073D-05	0.0
ROW 0.0	4	3.500036D-05	5.500073D-05	4.500110D-05	0.0
ROW 0.0	5	0.0	0.0	0.0	0.0

THE TABULATED FORCE INFLUENCE COEFFICIENTS ABOVE ARE THOSE WITH RESPECT TO THE STRAIGHT LINE CONNECTING THE FIRST AND LAST JOURNAL CENTERS OF THE NON-LINEAR STIFFNESS BEARINGS. THE ROW NUMBER REPRESENTS THE ROTOR STATION NUMBER WHERE A UNIT TRANSVERSE LOAD OF 1 LB IS APPLIED, AND THE COLUMN NUMBER NOT SHOWN ABOVE, DENOTES THE ROTOR STATION NUMBER WHERE THE RESULTING DEFLECTIONS IN INCHES ARE DESCRIBED. SINCE THE FORCE INFLUENCE COEFFICIENTS ARE SYMMETRIC ABOUT THE DIAGONALS, ROW AND COLUMN NUMBERS MAY BE INTERCHANGED.

MOMENT INFLUENCE COEFFICIENTS

ROW 1 0.0	5.249999D-05	5.999999D-05	3.749999D-05	0.0
ROW 2 0.0	2.999999D-05	4.499999D-05	2.999999D-05	0.0
ROW 3 0.0	-7.499999D-06	3.388132D-21	7.499999D-06	0.0
ROW 4 0.0	-2.999999D-05	-4.499999D-05	-2.999999D-05	0.0
ROW 5 0.0	-3.749999D-05	-5.999999D-05	-5.249999D-05	0.0

139 THE SIGNIFICANCE OF THE TABULATED MOMENT INFLUENCE COEFFICIENTS CAN BE
 SIMILARLY INTERPRETED AS THAT OF THE FORCE INFLUENCE COEFFICIENTS, EXCEPT THAT
 A UNIT MOMENT (1 IN-LB) APPLICATION IS USED INSTEAD OF A FORCE APPLICATION.
 THE COLUMN NUMBER REPRESENTS THE ROTOR STATION NUMBER WHERE THE RESULTING
 DEFLECTIONS IN INCHES ARE DESCRIBED. SINCE THE MOMENT INFLUENCE COEFFICIENTS
 ARE NO LONGER SYMMETRIC ABOUT A DIAGONAL, INTERCHANGE OF THE COLUMN AND ROW
 IS NOT PERMISSIBLE.

ELEMENTS IN ROTATING COORDINATES:

3.3499D-07	1.2365D-06	1.6037D-06	1.2365D-06	3.1429D-07	7.3540D-01
7.8540D-01	7.8540D-01	7.3540D-01	7.8540D-01	1.1302D-22	1.0
1.2592D-22	2.0	1.5071D-22	1.3472D-01	1.3472D-01	1.0472D-01
1.0472D-01	1.0472D-01	0.0	1.0472D-01		

THE ROTATING ELEMENTS SHOWN ABOVE ARE THOSE UNDER THE INITIAL TIME CONDITIONS, i.e., AT A STARTING TIME.

THE FIRST NS ELEMENTS IN ROTATING COORDINATES REPRESENT THE ROTOR RADIAL DISPLACEMENTS AT STATIONS 1 THROUGH NS WITH RESPECT TO RESPECTIVE STATIC ROTOR DEFLECTION CENTERS IN INCHES. THE STATIC ROTOR DEFLECTION CENTERS ARE THE ROTOR POSITIONS UNDER ZERO SPIN SPEED CONDITIONS BUT INCLUDING THE EFFECTS OF g_x AND g_y .

THE SECOND NS ELEMENTS DENOTE THE ABSOLUTE ANGULAR DISPLACEMENTS OF THE RADIAL DISPLACEMENT VECTORS IN RADIANS.

THE THIRD NS ELEMENTS REPRESENT THE RADIAL VELOCITIES OF THE RADIAL DISPLACEMENT VECTORS IN INCHES PER SECOND.

THE FOURTH NS ELEMENTS DENOTE THE ANGULAR VELOCITIES OF THE DISPLACEMENT VECTORS IN RADIANS PER SECOND.

THE SECOND LAST ELEMENT REPRESENTS THE ROTOR SPIN ANGULAR DISPLACEMENT IN RADIANS, AND THE LAST ELEMENT DENOTES THE ROTOR SPIN VELOCITY IN RADIANS PER SECOND.

CONT

THE APPROPRIATE DERIVATIVES BD'S

1.130344D-22	1.0	1.259229D-22	0.0	1.507126D-22	1.047198D-01	1.0471980_01
1.047199D-01	1.047198D-01	1.047193D-01	-2.458659D-13	4.614597D-14	1.538291D-13	4.6153590-14
-2.458610D-13	1.465497D-11	6.001129D-11	-5.327394D-11	5.968109D-11	3.114569D-11	1.047198D-01
-1.406093D-21						

THE BD'S LISTED ABOVE ARE THE CORRESPONDING TIME (SECOND) DERIVATIVES OF THE PREVIOUSLY LISTED ELEMENTS IN ROTATING COORDINATES. THESE BD'S ARE EVALUATED AT THE LAST SPECIFIED REAL TIME.

AT THE TIME: T = 1.02E625D-03 SEC.

X DISPLACEMENT ARRAY IN STATIONARY COORDINATES,(IN.)
2.320E0-07 8.5652D-07 1.1109D-06 8.5652D-07 2.320E0-07

Y DISPLACEMENT ARRAY IN STATIONARY COORDINATES,(IN.)
2.4160D-07 8.9178D-07 1.1567D-06 8.9178D-07 2.4160D-07

ROTOR DEFLECTION VECTOR ARRAY (IN.)
3.3499D-07 1.2365D-06 1.6037D-06 1.2365D-06 3.3499D-07

PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, DEGREES
4.6155D 01 4.6155D 01 4.6155D 01 4.6155D 01 4.6155D 01

X VELOCITY ARRAY IN STATIONARY COORDINATES,(IN.)
-2.5300D-06 -9.3387D-06 -1.2112D-05 -9.3387D-06 -2.5300D-06

Y VELOCITY ARRAY IN STATIONARY COORDINATES,(IN.)
2.4300D-06 8.9695D-06 1.1634D-05 8.9695D-06 2.4300D-06

WHIRL FREQUENCY ARRAY,RPM
1.0000D 02 1.0000D 02 1.0000D 02 1.0000D 02 1.0000D 02

WHIRL TO SPIN FREQUENCY RATIO ARRAY
1.0000D 00 1.0000D 00 1.0000D 00 1.0000D 00 1.0000D 00

TOTAL NO. OF REVOLUTIONS = 3.2094D-03
SPIN SPEED, RPM = 1.0000D 02

XB ARRAY	BEARING DISPLACEMENT COORDINATE, IN.
1.16020D-07	1.16020D-07
YB ARRAY	BEARING DISPLACEMENT COORDINATE, IN.
1.20800D-07	1.20800D-07
XBDOT ARRAY	BEARING VELOCITY COORDINATE, IN/SEC.
-1.26500D-06	-1.26500D-06
YBDOT ARRAY	BEARING VELOCITY COORDINATE, IN/SEC.
1.21500D-06	1.21500D-06
BRGPO ARRAY	JOURNAL DISPLACEMENT FROM BEARING-CENTER ARRAY, IN.
1.67490D-07	1.67490D-07
PHABRO ARRAY	JOURNAL DISPLACEMENT VECTOR PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES
4.61550 01	4.61550 01
BRGFOR ARRAY	BEARING REACTION ARRAY, LB.
1.34000D-02	1.34000D-02
PHAFOR ARRAY	BEARING REACTION VECTOR PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES
4.61550 01	4.61550 01
STIFF. ARRAY	BEARING FORCE TO JOURNAL DEFLECTION RATIO, OR EQUIVALENT LINEAR BEARING STIFFNESS, LB/IN/BEARING
8.00000 04	8.00000 04

THE APPROPRIATE DERIVATIVES BD,S

-3.401671D-16	1.813953D-16	3.897550D-16	1.814151D-16	-3.401572D-16	1.0471980 01	1.0471980 01
1.0471980 01	1.0471980 01	1.0471980 01	-7.113311D-14	1.673471D-13	2.749397D-13	1.673471D-13
-7.112303D-14	-3.799449D-09	-1.679891D-09	-1.629238D-09	-1.679542D-09	-3.766101D-09	1.673471D-09
-1.211355D-20						

APPENDIX B
COMPUTER PROGRAM LISTINGS

A complete set of the computer program Fortran Listing is attached. This includes also three library subroutines "ISIMDD," "TIMEV" and COUNTV" which may not be available in all the computation facilities. Subroutine CONTV is a part of TIMEV subroutine.

The main program and subroutine names and corresponding memory length required are:

<u>NAME</u>	<u>*MEMORY REQUIRED IN HEXADECIMAL BYTES</u>
MAIN	2EC2
READIN	D34
WRIO UT	38AC
STARUP	1581A
FUND	2EEA
BRGXY	9EE
INFLC Ø	B12
SZMASS	5F8
PL Ø T1	29A
RKADAM	3CC3
RUNKUT	6DE
ADAMLT	D2C
TIME	20C
ISIMDD	7D6
TIMEV	254

*The memory capacity required for "MAIN" "WRIO~~UT~~" and "PL~~Ø~~T1" may slightly deviate from the exact values resulting from changing to NASA CRT Subroutines and minor modifications after memory capacity counts.

C A SIMPLIFIED FLEXIBLE ROTOR DYNAMICS COMPUTER PROGRAM WRITTEN FOR 00768000
 C NASA LEWIS RESEARCH CENTER, CLEVELAND, OHIO, BY FRED. A. SHEN, 00769000
 C POWER SYSTEMS DIVISIONS, NORTH AMERICAN ROCKWELL CORP., LOS ANGELES 90070000
 C CALIFORNIA, MAY 1970. 00771000
 IMPLICIT REAL*8 (A-H,O-Z) 00772000
 INTEGER CRF,ACCEL,CONTIN 00773000
 REAL TITLE,INPRPM,Z,ROU,TT,RPM,WHRATI,FORC,BRGR,RUMAX,RUSTA 00774000
 1,RSTAR4,RSTAR5,RSTAR6 00775000
 DIMENSION DU(25),U(25),QL(25),DN(25), AM(25),AID(25)00776000
 1),AIRU(25),ECC(25),ALFA(25),BETA(25),GAMMA(25) 00777000
 2,TITLE(36),YSAVE(102),SP(3) 00778000
 DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25)00781000
 1),QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) 00782000
 2,IB(12),K(25) 00783000
 3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25), RO(25) 00784000
 4,BN8(25,6),BBB(25,6), BRUB(25,7), 00785000
 5C(25,25),B(25,25) 00786000
 DIMENSION XX(25),YY(25),XDUT(25),YDUT(25) 00789000
 1,XB(25),YB(25),XBDUT(25),YBDUT(25) 00790000
 1,QM(25),DELT(X(25),DELT(Y(25),WHRVLD(25),WHIRPM(25),Y(102),KK(25), 00793000
 2BRGRU(25),PHABRU(25),BRGFOR(25),PHAFUR(25),WHRATI(50),STIFF(25) 00794000
 4,ROSQ(25),WHRATU(25),RUMAX(50),ISTATN(50),RPM(50),RUSTA(50), INPRO00797000
 5PM(50),ROU(25),PHAROU(25),Z(25),BRGR(50,12),FORC(50,12),TT(50), 00798000
 6SZ(25),ZQ(25),SZUL(25),ZQUL(25),QZ(25),ZSOL(25),QZUL(25) 00799000
 COMMON/MAREAD/TITLE,INPRPM, 1,DT,TMAX, TMZ,IMZ1,TMZ2,TMZ300803000
 1, TOLI, NUORU, IASIGN,CRF,ACCEL 00804000
 COMMON/MAFUF1/DD,D,QL, NS,NB,IB 00806000
 COMMON/MAFUF2/ SZ,QZ,ZQ,SZUL,ZSOL,QZUL,QMLDV,QLL 00807000
 COMMON/MAFUF3/ IB1,IBNB 00808000
 COMMON/MAFUF4/ Z 00809000
 1 /MAFU0/ITIM,INT,KK 00810000
 COMMON/MAFU14/ FDUFIX 00810500
 COMMON/MAFU1/ TSTOP,K 00811000
 COMMON/MAFU2/ DN,AM,AID,AIRU,ECC,ALFA,BETA,GAMMA,GX,GY 00812000
 1 /MAFU3/WHIVEL 00813000
 COMMON/MAFU5/ QK,QC,OKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC00815000
 1HDF, BKMX,BKMY,BCMX,BCMY,BCB 00816000
 COMMON/MAFU6/ BN8,BBB,BRUB 00817000
 COMMON/MAFU7/ROSQ, WHRVLU,DELT(X,DELT(Y 00818000
 COMMON/MAFU9/ C,B 00820000
 1 /MAFU10/RU,PHAROU 00821000
 COMMON/MAFU11/FD,FDUT 00822000

```

1      /MAFU13/Y                                00823000
X      /SFP1/RPM,NPOINT,ICOND,CUNFIN          00824000
1      /SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,N,NS2P1 00824100
1      /GARBG7/XX,YY,XB,YB,XDUT,YDUT,XBDOT,YBDOT          00824200
2      /GARBG9/YSAVE,WHIRPM,BRGRO,PHABRO,BRGFOR,PHAFOR,WHRATO, 00824300
3PHARO(25),RUMAX,ISTATN,RUO,BRGR,FORC,T,WHRATI          00824400
  COMMON/MAFU12/ QM
  DATA PI/3.14159265358979324/                      00825000
404  FORMAT (7X,1P6E13.4)                          00825500
      A=180/PI-
      H=30/PI
      V=1/(2*PI)
      GU TU 2 '
1  KA=0                                         00826000
  DO 5 I=1,N,3
  SP(1)=Y(I)
  SP(2)=Y(I+1)
  SP(3)=Y(I+2)
  KA=KA+1
5  PUNCH 106,SP,KA                            00826100
106  FORMAT(1P3D22.15,6X,I8).                  00826200
2  CALL COUNTV
      TMIN=0
      ICC=0
      CALL READIN
      CALL WRIDUT
      TR=T
      IC=0
      ITIM=1
      IND=0
      INT=0
      MM=1
      MAXSHF=0
      WHIVEL=WHIVEL/H
      IF(ICOND.EQ.0),GU TU 3
      DO 33 I=1,NB
          IBI=IB(I)
33      KK(IBI)=1
          IBI=IB(1)
          IBNB=IB(NB)
          CALL SZMASS
          CALL INFLCO(C,B)

```

```

      DO 22 J=1,NS
      DELT=0.
      DO 23 I=1,NS
23  DELT=DELT+C(I,J)*QM(I)
      DELTX(J)=DELT*GX
22  DELTY(J)=DELT*GY
3   ICOND=0
      IF(CONTIN.EQ.1)GOTO80
      GO TO 8
7   ITIM=1
      DT=0.1*DT
8   CALL STARUP
      DO 95 I=1,NS
      I=NS=I+NS
      I2NS=I+NS2
      I3NS=I+NS3
      PHAROU(I)=PHAROU(I)/A
      XNUTR=R0(I)*DCOS(PHAROU(I))
      YNUTR=R0(I)*DSIN(PHAROU(I))
      XDOT(I)=WHIVEL*YNUTR
      YDOT(I)=WHIVEL*XNUTR
      Y(I)=DSQRT(XNUTR**2+YNUTR**2)
      Y(NS)=DATAN(YNUTR/XNUTR)
      Y(I2NS)=(XNUTR*XDOT(I)+YNUTR*YDOT(I))/Y(I)
      95 Y(I3NS)=(XNUTR*YDOT(I)-YNUTR*XDOT(I))/Y(I)**2
      Y(NS4P1)=FD/A
      Y(N)=FDUT/H
80  WRITE(6,1555)
1555 FORMAT(1H04X'ELEMENTS IN ROTATING COORDINATES: ')
      WRITE(6,404)(Y(I),I=1,N)
100  IF(CRT.EQ.0) GO TO 107
      IF(T.GE.TMAX.OR.TMIN.GE.TSTOP) GO TO 1040
      PPM=0
      P1=FDOT
107  TSAVE=1
      DO 41 I=1,N
41   YSAVE(I)=Y(I)
24   IF(T.GE.TMAX) GO TO 900
      ICC=ICC+1
      IF(ICC.LT.10) GO TO 97
      CALL TIMEV(TSEC)
      TMIN=TSEC/60
      00861000
      00862000
      00863000
      00865000
      00866000
      00867000
      00867050
      00867100
      00867110
      00867120
      00867130
      00867200
      00868000
      00869000
      00870000
      00871000
      00871100
      00871200
      00871300
      00871400
      00871500
      00874000
      00875000
      00876000
      00877000
      00879000
      00880000
      00880100
      00880200
      00880300
      00881000
      00881500
      00882000
      00883000
      00884000
      00885000
      00886000
      00887000
      00887100
      00887200
      00887300
      00887400

```

```

IF(TMIN.GE.TSTOP) GO TO 1
ICC=0
97 CALL RKADAM(N,TR,Y,DT,IND,ITIM,TOLI,IERR)
DO 4 I=1,N
  IF(Y(I).GE.1.D14) GO TO 7
4 CONTINUE
  CALL TIME(TR,DT,T,ITIM)
  IF(IERR.EQ.0)GOTO108
  WRITE(6,311)T
311 FORMAT('UNSUCCESSFUL SOLUTION AT T ='1PD12.5)
  STOP
108  IF(CRT.EQ.0) GO TO 110
    PM1=INPRPM(MM)-P1*H
    PM2=INPRPM(MM)-H*Y(N)
    P12=PM1*PM2
    IF(P12.GT.0) GO TO 110
    MM=MM+1
    IF(DABS(PM2).GT.DABS(PM1))GOTO1000
110  DO 32 I=1,NS
    INS=I+NS
    I2NS=I+NS2
    I3NS=I+NS3
    COSAB=DCOS(Y(INS))
    SINAB=DSIN(Y(INS))
    XNUTR=Y(I)*COSAB
    YNUTR=Y(I)*SINAB
    XX(I)=XNUTR-DELT(X(I))
    YY(I)=YNUTR-DELT(Y(I))
    PHARO(I)=Y(INS)
    XDOT(I)=Y(I2NS)*CUSAB-Y(I)*Y(I3NS)*SINAB
    YDOT(I)=Y(I2NS)*SINAB+Y(I)*Y(I3NS)*COSAB
32  WHRVLO(I)=Y(I3NS)
    FDOT=Y(N)
    CALL BRGXY
    A1=0
    B1=0
    DO 317 I=1,NB
      IBI=IB(I)
      BRGRU(IBI)=DSQRT(XB(IBI)**2+YB(IBI)**2)
      KP1=K(IBI)+1
      IF(BRGRU(IBI).GT.BROB(IBI,KP1)) GO TO 27
      KKB=KK(IBI)
00887500
00887600
00888000
00888100
00888200
00888300
00888600
00889000
00890000
00891000
00892000
00893000
00895000
00896000
00897000
00898000
00899000
00900000
00904000
00905000
00906000
00907000
00908000
00909000
00910000
00911000
00912000
00913000
00914000
00915000
00916000
00917000
00922000
00923000
00924000
00925000
00926000
00927000
00928000
00929000
00930000
00931000

```

```

25 IF(BRGRO(IBI).LT.BROB(IBI,KKB))GOTO29          00932000
26 KKB1=KKB+1                                     00933000
  IF(BRGRO(IBI).LE.BROB(IBI,KKB1))GO TO 28        00934000
  KKB=KKB+1                                     00935000
    A1=A1+1                                     00936000
  GO TO 26                                     00936700
27 WRITE(6,200)                                     00938000
200 FORMAT(' BEARING DEFLECTION EXCEEDS BEARING CLEARANCE') 00939000
  GU TO 1                                     00940000
29   KKB=KKB-1                                     00941000
    B1=B1+1                                     00942000
  GO TO 25                                     00943000
28   KK(IBI)=KKB                                     00944000
317  CONTINUE                                     00945000
    A1B1=A1+B1                                     00947000
  IF(A1B1.EQ.0) GU TO 318                      00948000
  ITIM=1                                     01034000
  T=TSAVE                                     01035000
  TR=T                                     01035100
  DO 102 I=1,N                                     01036000
102  Y(I)=YSAVE(I)                                     01037000
  MAXSHF=MAXSHF+1                                     01038000
  IF(MAXSHF.GE.10) GO TO 320                      01039000
  GOTO24                                     01040000
320 WRITE(6,330)                                     01041000
330 FORMAT(1H0,4X,'THE MAXIMUM NUMBER OF SHIFTING OF STIFFNESS SECTION 01042000
  1 IS TENTATIVELY LIMITED TO 10'/4X,'TO AVOID INFINITE CYCLIC COMPUTATION 01043000
  2 STATIONS DUE TO INADEQUATE INPUT DATA.')        01044000
  GU TO 1                                     01045000
318  IC=IC+1                                     01046000
  ITIM=0                                     01047000
  DO 43 I=1,NB                                     01048000
    IBI=IB(I)                                     01049000
  BRGRD(IBI)=DSQRT(XB(IBI)**2+YB(IBI)**2)        01050000
  PHABRD(IBI)=DATN2D(YB(IBI),XB(IBI))          01051000
  IF(PHABRD(IBI).LT.0) PHABRD(IBI)=360.+PHABRD(IBI) 01052000
  BRGFOX=BKMX(IBI)*(XX(IBI)-XB(IBI))          01053000
  BRGFOY=BKMY(IBI)*(YY(IBI)-YB(IBI))          01054000
  BRGFUR(IBI)=DSQRT(BRGFOX**2+BRGFOY**2)        01055000
  STIFF(IBI)=BRGFUR(IBI)/BRGRD(IBI)          01055100
  BRGFOX=BRGFOX+XDUT(IBI)*BCB(IBI)          01055200
  BRGFOY=BRGFOY+YDUF(IBI)*BCB(IBI)          01055300

```

```

BRGFUR(IBI)=DSQRT(BRGFOX**2+BRGF0Y**2) 01055400
PHAFUR(IBI)=DATN2D(BRGF0Y,BRGFOX) 01056000
43 IF(PHAFUR(IBI).LT.0) PHAFOR(IBI)=360.+PHAFUR(IBI) 01057000
FDOT=Y(N)
REVULN=V*Y(NS4P1) 01059000
DO 45 I=1,NS 01060000
  WHIRPM(I)= H*WHRVLD(I) 01061000
45 WHRATO(I)=WHRVLD(I)/FDOT 01062000
  WHRATI(IC)=WHRATO(IASIGN) 01063000
  RPMM= H*FDOT 01064000
  RPM(IC)=RPMM 01065000
  TT(IC)=T 01066000
  WRITE(6,1001)T 01067000
  01068000
1001 FORMAT (1H1//34X, 16HAT THE TIME: T=1PD13.6,2X,'SEC.') 01069000
  WRITE (6,701) 01070000
701 FORMAT(1H0,4X,52HX DISPLACEMENT ARRAY IN,STATIONARY COORDINATES,(I01071000
  1N.)) 01072000
  WRITE (6,404) (XX(I),I=1,NS) 01073000
  WRITE (6,702) 01074000
702 FORMAT(1H0,4X,52HY DISPLACEMENT ARRAY IN STATIONARY COORDINATES,(I01075000
  1N.)) 01076000
150  WRITE (6,404) (YY(I),I=1,NS) 01077000
  WRITE(6,707) 01078000
707 FORMAT(1H0,4X,35HROTOR DEFLECTION VECTOR ARRAY (IN.)) 01079000
  DO 34 I=1,NS 01080000
  RO(I)=Y(I) 01081000
  PHARO(I)=DATN2D(YY(I),XX(I)) 01082000
34 IF(PHARO(I).LT.0) PHARO(I)=360.+PHARO(I) 01083000
  WRITE(6,404) (RO(I),I=1,NS) 01084000
  WRITE(6,708) 01085000
708 FORMAT(1H0,4X, 'PHASE ANGLE ARRAY FOR ROTOR DEFLECTION VECTORS, 001086000
  1EGREES') 01087000
  WRITE(6,404) (PHARO(I),I=1,NS) 01088000
  WRITE (6,703) 01089000
703 FORMAT(1H0,4X, 'X VELOCITY ARRAY IN STATIONARY COORDINATES,(IN/SE01090000
  1C*)') 01090100
  WRITE (6,404) (XDOT(I),I=1,NS) 01091000
  WRITE (6,704) 01092000
704 FORMAT(1H0,4X, 'Y VELOCITY ARRAY IN STATIONARY COORDINATES,(IN/SE01093000
  1C*)') 01093100
  WRITE (6,404) (YDOT(I),I=1,NS) 01094000
  WRITE (6,705) 01095000

```

```

705 FORMAT (1H0,4X,25HWHIRL FREQUENCY ARRAY,RPM) 01096000
  WRITE (6,404) (WHIRPM(I),I=1,NS) 01097000
  WRITE (6,706) 01098000
706 FORMAT (1H0,4X,35HWHIRL TO SPIN FREQUENCY RATIO ARRAY) 01099000
  WRITE (6,404) (WHRATO(I),I=1,NS) 01100000
  WRITE(6,1002)REVDLN,RPMM 01101000
1002 FORMAT(//5X,26HTOTAL NO. OF REVOLUTIONS =1PD13.4/ 5X,26HSPIN SPEED 01102000
  1D, RPM =1PD13.4) 01103000
  WRITE (6,457) 01104000
457 FORMAT(1H1//5X,8HXB ARRAY,10X, 36HBEARING DISPLACEMENT COORDINATE 01105000
  1E, IN.) 01106000
  WRITE (6,404) (XB(IB(I)),I=1,NB) 01107000
  WRITE (6,458) 01108000
458 FORMAT (1H0,4X,8HYB ARRAY,10X,36HBEARING DISPLACEMENT COORDINATE, 01109000
  1IN.) 01110000
  WRITE (6,404) (YB(IB(I)),I=1,NB) 01111000
  WRITE (6,459) 01112000
459 FORMAT (1H0,4X,11HXB DOT ARRAY,7X,  'BEARING VELOCITY COORDINATE, 01113000
  1IN/SEC.') 01114000
551 WRITE (6,404) (XBDOF(IB(I)),I=1,NB) 01115000
  WRITE(6,460) 01116000
460 FORMAT (1H0,4X,11HYB DOT ARRAY,7X,  'BEARING VELOCITY COORDINATE, 01117000
  1IN/SEC.') 01118000
  WRITE (6,404) (YBDOT(IB(I)),I=1,NB) 01119000
  WRITE (6,1003) 01120000
1003 FORMAT (//1H0,4X, 11HBRGRD ARRAY,7X,  'JOURNAL DISPLACEMENT FROM 01121000
  1BEARING-CENTER ARRAY, 'IN.') 01122000
  WRITE (6,404) (BRGRD(IB(I)),I=1,NB) 01123000
  WRITE (6,1010) 01124000
1010 FORMAT (1H0,4X,12HPHABR0 ARRAY,7X,  'JOURNAL DISPLACEMENT VECTOR 01125000
  1PHASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES') 01126000
  WRITE (6,404) (PHABR0(IB(I)),I=1,NB) 01127000
  WRITE (6,1008) 01128000
1008 FORMAT(1H0,4X,12HBRGFOR ARRAY,6X,'BEARING REACTION ARRAY, LB.') 01129000
  WRITE (6,404) (BRGFOR(IB(I)),I=1,NB) 01130000
  WRITE (6,1009) 01131000
1009 FORMAT (1H0,4X, 12PHAFOR ARRAY, 6X,  'BEARING REACTION VECTOR PH01132000
  1ASE ANGLE COUNTER-CLOCKWISE FROM X-AXIS, DEGREES') 01133000
  WRITE (6,404) (PHAFOR(IB(I)),I=1,NB) 01134000
  WRITE(6,1020) 01134100
1020 FORMAT(1H0,4X,11HSTIFF ARRAY,7X,'BEARING FORCE TO JOURNAL DEFLECTION 01134200
  IN RATIO, OR EQUIVALENT LINEAR BEARING'/24X,'STIFFNESS, LB/IN/BEARI01134300

```

```

2NG')
      WRITE(6,404) (STIFF(IB(I)),I=1,NB)
      IF(CRT.NE.0) GO TO 105
      IC=1
      GO TO 107
105  DO 500 I=1,NB
      J=IB(I)
      FORC(IC,I)=BRGFDR(J)
      500 BRGR(IC,I)=BRGRO(J)
      IF(ACCEL.EQ.0)GOTO1000
      IF(P12.GT.0.0) GO TO 1027
      GO TO 1000
900  IF(CRT.EQ.0.OR.ACCEL.NE.0)GOTO1
1000 ACCEL =1
      DO 1104 I=1,NS
1104 R00(I) =R0(I)
      C PLOT 1
      "REAL CHAR11(21),CHAR21(7),CHAR31(8),CHARSS(4),SYMBOL/**/
      DATA CHAR11/*ROTUR 3-DIMENSIONAL MODE SHAPE WITH PHASE ANGLES (DEG)1147020
1REES) LABELED AS SHOWN, AT RPM=1/,CHAR21/*ROTOR AXIAL LENGTH, INCO1147030
152  2HES/*,CHAR31/*ROTOR DEFLECTION VECTOR, INCHES/*
      CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)
      CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)
      CALL LRLEGN(CHAR11,84,0,1.463,9.67,0.)
      CALL LRCNVT(RPM,3,CHARS ,4,13,5)
      CALL LRLEGN(CHARSS,13,0,8.6,9.67,0.)
      CALL LRLEGN(CHAR21,27,0,4.31,0.,0.)
      CALL LRCURV(Z,R00,NS,2,SYMBOL,0.)
      DO 1005 I=1,NS
      RSTAR4=PHARD(I)
      CALL LRCNVT(RSTAR4,3,CHARSS,3,4,0)
      CALL LRLABL(CHARSS,4,0,Z(I),R00(I),0.)
1005 CONTINUE
      CALL LRLEGN(CHAR31,31,1,0.,4.6,1.)
      IF(DABS(PM2).LE.DABS(PM1)) GO TO 1027
      IF(PPM.EQ.1) GO TO 100
      PPM=1
      GO TO 110
1027 PPM=0
      J=1
      DO 1209 I=1,NS
1209 IF(R0(J).LT.R0(I))J=I

```

```

ROMAX(IC)=RO(J)                                01170000
ISTATN(IC)=J                                   01171000
RUSTA(IC)=RO(IASIGN)                           01171500
IF(IC.LT.NPOINT) GO TO 100                      01171600
1040 NPOINT = IC                                01172000
C      PLUT 2                                    01173000
      CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)       01173500
      CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)    01174000
      REAL CHAR12(7)/*ROTUR SPIN SPEED VERSUS TIME/,CHAR22(4)/*TIME, SE01174500
1COND$  '/*,CHAR32(6)/*ROTUR SPIN SPEED, RPM  '/
      CALL LRCURV(TT,RPM,NPOINT,2,SYMBOL,0.)      01175000
      CALL LRCURV(TT,RPM,NPOINT,3,SYMBOL,0.)      01175500
      CALL LRLEGN(CHAR12,28,0,3.50,9.67,0.)       01176000
      CALL LRLEGN(CHAR22,13,0,4.86,0.,0.)        01176500
      CALL LRLEGN(CHAR32,21,1,0.,4.95,1.)        01177000
      IF(ACCEL.EQ.0)GOTO2000                      01177100
C      PLOT 3                                    01177200
      CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)       01177400
      CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)    01177500
      REAL CHAR13(19)/*ROTUR WHIRL-TO-SPIN FREQUENCY RATIO VERSUS ROTUR 01178500
153   1SPIN SPEED AT ROTUR STATION'/          01179000
      CALL LRCURV(RPM,WHRATI,NPOINT,2,SYMBOL,0.)  01179500
      CALL LRCURV(RPM,WHRATI,NPOINT,3,SYMBOL,0.)  01180000
      CALL LRCNVT(IASIGN,1,CHARSS,1,3,0)          01181000
      CALL LRLEGN(CHARSS,3,0,7.85,9.67,0.)        01181500
      CALL LRLEGN(CHAR13,76,0,1.795,9.67,0.)      01182000
      CALL LRLEGN(CHAR13,35,1,0.,3.47,0.)        01182500
      CALL LRLEGN(CHAR32,21,0,4.55,0.,1.)        01183000
C      PLOT 4                                    01184000
C      PLOT RPM VS FURC FUNCTIONS               01185000
      CALL PLUT1(FORC)                           01186000
      CALL LRLEGN(TITLE,72,0,1.15,9.99,0.)       01186100
      CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.)    01186200
      REAL CHAR14(24)/*BEARING REACTIONS VERSUS ROTUR SPIN SPEED WITH BE01186300
1ARING LOCATION STATION NUMBERS LABELED AS SHOWN/,CHAR24(7)/*BEARI01186400
2NG REACTIONS, POUNDS  '/
      CALL LRLEGN(CHAR14,96,0,1.385,9.67,0.)      01186500
      CALL LRLEGN(CHAR32,21,0,4.55,0.,0.)        01186600
      CALL LRLEGN(CHAR24,25,1,0.,4.37,1.)        01186700
C      PLOT 5                                    01186800
C      PLOT RPM VS BRGR FUNCTIONS               01192000
      CALL PLOT1(BRGR)                           01193000
C

```

```

CALL LRLEGN(TITLE,72,0,1.15,9.99,0.) 01194100
CALL LRLEGN(TITLE(19),72,0,1.15,9.83,0.) 01194200
REAL CHAR15(25)/*JOURNAL DISPLACEMENT VERSUS ROTOR SPIN SPEED WITH 01194300
1 BEARING LOCATION STATION NUMBERS LABELED AS SHOWN */,CHAR25(8)/ 01194400
2 JOURNAL DISPLACEMENTS, INCHES 01194500
CALL LRLEGN(CHAR15,99,0,1.21,9.67,0.) 01194600
CALL LRLEGN(CHAR32,21,0,4.55,0.,0.) 01194700
CALL LRLEGN(CHAR25,29,1,0.,4.52,1.) 01194800
C PLOT 6 01200000
CALL LRANGE(0.,0.,0.,0.) 01200005
REAL CHAR16(13)/*MAXIMUM ROTOR DEFLECTIONS VERSUS ROTOR SPIN SPEED 01201000
1 */,CHAR26(22)/*(THE STATION NUMBERS WHERE THE MAXIMUM DEFLECTION 01202000
2NS OCCUR ARE SHOWN) */,CHAR36(9)/*MAXIMUM ROTOR DEFLECTIONS, INCHES 01203000
3S */
CALL LRCURV(RPM,RUMAX,NPOINT,2,SYMBOL,0.) 01204000
DO 1006 I=1,NPOINT 01205000
CALL LRCNVT(ISTATN(I),1,CHARSS,1,3,0) 01206000
1006 CALL LRLABL(CHARSS,3,0,RPM(I),ROMAX(I),0.) 01207000
CALL LRLEGN(TITLE,72,0,1.15,9.99,0.) 01208000
CALL LRLEGN(TITLE(19),72,0,1.15,9.873,0.) 01209000
CALL LRLEGN(CHAR16,49,0,3.45,9.756,0.) 01210000
CALL LRLEGN(CHAR26,67,0,2.75,9.639,0.) 01211000
CALL LRLEGN(CHAR32,21,0,4.55,0.,0.) 01212000
CALL LRLEGN(CHAR36,33,1,0.,4.52,1.) 01213000
C PLOT 7 01214000
REAL CHAR17(17)/*(THE STATION NUMBER WHERE THE ROTOR DEFLECTIONS 001214020
1CCUR IS SHOWN) */
CALL LRCURV(RPM,ROSTA,NPOINT,2,SYMBOL,0.) 01214030
DO 1007 I=1,NPOINT 01214040
CALL LRCNVI(IASIGN,1,CHARSS,1,3,0) 01214050
1007 CALL LRLABL(CHARSS,3,0,RPM(I),ROSTA(I),0.) 01214060
CALL LRLEGN(TITLE,72,0,1.15,9.99,0.) 01214070
CALL LRLEGN(TITLE(19),72,0,1.15,9.873,0.) 01214080
CALL LRLEGN(CHAR16(3),41,0,3.45,9.756,0.) 01214090
CALL LRLEGN(CHAR17,63,0,2.75,9.639,0.) 01214100
CALL LRLEGN(CHAR32,21,0,4.55,0.,0.) 01214110
CALL LRLEGN(CHAR36(3),25,1,0.,4.25,1.) 01214120
2000 IC=0 01214130
      IF(T.GE.TMAX.OR.TMIN.GE.TSTOP) GO TO 1 01215000
      GOTO100 01215100
      END 01216000
                                         01217000

```

```

SUBROUTINE READIN                               01218000
IMPLICIT REAL*8 (A-H,O-Z)                      01219000
INTEGER CRT,ACCEL,CONTIN                      01220000
REAL TITLE,INPRPM, RPM                         01221000
      DIMENSION DD(25),D(25),QL(25),DN(25),EE(25),GG(25),AM(25),AID(250) 01222000
1),AIR0(25),ECC(25),ALFA(25),BETA(25),GAMMA(25), EI(25),GAK(25) 01223000
2,TITLE(36),INPRPM(50),RPM(50)                01224000
      DIMENSION CZ(25),CZ1(25),CZ2(25)          01225000
      DIMENSION XKF(25),XCF(25),XKFF(25),XCFF(25) 01226000
      DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(250) 01227000
1),QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) 01228000
2,IB(12),K(25)                                01229000
3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),          BCB(25), 01230000
4 BNB(25,6),BBB(25,6),BRDB(25,7),BKB(25,6),BHB(25,6),BDB(25,6), 01231000
5BEB(25,6),Y(102)                            01232000
COMMON/MAREAD/ TITLE,INPRPM,      T,DT,TMAX, TMZ,TMZ1,TMZ2,TMZ3 01234000
1,      TOLI, NOURPM,      IASIGN,CRT,ACCEL 01235000
1      ./SFP1/RPM,NPOINT,ICOND,CONTIN 01236000
COMMON/MAFC/  EE,GG, EI,GAK 01237000
COMMON/MAFU1/DD,D,QL,  NS,NB,IB 01238000
COMMON/MAFU14/ FDOFIX 01238500
COMMON/MAFU1/      TSTOP,K 01239000
COMMON/MAFU2/ DN,AM,AID,AIR0,ECC,ALFA,BETA,GAMMA,GX,GY 01240000
1      /MAFU3/WHIVEL 01241000
COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCFF 01242000
COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC0 01243000
1HDF,      BKMX,BKMY,BCMX,BCMY,BCB 01244000
COMMON/MAFU6/ BNB,BBB,BRDB,BKB,BHB,BDB,BEB 01245000
COMMON/MAFU8/ TOLB 01246000
1      /MAFU13/Y 01247000
COMMON/MAFU11/FD,FDOT 01248000
1      ./SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,NS4P2,NS2P1 01248100
NAMELIST/DATA1/NS,      TMAX,TSTOP,DD,QL,NB,IB,K,ICOND,CONTIN, FDOT,W0 01249000
1HIVEL 01249100
NAMELIST/DATA2/ T,FD,DT,FDOFIX, 01250000
1      IASIGN,NOURPM,NPOINT,CRT,ACCEL,INPRPM,TOLI,TOLB,GX,01251000
1GY,TMZ,TMZ1,TMZ2,TMZ3,      D,DN,EE,GG, EI,GAK,AM,AID,AIR0,ECC,ALFA0 01252000
2,BETA,GAMMA,CZ,CZ1,CZ2,XKF,XCF,XKFF,XCFF,QK,QC,QKP,QCP,QKHD,QCHD,Q0 01253000
3KF,QCF,QKPF,QCPF,QKHDF,QCHDF,      BKMX,BKMY,BCMX,BCMY,BCB,BNB,BRDB 01254000
4B,BKB,BHB,BDB,BEB 01255000
      READ (5,5) TITLE 01257000
5 FORMAT(18A4) 01258000

```

```

READ(5,DATA1)                                01259000
  IF(NS.LT.5) GO TO 12                      01261000
  GO TO 11                                    01262000
12 WRITE (6,377)                                01263000
377 FORMAT(//1H0,10X, 47HMINIMUM ALLOWABLE NUMBER OF ROTOR STATIONS I01264000
1S 5)                                         01265000
  CALL EXIT                                    01266000
11   FD=0                                      01266500
  IASIGN=1                                     01267000
  NOORPM=1                                     01268000
  NPOINT=25                                    01269000
  CRT=0                                         01270000
  ACCEL=0                                       01271000
  INPRPM(1)=0                                 01272000
  T=0                                           01273000
  DT=.00001                                    01273500
  TOLI=.01                                      01274000
  TOLB=.001                                    01275000
  GX=0                                         01276000
  GY=0                                         01277000
  TZ=0                                         01278000
  TZ1=0                                        01279000
  TZ2=0                                        01280000
  TZ3=0                                         01281000
156 NS1=NS-1                                     01281200
  NSM2=NS-2                                     01281400
  NSM3=NS-3                                     01281600
  NSP1=NS+1                                     01281800
  NS2=2*NS                                      01282000
  NS3=3*NS                                      01282200
  NS4=4*NS                                      01282400
  NS4P1=NS4+1                                   01282600
  NS4P2=NS4+2                                   01282800
  NS2P1=NS2+1                                   01283000
  DO 14  I=1,NSM1                                01283200
    D(I)=0                                       01284000
    DN(I)=.283                                    01285000
    EE(I)=3.D7                                    01286000
    GG(I)=1.15D7                                 01287000
    EI(I)=0                                       01288000
14    GAK(I)=0                                    01289000
    DO 16  I=1,NS                                01290000

```

```

AM(I)=1.D-16          01291000
AID(I)=0              01292000
AIRO(I)=1.D-16        01293000
ECC(I)=1.D-16         01294000
ALFA(I)=0             01295000
BETA(I)=0             01296000
GAMMA(I)=0            01297000
CZ(I)=0               01298000
CZ1(I)=0              01299000
CZ2(I)=0              01300000
XKF(I)=0              01301000
XCF(I)=0              01302000
XKFF(I)=0             01303000
XCFF(I)=0             01304000
QK(I)=0               01305000
QC(I)=0               01306000
QKP(I)=0              01307000
QCP(I)=0              01308000
QKHD(I)=0             01309000
QCHD(I)=0             01310000
QKF(I)=0              01311000
QCF(I)=0              01312000
QKPF(I)=0             01313000
QCPF(I)=0             01314000
QKHD(I)=0             01315000
QCHD(I)=0             01316000
01317000
DO 20 N1=1,NB
I=IB(N1)
BKMX(I)=1.D10          01318000
BKMY(I)=1.D10          01319000
BCMX(I)=1.D-16          01321000
BCMY(I)=1.D-16          01322000
20   BCB(I)= 1.D-16        01323000
FDOFIX=0               01324000
01325000
01325500
DO 21 N1=1,NB
I=IB(N1)
KI=K(I)
DO 21 J=1,KI
BNB(I,J)=0              01326000
3BB(I,J)=0              01327000
3KB(I,J)=1.D6            01328000
01329000
01330000
01331000
01332000

```

BHB(I,J)=1	01333000
BDB(I,J)=0	01334000
21 BEB(I,J)=0	01335000
DO 22 N1=1,NB	01336000
I=IB(N1)	01337000
22 BROB(I,1)=.005	01340000
15 FORMAT(6E12.8)	01341000
READ(5,DATA2)	01342000
DT=2.*DT	01343000
DO 24 N1=1,NB	01343100
I=IB(N1)	01343200
KI=K(I)	01343300
DO 24 J=1,KI	01343400
JP1=J+1	01343500
24 BROB(I,JP1)=BROB(I,J)	01343600
DO 26 N1=1,NB	01343700
I=IB(N1)	01343800
26 BROB(I,1)=0	01343900
DD(NS)=1.	01344000
D(NS)= 0.	01345000
QL(NS)=1.	01346000
DN(NS)=0.	01347000
EE(NS)=1.	01348000
GG(NS)=1.	01349000
EI(NS)=0.	01350000
GAK(NS)=0.	01351000
IF(CONTIN.EQ.0) GO TO 17	01352000
READ(5,410) (Y(I),I=1,NS4P2)	01355000
410 FORMAT(3D22.15)	01356000
17 DO 10 I=1,NS	01356100
IF(EI(I).NE.0) EE(I)=0	01356200
IF(GAK(I).NE.0) GG(I)=0	01356300
10 CONTINUE	01356400
RETURN	01367000
END	01368000

SUBROUTINE WRITOUT 01570000
 IMPLICIT REAL*8 (A-H,O-Z) 01571000
 INTEGER CRT,ACCEL,CONTIN 01572000
 REAL TITLE, INPRPM,RPM 01573000
 DIMENSION DD(25),D(25),QL(25),DN(25),EE(25),GG(25),AM(25),AID(25) 01575000
 1),AIRO(25),ECC(25),ALFA(25),BETA(25),GAMMA(25), E1(25),GAK(25) 01576000
 2,TITLE(36) 01577000
 DIMENSION CZ(25),CZ1(25),CZ2(25) 01578000
 DIMENSION XKF(25),XCF(25),XKFF(25),XCFF(25) 01579000
 DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25) 01580000
 1),QCF(25),QKPF(25),QCPF(25),QKHD(25),QCHDF(25) 01581000
 2,IB(12),K(25) 01582000
 3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25) 01583000
 4,BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6),BDB(25,6), 01584000
 5BEB(25,6), RPM(50),INPRPM(50) 01585000
 COMMON/MAREAD/ TITLE,INPRPM, T,DT,TMAX, TMZ,TMZ1,TMZ2,TMZ3 01587000
 1, TOLI, NDURPM, IASIGN,CRT,ACCEL 01588000
 COMMON/MAFC/ EE,GG, EI,GAK 01589000
 COMMON/MAFU1/ DD,D,QL, NS,NB,IB 01590000
 COMMON/MAFU14/ FDOFIX 01590500
 COMMON/MAFU1/ TSTOP,K 01591000
 COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY 01592000
 1 /MAFU3/WHIVEL 01593000
 COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCFF 01594000
 COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHD(25),QCF(25),QCHD(25),QKF(25) 01595000
 1HDF, BKMX,BKMY,BCMX,BCMY,BCB 01596000
 COMMON/MAFU6/ BNB,BBB,BROB,BKB,BHB,BDB,BEB 01597000
 COMMON/MAFU8/ TOLB 01598000
 COMMON/MAFU11/ FD,FDOT 01601000
 X /MAFU13/Y(102) 01601100
 X /SFP1/RPM,NPOINT,ICOND,CONTIN 01602000
 1 /SFP2/N1,NSM2,NSM3,NSP1,NS2,NS3,NS4, IDUM1(3) 01602100
 2 /GARBG3/ICO,M,KI1,KI,I 01602200
 WRITE(6,471) 01602300
 WRITE(6,400) TITLE 01603000
 399 FORMAT(10I10) 01604000
 400 FORMAT (1H0/ (20X,18A4)) 01605000
 404 FORMAT (7X,1P6E13.4) 01606000
 490 FORMAT (14X,1P6E13.4) 01607000
 WRITE (6,401) NS, IASIGN, NDURPM, NPOINT, CRT, ACCEL 01608000
 401 FORMAT (1H0/, 4X,8HNS =I3,14X, 'NUMBER OF RUTUR STATIONS (AL 01609000
 1LOWABLE RANGE:5=<NS=<25)'/4X, 01610000

18HIASIGN =I3,14X, 70HA ROTOR STATION NUMBER AT WHICH THE WHIRL/SP01611000
 2IN FREQUENCY RATIO WILL BE/34X,'PLOTTED ON CRT'/4X,8HNOORPM =I3,1401612000
 3X,'THE NUMBER OF SPIN SPEEDS IN RPM AT OR NEAR WHICH 3-DIMENSIONAL01613000
 1 ABSOLUTE ROTOR'34X'MODE SHAPE CRT GRAPHS ARE REQUIRED. THE SPI01614000
 2N SPEED RPM VALUES ARE LISTED'34X'UNDER INPRPM ARRAY.'34X,'(ALLO01615000
 2WABLE RANGE: 0=<NOORPM=<50)'/ 4X,8HNP0IN01616000
 3T =I3, 01617000
 614X,'THE NUMBER OF POINTS (ONE PER EACH INTEGRATION STEP) FOR EACH01618000
 7 CRT GRAPH.'34X,'(ALLOWABLE RANGE: 1=<NPOINT=<50)'/ 01619000
 8 4X,8HCRT =I3,14X,31HCRT=0 MEANS CRT IS NOT REQUIRE01620000
 8D/29X, 27HCRT=1 MEANS CRT IS REQUIRED/4X,8HACCEL =I3,14X,'ACCEL=01621000
 90 MEANS A 3-DIMENSIONAL ROTOR MODE SHAPE CRT CORRESPONDING TO THAT01622000
 9 AT'34X, 'BEGINNING OF THE RUN WILL BE PROVIDED IF CONCURRENTLY C01623000
 9RT=1.'/29X, 'ACCEL=1 MEANS ONLY THE TRANSIENT-SPEED ROTOR MODE SHAP01624000
 9ES AT OR NEAR INPRPM'34X,'VALUES WILL BE PROVIDED IF CONCURRENTLY01625000
 9 CRT=1.') 01625100
 WRITE (6,453) 01626000
 453 FORMAT(1H0, 3X,12HINPRPM ARRAY, 13X,'THE ROTOR SPIN SPEED RPM VAL01627000
 1UES AT OR NEAR WHICH CRT GRAPHS FOR'29X,53H3-DIMENSIONAL ABSOLUTE01628000
 2 ROTOR MODE SHAPES ARE REQUIRED) 01629000
 WRITE (6,404) (INPRPM(I),I=1,NOORPM) 01630000
 DT=DT/2. 01630500
 WRITE (6,402) T, DT, TMAX, TOLI, TOLB,TSTOP 01631000
 402 FORMAT (1H0,/4X,8HT =1PD13.4,4X, 23HINITIAL REAL TIME, SEC01632000
 1./4X,8HDT =1PD13.4,4X'ESTIMATED INITIAL ' 01633000
 2 'INTEGRATION STEP (REAL) TIME, SEC.'/4X8H01634000
 2TMAX =1PD13.4,4X,31HTOTAL REAL TIME TO BE RUN, SEC./4X,8HTOLI 01635000
 3=1PD13.4,4X, 31HINTEGRATION TOLERANCE, FRACTION/4X,8HTOLB =1PD1301636000
 4.,4,4X,54HTOLERANCE IN COMPUTING BEARING DISPLACEMENTS, FRACTION/4X01637000
 5,8HTSTOP =1PD13.4,4X,'THE COMPUTER TIME ALLOWED FOR EACH SET OF D01637100
 5ATA, MINUTES.') 01637200
 WRITE (6,403) GX, GY, TMZ, TMZ1, TMZ2, TMZ3 01638000
 403 FORMAT (1H0/,4X,8HGX. =1PD13.4,4X, 46HGRAVITY OR G-LOADING I01639000
 1N X DIRECTION, IN/SEC**2/4X,8HGY =1PD13.4, 4X, 46HGRAVITY OR 01640000
 2G-LOADING IN Y DIRECTION, IN/SEC**2/4X,8HTMZ =1PD13.4,4X, 47H01641000
 3EXPONENT FOR SPEED SENSITIVE ROTOR DRIVE TORQUE/4X,8HTMZ1 =1PD1301642000
 4.,4,4X, 34HCOEFFICIENT FOR ROTOR DRIVE TORQUE/4X,8HTMZ2 =1PD13.401643000
 5.,4X, 52HCOEFFICIENT FOR ROTOR DRIVE TORQUE, (IN-LB-SEC)/RAD./4X01644000
 6.,8HTMZ3 =1PD13.4,4X, 43HCOEFFICIENT FOR ROTOR DRIVE TORQU01645000
 7E, IN-LB) 01646000
 WRITE (6,405) 01647000
 405 FORMAT (1H1/ 5X,'DD ARRAY',10X,'OUTSIDE DIAMETERS OF ROTOR SECTI001648000

1NS BETWEEN ADJACENT ROTOR STATIONS, IN.')	01649000
WRITE (6,490) (DD(I),I=1,N1)	01651000
WRITE (6,406)	01652000
406 FORMAT (1H0,4X,'D ARRAY',11X, 'INSIDE DIAMETERS OF ROTOR SECTION')	01653000
1S BETWEEN ADJACENT ROTOR STATIONS, IN.')	01654000
WRITE (6,490) (D(I),I=1,N1)	01655000
WRITE (6,407)	01656000
407 FORMAT (1H0,4X,'QL ARRAY', 10X, 'ROTOR SECTION LENGTHS BETWEEN ADJACENT ROTOR STATIONS, IN.')	01657000
WRITE (6,490) (QL(I),I=1,N1)	01658000
WRITE (6,408)	01659000
408 FORMAT (1H0,4X,'DN ARRAY',10X,'MATERIAL DENSITIES OF ROTOR SECTION')	01661000
1S BETWEEN ADJACENT STATIONS, LB/IN**3')	01662000
WRITE (6,490) (DN(I),I=1,N1)	01663000
WRITE (6,409)	01664000
409 FORMAT (1H0,4X,'EE ARRAY',10X, 'YOUNGS MODULI OF RUTOR SECTIONS')	01665000
1BETWEEN ADJACENT STATIONS, LB/IN**2')	01666000
WRITE (6,490) (EE(I),I=1,N1)	01667000
WRITE (6,410)	01668000
410 FORMAT (1H0,4X,'GG ARRAY',10X, 'SHEAR MODULI OF ROTOR SECTIONS')	01669000
1ETWEEN ADJACENT STATIONS, LB/IN**2')	01670000
WRITE (6,490) (GG(I),I=1,N1)	01671000
WRITE (6,418)	01672000
418 FORMAT (1H0,4X,'EI ARRAY',9X, 'DIRECT INPUT OF THE PRODUCTS OF YOUNGS MODULI AND AREA MOMENTS OF INERTIA OF ROTOR'/23X,'SECTIONS BETWEEN ADJACENT STATIONS, LB-IN**2')	01673000
WRITE (6,490) (EI(I),I=1,N1)	01674000
WRITE (6,419)	01675000
419 FORMAT (1H0,4X,'GAK ARRAY',9X, 'DIRECT INPUT OF THE PRODUCTS OF SHEAR MODULI, CROSS-SECTIONAL AREAS'/23X'AND RECIPROCAIS OF SHEAR STRESS CONCENTRATION FACTORS BETWEEN ADJACENT STATIONS, LB.')	01676000
WRITE (6,490) (GAK(I),I=1,N1)	01677000
WRITE (6,411)	01678000
411 FORMAT(1H0, 4X,'AM ARRAY', 10X, 'ADDITIONAL ROTOR MASSES AT ROTOR STATIONS, (LB-SEC**2)/IN.')	01679000
WRITE (6,404) (AM(I),I=1,NS)	01680000
WRITE (6,412)	01681000
412 FORMAT (1H0,4X,'AID ARRAY', 9X, 'ADDITIONAL ROTOR TRANSVERSE MASS MOMENTS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)')	01682000
WRITE (6,404) (AID(I),I=1,NS)	01683000
WRITE (6,413)	01684000
413 FORMAT (1H0,4X,'AIRO ARRAY',8X, 'ADDITIONAL ROTOR POLAR MASS MOMEN')	01685000
	01686000
	01687000
	01688000
	01689000
	01690000
	01691000

```

ITS OF INERTIA AT ROTOR STATIONS, (LB-IN-SEC**2)) 01692000
  WRITE (6,404) (AIRO(I),I=1,NS) 01693000
  WRITE (6,414) 01694000
414 FORMAT (1H0,4X,'ECC ARRAY', 9X, 'ROTOR MASS ECCENTRICITIES AT ROT001695000
1R STATIONS, IN.') 01696000
  WRITE (6,404) (ECC(I),I=1,NS) 01697000
  WRITE (6,415) 01698000
415 FORMAT (1H0,4X,'ALFA ARRAY',8X, 'PHASE ANGLES FOR ROTOR MASS ECCEN01699000
1TRICITY VECTORS AT ROTOR STATIONS MEASURED FROM THE'/23X,'INITIAL 01700000
2ROTOR SPIN ANGULAR POSITION, DEGREES') 01701000
  WRITE (6,404) (ALFA(I),I=1,NS) 01702000
  WRITE (6,416) 01703000
416 FORMAT (1H0//,5X,10HBETA ARRAY, 8X, 'INITIAL MISALIGNMENTS B01704000
1BETWEEN THE AXES OF THE MASS MOMENTS OF INERTIA'/23X, 'AND THE ELA01705000
2STIC AXES AT ROTOR STATIONS, DEGREES') 01706000
  WRITE (6,404) (BETA(I),I=1,NS) 01707000
  WRITE (6,417) 01708000
417 FORMAT (1H0,4X,'GAMMA ARRAY',7X,'ANGULAR POSITIONS OF THE X-Y PLANO1709000
1E PROJECTIONS ' 01710000
  1 'OF THE AXES OF MASS MOMENTS OF'/23X,'INERTIA AT ROTOR 01711000
2STATIONS MEASURED FROM THAT AT THE FIRST ROTOR STATION, DEGREES') 01712000
  WRITE (6,404) (GAMMA(I),I=1,NS) 01713000
  WRITE (6,422) 01714000
422 FORMAT (1H0,4X,'CZ ARRAY', 10X,'TORSIONAL FRICTION EXPONENTS AT ROT01715000
1ROT STATIONS, DIMENSIONLESS') 01716000
  WRITE (6,404) (CZ(I),I=1,NS) 01717000
  WRITE (6,423) 01718000
423 FORMAT (1H0,4X,'CZ1 ARRAY',9X, 'TORSIONAL FRICTION COEFFICIENTS AT01719000
1 ROTOR STATIONS, DIMENSION OF CZ1(I)*FDOT**CZ(I)'/23X,'IS IN-LB.') 01720000
  WRITE (6,404) (CZ1(I),I=1,NS) 01721000
  WRITE (6,424) 01722000
424 FORMAT (1H0,4X,'CZ2 ARRAY',9X, 'TORSIONAL FRICTION COEFFICIENTS AT01723000
1 ROTOR STATIONS, (IN-LB-SEC)/RAD.') 01724000
  WRITE (6,404) (CZ2(I),I=1,NS) 01725000
  WRITE (6,425) 01726000
425 FORMAT(1H0, 4X,9HXKF ARRAY,9X, 'WHIRL-FREQUENCY FACTORS FOR STIFF01727000
1NESS FORCE COEFFICIENTS AT ROTOR STATIONS,'/23X, 'DIMENSIONLESS') 01728000
  WRITE (6,404) (XKF(I),I=1,NS) 01729000
  WRITE (6,426) 01730000
426 FORMAT (1H0,4X,9HXCF ARRAY, 9X, 'WHIRL-FREQUENCY FACTORS FOR DAMPI01731000
1NG FORCE COEFFICIENTS AT ROTOR STATIONS,'/ 23X, 'DIMENSIONLESS') 01732000
  WRITE (6,404) (XCF(I),I=1,NS) 01733000

```

```

      WRITE (6,427) 01734000
427 FORMAT (1HO,4X,10HXXFF ARRAY,8X,'WHIRL-FREQUENCY FACTORS FOR STIFF01735000
1NESS MOMENT COEFFICIENTS AT ROTOR STATIONS,'/23X,'DIMENSIONLESS') 01736000
      WRITE (6,404) (XKFF(I),I=1,NS) 01737000
      WRITE (6,428) 01738000
428 FORMAT (1HO,4X,10HXCFF ARRAY,8X,'WHIRL-FREQUENCY FACTORS FOR DAMPI01739000
1NG MOMENT COEFFICIENTS AT ROTOR STATIONS,'/23X,'DIMENSIONLESS') 01740000
      WRITE (6,404) (XCFF(I),I=1,NS) 01741000
      WRITE (6,429) 01742000
429 FORMAT (1HO, 4X,'QK ARRAY',10X,'IN-PHASE STIFFNESS FORCE COEFFICI01743000
1ENTS AT ROTOR STATIONS, LB/IN.') 01744000
      WRITE (6,404) (QK(I),I=1,NS) 01745000
      WRITE (6,430) 01746000
430 FORMAT (1HO,4X,8HQC ARRAY,10X, 'IN-PHASE DAMPING FORCE COEFFICIEN01747000
1TS AT ROTOR STATIONS, (LB-SEC)/IN.') 01748000
      WRITE (6,404) (QC(I),I=1,NS) 01749000
      WRITE (6,431) 01750000
431 FORMAT (1HO,4X,9HQKP ARRAY, 9X, 'OUT-OF-PHASE STIFFNESS FORCE COEF01751000
1ICIENTS AT ROTOR STATIONS, LB/IN.') 01752000
163      WRITE (6,404) (QKP(I),I=1,NS) 01753000
      WRITE (6,432) 01754000
432 FORMAT (1HO,4X,9HQCP ARRAY, 9X, 'OUT-OF-PHASE DAMPING FORCE COEFFICI01755000
1ICIENTS AT ROTOR STATIONS, (LB-SEC)/IN.') 01756000
      WRITE (6,404) (QCP(I),I=1,NS) 01757000
      WRITE (6,433) 01758000
433 FORMAT (1HO,4X,10HQKHD ARRAY,8X,'OUT-OF-PHASE WHIRL AND SPIN VELOC01759000
1ITY SENSITIVE STIFFNESS FORCE COEFFICIENTS AT ROTOR'/23X,'STATIONS01760000
1,(LB-SEC)/(IN-RAD)') 01761000
      WRITE (6,404) (QKHD(I),I=1,NS) 01762000
      WRITE (6,434) 01763000
434 FORMAT (1HO,4X,10HQCHD ARRAY,8X,'OUT-OF-PHASE WHIRL AND SPIN VELOC01764000
1ITY SENSITIVE DAMPING FORCE COEFFICIENTS AT ROTOR'/23X,'STATIONS,(01765000
2LB-SEC**2)/(IN-RAD)') 01766000
      WRITE (6,404) (QCHD(I),I=1,NS) 01767000
      WRITE (6,435) 01768000
435 FORMAT (1HO,4X,9HQKF ARRAY,9X, 'IN-PHASE STIFFNESS MOMENT COEFFICI01769000
1IENTS AT ROTOR STATIONS, IN-LB.') 01770000
      WRITE (6,404) (QKF(I),I=1,NS) 01771000
      WRITE (6,436) 01772000
436 FORMAT (1HO,4X,9HQCF ARRAY,9X, 'IN-PHASE DAMPING MOMENT COEFFICIEN01773000
1NTS AT ROTOR STATIONS, IN-LB-SEC.') 01774000
      WRITE (6,404) (QCF(I),I=1,NS) 01775000

```

56

```
        WRITE (6,437) 01776000
437 FORMAT (1H0,4X,10HQKPF ARRAY,8X,'OUT-OF-PHASE STIFFNESS MOMENT COE01777000
1EFFICIENTS AT ROTOR STATIONS, IN-LB.') 01778000
        WRITE (6,404) (QKPF(I),I=1,NS) 01779000
        WRITE (6,438) 01780000
438 FORMAT (1H0,4X,10HQCPF ARRAY,8X,'OUT-OF-PHASE DAMPING MOMENT COEFF01781000
1ICIENTS AT ROTOR STATIONS, IN-LB-SEC.') 01782000
        WRITE (6,404) (QCPF(I),I=1,NS) 01783000
        WRITE (6,439) 01784000
439 FORMAT (1H0,4X,'QKHDF ARRAY',7X,'OUT-OF-PHASE WHIRL AND SPIN VELOC01785000
1ITY SENSITIVE STIFFNESS MOMENT COEFFICIENTS AT'//23X, 'ROTOR STATI01786000
2ONS, (LB-IN-SEC)/RAD.') 01787000
        WRITE (6,404) (QKHDF(I),I=1,NS) 01788000
        WRITE (6,440) 01789000
440 FORMAT (1H0,4X,'QCHDF ARRAY',7X,'OUT-OF-PHASE WHIRL AND SPIN VELOC01790000
1ITY SENSITIVE DAMPING MOMENT COEFFICIENTS AT'//23X, 'ROTOR STATION01791000
2, (LB-IN-SEC**2)/RAD.') 01792000
        WRITE (6,404) (QCHDF(I),I=1,NS) 01793000
        WRITE (6,461) NB 01794000
461 FORMAT (1H1/, 5X,12HNB      =  I3, 3X,'NUMBER OF NON-LINEAR STIFF01795000
1NESS BEARINGS. (ALLOWABLE RANGE: 2=<NB=<12)') 01796000
        WRITE (6,462) 01797000
462 FORMAT( 1H0,4X8HIB ARRAY,10X,'ROTOR STATION NUMBERS FOR NON-LINEA01798000
1R STIFFNESS BEARINGS') 01799000
        WRITE (6,399) (IB(I),I=1,NB) 01800000
        WRITE (6,463) 01801000
463 FORMAT( 1H0,4X,'K ARRAY',11X, 'TOTAL NUMBER OF STIFFNESS SECTION01802000
1S FOR EACH OF THE NON-LINEAR STIFFNESS BEARINGS.'//23X,'(ALLOWABLE 01803000
2RANGE: 1=<K=<6)') 01804000
        WRITE (6,399) (K(IB(I)),I=1,NB) 01805000
        WRITE (6,441) 01806000
441 FORMAT( 1H0,4X,10HBKMX ARRAY,8X,63HNUN-ISOTROPIC MOUNT STIFFNESS 01807000
1COEFFICIENTS IN X-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS BEARI01808000
2NGS, LB/IN.') 01809000
        WRITE (6,404) (BKMX(IB(I)),I=1,NB) 01810000
        WRITE (6,442) 01811000
442 FORMAT( 1H0,4X,'BKMY ARRAY'8X, 63HNUN-ISOTROPIC MOUNT STIFFNESS 01812000
1COEFFICIENTS IN Y-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS BEARI01813000
2NGS, LB/IN.') 01814000
        WRITE (6,404) (BKMY(IB(I)),I=1,NB) 01815000
        WRITE (6,443) 01816000
443 FORMAT( 1H0,4X'BCMX ARRAY',8X, 'NON-ISOTROPIC MOUNT DAMPING COEFF01817000
```

```

1ICIENTS IN X-DIRECTION FOR' /23X, 'NON-LINEAR STIFFNESS BEARINGS,(LB01818000
1-SEC)/IN.') 01819000
  WRITE (6,404) (BCMX(IB(I)),I=1,NB) 01820000
  WRITE (6,444) 01821000
444 FORMAT( 1H0,4X,10HBCMY ARRAY,8X,68HNON-ISOTROPIC MOUNT DAMPING C001822000
1EFFICIENTS IN Y-DIRECTION FOR /23X, 'NON-LINEAR STIFFNESS B01823000
2EARINGS, (LB-SEC)/IN.') 01824000
  WRITE (6,404) (BCMY(IB(I)),I=1,NB) 01825000
  WRITE (6,445) 01826000
445 FORMAT( 1H0,4X,9HBBCB ARRAY,9X, 'BEARING DAMPING COEFFICIENTS FOR 01827000
1NON-LINEAR STIFFNESS BEARINGS, (LB-SEC)/IN.') 01828000
  WRITE (6,404) (BCB(IB(I)),I=1,NB) 01829000
  WRITE(6,420) 01830000
420 FORMAT(1H1//37X,'NONLINEAR BEARING SPECIFICATIONS') 01830500
471 FORMAT(1H1) 01831000
  DO 19 I=1,NB 01832000
    KI=K(IB(I)) 01833000
    WRITE (6,464) I 01834000
464 FORMAT(1H0,4X, 9HBNB ARRAY,9X, 'ROTOR SPIN-SPEED SENSITIVE BEARIN01835000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS' /23X, 'SECTIONS OF 01836000
1. 2NON-LINEAR STIFFNESS BEARING NUMBER' I3) 01837000
501 19 WRITE (6,404) (BNB(IB(I),M),M=1,KI) 01838000
  DO 20 I=1,NB 01839000
    KI=K(IB(I)) 01840000
    WRITE (6,465) I 01841000
465 FORMAT (1H0,4X,9HBBB ARRAY,9X, 'ROTOR SPIN-SPEED SENSITIVE BEARIN01842000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS' /23X, 'SECTIONS OF 01843000
2BEARINGS FOR NON-LINEAR STIFFNESS BEARING NUMBER' I3) 01844000
20 WRITE (6,404) (BBB(IB(I),M),M=1,KI) 01845000
  DO 13 I=1,NB 01846000
    KI1=K(IB(I))+1 01847000
    BROB(IB(I),1)=0.0 01848000
    WRITE (6,466) I 01849000
466 FORMAT (1H0, 4X,'BRUB ARRAY',8X,'UPPER BEARING-DISPLACEMENT LIMIT01850000
1S FOR K STIFFNESS SECTIONS OF BEARING '/23X,'NUMBER' I3, ', IN.') 01851000
13 WRITE (6,404) (BROB(IB(I),M),M=2,KI1) 01852000
  DO 14 I=1,NB 01853000
    KI=K(IB(I)) 01854000
    WRITE (6,467) I 01855000
467 FORMAT (1H0,4X,9HBKB ARRAY,9X, 'NON-LINEAR BEARIN01856000
1G STIFFNESS COEFFICIENTS FOR K STIFFNESS' /23X, 'SECTIONS OF 01857000
2 BEARING '/23X,'NUMBER' I3) 01858000

```

```

14 WRITE (6,404) (BKB(IB(I),M),M=1,KI) 01859000
DO 21 I=1,NB 01860000
  KI=K(IB(I)) 01861000
  WRITE (6,468) I 01862000
468 FORMAT (1H0,4X,9HBHB ARRAY,9X, 'NON-LINEAR BEARING STIFFNESS EXP001863000
1NENTS FOR K STIFFNESS SECTIONS OF BEARING'/23X, 'NUMBER'13) 01864000
21 WRITE (6,404) (BHB(IB(I),M),M=1,KI) 01865000
  DO 16 I=1,NB 01866000
    KI=K(IB(I)) 01867000
    WRITE (6,469) I 01868000
469 FORMAT (1H0,4X,9HBDB ARRAY,9X, 'NON-LINEAR BEARING STIFFNESS COEF01869000
1FICIENTS FOR K STIFFNESS SECTIONS OF BEARING'/23X, 'NUMBER'13) 01870000
16 WRITE (6,404) (BDB(IB(I),M),M=1,KI) 01871000
  DO 17 I=1,NB 01872000
    KI=K(IB(I)) 01873000
    WRITE (6,470) I 01874000
470 FORMAT (1H0,4X,9HBEB ARRAY,9X, 'NON-LINEAR BEARING STIFFNESS COEF01875000
1FICIENTS FOR K STIFFNESS SECTIONS OF BEARING'/23X, 'NUMBER'13) 01876000
17 WRITE (6,404) (BEB(IB(I),M),M=1,KI) 01877000
  WRITE (6,448) 01878000
448 FORMAT (1H1 //37X, 35HINITIAL ROTOR MOTION SPECIFICATIONS) 01879000
99T ICO=ICOND 01880000
  WRITE(6,450) ICO,CONTIN 01881000
450 FORMAT(1H0/,5X'ICOND = 'I3,7X,'ICOND=1 MEANS STARTING A NEW ROTOR-01881100
2BEARING CONFIGURATION'/23X'ICOND=0 MEANS READ-IN ALTERNATE INITIAL01881200
3 CONDITIONS FOR THE SAME'/32X,'ROTOR-BEARING CONFIGURATION'//5X,'C01881300
40NTIN ='I3,7X'CONTIN=0 MEANS STARTING A NEW ROTORDYNAMICS ANALYSIS01881400
5 WITH INITIAL CONDITIONS'/32X,'PROVIDED BY THE STARUP SUBROUTINE'/201881500
63X,'CONTIN=1 MEANS CONTINUITION OF A PREVIOUS ANALYSIS BY USING TH01881600
7E PREVIOUS'//32X,'RESULTS ON PUNCHED CARDS AS THE INITIAL CONDTIONS01881610
8') 01881620
  IF(CONTIN.EQ.1) GO TO 480 01881700
  WRITE (6,446) FD,FDOT,WHIVEL,FDOFIX 01881800
446 FORMAT( 1H0,4X,
1      15HFD      =      1PD13.4,5X'ROTOR SPIN ANGULAR DIS01882000
1PLACEMENT COORDINATE, DEGREES.'/5X,      15HFDDOT      =      1PD13.4,01882100
25X,'ROTOR SPIN FREQUENCY, RPM.'/5X,15HWHIVEL      =      1PD13.4, 5X,'01882200
3ROTOR WHIRL FREQUENCY,RPM.'/5X,15HFDOFIX      =      1PD13.4, 5X,      01882300
4  'A BEARING STIFFNESS SPEED SENSITIVE PARAMETER,  RPM') 01882350
  R=3.14159265358979324/30 01882310
  FDOFIX=FDOFIX*R 01882320
  GO TO 500 01882400

```

480 WRITE(6,491) 01882500
 491 FORMAT(1HO,30X,'RESTART ROTOR DEFLECTION AND VELOCITY ARRAY') 01882600
 WRITE(6,492) 01882700
 492 FORMAT(1HO,4X,'ROTOR DEFLECTION VECTOR LENGTH, IN.') 01882800
 WRITE(6,404) (Y(I),I=1,NS) 01882900
 WRITE(6,493) 01883000
 493 FORMAT(1HO,4X,'ROTOR DEFLECTION VECTOR ANGULAR POSITION, RAD.') 01883100
 NS2P1=NSP1+NS 01883300
 NS3P1=NS2P1+NS 01883400
 NS4P1=NS3P1+NS 01883500
 NS4P2=NS4P1+1 01883600
 WRITE(6,404) (Y(I),I=NSP1,NS2) 01884000
 WRITE(6,494) 01884100
 494 FORMAT(1HO,4X,'ROTOR DEFLECTION VECTOR VELOCITIES') 01884200
 WRITE(6,404) (Y(I),I=NS2P1,NS3) 01884300
 WRITE(6,495) 01884400
 495 FORMAT(1HO,4X,'ROTOR DEFLECTION VECTOR ANGULAR VELOCITIES') 01884500
 WRITE(6,404) (Y(I),I=NS3P1,NS4) 01884600
 WRITE(6,496) Y(NS4P1),Y(NS4P2) 01884700
 496 FORMAT(1HO,4X,15HF = 1PD13.4,5X,'ROTOR SPIN ANGULAR DISPLACEMENT COORDINATE, RAD.')/5X, 15HFDOT = 1PD13.4,5X,'ROTATION RATE, RAD./SEC.') 01884800
 2R SPIN VELOCITY, RAD./SEC.') 01884900
 500 WRITE(6,501) 01885000
 501 FORMAT(//1HO,7X,'.....THE END OF INPUT DATA.....') 01885010
 RETURN 01885020
 END 01885100
 01885200

167

Card Count 362

SUBROUTINE STARUP
 IMPLICIT REAL*8 (A-H,O-Z)
 REAL Z,TITLE,INPRPM
 DIMENSION QL(25),DD2(25),D2(25),DD4(25),D4(25),QL2(25) 00001000
 1,QLDNDD(25),DN(25),Q6LDND(25),DDPLD(25),DDL(25),Q1LDND(25),QM(25), 00002000
 2AM(25),QID(25),AID(25),QIRO(25),AIRO(25) 00004000
 DIMENSION Z(25), QME(25), ZS(25),SZ(25),ZQ 00005000
 1(25),RD(25) 00006000
 DIMENSION C(25,25),B(25,25) 00007000
 DIMENSION Y(102) 00008000
 DIMENSION XX(25),YY(25),IB(12) 00009000
 DIMENSION AA(51,25),BB(51,25), IA(51) 00010000
 DIMENSION COSFAL(25),SINFAL(25),QMECOS(25),QM 00011000
 1ESIN(25) 00013000
 DIMENSION CMESIN(25,25),CMECOS(25,25) 00015000
 DIMENSION QIRFDO(25) 00016000
 DIMENSION DD(25),D(25),ECC(25),ALFA(25), BETA(25),GAMMA(25) 00017000
 DIMENSION QZ(25),QZOL(25),ZQOL(25),SZOL(25),ZSOL(25) 00018000
 DIMENSION QK(25),QC(25),QKP(25),QCP(25),QKHD(25),QCHD(25),QKF(25) 00019000
 1,QCF(25),QKPF(25),QCPF(25),QKHDF(25),QCHDF(25) 00022000
 3,BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25),BKB(25,6),QKSAVE(25), 00023000
 4QCSAVE(25),BNB(25,6),BBB(25,6),BROB(25,7),BHB(25,6),BDB(25,6), 00025000
 4BEB(25,6) 00026000
 DIMENSION CZ(25),CZ1(25),CZ2(25),BRO(25,25),QRD(25) 00027000
 DIMENSION CQME(25,25),CONST(50),AS(50,50),PHAROO(25),BS(25,50) 00027500
 DIMENSION TITLE(36), INPRPM(50) 00028000
 COMMON/MAFUF1/DD,D,QL, NS,NB,IB 00029000
 COMMON/MAFUF2/ SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZOOL,QMLOV,QLL 00030000
 COMMON/MAFUF3/ IB1,IBNB 00031000
 COMMON/MAFUF4/ Z 00032000
 COMMON/MAFUF4/ Z 00033000
 COMMON/MAFUF4/ Z 00034000
 COMMON/MAFUF4/ FDOFIX 00035000
 COMMON/MAFUF2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA 00036000
 COMMON/MAFUF4/ CZ,CZ1,CZ2 00037000
 COMMON/MAFUF5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC 00038000
 1HDF,BKMX,BKMY,BCMX,BCMY,BCB 00039000
 COMMON/MAFUF6/BNB,BBB,BROB,BKB,BHB,BDB,BEB 00040000
 COMMON/MAFUF8/ TOLB 00042000
 COMMON/MAFUF9/ C,B 00043000
 COMMON/MAFUF10/ RD,PHAROO 00044000
 COMMON/MAFUF11/ FD,FDOT 00045000
 COMMON/MAIFU/ Q,S 00046000
 COMMON/MAREAD/ TITLE,INPRPM, TTT,DT,TMAX, TMZ,TMZ1,TMZ2,TMZ3 00047000

```

1,      TOLI
COMMON/MAFU13/Y.
  FDSAVE=FD
  FOSAVE=FDOT
  PI= 3.14159265358979324
  G=386.088
  U=4./3.
  V=PI/180.
  W=PI/(128.*G)
  E=PI/(8.*G)
  F=FD*V
  FDOT=FDOT*V*6..
  DO 90 I=1,NB
    IBI=IB(I)
    QKSAVE(IBI)=QK(IBI)
    QCSAVE(IBI)=QC(IBI)
    QK(IBI)=QK(IBI)+1./(2.(BKMX(IBI)+BKMY(IBI))+1/(
    1)+BNB(IBI,1)*(FDOT-FDOFIX)))
    IF(BCB(IBI).EQ.0.)BCB(IBI)=1.D-16
    IF(BCMX(IBI).EQ.0.)BCMX(IBI)=1.D-16
    IF(BCMY(IBI).EQ.0.)BCMY(IBI)=1.D-16
    QC(IBI)=1./(1./BCB(IBI)+2./(BCMX(IBI)+BCMY(IBI)))+QC(IBI)
    QC(IBI)=0.
  90  QC(IBI)=0.
  DO 50   I=1,NS.
    ZS(I)=-SZ(I)
    DD2(I)= DD(I)**2
    D2 (I)= D(I)**2
    DD4(I)=DD2(I)**2
    D4(I) =D2(I)**2
    QL2(I)= QL(I)**2..
    QLDND(I)=QL(I)* DN(I)*(DD2(I)-D2(I))
    Q6LDND(I) =          W*QLDND(I)
    DDPLD(I) = DD2(I)+D2(I)
    DDL(I) = DDPLD(I)+          U*QL2(I)
    50  QILDND(I) =          E*QLDND(I)
    QM(1)=Q1LDND(1) + AM(1)
    QM(NS)=Q1LDND(NS-1) +AM(NS)
    QID(1) = Q6LDND(1)*DDL(1) + AID(1)
    QID(NS)= Q6LDND(NS-1)*DDL(NS-1) +AID(NS)
    QIRO(1)=2.*Q6LDND(1)*DDPLD(1)+AIRO(1)
    QIRO(NS)=2.*Q6LDND(NS-1)*DDPLD(NS-1) +AIRD(NS)
    ..NSM1=NS-1
                                         00048000
                                         00049000
                                         00051000
                                         00052000
                                         00059000
                                         00060000
                                         00061000
                                         00062000
                                         00063000
                                         00064000
                                         00065000
                                         00066000
                                         00053000
                                         00054000
                                         00055000
                                         00056000
                                         (BKB(IBI,00057000
                                         00057500
                                         00058000
                                         00058100
                                         00058200
                                         00058300
                                         00058400
                                         00067000
                                         00068000
                                         00069000
                                         00070000
                                         00071000
                                         00072000
                                         00073000
                                         00074000
                                         00075000
                                         00076000
                                         00077000
                                         00078000
                                         00079000
                                         00080000
                                         00081000
                                         00082000
                                         00083000
                                         00084000
                                         00085000

```

```

DO 55 I=2,NSM1 00086000
QM(I) =Q1LDND(I-1)+Q1LDND(I) + AM(I) 00087000
QID(I)=Q6LDND(I-1)*DDL(I-1)+Q6LDND(I)*DDL(I)+AID(I) 00088000
55 QIRO(I) = 2.*(Q6LDND(I-1)*DDPLD(I-1)+Q6LDND(I)*DDPLD(I)) +AIRU(I) 00089000
DO 103 I=1,NS 00100000
103 QME(I)=QM(I)*ECC(I) 00101000
WRITE(6, 80 ) 00103000
80 FORMAT (1H1///14X,28HFORCE INFLUENCE COEFFICIENTS) 00104000
DO 81 I=1,NS 00105000
81 WRITE(6,592) I,(C(I,J),J=1,NS) 00106000
592 FORMAT(1H04X3HROWI3/(1P7D15.6)) 00107000
WRITE (6,82 ) 00108000
82 FORMAT (1H1///14X,29HMOMENT INFLUENCE COEFFICIENTS) 00109000
DO 83 I=1,NS 00110000
83 WRITE(6,592) I,(B(I,J),J=1,NS) 00111000
DO 45 J=1,NS 00112000
AA(J,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+QL(1)) 00113000
1 +QL(1) 00114000
AA(J,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+QL(2)) 00115000
1QL(2)) 00116000
NSM2=NS-2 00117000
NSM3=NS-3 00118000
AA(J,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL(NSM1)) 00119000
1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00120000
45 AA(J,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2)) 00121000
1)-QID(NS)*B(NS,J)/QL(NSM1) 00122000
DO 47 J=1,NS 00123000
DO 47 I=3,NSM2 00124000
47 AA(J,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00125000
1+1) 00126000
AA(IB1,1) = -(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00127000
AA(IB1,2) = -(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00128000
AA(IB1,NSM1)=-(Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM3)) 00129000
1)-QID(NS)/QL(NSM1) 00130000
AA(IB1,NS) = -(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID(NSM3)/QL(NSM1) 00131000
1D(NS)/QL(NSM1) 00132000
DO 49 I=3,NSM2 00133000
49 AA(IB1,I)=-(Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/QL(I+1)+QL(I)) 00134000
1 00135000
AA(IBNB,1) = -(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00136000
AA(IBNB,2) = -(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00137000
AA(IBNB,NSM1)=-(Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM3)) 00138000

```

```

1)+QID(NS)/QL(NSM1) 00139000
AA(IBNB,NS) = -(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID00140000
1D(NS)/QL(NSM1) 00141000
DO 48 I=3,NSM2 00142000
48 AA(IBNB,I)=-(Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/00143000
1(QL(I+1)+QL(I)) 00144000
NSP1=NS+1 00145000
NS2 =2*NS 00147000
DO 53 K=NSP1,NS2 00148000
J=K-NS 00149000
BB(K,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+00150000
1QL(1) 00151000
BB(K,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+00152000
1QL(2) 00153000
BB(K,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL00154000
1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00155000
53 BB(K,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2)+00156000
1)-QID(NS)*B(NS,J)/QL(NSM1) 00157000
DO 65 K=NSP1,NS2 00158000
J=K-NS 00159000
DO 65 I=3,NSM2 00160000
BB(K,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(100161000
1+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00162000
NSB1=NS+IB1 00163000
BB(NSB1,1)=-(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00164000
BB(NSB1,2)=-(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00165000
BB(NSB1,NSM1)=-(Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM00166000
13))-QID(NS)/QL(NSM1) 00167000
BB(NSB1,NS)=-(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID00168000
1(NS)/QL(NSM1) 00169000
DO 66 I=3,NSM2 00170000
66 BB(NSB1,I)=-(Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/00171000
1(QL(I+1)+QL(I)) 00172000
NSNB=NS+IBNB 00173000
BB(NSNB,1)=-(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00174000
BB(NSNB,2)=-(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00175000
BB(NSNB,NSM1)=-(Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM00176000
13))+QID(NS)/QL(NSM1) 00177000
BB(NSNB,NS)=-(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID00178000
1(NS)/QL(NSM1) 00179000
DO 56 I=3,NSM2 00180000
56 BB(NSNB,I)=-(Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/00181000

```

```

    I(QL(I+1)+QL(I))
      DO 67 J=1,NS
      DO 67 I=1,NS
      QME(I)=QM(I)*ECC(I)
      CQME(I,J)=C(I,J)*QME(I)
      FALFA = F+ ALFA(I)*V
      SINFAL(I)=DSIN(FALFA)
      QMESIN(I)=QME(I)*SINFAL(I)
      COSFAL(I)=DCOS(FALFA)
      QMECOS(I)=QME(I)*COSFAL(I)
      CMESIN(I,J)=CQME(I,J)*SINFAL(I)
      67     CMECOS(I,J)=CQME(I,J)*COSFAL(I)
      DO 71 I=1,NS
      71     QIRFD0(I)=QIRO(I)*FDOT
      NS2 =2*NS
      FDOTSQ=FDOT**2
      FF=0.0
      DO 690 J=1,NS2
      DO 690 I=1,NS2
      690   AS(J,I)=0
      DO 700 J=1,NS
      700   BRO(J,1)= B(1,J)*QIRO(1)/QL(1) +B(2,J)*QIRO(2)/(QL(1)+QL(2)) 00206000
      BRO(J,2)=-B(1,J)*QIRO(1)/QL(1) +B(3,J)*QIRO(3)/(QL(2)+QL(3)) 00207000
      BRO(J,NSM1)=-B(NSM2,J)*QIRO(NSM2)/(QL(NSM3)+QL(NSM2))+ B(NS,J)* 00208000
      1QIRO(NS)/QL(NSM1) 00209000
      700   BRO(J,NS) = -B(NSM1,J)*QIRO(NSM1)/(QL(NSM2)+QL(NSM1)) -B(NS,J)*QIRO( 00210000
      1IRO(NS)/QL(NSM1) 00211000
      DO 710 J=1,NS
      DO 710 I=3,NSM2
      710   BRO(J,I)=-B(I-1,J)*QIRO(I-1)/ (QL(I-2)+QL(I-1))+ B(I+1,J)*QIRO(00214000
      1I+1)/(QL(I)+QL(I+1)) 00215000
      QRO(1)= QIRO(1)/QL(1)+ QIRO(2)/ (QL(1)+QL(2)) 00216000
      QRO(2)= -QIRO(1)/QL(1) +QIRO(3)/ (QL(2)+QL(3)) 00217000
      QRO(NSM1)= -QIRO(NSM2)/ (QL(NSM3)+QL(NSM2)) +QIRO(NS)/QL(NSM1) 00218000
      QRO(NS) = -QIRO(NSM1)/ (QL(NSM2)+QL(NSM1)) -QIRO(NS)/ QL(NSM1) 00219000
      DO 720 I=3,NSM2
      720   QRO(I)= -QIRO(I-1) / (QL(I-2)+QL(I-1)) +QIRO(I+1)/ (QL(I)+QL(I+1)) 00221000
      1)
      DO 730 J=1,NS
      CONST(J)=0
      DO 740 I=1,NS
      740   CONST(J)=CONST(J)-CMECOS(I,J) 00226000

```

```

730 CONST(J)= FDOTSQ*CONST(J) 00227000
    CONST(IB1)=0 00228000
    CONST(IBNB)=0 00229000
DO 750 I=1,NS 00230000
    CONST(IB1)=CONST(IB1) -QZ(I)* QMECUS(I) 00231000
750 CONST(IBNB)=CONST(IBNB) -ZS(I) *QMECOS(I) 00232000
    CONST(IB1)= FDUTSQ*CONST(IB1) 00233000
    CONST(IBNB)= FDUTSQ*CONST(IBNB) 00234000
DO 760 J=NSP1,NS2 00235000
    K=J-NS 00236000
    CONST(J)=0 00237000
DO 770 I=1,NS 00238000
    CONST(J)= CONST(J)-CMESIN(I,K) 00239000
760 CONST(J)= FDUTSQ*CONST(J) 00240000
    CONST(NSB1)=0 00241000
    CONST(NSNB)=0 00242000
DO 775 I=1,NS 00243000
    CONST(NSB1)= CONST(NSB1) -QZ(I)*QMESIN(I) 00244000
775 CONST(NSNB)= CONST(NSNB) -ZS(I)*QMESIN(I) 00245000
    CONST(NSB1)= FDUTSQ* CONST(NSB1) 00246000
    CONST(NSNB)= FDUTSQ* CONST(NSNB) 00247000
DO 790 J=1,NS 00248000
DO 790 I=1,NS 00249000
    AS(J,I) =-C(I,J)*QK(I) +FDUTSQ*BRO(J,I) 20052000
790 BS(J,I)=AS(J,I) 20052500
DO 800 J=1,NS 00251000
DO 800 I=NSP1,NS2 00252000
    M=I-NS 00253000
    AS(J,I)= C(M,J)*FDOT*QC(M) 20055000
800 BS(J,I)=-AS(J,I) 20055500
DO 810 J=1,NS 00255000
    AS(J,IB1)= AS(J,IB1)+1-ZS(J)/QLL 00256000
    BS(J,IB1)=AS(J,IB1) 20057500
    AS(J,IBNB)=AS(J,IBNB)+ZS(J)/QLL 00257000
    BS(J,IBNB)=AS(J,IBNB) 20058500
    AS(J,J) = AS(J,J)-1 20059000
810 BS(J,J)=AS(J,J) 20059500
DO 820 I=1,NS 00259000
    AS(IB1,I)=-QZ(I)*QK(I)- FDUTSQ*QRO(I) 20065000
820 BS(IB1,I)=AS(IB1,I) 20065500
DO 830 I=NSP1,NS2 00261000
    M=I-NS 00262000

```

```

      AS(IB1,I)= QZ(M)*FDOT*QC(M)          20068000
830    BS(IB1,I)=-AS(IB1,I)                  20068500
      DO 840 I=1,NS
        AS(IBNB,I)= -ZS(I)*QK(I) +FDOTSQ* QRD(I) 00264000
840    BS(IBNB,I)=AS(IBNB,I)                20075000
      DO .850 I=NSP1,NS2
        M=I-NS
        AS(IBNB,I)= ZS(M)*QC(M)*FDOT 00266000
850    BS(IBNB,I)=-AS(IBNB,I)                20077000
      DO 860 J=1,NS
      DO 860 I=1,NS
860    AS(J,I)= AS(J,I) -FDOTSQ*AA(J,I) 00271000
      DO 950 J=1,NS
      DO 950 I=1,NS
        K=J+NS
        BS(J,I)= BS(J,I) -FDOTSQ*BB(K,I) 00272000
950    DO 630 J=NSP1,NS2
        M=J-NS
      DO 630 I=NSP1,NS2
        K=I-NS
        AS(J,I)=BS(M,K)                  00273000
630    AS(J,K)=BS(M,I)                  00297000
      KL=1$IMDD(50,NS2,1,AS,CONST,FF,IA) 00298000
      174
      DO 620 I=1,NS
        IPNS=I+NS
        XX(I)=AS(I,1)                  00299000
620    YY(I)=AS(IPNS,1)                  00300000
      WRITE(6,471)                      00301000
      WRITE(6,600)                      00302000
600  FORMAT(1H0,4X,'COMPUTED STARTING XX ARRAY')
      WRITE(6,404) (XX(I),I=1,NS)        00303000
      WRITE(6,610)                      00304000
610  FORMAT(1H0,4X,'COMPUTED STARTING YY ARRAY')
      WRITE(6,404) (YY(I),I=1,NS)        00305000
      DO 870 I=1,NS
        RO(I)=DSQRT(XX(I)**2+YY(I)**2) 00306000
        PHAROO(I)=DATN2D(YY(I),XX(I)) 00321000
870    IF(PHAROO(I).LT.0) PHAROO(I)=360+PHAROO(I) 00322000
404  FORMAT (7X,1P6E13.4)                00323000
471  FORMAT(1H1)
      WRITE(6,447)                      00324000
447  FORMAT(1H0,4X,

```

8HRO 00339000

```

1ARRAY,10X,'INITIAL DISPLACEMENT VECTORS (WHICH CAN NOT BE ZERO) FRO0340000
10M THEIR RESPECTIVE ZERO LOAD' /23X,'POSITIONS, IN.') 00341000
  WRITE (6,404) (RO(I),I=1,NS) 00342000
  WRITE (6,449) 00343000
449 FORMAT(1H0,4X'PHAROO ARRAY',6X'INITIAL PHASE ANGLES FOR THE DISPLAO0344000
1CEMENT VECTORS, DEGREES.' /23X,'THE PHASE ANGLES CAN NOT BE ZERO OR00345000
2 MULTIPLES OF 90 DEGREES.') 00346000
  WRITE (6,404) (PHAROO(I),I=1,NS) 00347000
  WRITE(6,471) 00350000
  IF (KL.NE.3) GO TO 560 00351000
  WRITE (6,540) KL 00352000
540 FORMAT(1H1/10X,4HKL =I1) 00353000
  CALL EXIT 00354000
560  FD=FDSAVE 00355000
  FDOT=FOSAVE 00356000
    DO 980 I=1,NB 00357000
    IBI=IB(I)
    QK(IBI)=QKSAVE(IBI) 00358000
980  QC(IBI)=QCSAVE(IBI) 00359000
  RETURN 00360000
  END 00361000
                           00362000

```

```

SUBROUTINE FUND(N,T,Y,BD)                               00102000
IMPLICIT REAL*8 (A-H,O-Z)                           00103000
REAL Z,TITLE,INPRPM                                00105000
DIMENSION QL(25),DN(25),                           00106000
1QM(25),Z(25),QME(25),SZ(25),ZQ(25),QMZ(25),ROSQ(25),      Y(102),00107000
2AM(25),QID(25),AID(25),QIRO(25),AIRO(25)           00108000
3,DELTX(25),DELY(25),COSAB(25),SINAB(25),XDOT(25),YDOT(25),XX(25),00109000
4YY(25),IB(12),XBDOT(25),YBDOT(25),BCB(25),BCMX(25),BCMY(25),KK(25)00110000
5,C(25,25),B(25,25),BKMX(25),BKMY(25)           00111000
DIMENSION BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6),BDB00122000
1 (25,6),BEB(25,6)                                00123000
DIMENSION AA(51,25),BB(51,25),AROT(51,51),IA(51)      00124000
1,CC(51),DRD(51),DSTA(51)                         00125000
DIMENSION COSFAL(25),SINFAL(25),QMECOS(25),QMO00126000
1ESIN(25),QIDBSN(25),QIDBCS(25),QMEFSQ(25),QKHDF(25),QCHDF(25), 000127000
2IRFD(25),QKHDF(25),QCHDF(25),BRGRGP(25),XB(25),YB(25),AMX(25), 00131000
3AMY(25),CZ(25),CZ1(25),CZ2(25),COSFGA(25),SINFGA(25) 00131100
DIMENSION APX(25),AFY(25),AMX1(25),AMY1(25),AMX2(25),AMY2(25) 00132000
DIMENSION BD(102),WHRVLO(25), QKHDD(25),QCHDD(25) 00133000
DIMENSION DD(25),D(25),ECC(25),ALFA(25), BETA(25),GAMMA(25), 00134000
1 A(25)                                              00135000
DIMENSION QZ(25),QZOL(25),ZQOL(25),SZOL(25),ZSOL(25) 00136000
DIMENSION XKF(25),XCF(25),XKFF(25),XCFF(25), QK(25),QC(25),QKP00139000
1(25),QCP(25),QKHD(25),QCHD(25),QKF(25),QCF(25),QKPF(25),QCPF(25) 00140000
DIMENSION TITLE(36), INPRPM(50)                      00143000
COMMON/MAFUF1/DD,QL, NS,NB,IB                      00144000
COMMON/MAFUF2/ SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZQUL,QML0V,QLL 00145000
COMMON/MAFUF3/ IB1,IBNB                            00146000
COMMON/MAFUF4/ Z                                  00147000
COMMON/MAFUF4/ FDOFIX                            00147600
COMMON/MAFU0/ ITIM,INT, KK                         00148000
COMMON/MAFU2/ DN,AM,AID,AIRO,ECC,ALFA,BETA,GAMMA,GX,GY 00149000
COMMON/MAFU4/ CZ,CZ1,CZ2, XKF,XCF,XKFF,XCFF        00150000
COMMON/MAFU5/ QK,QC,QKP,QCP,QKHD,QCHD,QKF,QCF,QKPF,QCPF,QKHDF,QC00151000
1HDF,      BKMX,BKMY,BCMX,BCMY,BCB                00152000
COMMON/MAFU6/ BNB,BBB,BROB,BKB,BHB,BDB,BEB          00153000
COMMON /MAFU7/ROSQ,      WHRVLO,DELTX,DELY        00154000
COMMON/MAFU8/ TOLB                                00155000
COMMON/MAFU9/ C,B                                00156000
X      /MAFU12/QM,QID,QIRO                      00157000
1      /MAFU11/FD,FDOT                         00157100
COMMON/MAIFU/ Q,S                                00158000

```

```

COMMON/MAREAD/ TITLE,INPRPM,          TTT,DT,TMAX, TMZ,TMZ1,TMZ2,TMZ300159000
1,      TOL I                      00160000
2      /SFP2/NSM1,NSM2,NSM3,NSP1,NS2,NS3,NS4,NS4P1,NS4P2,NS2P1 00160010
3      /GARBG5/QME                      00160020
4      /GARBG6/QMZ,AA,BB,AROT,CC,DRO,DSTA,                      00160030
5COSAB,SINAB,QIRFDU,QKHUFF,QCHDFB,BRGRDP,AFX,AFY,AMX1,AMX2,AMY1, 00160040
6AMY2,QCHDD,QKHDD,A,AMX,AMY,COSFGA,STNFGA,                      00160050
7TRIG,COSFAL,SINFAL,QMECOS,QMESIN,QIDBET,QIDBSN,QIDBCS,QMEFSQ, 00160060
8IA
9      /GARBG7/XX,YY,XB,YB,XDOT,YDOT,XBDOT,YBDOT              00160070
IF (INT .EQ.2) GO TO 406          00160080
PI= 3.14159265358979324          00161000
V=PI/180.                         00162000
DO 103 I=1,NS                      00165000
103 QME(I)=QM(I)*ECC(I)          00200000
INT=2
406 DO 2 I=1,N                      00201000
IF (Y(I).GE.1.D14) GO TO 3          00203000
2 CONTINUE                         00204000
GO TO 4                           00205000
3 RETURN                            00206000
4 FDOT=Y(NS4P2)                   00207000
DO 79 I=1,NS                      00210000
INS=I+NS                         00211000
I2NS=I+NS2                        00212000
I3NS=I+NS3                        00213000
00214000
COSAB(I)=DCOS(Y(INS))           00215000
SINAB(I)=DSIN(Y(INS))           00216000
00217000
XNUTR=Y(I)*COSAB(I)             00218000
YNUTR=Y(I)*SINAB(I)              00219000
XX(I)=XNUTR-DELTX(I)             00220000
YY(I)=YNUTR-DELTY(I)
     XDOT(I)=Y(I2NS)*COSAB(I)-Y(I)*Y(I3NS)*SINAB(I) 00221000
     YDOT(I)=Y(I2NS)*SINAB(I)+Y(I)*Y(I3NS)*COSAB(I) 00222000
79  WHRVLO(I)=Y(I3NS)             00223000
80  CALL BRGXY                      00223100
     DO 45 J=1,NS                  00312000
     AA(J,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)*00313000
1     +QL(1))                      00314000
     AA(J,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+00315000
1QL(2))                           00316000
     AA(J,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL00319000

```

```

1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00320000
45 AA(J,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2)) 00321000
  1)-QID(NS)*B(NS,J)/QL(NSM1) 00322000
    DO 47 J=1,NS 00323000
    DO 47 I=3,NSM2 00324000
47 AA(J,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I) 00325000
  1+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00326000
    AA(IB1,1) = -(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00327000
    AA(IB1,2) = -(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00328000
    AA(IB1,NSM1)=-(Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM3)) 00329000
  1)-QID(NS)/QL(NSM1) 00330000
    AA(IB1,NS) = -(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID(NS)/QL(NSM1) 00331000
  1D(NS)/QL(NSM1) 00332000
    DO 49 I=3,NSM2 00333000
49 AA(IB1,I)=-(Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/ 00334000
  1(QL(I+1)+QL(I)) 00335000
    AA(IBNB,1) = -(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00336000
    AA(IBNB,2) = -(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00337000
    AA(IBNB,NSM1)=-(Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM3)) 00338000
  1)+QID(NS)/QL(NSM1) 00339000
    AA(IBNB,NS) = -(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID(NS)/QL(NSM1) 00340000
  1D(NS)/QL(NSM1) 00341000
    DO 48 I=3,NSM2 00342000
48 AA(IBNB,I)=-(Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/ 00343000
  1(QL(I+1)+QL(I)) 00344000
    DO 53 K=NSP1,NS2 00348000
    J=K-NS 00349000
    BB(K,1)=-C(1,J)*QM(1)+QID(1)*B(1,J)/QL(1)+QID(2)*B(2,J)/(QL(2)+00350000
  1QL(1)) 00351000
    BB(K,2)=-C(2,J)*QM(2)-QID(1)*B(1,J)/QL(1)+QID(3)*B(3,J)/(QL(3)+00352000
  1QL(2)) 00353000
    BB(K,NSM1)=-C(NSM1,J)*QM(NSM1)-QID(NSM2)*B(NSM2,J)/(QL(NSM2)+QL(NSM3)) 00354000
  1(NSM3))+QID(NS)*B(NS,J)/QL(NSM1) 00355000
53 BB(K,NS)=-C(NS,J)*QM(NS)-QID(NSM1)*B(NSM1,J)/(QL(NSM1)+QL(NSM2)) 00356000
  1)-QID(NS)*B(NS,J)/QL(NSM1) 00357000
    DO 65 K=NSP1,NS2 00358000
    J=K-NS 00359000
    DO 65 I=3,NSM2 00360000
65 BB(K,I)=-C(I,J)*QM(I)-QID(I-1)*B(I-1,J)/(QL(I-1)+QL(I-2))+QID(I) 00361000
  1+1)*B(I+1,J)/(QL(I+1)+QL(I)) 00362000
    NSB1=NS+IB1 00363000
    BB(NSB1,1)=-(Q-Z(1))*QM(1)-QID(1)/QL(1)-QID(2)/(QL(2)+QL(1)) 00364000

```

```

        BB(NSB1,2)=-(Q-Z(2))*QM(2)+QID(1)/QL(1)-QID(3)/(QL(3)+QL(2)) 00365000
        BB(NSB1,NSM1)=-(Q-Z(NSM1))*QM(NSM1)+QID(NSM2)/(QL(NSM2)+QL(NSM00366000
13))-QID(NS)/QL(NSM1) 00367000
        BB(NSB1,NS)=-(Q-Z(NS))*QM(NS)+QID(NSM1)/(QL(NSM1)+QL(NSM2))+QID00368000
1(NS)/QL(NSM1) 00369000
        DO 66 I=3,NSM2 00370000
66  BB(NSB1,I)=-(Q-Z(I))*QM(I)+QID(I-1)/(QL(I-1)+QL(I-2))-QID(I+1)/00371000
1(QL(I+1)+QL(I)) 00372000
        NSNB=NS+IBNB 00373000
        BB(NSNB,1)=-(Z(1)-S)*QM(1)+QID(1)/QL(1)+QID(2)/(QL(2)+QL(1)) 00374000
        BB(NSNB,2)=-(Z(2)-S)*QM(2)-QID(1)/QL(1)+QID(3)/(QL(3)+QL(2)) 00375000
        BB(NSNB,NSM1)=-(Z(NSM1)-S)*QM(NSM1)-QID(NSM2)/(QL(NSM2)+QL(NSM00376000
13))+QID(NS)/QL(NSM1) 00377000
        BB(NSNB,NS)=-(Z(NS)-S)*QM(NS)-QID(NSM1)/(QL(NSM1)+QL(NSM2))-QID00378000
1(NS)/QL(NSM1) 00379000
        DO 56 I=3,NSM2 00380000
56  BB(NSNB,I)=-(Z(I)-S)*QM(I)-QID(I-1)/(QL(I-1)+QL(I-2))+QID(I+1)/00381000
1(QL(I+1)+QL(I)) 00382000
        DO 76 I=1,NS 00383000
        FGAMMA = Y(NS4P1)+GAMMA(I)*V 00384000
        COSFGA(I)=DCOS(FGAMMA) 00385000
179  SINFGA(I)=DSIN(FGAMMA) 00386000
        FALFA=Y(NS4P1)+V*ALFA(I) 00386020
        SINFAL(I)=DSIN(FALFA) 00386030
        COSFAL(I)=DCOS(FALFA) 00386040
        QMESIN(I)=QME(I)*SINFAL(I) 00386050
        QMECOS(I)=QME(I)*COSFAL(I) 00386060
        BETA(I)=V*BETA(I) 00386070
        QIDBET=QID(I)*BETA(I) 00386080
        QIDBSN(I)=QIDBET*SINFGA(I) 00386090
76  QIDBCS(I)=QIDBET*COSFGA(I) 00386100
        DO 67 J=1,NS 00387000
        CC(J)=0.0 00388000
        DO 67 I=1,NS 00389000
67  CC(J)=CC(J)+B(I,J)*QIDBSN(I)+C(I,J)*QME(I)*SINFAL(I) 00406000
        CC(IB1)=0.0 00407000
        CC(IBNB)=0.0 00408000
        DO 78 I=1,NS 00409000
        QMZ(I)=Q-Z(I) 00410000
        CIB1=QMZ(I)*QMESIN(I)-QIDBSN(I) 00411000
        CIBNB=(Z(I)-S)*QMESIN(I)+QIDBSN(I) 00412000
        CC(IB1)=CC(IB1)+CIB1 00413000

```

```

78      CC(IBNB)=CC(IBNB)+CIBNB          00414000
      DO 71 J=NSP1,NS2                  00416000
      K=J-NS
      CC(J)=0.0                         00417000
      DO 71 I=1,NS                      00418000
71      CC(J)=CC(J)-COSFAL(I)*C(I,K)*QME(I)-B(I,K)*QIDBCS(I) 00421000
      CC(NSB1)=0.0                      00422000
      CC(NSNB)=0.0                      00423000
      DO 73 I=1,NS                      00424000
      CIB1=-QMZ(I)*QMECOS(I)+QIDBCS(I) 00425000
      CIBNB=-(Z(I)-S)*QMECOS(I)-QIDBCS(I) 00426000
      CC(NSB1)=CC(NSB1)+CIB1          00427000
73      CC(NSNB)=CC(NSNB)+CIBNB          00428000
      CC(NS2P1)=0.0                      00430000
      DO 75 I=1,NS                      00431000
      AA(NS2P1,I)=-QMESIN(I)          00432000
      BB(NS2P1,I)=QMECOS(I)          00433000
      C2NP1=QIRO(I)+QME(I)*ECC(I)    00434000
75      CC(NS2P1)=CC(NS2P1)+C2NP1      00435000
      FDOTSQ=FDOT**2                  00436000
      DO 201 I=1,NS                  00437000
      I3NS=I+NS3
      QMEFSQ(I)=QME(I)*FDOTSQ        00438000
      QKHDD(I)=QKHD(I)*(FDOT-XKF(I)*Y(I3NS)) 00439000
      QCHDD(I)=QCHD(I)*(FDOT-XCF(I)*Y(I3NS)) 00440000
      QIRO(I)=QIRO(I)*FDOT          00441000
      QKHDDF(I)=QKHD(F(I)*(FDOT-XKFF(I)*Y(I3NS)) 00442000
      QCHDDF(I)=QCHDF(I)*(FDOT-XCFF(I)*Y(I3NS)) 00443000
201      QOSAVE=FDOT                  00444000
      FDOT=FDOT-FDOFIX              00445000
      DO 202 I=1,NB                  00446000
      IBI=IB(I)                      00446500
      KKB=KK(IBI)                  00446550
      BRO=DSQRT(XB(IBI)**2+YB(IBI)**2) 00447000
202      BRGRDP(IBI)=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BRO-BROB(IBI,KKB)))+00452000
      1BKB(IBI,KKB))*(BRO***(BHB(IBI,KKB)-1.0)+BDB(IBI,KKB)+BEB(IBI,KKB)/ 00454000
      2BRO)                          00455000
      FDOT=FOSAVE                  00455500
      DO 63 I=1,NS                  00456000
      A(I)=0.0                      00457000
      DO 54 I=1,NB                  00458000
      IBI=IB(I)                      00459000

```

```

54      A(IBI)=BRGROP(IBI)*XB(IBI)+BCB(IBI)*XBDOT(IBI)          00460000
      DO 203 I=1,NS                                         00461000
203      AFX(I)=-(QM(I)*GX-QMEFSQ(I)*COSFAL(I)+QK(I)*XX(I)+QC(I)*XDOT(I)) 00462000
      1)+QKPF(I)*YY(I)+QCP(I)*YDOT(I)+QKHDD(I)*YY(I)+QCHDD(I)*YDOT(I))-A(I) 00463000
      2).                                         00464000
      DO 204 I=1,NB                                         00465000
      IBI=IB(I)                                         00466000
204      A(IBI)=BRGROP(IBI)*YB(IBI)+BCB(IBI)*YBDOT(IBI)          00467000
      DO 205 I=1,NS                                         00468000
      TRIG=(QID(I)-QIRO(I))*FDOTSQ*BETA(I)                  00469010
      APY(I)=-(QM(I)*GY-QMEFSQ(I)*SINFAL(I)+QK(I)*YY(I)+QC(I)*YDOT(I)) 00469000
      1)-QKPF(I)*XX(I)-QCP(I)*XDOT(I)-QKHDD(I)*XX(I)-QCHDD(I)*XDOT(I))-A(I) 00470000
      2)                                         00471000
      AMY1(I)=TRIG*CUSFGA(I)                               00472000
205      AMX1(I)=-TRIG*SINFGA(I)                           00473000
      I=1                                         00474000
      J=2                                         00475000
      K=1                                         00476000
57      AMY2(I)=-1./QL(K)*(QIRFDO(I)*(YDUT(J)-YDOT(K))+QKF(I)*(XX(J)-X 00477000
      1X(K))+                                         00478000
      1-QCF(I)*(XDOT(J)-XDOT(K))+QKPF(I)*(YY(J)-YY(K))+QCPF(I)*(YDOT( 00479000
      2J)-YDOT(K))+QKHDF(I)*(YY(J)-YY(K))+QCHDF(I)*(YDUT(J)-YDOT(K))) 00480000
      AMX2(I)=1./QL(K)*(-QIRFDO(I)*(XDOT(J)-XDOT(K))+QKF(I)*(YY(J)-Y 00481000
      1Y(K))+QCF(I)*(YDOT(J)-YDOT(K))-QKPF(I)*(XX(J)-XX(K))-QCPF(I)*(XDOT( 00482000
      2(J)-XDOT(K))-QKHDF(I)*(XX(J)-XX(K))-QCHDF(I)*(XDOT(J)-XDOT(K))) 00483000
      IF (I.GT.1) GO TO 58                               00484000
      I=NS                                         00485000
      J=NS                                         00486000
      K=NSM1                                         00487000
      GO TO 57                                         00488000
58 DO 59 I=2,NSM1                                     00490000
      AMY2(I)=-1./QL(I)*(QIRFDO(I)*(YDUT(I+1)-YDOT(I-1))+ 00491000
      1QKF(I)*(XX(I+1)-XX(I-1))+QCF(I)*(XDOT(I+1)-XDOT(I-1))+QKPF(I)*(YY( 00492000
      2I+1)-YY(I-1))+QCPF(I)*(YDOT(I+1)-YDOT(I-1))+QKHDF(I)*(YY(I+1)-YY( 00493000
      3I-1))+QCHDF(I)*(YDUT(I+1)-YDOT(I-1)))          00494000
59      AMX2(I)=1./QL(I)*(-QIRFDO(I)*(XDOT(I+1)-XDOT(I-1))+ 00495000
      1KF(I)*(YY(I+1)-YY(I-1))+QCF(I)*(XDOT(I+1)-XDOT(I-1))-QKPF(I)*(XX( 00496000
      2+1)-XX(I-1))-QCPF(I)*(XDOT(I+1)-XDOT(I-1))-QKHDF(I)*(XX(I+1)-XX( 00497000
      3-1))-QCHDF(I)*(XDOT(I+1)-XDOT(I-1)))          00498000
      DO 60 I=1,NS                                         00499000
      AMY(I)=AMY1(I)+AMY2(I)                           00500000
60      AMX(I)=AMX1(I)+AMX2(I)                         00501000

```

```

DO 69 J=1,NS
DST=0.
DO 44 I=1,NS
DSTAT = -C(I,J)*AFX(I)-B(I,J)*AMY(I)
44 DST=DST+DSTAT
69 DSTA(J)=DST-XX(IB1)-(XX(IBNB)-XX(IB1))*(Z(J)-S)/QLL+XX(J)
      DSTA(IB1)=0.0
      DSTA(IBNB)=0.0
      DO 43 I=1,NS
      DST1 = -((Q-Z(I))*AFX(I)-AMY(I))
      DSTA(IB1)=DSTA(IB1)+DST1
      DSTNB=-(Z(I)-S)*AFX(I)-AMY(I)
43  DSTA(IBNB)=DSTA(IBNB)+DSTNB
      DO 42 J=NSP1,NS2
      K=J-NS
      DST=0.
      DO 41 I=1,NS
      DSTAT=-C(I,K)*AFY(I)+B(I,K)*AMX(I)
41  DST=DST+DSTAT
42  DSTA(J)=DST-YY(IB1)-(YY(IBNB)-YY(IB1))*(Z(K)-S)/QLL+YY(K)
      DSTA(NSB1)=0.0
      DSTA(NSNB)=0.0
      DO 70 I=1,NS
      DSTN1=-(Q-Z(I))*AFY(I)-AMX(I)
      DSTA(NSB1)=DSTA(NSB1)+DSTN1
      DSTNB=-(Z(I)-S)*AFY(I)+AMX(I)
70  DSTA(NSNB)=DSTA(NSNB)+DSTNB
      DSTA(NS2P1)=          TMZ1*FDOT**TMZ+TMZ2*FDOT+TMZ3
      DO 72 I=1,NS
      DSR=-(QME(I)*(GY*COSFAL(I)-GX*SINFAL(I))+(CZ1(I)*FDOT**CZ(I)+1
      CZ2(I)*FDOT))
72  DSTA(NS2P1)=DSTA(NS2P1)+DSR
      DO 172 J=1,NS
      JNS=J+NS
      DO 172 I=1,NS
      INS=I+NS
      AROT(J,I)=AA(J,I)*COSAB(I)
      AROT(J,INS)=-AA(J,I)*SINAB(I)*Y(I)
      AROT(JNS,I)= BB(JNS,I)*SINAB(I)
      AROT(JNS,INS)=BB(JNS,I)*COSAB(I)*Y(I)
      AROT(NS2P1,I)=AA(NS2P1,I)*COSAB(I)+BB(NS2P1,I)*SINAB(I)
172 AROT(NS2P1,INS)=-AA(NS2P1,I)*SINAB(I)*Y(I)+BB(NS2P1,I)*COSAB(I)*Y(I)
172 AROT(NS2P1,INS)=AA(NS2P1,I)*SINAB(I)*Y(I)+BB(NS2P1,I)*COSAB(I)*Y(I)
00502000
00503000
00504000
00505000
00506000
00507000
00508000
00509000
00510000
00511000
00512000
00513000
00514000
00517000
00518000
00519000
00520000
00521000
00522000
00523000
00524000
00525000
00526000
00527000
00528000
00529000
00530000
00532000
00533000
00534000
00535000
00536000
00537000
00538000
00539000
00540000
00541000
00542000
00543000
00544000
00546000
00547000

```

```

11)          00548000
      DO 173 J=1,NS
      TRIG=0.          00549000
      DO 174 I=1,NS
      INS =I+NS          00550000
      I2NS=I+NS2          00551000
      I3NS=I+NS3          00552000
      DROT=AA(J,I)*(YDUT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*SINAB(I))          00553000
174  TRIG=TRIG+DROT          00554000
173  DRO(J)=TRIG+DSTA(J)          00555000
      DO 175 J=NSP1,NS2
      TRIG=0.          00556000
      DO 176 I=1,NS
      INS=I+NS          00557000
      I2NS=I+NS2          00558000
      I3NS=I+NS3          00559000
      DROT=-BB(J,I)*(XDOT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*COSAB(I))          00560000
176  TRIG=TRIG+DROT          00561000
175  DRO(J)=TRIG+DSTA(J)          00562000
      DRO(NS2P1)=DSTA(NS2P1)          00563000
      DO 177 I=1,NS
      INS=I+NS          00564000
      I2NS=I+NS2          00565000
      I3NS=I+NS3          00566000
      DROT=AA(NS2P1,I)*(YDUT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*SINAB(I))-BB00574000
1(NS2P1,I)*(XDOT(I)*Y(I3NS)+Y(I2NS)*Y(I3NS)*COSAB(I))          00567000
177  DRO(NS2P1)=DRO(NS2P1)+DROT          00569000
      DO 39 J=1,NS2P1
      AROT(J,NS2P1)=CC(J)          00570000
      FF=0.0          00571000
      183
      KL=ISIMDD(51,NS2P1,1,AROT,DRO,FF,IA)          00572000
      DO 600 I=1,NS
      INS=I+NS          00573000
      I2NS=I+NS2          00574000
      I3NS=I+NS3          00575000
      BD(I)=Y(I2NS)          00576000
      BD(INS)=Y(I3NS)          00577000
      BD(I2NS)=ARUT(I,1)          00578000
      BD(I3NS)=ARUT(INS,1)          00579000
      BD(NS4P1)=Y(NS4P2)          00581000
      BD(NS4P2)=ARUT(NS2P1,1)          00582000
      WRITE (6,620) (BD(I),I=1,N)          00583000
      00584000
      00585000
      00586000
      00587000
      00588000
      00589000
      00590000
      00591000
      00592000

```

```
620 FORMAT (1H1///14X,33HTHE APPROPRIATE DERIVATIVES BD,S//(1P7D15.6)00593000
1) 00594000
    IF (KL.NE.3) GO TO 560 00595000
    WRITE (6,540) KL 00596000
540 FORMAT(1H1/10X,4HKL =I1) 00597000
    CALL EXIT 00598000
560 RETURN 00599000
    END 00600000
```

Card Count 344

```

SUBROUTINE BRGXY                               00223020
  IMPLICIT REAL*8 (A-H,U-Z)                   00223030
  COMMON/MAFU14/ FDOFIX                      00223035
  COMMON/MAFU1/ DUM1(75),NS,NB,IB(12)          00223040
  X      /MAFU0/IDUM1(2),KK(25)                00223045
  1      /MAFU5/DUM2(300),BKMX(25),BKMY(25),BCMX(25),BCMY(25),BCB(25) 00223050
  2      /MAFU6/BNB(25,6),BBB(25,6),BROB(25,7),BKB(25,6),BHB(25,6), 00223060
  3BDB(25,6),BEB(25,6)                      00223070
  4      /MAFU8/TULB                           00223080
  5      /MAFU11/FD,FDOT                      00223090
  6      /GARBG7/XX(25),YY(25),XB(25),YB(25),XDOT(25),YDOT(25), 00223100
  7XBDUT(25),YBDOT(25)                      00223110
  8      /GARBG8/XB0,XB1,XB2,YB0,YB1,YB2,BROX0,BROX1,BROX2,BROY0, 00223120
  9BROY1,BROY2,FBRGX0,FBRGX1,FBRGX2,FBRGY0,FBRGY1,FBRGY2,I,IBI,KKB
    FUSAVE=FDUT                            00223130
    FDUT=FDOT-FDOFIX                         00223200
  10 DU 162 I=1,NB                           00223300
    IBI=IB(I)                                00224000
    IF (BCMX(IBI).NE.0)  GO TO 91            00225000
    BCMX(IBI)=1.D-16                         00226000
  91    XBDOT(IBI)=XDOT(IBI)/(1+BCB(IBI)/BCMX(IBI)) 00227000
  185   IF (BCMY(IBI).NE.0)  GO TO 101          00228000
    BCMY(IBI)=1.D-16                         00229000
  101   YBDOT(IBI)=YDOT(IBI)/(1+BCB(IBI)/BCMY(IBI)) 00230000
    IF(BCB(IBI).EQ.0.)BCB(IBI)=1.D-16        00231000
    KKB=KK(IBI)                                00234000
    XB2=0.3D0*XX(IBI)                         00240000
    XB1=0.1D0*XX(IBI)                         00241000
    BROX2=DSQRT(XB2**2+(YY(IBI)/(BKMX(IBI)/BKMY(IBI)*(XX(IBI)/XB2-1.)) 00242000
  1+1.))**2)                                  00244000
    BROX1=DSQRT(XB1**2+(YY(IBI)/(BKMX(IBI)/BKMY(IBI)*(XX(IBI)/XB1-1.)) 00245000
  1+1.))**2)                                  00246000
    FBRGX2=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROX2-BRUB(IBI,KKB))) + 00247000
  1BKB(IBI,KKB))*(BROX2**2*(BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB)/00248000
  2BROX2)*XB2-BKMX(IBI)*(XX(IBI)-XB2)        00249000
    FBRGX1=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROX1-BROB(IBI,KKB))) + 00250000
  1BKB(IBI,KKB))*(BROX1**2*(BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB)/00251000
  2BROX1)*XB1-BKMX(IBI)*(XX(IBI)-XB1)        00252000
  52  XB0=XB1-FBRGX1/(FBRGX2-FBRGX1)*(XB2-XB1) 00253000
    BROX0=DSQRT(XB0**2+(YY(IBI)/(BKMX(IBI)/BKMY(IBI)*(XX(IBI)/XB0-1.)) 00254000
  1+1.))**2)                                  00255000
    FBRGX0=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BROX0-BROB(IBI,KKB))) + 00256000

```

```

1BKB(IBI,KKB))*(BRUX0**((BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB))/00257000
2BRUX0)*XBO-BKMX(IBI)*(XX(IBI)-XBO) 00258000
PERCEX=XBO/XX(IBI) 00258500
IF(DABS(1.-PERCEX).LE.1.D-8)GOTO152 00259000
IF(DABS(PERCEX).LT.1.D-8)GOTO152 00259500
PERCEX=DABS(FBRGX0/(BKMX(IBI)*(XX(IBI)-XBO))) 00260000
IF(PERCEX.LT.TOLB) GO TO 152 00265000
IF(DABS(XB2).GE.DABS(XB1))XB2=XB1 00265500
XB1=XBO 00267000
FBRGX2=FBRGX1 00268000
FBRGX1=FBRGX0 00269000
GO TO 52 00270000
152 XB(IBI)=XBO 00271000
YB2=0.3D0*YY(IBI) 00275000
YB1=0.1D0*YY(IBI) 00276000
BRDY2=DSQRT((YB2**2+(XX(IBI)/(BKMY(IBI)/BKMX(IBI)*(YY(IBI)/YB2-1.)) 00277000
1+1.))***2) 00278000
BRDY1=DSQRT((YB1**2+(XX(IBI)/(BKMY(IBI)/BKMX(IBI)*(YY(IBI)/YB1-1.)) 00279000
1+1.))***2) 00280000
FBRGY2=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BRUY2-BROB(IBI,KKB)))+ 00281000
1BKB(IBI,KKB))*(BRDY2**((BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB))/00282000
2BRDY2)*YB2-BKMY(IBI)*(YY(IBI)-YB2) 00283000
FBRGY1=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BRUY1-BROB(IBI,KKB)))+ 00284000
1BKB(IBI,KKB))*(BRDY1**((BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB))/00285000
2BRDY1)*YB1-BKMY(IBI)*(YY(IBI)-YB1) 00286000
64 YBO=YB1-FBRGY1/(FBRGY2-FBRGY1)*(YB2-YB1) 00287000
BRUY0=DSQRT((YB0**2+(XX(IBI)/(BKMY(IBI)/BKMX(IBI)*(YY(IBI)/YB0-1.)) 00288000
1+1.))***2) 00289000
FBRGY0=(FDOT*(BNB(IBI,KKB)+BBB(IBI,KKB)*(BRUY0-BRUB(IBI,KKB)))+ 00290000
1BKB(IBI,KKB))*(BRYO**((BHB(IBI,KKB)-1.))+BDB(IBI,KKB)+BEB(IBI,KKB))/00291000
2BRYO)*YB0-BKMY(IBI)*(YY(IBI)-YB0) 00292000
PERCEY=YB0/YY(IBI) 00292500
IF(DABS(1.-PERCEY).LE.1.D-8)GOTO162 00293000
IF(DABS(PERCEY).LT.1.D-8)GOTO162 00293500
PERCEY=DABS(FBRGY0/(BKMY(IBI)*(YY(IBI)-YB0))) 00294000
IF(PERCEY.LT.TOLB) GO TO 162 00299000
IF(DABS(YB2).GE.DABS(YB1))YB2=YB1 00300000
YB1=YB0 00301000
FBRGY2=FBRGY1 00302000
FBRGY1=FBRGY0 00303000
GO TO 64 00304000
162 YB(IBI)=YB0 00305000
FDOT=FOSAVE 00305500
RETURN 00306000
END 00307000

```

```

SUBROUTINE INFLCO (C,B) 00601000
IMPLICIT REAL*8 (A-H,O-Z) 00602000
REAL*8 MOF,MOM 00603000
REAL Z 00604000
DIMENSION DD(25), D(25), QL(25),EE(25), GG(25), EIO(25), GAK(25),00605000
1SZ(25),Z0(25),SZOL(25),ZQOL(25),QZ(25),ZSOL(25),QZOL(25),IB(12), 00606000
2SHERGA(25),ROF(25,25),ROM(25,25),AFALEI(25),QLEI(25),SQL2EI(25) 00606010
2,C(25,25),B(25,25), Z(25), FOF(25,25), MOF(25,25), FOM(25,25)0060607000
3),MOM(25,25) 00608000
COMMON/MAFUF1/DD,D,QL, NS,NB,IB 00613000
COMMON/MAFUF2/ SZ,QZ,Z0,SZOL,ZSOL,QZOL,ZQOL,QMLDV,QLL 00614000
COMMON/MAFUF3/ IB1,IB2 00615000
COMMON/MAFUF4/ Z 00616000
COMMON/MAFC/ EE,GG,EIO,GAK 00617000
1 /SFP2/NSM1, IDUM1(9) 00617030
2 /GARBG2/FOF,MOF,FOM,MOM,QLEI,SQL2EI,ROF,ROM,AFALEI,SHERGA,A,00617060
3E,F,G,GAQL,THETAM,THETAF,DSQ,DDSQ,D1,D2,I,J,K 00617090
DATA PI/3.1415926535897932/ 00617120
A=PI/4. 00619000
E=A/16. 00620000
F=4./3.D0 00621000
G=F/2. 00622000
187
DO 200 J=1,NSM1 00624000
DSQ=D(J)**2 00624500
DDSQ=DD(J)**2 00625000
D1=DDSQ-DSQ 00625500
D2=DDSQ+DSQ 00626000
QLEI(J)=QL(J)/(E*D1*D2*EE(J)+EIO(J)) 00626500
SQL2EI(J)=0.5* QL(J)* QLEI(J) 00627000
GAQL=GAK(J)/QL(J)+A*GG(J)*D1/(QL(J)*(F*(DDSQ*DD(J)-DSQ*D(J))/( D2*00630000
1(DD(J)-D(J)))) 00631000
SHERGA(J)=1./GAQL 00632000
200 AFALEI(J)=SHERGA(J)+G*QL(J)*SQL2EI(J) 00633000
DO 400 I=1,NS 00642000
DO 400 J=1,NS 00643000
FOF(I,J)=0. 00643200
MOF(I,J)=0. 00643400
FOM(I,J)=0. 00643600
MOM(I,J)=0. 00643800
C(I,J)=0 00644000
400 B(I,J)=0 00645000
DO 140 I=1,IB1 00646000

```

112 IF(IB1.EQ.1) GO TO 122	00647000
112 K=I+1	00654000
112 DO 120 J=K,IB1	00655000
112 FOF(I,J) =-1.0	00656000
112 MOF(I,J) =Z(J)-Z(I)	00657000
120 MOM(I,J) = -1.0	00659000
122 K=IB1+1	00660000
122 DO 130 J=K,IB2	00661000
122 FOF(I,J)= SZOL(I)	00662000
122 MOF(I,J)=QZ(J)*SZOL(I)	00663000
122 FOM(I,J)=QML0V	00664000
130 MOM(I,J)=ZQOL(J)	00665000
140 CONTINUE	00673000
140 KK=IB1+1	00674000
140 DO 440 I=KK, IB2	00675000
412 K=IB1+1	00682000
412 DO 420 J=K, I	00683000
412 FOF(I,J) = QZOL(I)	00684000
412 MOF(I,J)=SZ(J)*QZOL(I)	00685000
412 FOM(I,J) =QML0V	00686000
420 MOM(I,J) =ZSOL(J)	00687000
420 K=I+1	00688000
188 IF(K.GT.IB2)GOT0440	00689000
188 DO 430 J=K, IB2	00690000
188 FOF(I,J) =SZOL(I)	00691000
188 MOF(I,J)=-ZQ(J)*SZOL(I)	00692000
188 FOM(I,J)=QML0V	00693000
430 MOM(I,J)=ZQOL(J)	00694000
440 CONTINUE	00702000
440 IF (IB2.EQ.NS)GO TO 542	00703000
440 KK=IB2+1	00704000
440 DO 540 I=KK, NS	00705000
512 K=IB1+1	00712000
512 DO 520 J=K, IB2	00713000
512 FOF(I,J) =QZOL(I)	00714000
512 MOF(I,J)=SZ(J)*QZOL(I)	00715000
512 FOM(I,J)=QML0V	00716000
520 MOM(I,J) =ZSOL(J)	00717000
520 K=IB2+1	00718000
520 DO 530 J=K, I	00719000
520 FOF(I,J) =1.	00720000
520 MOF(I,J) =Z(I)-Z(J)	00721000

```

530  MOM(I,J) =1.0          00723000
540  CONTINUE                00731000
542  DO 600  I=1,NS          00732000
     ROF(I,1)=0.              00732200
     ROM(I,1)=0.              00732400
     THETAF=0.                00732600
     THETAM=0.                00732800
     DO 600  J=2,NS          00733000
     ROF(I,J)=ROF(I,J-1)+QL(J-1)*THETAF+AFALEI(J-1)*FOF(I,J)+MOF(I,J)* 00734000
     1SQL2EI(J-1)              00735000
     ROM(I,J)=ROM(I,J-1)+QL(J-1)*THETAM+AFALEI(J-1)*FOM(I,J)+MOM(I,J)* 00736000
     1SQL2EI(J-1)              00737000
     THETAF=THETAF+SQL2EI(J-1)*FOF(I,J)+QLEI(J-1)*MOF(I,J)          00738000
600  THETAM=THETAM+SQL2EI(J-1)*FOM(I,J)+QLEI(J-1)*MOM(I,J)          00739000
     DO 700  I=1,NS          00742000
     DO 700  J=1,NS          00743000
     C(I,J)=ROF(I,J)-ROF(I,IB1)+SZOL(J)*(ROF(I,IB2)-ROF(I,IB1)) 00748000
700  B(I,J)=ROM(I,J)-ROM(I,IB1)+SZOL(J)*(ROM(I,IB2)-ROM(I,IB1)) 00749000
     RETURN
     END                      00750000
                                         00751000

```

```

SUBROUTINE SZMASS                               01496000
  IMPLICIT REAL*8 (A-H,O-Z)                   01497000
  REAL Z                                       01498000
  DIMENSION Z(25),QL(25),SZ(25),QZ(25),ZQ(25),SZOL(25),ZSOL(25),  Q01500000
1 ZOL(25),ZQOL(25), DD(25),D(25),DN(25),  Q6LDND(25),DDPLD(25),DDL(201501000
25),Q1LDND(25), QM(25),AM(25),QID(25),AID(25),QIRO(25),AIRO(25)  01502000
  DIMENSION ECC(25),ALFA(25),BETA(25),GAMMA(25)                   01503000
1,IB(12)                                      01503100
  COMMON/MAFU2/ DN,AM,AID,ECC,ALFA,BETA,GAMMA,GX,GY 01504000
  COMMON/MAFU12/ QM,QID,QIRO                   01505000
1     ./SFP2/NSM1,1DUM1(9)                      01505100
2     /GARBG1/QMASS,QLDNDD,QL2,Q1LDND,DDL,DDPLD,Q6LDND,D2,  01505200
3DD2,WEIT,PDOLARA,I                           01505300
  COMMON/MAFUF1/DD,D,QL,NS,NB,IB               01506000
  COMMON/MAFUF2/SZ,QZ,ZQ,SZOL,ZSOL,QZOL,ZQOL,QMLOV,QLL 01507000
  COMMON/MAFUF3/ IB1,IBNB                      01508000
  COMMON/MAFUF4/Z/MAIFU/Q,S                  01509000
  DATA PI/3.1415926535897932/                 01510000
  U=4./3.D0                                     01512000
  W=PI/(128.*386.088)                         01514000
  E=16.*W                                       01515000
  Z(1)=0                                         01518000
  DO 105 I=2,NS                                01519000
105  Z(I)=Z(I-1)+QL(I-1)                      01520000
  Q=Z(IBNB)                                     01522000
  S=Z(IB1)                                      01521000
  DO 103 I=1,NS                                01523000
  SZ(I)=Z(IB1)-Z(I)                           01525000
  ZQ(I)=Z(I)-Z(IBNB)                         01526000
103  QZ(I)=-ZQ(I)                            01527000
  QLL=Z(IBNB)-Z(IB1)                         01528000
  QMLOV=-1./QLL                                01530000
    DO 104 I=1,NS                                01531000
  SZOL(I)=SZ(I)/QLL                           01532000
  ZSOL(I)=-SZOL(I)                           01533000
  QZOL(I)=QZ(I)/QLL                           01534000
104  ZQOL(I)=-QZOL(I)                         01535000
  DO 50 I=1,NS                                01536000
  DD2    = DD(I)**2                           01537000
  D2     = D(I)**2                           01538000
  QL2    = QL(I)**2                           01541000
  QLDNDD = QL(I)* DN(I)*(DD2      -D2      )  01542000

```

```

Q6LDND(I) = W*QLDNDD          01543000
DDPLD(I) = DD2    +D2          01544000
DDL(I) = DDPLD(I)+           U*QL2          01545000
50 Q1LDND(I) =           E*QLDNDD          01546000
QM(1)=Q1LDND(1) + AM(1)        01547000
QM(NS)=Q1LDND(NS-1) +AM(NS)    01548000
QID(1) = Q6LDND(1)*DDL(1) + AID(1) 01549000
QID(NS)= Q6LDND(NS-1)*DDL(NS-1) +AID(NS) 01550000
QIRO(1)=2.*Q6LDND(1)*DDPLD(1)+AIRO(1) 01551000
QIRO(NS)=2.*Q6LDND(NS-1)*DDPLD(NS-1) +AIRO(NS) 01552000
DO 55 I=2,NSM1                01554000
QM(I) =Q1LDND(I-1)+Q1LDND(I) + AM(I)          01555000
QID(I)=Q6LDND(I-1)*DDL(I-1)+Q6LDND(I)*DDL(I)+AID(I) 01556000
55 QIRO(I) = 2.*(Q6LDND(I-1)*DDPLD(I-1)+Q6LDND(I)*DDPLD(I)) +AIRO(I) 01557000
62 QMASS=0.0                     01558000
POLARA=0.0                      01559000
DO 40 I=1,NS                     01560000
QMASS=QMASS+QM(I)              01561000
40 POLARA=POLARA+QIRO(I)        01562000
WEIT =386.088*QMASS            01563000
WRITE (6,77) WEIT,QMASS,POLARA 01564000
77 FORMAT(1H0///,9X'TOTAL ROTOR WEIGHT,LB ='1PD13.4/ 9X,33HTOTAL ROT001565000
1R MASS,(LB*SEC**2)/IN =1PD13.4/ 9X,57HTOTAL ROTOR POLAR MASS MOMEN01566000
2T OF INERTIA, LB*IN*SEC**2 =1PD13.4/)          01567000
      RETURN                      01568000
      END                         01569000

```

```

SUBROUTINE PLUT1(F)                               00752000
REAL F(50,12), Y(2)                            00753000
COMMON/SFP1/RPM(50),NPOINT                      00754000
1      /GARBG4/Y,I,J                            00755100
2      /MAFUF1/DUM1(151),NB,IB(12)              00755200
C      GET LIMITS OF F & RPM                   00757000
Y(1)=1.E70                                     00760000
Y(2)=-1.E70                                    00761000
DO 10 I=1,NB                                    00762000
DO 10 J=1,NPOINT                                00763000
Y(1)=AMIN1(Y(1),F(J,I))                      00764000
10 Y(2)=AMAX1(Y(2),F(J,I))                   00765000
CALL LRANGE(RPM(1),RPM(NPOINT),Y(1),Y(2))    00767000
C      PLOT EACH OF THE NB FUNCTIONS           00768000
DO 20 I=1,NB                                    00769000
CALL LRCURV(RPM,F(1,I),NPOINT,2,Y,0.)        00769100
CALL LRCNVT(IB(I),1,Y,1,3,0)                  00769200
DO 20 J=1,NPOINT                                00769300
20 CALL LRLABL(Y,3,0,RPM(J),F(J,I),0.)       00769400
RETURN                                         00770100
END                                            00771000

```

192

Card Count 21

C SOLUTION TO A SYSTEM OF 1ST ORDER ORDINARY DIFFERENTIAL EQUATIONS RKAD0010
 C OF THE INITIAL VALUE TYPE. THE FOLLOWING METHODS ARE AVAILABLE---RKAD0020
 C 1. ADAMS-MOULTON PREDICTOR-CORRECTOR FIXED INCREMENT RKAD0030
 C 2. ADAMS-MOULTON PREDICTOR-CORRECTOR VARIABLE INCREMENT RKAD0040
 C 3. RUNGE-KUTTA (ALSO USED TO GENERATE STARTING VALUES FOR A-M METHODS)RKAD0050
 SUBROUTINE RKADAM(N,T,Y,H,IND,ITIM,TOL,NERR) RKAD0060
 IMPLICIT REAL*8 (A-H,O-Z) RKAD0070
 REAL*8 T,Y,H,TOL RKAD0080
 DIMENSION Y(1),F(102,7),YB(102,5),A(102,4),YSAVE(102) RKAD0090
 C ORDER OF SYSTEM (IF REDIMENSIONING REQUIRED, CHANGE NN IN DATA RKAD0100
 C STATEMENT AND ALSO THE 1ST SUBSCRIPT OF F AND YB IN DIMENSION RKAD0110
 C STATEMENT). FROM CALLING PROGRAMRKAD0115
 C T INDEPENDENT VARIABLE -- UPON ENTRY TO RKADAM FROM CALLING RKAD0120
 C PROGRAM X IS AT BEGINNING OF STEP. UPON RETURN TO CALLING RKAD0130
 C PROGRAM X IS AT END OF STEP. RKAD0140
 C Y SOLUTION VECTOR OF DEPENDENT VARIABLES AS A FUNCTION OF X RKAD0150
 C H INCREMENT(ALGEBRAIC) -- UPON ENTRY TO RKADAM FROM CALLING RKAD0160
 C PROGRAM H IS THE TRIAL INCREMENT FOR THIS STEP. UPON RETURN TURKAD0170
 C CALLING PROGRAM H IS THE TRIAL INCREMENT FOR THE NEXT STEP. RKAD0180
 C IND FLAG TO SELECT METHOD FROM CALLING PROGRAMRKAD0190
 C =0 ADAMS-MOULTON PREDICTOR-CORRECTOR VARIABLE INCREMENT H RKAD0200
 C =1 RUNGE-KUTTA FIXED INCREMENT H RKAD0210
 C =2 ADAMS-MOULTON FIXED INCREMENT H RKAD0220
 C ITIM RESTART FLAG (APPLIES ONLY FOR IND=0,2) FROM CALLING PROGRAMRKAD0230
 C =+1 RESTART WITH FORWARDS (IN THE DIRECTION OF SIGN(H)) RKAD0240
 C INTEGRATION BY RUNGE-KUTTA TO GET STARTING VALUES RKAD0250
 C =-1 SAME AS +1 EXCEPT USES BACKWARDS (-SIGN(H)) INTEGRATION RKAD0260
 C =0 CONTINUE INTEGRATING RKAD0270
 C TOL APPLIES ONLY TO IND=0. ALLOWABLE RELATIVE ERROR BETWEEN THE RKAD0280
 C PREDICTED AND CORRECTED SOLUTIONS. FROM CALLING PROGRAMRKAD0290
 C NERR ERROR FLAG RETURNED TO CALLING PROGRAM RKAD0300
 C =0 SOLUTION IS VALID RKAD0310
 C =1 SOLUTION INVALID -- N IS INVALID OR ELSE H HAS GONE TO 0 RKAD0320
 DATA NN/102/ RKAD0330
 NERR=0 RKAD0340
 DO 200 I=1,N RKAD0341
 200 YSAVE(I)=Y(I) RKAD0342
 IF(IND.EQ.1)GOTO10 RKAD0350
 IF(ITIM.EQ.0)GOTO170 RKAD0360
 10 NS=0 RKAD0370
 CALL OVERFL(K) RKAD0375
 C TEST VALIDITY OF N RKAD0380

```

      IF(N.GE.1.AND.N.LE.NN)GOTO40
20  NERR=1
30  DO 220 I=1,N
220  Y(I)=0.5*(Y(I)+YSAVE(I))
      RETURN
40  IF(IND.NE.1)GOTO50
C    RUNGE-KUTTA
      CALL RUNKUT(N,H,T,Y,Y,A)
      T=T+H
      GOT030.
C    ADAMS-MOULTON
50  IF(ITIM.LT.0)GOTO180
C    RESTART FORWARDS INTEGRATION (IN DIRECTION OF X+H
      .ISTFLG=1
      IF(IND.EQ.2)ISTFLG=0
      XB=T
      DO60I=1,N
60  YB(I,1)=Y(I)
      L=0
70  DO80I=1,3
      T=XB+H*(I-1)
      J=5-I
      CALL RUNKUT(N,H,T,Y,YB(1,I+1),A)
      DO 75 K=1,N
      IF(DABS(YB(K,I+1)).LE.1.D12)GOTO73
      H=H/10.
      GOTO 83
73  IF(L.NE.0.AND.I.EQ.1)GOTO75
      F(K,J)=A(K,1)
75  Y(K)=YB(K,I+1)
80  CONTINUE
      T=T+H
      CALL FUND(N,T,Y,F)
      DO 2 I=1,N
      IF(Y(I).GE.1.D14) GO TO 3
2  CONTINUE
      GO TO 4
3  RETURN
4  CALL ADAMLT(N,T,Y,H,IND,ITIM,TOL,NERR,ISTFLG,F,A)
      IF(ISTFLG.EQ.0)GOTO110
      H=H/2
83  T=XB
      RKAD0390
      RKAD0400
      RKAD0401
      RKAD0402
      RKAD0411
      RKAD0420
      RKAD0430
      RKAD0440
      RKAD0450
      RKAD0460
      RKAD0470
      RKAD0480
      RKAD0490
      RKAD0500
      RKAD0505
      RKAD0510
      RKAD0520
      RKAD0530
      RKAD0535
      RKAD0540
      RKAD0560
      RKAD0567
      RKAD0570
      RKAD0575
      RKAD0577
      RKAD0579
      RKAD0581
      RKAD0583
      RKAD0585
      RKAD0590
      RKAD0595
      RKAD0605
      RKAD0607,
      RKAD0608
      RKAD0609
      RKAD0610
      RKAD0611
      RKAD0612
      RKAD0615
      RKAD0620
      RKAD0640
      RKAD0645

```

```

    CALL OVERFL(K) RKAD0650
    IF(K.EQ.3)GOTO100 RKAD0660
    D0901=1,N RKAD0670
90  Y(I)=YB(I,1) RKAD0680
    L=1 RKAD0685
    GOT070 RKAD0690
100 H=0. RKAD0700
    GOT020 RKAD0710
C   FOR ITIM = +1, FEED STARTING VALUES BACK TO CALLING PROGRAM ONE ATRRKAD0720
C   A TIME (RUNGE-KUTTA SOLNS 1ST 4 PTS AND THEN A-M SOLN FOR 5TH PT) RKAD0730
110 D0120I=1,N RKAD0740
120 YB(I,5)=Y(I) RKAD0750
    NS=2 RKAD0760
130 T=XB+H*(NS-1) RKAD0770
    D0140I=1,N RKAD0780
140 Y(I)=YB(I,NS) RKAD0790
    NS=NS+1 RKAD0800
    IF(NS.GT.5)NS=0 RKAD0810
    GOT030 RKAD0820
C   CONTINUE INTEGRATION PROCEDURE (A-M) RKAD0830
170 IF(NS)190,190,130 RKAD0840
180 ISTFLG=0 RKAD0850
190 CALL ADAMLT(N,T,Y,H,IND,ITIM,TOL,NERR,ISTFLG,F,A) RKAD0860
    GOT030 RKAD0870
    END RKAD0880

```

C 4TH ORDER RUNGE-KUTTA INTEGRATION FOR A SYSTEM OF 1ST ORDER,
 C ORDINARY DIFFERENTIAL EQNS.
 C SUBROUTINE RUNKUT(N,H,X,Y,YY,A)
 C SEE DIMENSION STATEMENT FOR LIMITATION ON ORDER OF SYSTEM
 C CHANGE DIMENSION AS IS REQUIRED, I.E.,A(MAXORDER,4), V(MAXORDER)
 C IMPLICIT REAL*8 (A-H,O-Z)
 REAL*8 X,H,Y(1),YY(1)
 DIMENSION A(102,4),V(102)
 C N = ORDER OF SYSTEM
 C H = INTEGRATION STEP
 C X = INDEPENDENT VARIABLE AT BEGINNING OF STEP
 C Y = VECTOR OF DEPENDENT VARIABLES AT BEGINNING OF STEP
 C YY = SOLUTION VECTOR OF DEPENDENT VARIABLES AT END OF STEP
 X1=H/2.
 X2=X+X1
 X3=X+H
 CALL FUND(N,X,Y,A(1,1))
 DO 2 I=1,N
 IF(Y(I).GE.1.D14) GO TO 3
 2 CONTINUE
 GO TO 4
 3 RETURN
 4 DO 10 I=1,N
 10 V(I)=Y(I)+X1*A(I,1)
 CALL FUND(N,X2,V,A(1,2))
 DO 12 I=1,N
 IF(Y(I).GE.1.D14) GO TO 13
 12 CONTINUE
 GO TO 14
 13 RETURN
 14 DO 20 I=1,N
 20 V(I)=Y(I)+X1*A(I,2)
 CALL FUND(N,X2,V,A(1,3))
 DO 22 I=1,N
 IF(Y(I).GE.1.D14) GO TO 23
 22 CONTINUE
 GO TO 24
 23 RETURN
 24 DO 30 I=1,N
 30 V(I)=Y(I)+H*A(I,3)
 CALL FUND(N,X3,V,A(1,4))
 DO 32 I=1,N

RUNK0010
 RUNK0015
 RUNK0020
 RUNK0022
 RUNK0024
 RUNK0025
 RUNK0030
 RUNK0040
 RUNK0050
 RUNK0060
 RUNK0070
 RUNK0080
 RUNK0090
 RUNK0100
 RUNK0110
 RUNK0120
 RUNK0130
 RUNK0131
 RUNK0132
 RUNK0133
 RUNK0134
 RUNK0135
 RUNK0140
 RUNK0150
 RUNK0160
 RUNK0161
 RUNK0162
 RUNK0163
 RUNK0164
 RUNK0165
 RUNK0170
 RUNK0180
 RUNK0190
 RUNK0191
 RUNK0192
 RUNK0193
 RUNK0194
 RUNK0195
 RUNK0200
 RUNK0300
 RUNK0310
 RUNK0311

```
IF(Y(I).GE.1.D14) GO TO 33          RUNK0312
32 CONTINUE                         RUNK0313
      GO TO 34                         RUNK0314
33 RETURN                           RUNK0315
34 DO 40 I=1,N                      RUNK0320
40 YY(I)=Y(I)+H*(A(I,1)+2.*(A(I,2)+A(I,3))+A(I,4))/6.0
      RETURN                         RUNK0330
      END                           RUNK0340
                                    RUNK0350
```

Card Count 50

```

ADAMS-MOULTON INTEGRATOR FOR A SYSTEM OF 1ST ORDER ORDINARY
C DIFFERENTIAL EQUATIONS OF THE INITIAL VALUE TYPE. ADAM0010
C CHOICE OF FIXED OR VARIABLE INCREMENTING. ADAM0020
C IF VARIABLE INCREMENTING THE FOLLOWING RULES WILL APPLY ---
C DEFINE YP=PREDICTOR SOLUTION VECTOR ADAM0030
C DEFINE YC=CORRECTOR SOLUTION VECTOR ADAM0040
C DEFINE RE=UIFF.RATIO OF YP & YC IF ABS(YC).GT.1 ELSE JUST DIFF. ADAM0050
C 1. IF TOL*0.02.LE.RE.LE.TOL IS TRUE THEN H IS UNCHANGED AND YC IS SAVED ADAM0060
C 2. IF RE(I).GT.TOL FOR ANY I OF RE THEN H IS HALVED AND YP AND YC ARE ADAM0070
C RECOMPUTED. HALVING THE INCREMENT IS NOT RESTRICTED. ADAM0080
C THIS STEP. H IS REPLACED BY 2*H AND RETURNED (WITH YC) AS NEW ADAM0090
C INCREMENT FOR THE NEXT STEP. H MAY BE DOUBLED ONLY AFTER 4 ADAM0100
C SUCCESSIVE STEPS USING THE SAME INCREMENT (AN ATTEMPT TO MAINTAIN ADAM0110
C STABILITY IN THE SOLUTION). ADAM0120
C 4. IF H IS HALVED AN ENTIRE SET OF DERIVATIVES ARE COMPUTED USING ADAM0130
C BACKWARDS (IN THE DIRECTION OF -SIGN(H)) INTEGRATION BY RUNGE-KUTA ADAM0140
C 5. IF H IS DOUBLED PREVIOUS SAVED DERIVATIVES ARE USED. ADAM0150
C
C SUBROUTINE ADAMLT(N,X,Y,H,IND,ITIM,TOL,NERR,ISTFLG,F,A) ADAM0160
C IMPLICIT REAL*8 (A-H,O-Z) ADAM0170
C REAL*8 X,Y,H,TOL,F ADAM0180
C DIMENSION Y(1),F(102,7),YP(102),YC(102),A(102,4) ADAM0190
C SEE COMMENTS IN SUBROUTINE RKADAM FOR DESCRIPTION OF ARGUMENTS. ADAM0200
C IF REDIMENSIONING IS DONE IN RKADAM THEN NN MUST BE CHANGED ADAM0210
C ACCORDINGLY IN F(NN,7), YP(NN), AND YC(NN) IN ADAMLT DIMENSION STMADAM0220
C IF(ITIM.NE.0)GOTO60 ADAM0230
C IF(IND.EQ.2)GOTO10 ADAM0240
C IF(IC.EQ.-1)GOTO30 ADAM0250
C
C 10 D020I=1,N ADAM0260
C   F(I,6)=F(I,5) ADAM0270
C   F(I,5)=F(I,4) ADAM0280
C   F(I,4)=F(I,3) ADAM0290
C   F(I,3)=F(I,2) ADAM0300
C 20 F(I,2)=F(I,1) ADAM0310
C   GOTO50 ADAM0320
C   DOUBLE INCREMENT BEING ATTEMPTED THIS STEP (THIS ENTRY) ADAM0330
C 30 IC=0 ADAM0340
C   D040I=1,N ADAM0350
C   F(I,3)=F(I,4) ADAM0360
C 40 F(I,4)=F(I,6) ADAM0370
C 50 CALL FUND(N,X,Y,F(1,1)) ADAM0380
C   GOTO90 ADAM0390

```

```

C KESIAKI
60 IC=0 ADAM0440
    IF(ITIM.GT.0)GOTO90 ADAM0460
    DO 70 I=1,N ADAM0470
    70 YP(I)=Y(I) ADAM0475
C GET NEW SET OF DERIVATIVES BY BACKWARDS INTEGRATIONS (RUNGE-KUTTA)ADAM0480
    XB=X ADAM0490
    DO80I=1,3 ADAM0500
    CALL RUNKUT(N,-H,XB,YP,YP,A) ADAM0510
    XB=X-H*I ADAM0520
    DO 78 K=1,N ADAM0530
    78 F(K,I)=A(K,1) ADAM0532
    80 CONTINUE ADAM0535
    CALL FUND(N,XB,YP,F(1,4)) ADAM0537
    90 HH=H/24.D0 ADAM0540
C PREDICTOR SOLUTION ADAM0550
    DO100I=1,N ADAM0560
    100 YP(I)=Y(I)+HH*((55.D0*F(I,1)-59.D0*F(I,2)+37.D0*F(I,3)-9.D0*F(I,4))ADAM0570
    130 CALL FUND(N,X+H,YP,F(1,7)) ADAM0650
C CORRECTOR SOLUTION ADAM0660
    DO140I=1,N ADAM0670
    140 YC(I)=Y(I)+HH*((19.D0*F(I,1)-5.D0*F(I,2)+F(I,3)+9.D0*F(I,7)) ADAM0680
C TEST FOR FIXED INCREMENT OPTION ADAM0682
    IF(IND.EQ.0)GOTO145 ADAM0684
    X=X+H ADAM0686
    GOTO170 ADAM0688
C TEST RELATIVE ERROR ADAM0690
    145 S=1 ADAM0700
    DO150I=1,N ADAM0710
    T=DABS(YC(I)-YP(I)) ADAM0720
    U=DMAX1(DABS(YC(I)),1.D0) ADAM0730
    V=U*TOL ADAM0740
    W=V*0.02D0 ADAM0750
    IF(T.GT.V)GOTO200 ADAM0760
    IF(W.LE.1)S=0.D0 ADAM0770
    150 CONTINUE ADAM0780
    ISTFLG=0 ADAM0790
    X=X+H ADAM0800
C TEST IF 4 STEPS HAVE ELAPSED. USING SAME INCREMENT FOR DOUBLNG TESTADAM0810
    IF(IC.LT.3)GOTO160 ADAM0820
    IF(S.EQ.0)GOTO170 ADAM0830
C SET H TO 2*D H FOR NEXT STEP ADAM0840

```

IC=-1	ADAM0850
H=2.00*H	ADAM0860
GOTO170	ADAM0870
160 IC=1+IC	ADAM0880
170 DO180 I=1,N	ADAM0890
180 Y(I)=YC(I)	ADAM0900
GOTO120	ADAM0910
200 IF(ISTFLG.NE.0)GOTO120	ADAM0920
HALVE THE INCREMENT AND RECOMPUTE	ADAM0930
H=H/2.00	ADAM0940
CALL OVERFL(K)	ADAM0950
IF(K.NE.3)GOTO60	ADAM0960
H=0	ADAM0970
NERR=1	ADAM0980
120 RETURN	ADAM0990
END	ADAM1000

Card Count 100

```

SUBROUTINE TIME(T,H,TA,ITIM)          00000100
IMPLICIT REAL*8 (A-H,U-Z)           00000200
IF(ITIM.EQ.0) GO TO 10              00000300
TA=0.5*H                            00000400
ICOUNT=1                            00000500
RETURN                               00000600
10 ICOUNT=ICOUNT+1                  00000700
IF(ICOUNT.EQ.2) GO TO 2             00000800
IF(ICOUNT.EQ.3) GO TO 3             00000900
IF(ICOUNT.EQ.4) GO TO 4             00001000
IF(ICOUNT.GE.5) GO TO 5             00001100
2  TA=1.25*H                         00001200
RETURN                               00001300
3  TA=2.125*H                        00001400
RETURN                               00001500
4  TA=3.0625*H                       00001600
TSAVE=T                            00001700
RETURN                               00001800
5  TA=TA+0.5*(T-TSAVE)              00001900
TSAVE=T                            00002000
ICOUNT=ICOUNT-1                     00002100
RETURN                               00002200
END                                  00002300

```

SUBPROGRAM TO SOLVE SIMULTANEOUS LINEAR EQUATIONS
ARGUMENTS-

DATE- 1/13/67 MODIFIED FOR COMPILED IN RELEASE 14

DSM	DIMENSIONED SIZE OF COEFFICIENT MATRIX	AWCU0080
NE	ACTUAL NUMBER OF EQUATIONS FOR THIS CALL	AWCU0090
NC	NUMBER OF COLUMNS IN CONSTANT MATRIX	AWCU0100
A	COEFFICIENT MATRIX	AWCU0110
B	CONSTANT MATRIX	AWCU0120
DET	INPUT - SCALE FACTOR, OUTPUT - FACTOR TIMES DETERMINANT VALUE OF COEFFICIENT MATRIX	AWCU0130
C	TEMPORARY STORAGE FOR SUBROUTINE	AWCU0140
ISIMEQ	RETURNS 1 IF OK, 2 IF OVFLO, 3 IF SINGULAR	AWCU0150
IF NC IS NEGATIVE, THE INVERSE OF THE COEFFICIENT MATRIX IS REQUIRED, MATRIX B IS SET UP AS IDENTITY.	AWCU0160	
MATRIX IS REQUIRED, MATRIX B IS SET UP AS IDENTITY.	AWCU0170	
FUNCTION ISIMDD(DSM, NE, NC, A, B, DET, C)	AWCU0180	
LOGICAL DVO	AWCU0190	
INTEGER DSM, C, T, SUB1, SUB2, R, D	AWCU0200	
DOUBLE PRECISION B, PIVOT, DET, S	AWCU0210	
DIMENSION B(1),C(1)	AWCU0220	
INITIALIZE	AWCU0230	
N = NE	AWCU0240	
D = DSM	AWCU0250	
M = IABS(NC)	AWCU0260	
ISIMDD = 1	AWCU0270	
DVO = .FALSE.	AWCU0280	
DO 1 I = 1, N	AWCU0290	
1 C(I) = I	AWCU0300	
IF(NC) 5, 15, 15	AWCU0310	
5 INVERSE REQUIRED	AWCU0320	
SUB2 = 0	AWCU0330	
DO 10 J = 1, N	AWCU0340	
SUB1 = SUB2	AWCU0350	
DO 6 I = 1, N	AWCU0360	
SUB1 = SUB1 + 1	AWCU0370	
6 B(SUB1) = 0.0	AWCU0380	
SUB1 = SUB2 + J	AWCU0390	
B(SUB1) = 1.0D0	AWCU0400	
10 SUB2 = SUB2 + 0	AWCU0410	
GO TO 15	AWCU0420	
ENTRY IDETDD(DSM, NE, A, DET)	AWCU0430	
	AWCU0440	

```

    DOUBLE PRECISION A
    DIMENSION A(1)
    N          = NE
    D          = DSM
    IDETDD   = 1
    DVO      = .TRUE.
C   START MAIN LOOP
15 DO 1000 L = 1,N
     LP1 = L+ 1
     DO 40 I = L,N
       PIVOT = 0.0D0
       SUB1 = (L-1) * D + I
       SUB2 = SUB1
       DO 20 J = L,N
         IF(DABS(PIVOT) .GE. DABS(A(SUB1))) GO TO 20
         PIVOT = A(SUB1)
         JB = J
20      SUB1 = SUB1 + D
C   COMPUTE DETERMINANT
C   CALL OVERFL(T)
     DET = DET * PIVOT
     IF(.NOT. DVO) GO TO 24
     CALL OVERFL(T)
     IF(T .EQ. 1) IDETDD = 2
C   TEST FOR SINGULAR MATRIX
24 IF(PIVOT .EQ. 0.0D0)GO TO 2000
     DO 25 J = L,N
       A(SUB2) = A(SUB2) / PIVOT
25      SUB2 = SUB2 + D
     IF (DVO) GO TO 35
     SUB1 = I
     DO 30 J = 1,M
       B(SUB1) = B(SUB1) / PIVOT
30      SUB1 = SUB1 + D
35      IF (I .EQ. L) JP = JB
40      CONTINUE
C   INTERCHANGE COLUMNS
100   IF (JP .EQ. L) GO TO 260
     IF (DVO) GO TO 110
     T = C(L)
     C(L) = C(JP)
     C(JP) = T

```

AWCU0450
AWCU0460
AWCU0470
AWCU0480
AWCU0490
AWCU0500
AWCU0510
AWCU0520
AWCU0530
AWCU0540
AWCU0550
AWCU0560
AWCU0570
AWCU0580
AWCU0590
AWCU0600
AWCU0610
AWCU0620
AWCU0630
AWCU0640
AWCU0650
AWCU0660
AWCU0670
AWCU0680
AWCU0690
AWCU0700
AWCU0710
AWCU0720
AWCU0730
AWCU0740
AWCU0750
AWCU0760
AWCU0770
AWCU0780
AWCU0790
AWCU0800
AWCU0810
AWCU0820
AWCU0830
AWCU0840
AWCU0850
AWCU0860

```

110      R = D * L - D          AWCU0870
        T = D * JP - D          AWCU0880
        DO 120 I = 1,N          AWCU0890
        SUB1 = R + I             AWCU0900
        SUB2 = T + I             AWCU0910
        S = A(SUB1)              AWCU0920
        A(SUB1) = A(SUB2)          AWCU0930
120      A(SUB2) = S             AWCU0940
        DET = -DET              AWCU0950
C      REDUCE PIVOT COLUMN      AWCU0960
260      R = D* L - D          AWCU0970
        DO 400 I = 1,N          AWCU0980
        IP = R + I              AWCU0990
        PIVOT = A(IP)            AWCU1000
        IF (I .EQ. L .OR. PIVOT .EQ. 0.0) GO TO 400
        SUB1 = L                 AWCU1010
        SUB2 = I                 AWCU1020
        DO 360 J = 1,N          AWCU1030
        IF (J .LT. LP1) GO TO 300
        S = PIVOT * A(SUB1)      AWCU1040
        A(SUB2) = A(SUB2) - S    AWCU1050
204      IF (DABS(A(SUB2)) .LT. DABS(3.0E-8*S)) A(SUB2) = 0.0
300      IF (DVO .OR. J .GT. M) GO TO 350          AWCU1060
        B(SUB2) = B(SUB2) - PIVOT * B(SUB1)          AWCU1070
        AWCU1080
350      SUB1 = SUB1 + D             AWCU1090
        SUB2 = SUB2 + D             AWCU1100
        AWCU1110
360      AWCU1120
400      CONTINUE                AWCU1130
        AWCU1140
1000     CONTINUE                AWCU1150
        IF (DVO) GO TO 1500          AWCU1160
C      REARRANGE VARIABLES      AWCU1170
1100     DO 1201 L=1,N          AWCU1180
        SUB1 = C(L)                AWCU1190
        SUB2 = L                  AWCU1200
        DO 1200 J = 1,M          AWCU1210
        A(SUB1) = B(SUB2)          AWCU1220
        SUB1 = SUB1 + D            AWCU1230
1200     SUB2 = SUB2 + D            AWCU1240
        AWCU1250
1201     CONTINUE                AWCU1260
1500     RETURN                  AWCU1270
C      SINGULAR COEFFICIENT MATRIX
2000     IF(DVO) GO TO 3000          AWCU1280
        ISIMDD = 3                AWCU1290
        GO TO 1500
3000     IDETDD = 3                AWCU1300
        GO TO 1500                AWCU1310
        END                      AWCU1320

```


TIME	TU	TIME IN REG 0	AWCG0450
ST	0,TZERO		AWCG0460
L	13,SAVE+4		AWCG0470
LM	14,12,12(13)		AWCG0480
MVI	12(13),X'FF'		AWCG0490
LA	15,0		AWCG0500
BR	14	RETURN	AWCG0510
EJECT			
* READ ELAPSED TIME IN SECONDS			
TIMERV	EQU *		AWCG0530
TIMEV	SAVE (14,12),,TIMEV		AWCG0540
	BALR - 10,0		AWCG0550
	USING *,10		AWCG0560
	ST 13,SAVE+4		AWCG0570
	LA .2,SAVE		AWCG0580
	ST 2,8(13)		AWCG0590
	L 4,0(1)	LOC OF USERS TIME CELL	AWCG0600
	TIME TU	TIME IN REG 0	AWCG0610
	TM FSTSW,1	DID COUNTV GET CALLED	AWCG0620
206	BNZ T1		AWCG0630
	ST 0,TZERO	COUNTV WASN'T CALLED	(N1) AWCG0640
	MVI FSTSW,1	SET FLAG TO PRETEND COUNTV NOW BEING CALLED	(N1) AWCG0650
	SER 2,2	ZERO FL PT REG	(N1) AWCG0660
	B T3		(N1) AWCG0670
*	*	*	(N1) AWCG0680
T1	SL 0,TZERO	COUNTV WAS CALLED	(N1) AWCG0690
	BC 3,T2	IS DIFFERENCE NEGATIVE	(N1) AWCG0700
	AL 0,DAY	YES ADD 24 HRS	(N1) AWCG0710
T2	ST 0,BITS2	ELAPSED TIME IN REG 0	(N1) AWCG0720
	LD 2,BITS1	ELAPSED TIME IN FREG 2	(N1) AWCG0730
	DD 2,FACTOR	CONVERT TIMER UNITS TO SECONDS.	(N1) AWCG0740
T3	STE 2,0(4)	STORE IN USERS TIME CELL	(N1) AWCG0750
	L 13,SAVE+4		AWCG0760
	LM 14,12,12(13)		AWCG0770
	MVI 12(13),X'FF'		AWCG0780
	LA 15,0		AWCG0790
	BR 14	RETURN	AWCG0800
EJECT			
FSTSW	DC X'0'	SET FIRST TIME THRU	AWCG0810
FACTOR	DC D'38400'	300*2**7= 38400	(N1) AWCG0820
BITS1	DC X'4E000000'	1-ST HALF OF DOUBLE-WORD 'BITS1'	(N1) AWCG0830
BITS2	DC X'00000000'	2-ND HALF OF DOUBLE-WORD 'BITS1'	(N1) AWCG0840

```

DAY      DC X'C5C10000' =3317760000= 38400*86400= 300*2**7*60*60*24(N1) AWCG0870
TZERO    DC   F'0t                      FIRST TIME READING .          AWCG0880
SAVE     DC   18F'0'                     SAVE AREA                 AWCG0890
*THE TIME TU MACRO RETURNS THE TIME OF DAY AS 32-BIT UNSIGNED INTEGER. AWCG0900
*BIT 31= 1/(300*2**7) SEC.= 26 MICRO-SEC.          AWCG0910
*BIT 24= 1/300 SEC.          AWCG0920
*BIT 0= 2**24/300 SEC.= 15.5 HOURS          AWCG0930
*MAX. TIME = 2**25/300 SEC. = 31 HOURS WHICH IS ENOUGH TO CONTAIN THE AWCG0940
* TIME OF DAY BASED ON A 24-HOUR CLOCK.          AWCG0950
*MAX. RESOLUTION = 1/60 SEC. STANDARD OR 26 MICRO-SEC. FOR AN OPTIONAL AWCG0960
* HIGH-RESOLUTION TIMER WHICH IS NOT STANDARD AT MOST INSTALLATIONS. AWCG0970
END
/*

```

Card Count 97